

NASA Contractor Report 191128

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# HIGH PERFORMANCE STORABLE PROPELLANT RESISTOJET

Contract NASA 3-24652

91-R-1553

## FINAL REPORT

Prepared for:

**NASA Lewis Research Center**  
Cleveland, OH 44135

January 17, 1992

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Unclass

N95-18195

STORABLE PROPELLANT RESISTOJET  
(NASA-CR-191128) HIGH PERFORMANCE  
Final Technical Report (Rocket  
Research Corp.) 314 p

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**HIGH PERFORMANCE STORABLE  
PROPELLANT RESISTOJET**

**Contract NAS3-24652**

Prepared for:

**NASA Lewis Research Center**  
Cleveland, OH 44135

Prepared by:

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Report No. 91-R-1553  
Date: January 17, 1992

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**Document Revision Record**  
Document No. 91-R-1553

REVISION AND DATE	DESCRIPTION OF CHANGE/ REVISION AND PAGES AFFECTED	EFFECTIVITY
Original January 17, 1992		<i>Mr. 1/30/92</i>



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## SUMMARY

From 1965 until 1985 resistojets were used for a limited number of space missions. Capability increased in stages from an initial application using a 90 W gN<sub>2</sub> thruster operating at 123 sec specific impulse (Isp) to a 830 W N<sub>2</sub>H<sub>4</sub> thruster operating at 305 sec Isp. Prior to 1985 fewer than 100 resistojets were known to have been deployed on spacecraft.

Building on this base NASA embarked upon the High Performance Storable Propellant Resistojet (HPSPR) program to significantly advance the resistojet state-of-the-art. Higher performance thrusters promised to increase the market demands for resistojets and enable space missions requiring higher performance.

The program goals were ambitious. At the beginning of the program the performance goals were to produce a resistojet design capable of producing 50 to 100 mlbf of thrust with an Isp of 380 seconds utilizing no more than 1500 watts of electrical power. The goals were refocused to the changing needs of users and a changing status of technology throughout the program. The performance goals during successive phases are summarized below:

Phase	Goals	Achieved
II	Isp = 380 sec 1500 W 50 – 100 mlbf	348 sec, vented cavity, radiative heater 322 sec, sealed cavity, radiative heater Reduced to 750 prior to test 39 – 62 mlbf at 500 to 850 W, vented cavity, radiative heater 37 – 56 mlbf at 500 to 850 W, sealed cavity, radiative heater
III	Isp = 315 sec Power < 750 W 100,000 lb-sec	322 sec, immersed heater 763 W Never attempted, mass loss was excessive at 315 + Isp

The key technology limitation turned out to be the tungsten wire temperature. For all configurations tested the wire temperatures required to achieve the desired performance resulted in excessive sublimation rates or in adverse chemical reactions resulting in filament mass loss. In addition, heat exchanger material creep drove throat diameter up. This diameter increase resulted in a lower nozzle efficiency and hence lower overall performance.

During the program three resistojets were fabricated and tested. High temperature wire and coupon materials tests were completed. A life test was conducted on an advanced gas generator.

The following are the notable accomplishments of the HPSPR program over its 5-year history. These items are also summarized graphically in Figure 1.

1. Advanced the understanding of low Reynolds number nozzles. Fabricated nozzles for NASA cold gas study and performed analysis in support of the study. Resulted in NASA TM-89858.
2. Comprehensive performance mappings of vented cavity, sealed cavity, and immersed filament resistojets were completed. Performance limits of each configuration were determined. An extensive data base in the moderate power (500 to 850 W) range for Isp, thrust efficiency, and thermal performance was obtained for the three thruster configurations.
3. High temperature tungsten filaments for three resistojet configurations were characterized. Material loss rates and mechanisms were investigated.
4. A better understanding of refractory metals was obtained. The mechanisms of attack of N<sub>2</sub>H<sub>4</sub> decomposition products were explored. The effects of high temperature atomic oxygen were investigated. Metals studied included rhenium, tungsten, Mo/4Re, and Mo/3Re.
5. Conducted a life test of an advanced low flow gas generator. The life test ultimately achieved 887 hours and the gas generator demonstrated reduced feed tube plugging compared to all previous designs.

## FOLDOUT FRAME 1.

# HPSPR Proc

### LOW REYNOLDS NUMBER NOZZLES

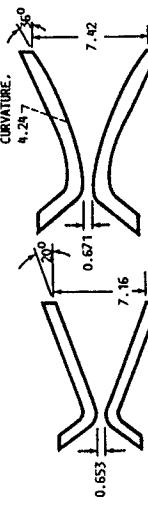
#### SPECIFIC IMPULSE EFFICIENCY AS A FUNCTION OF THROAT REYNOLDS NUMBER FOR $gH_2$ & $gN_2$ .

#### Experimental Data & Two-Dimensional Kinetics (TDK) Nozzle Analysis

All Dimensions are in Millimeters

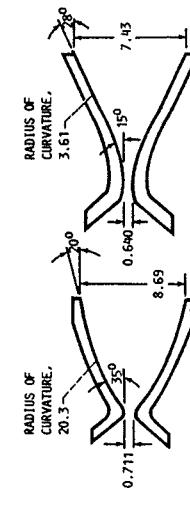
#### NOZZLE GEOMETRIES

All Dimensions are in Millimeters



#### CONICAL

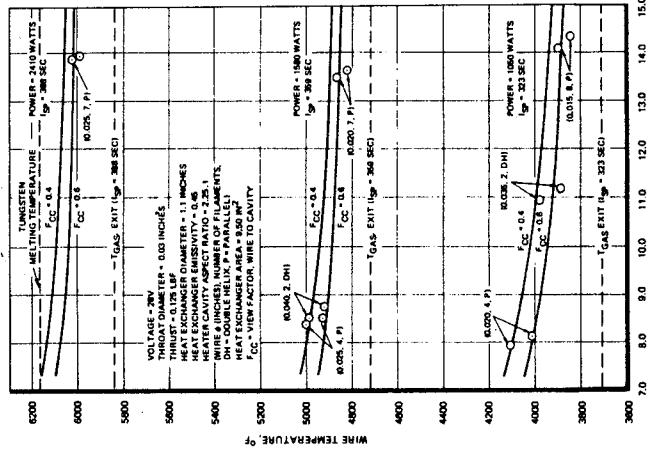
#### TRUMPET



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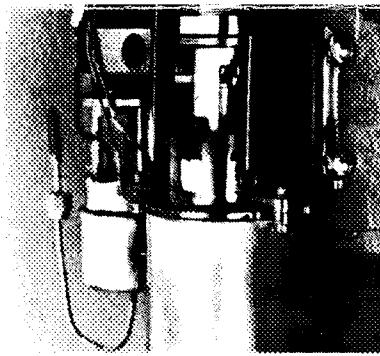
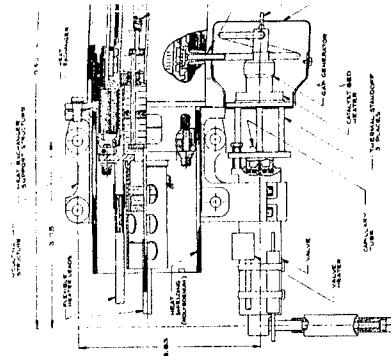
### TUNGSTEN FILAMENT BEHAVIOR

#### HIGH PERFORMANCE RESISTOJET HEATER SIZING STUDIES



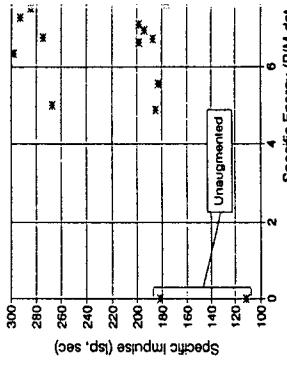
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### RESISTOJET DESIGN/IV



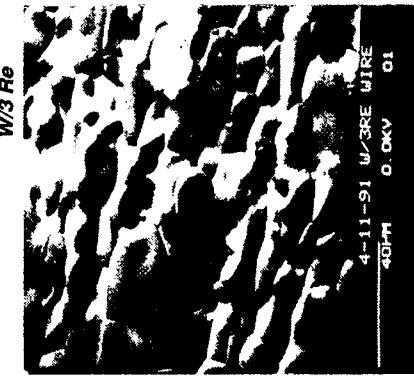
### HIGH PERFORMANCE RE:

#### Sealed Cavity Data

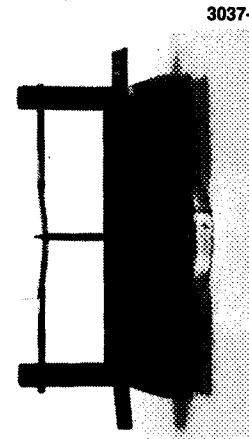


### REFRACTORY METAL

#### W/3 Re



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AFTER  
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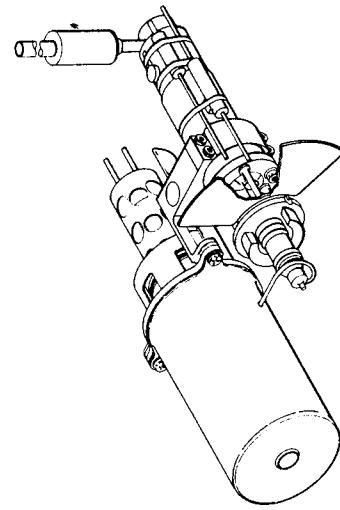
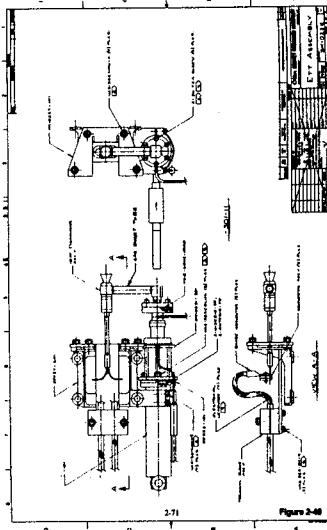


# in Highlights

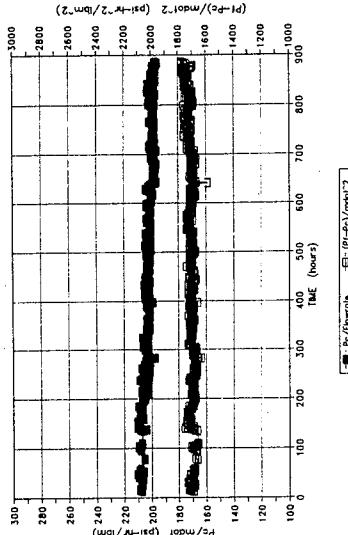
ON DUTY TIME

## APPLICATION & TESTING

### ADVANCED GAS GENERATOR



NASA FLIGHT DEVELOPMENT GG  
(Pf-Pc) / Flowrate'2 & P<sub>c</sub> / Flowrate



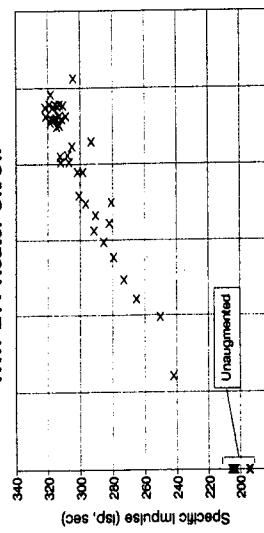
### HIGH PERFORMANCE RESISTOJET Preliminary Performance Specification

- Mission • N-S stationkeeping, geosynchronous communication satellite
- 10 years (250-hour total firing time)
- 0.125-lbf nominal design point
- 117,000 lbf-sec
- 360 lbf-sec/lbm average over feed pressure blowdown range (300 — 100 psia)
- Direct operation from S/C bus. Surge current limit circuit on start-up desirable but not required
- Regulated or battery voltage set-down
- 28 vdc nominal peak. adjustable as required
- 1600 watt at P<sub>f</sub> = 300 psia; 1400 watt at 100 psia
- Blowdown 300 to 100 psia
- Regulated 150 psia minimum
- ±5%
- Spectrum 0.2-7 Hz, 20 — 20,000 Hz
- Overall 13.0 g rms
- TBD

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HIGH PERFORMANCE RESISTOJET  
TRW-ETT Heater On/Off



### TESTING @ 2100°F WITH 1% MOISTURE IN AMMONIA

Sample	103 Hrs	493 Hrs	% Wt Gain	% Wt Gain
IrCVDW	+0.0078	+0.026*		
W/25Re	+0.0087	0		
Mo/41Re	-0.205	-0.366		
Re	+0.0038	+0.015		
W100	-0.022	-0.026		

\* If CVD W sample was split after the 103-hour test for metallographic analysis and exposed tungsten surface underwent a different reaction (turned black).

Sample	Appearance	
IrCVDW	Ir coating unchanged. Bare W shows dull greying.	
	Fractured surface is black, no visible cracks.	
W/25Re	Polished surface still has some reflective properties. No visible cracks.	
Mo/41Re	Polished surface still has some reflectivity. Sample has cracks that appear to penetrate through the piece, i.e. cracks continue from one surface into the adjoining surface.	
Re	Strip has original appearance, reflective. No visible cracks.	
W100	Polished surface totally nonreflective, dull, grey.	No visible cracks.



## 1.0 INTRODUCTION

This is the Final Technical Report for Contract NAS3-24652 entitled "High Performance Storable Propellant Resistojet" (HPSPR). The program technical effort began in January, 1986 and concluded in May, 1991.

### 1.1 HISTORY OF RESISTOJETS

In the early 1960's resistojets were one of several electric propulsion devices developed for a wide variety of space missions. Avco and Giannini developed resistojets concurrently with early research on arcjets. Power levels ranged from less than one watt to over 60 kW. High thrust efficiency and Isp up to 846 sec were measured. Most of these resistojets operated on NH<sub>3</sub> or H<sub>2</sub> (Jahn, 1970). The minimal available electrical power in early spacecraft, poor integration into ACS and delta-V propulsion systems, and finally a dearth of missions requiring the level of performance offered resulted in few space applications.

The earliest resistojet used in space propulsion was in 1965 when the Vela spacecraft carried two gN<sub>2</sub> resistojets into orbit. These thrusters used 90 W of electrical power and produced an Isp of 123 sec compared to 70 sec for a cold gas gN<sub>2</sub> system (Jackson, 1966). Other early flight systems included NH<sub>3</sub> fueled micro-lbf class thrusters developed for the Naval Research Laboratory and low-power, low thermal inertia resistojets flown as experiments on the ATS spacecraft series (Pugmire, 1971). While mission models indicated that increased performance was desirable in low thrust engines it was clear that a practical resistojet would have to be fully integrated into the ACS propulsion of a spacecraft. The need for lighter weight, long life, station keeping propulsion systems for Geosynchronous Equatorial Orbit (GEO) communication satellites provided the first user requirements that demanded technology advancements such as resistojets (Sackheim, 1979).

The technology was significantly advanced again in the late 1970's when a TRW team designed (Grabbi, 1976) and completed the qualification of the High Performance Electrothermal Hydrazine Thruster (HIPEHT) hydrazine resistojet (Dressler, 1981). The HIPEHT thruster was qualified for the Intelsat-V spacecraft. This thruster used N<sub>2</sub>H<sub>4</sub> propellant and was fully integrated with the catalytic thruster ACS propulsion system. This design used a decomposition heater in a separate chamber to break down the N<sub>2</sub>H<sub>4</sub> into N<sub>2</sub>, H<sub>2</sub>, and NH<sub>3</sub>, and an immersed tungsten filament to super-heat the decomposition gases in a vortex heat exchanger. The thrust range of this resistojet was 50 to 110 mlbf with a power requirement of 5 W per mlbf delivering an Isp of 295 sec. The thruster suffered from early heater failures and deposition of nonvolatile residue in the feed tube in the decomposition chamber.

Starting in 1981 Rocket Research Company (RRC) was involved in a series of developments extending resistojet technology further with an emphasis on flight hardware. The design approach used a catalytic gas generator (based on a 0.1 lbf thruster) to decompose the N<sub>2</sub>H<sub>4</sub> into N<sub>2</sub>, H<sub>2</sub>, and NH<sub>3</sub>. The decomposition gases were super heated within a refractory metal heat exchanger. The heat exchanger and heater filament were radiatively coupled with the

filament centrally located in an evacuated cavity within the heat exchanger. This feature allowed the heater filament to be hotter, but decreased the thrust efficiency.

RRC conducted a series of resistojet programs. In 1981 RRC was awarded a contract from RCA for the design, qualification, and production of an Augmented Catalytic Thruster (ACT). Qualification and first flight hardware deliveries of the MR-501 thruster were completed in about 13 months. This ACT flew on SATCOM F-1, Spacenet, G-Star, ASC I-III, and BS-3. Additional technology programs included an effort for AFRPL in which a gas generator was operated for 500 hours (equivalent to 130,000 lbf-sec of impulse). In a contract with Intelsat, RRC completed a comprehensive design study of scaling effects for electrothermal thrusters in the 0.5 to 5 N thrust range. For NASA-LeRC, RRC completed a demonstration program to map resistojet performance up to 1 kW using the ACT heat exchanger with gH<sub>2</sub>, gN<sub>2</sub>, and NH<sub>3</sub>. In another NASA-LeRC program, RRC fabricated an all rhodium heat exchanger to withstand higher temperatures and yield longer life times.

These programs resulted in a qualified flight thruster, the MR-501, and advanced technology in heaters, heat exchangers, materials, and gas generators. All of these efforts formed the technology basis for the HPSPR program. This background is summarized in Figure 1-1.

## 1.2 HPSPR PROGRAM GOALS/PLANS

The purpose of the HPSPR program was to advance the technology of resistojets used for auxiliary propulsion. The performance of resistojet thrusters to this point in time had been limited primarily by material temperature limits, heat exchanger and nozzle performance, and by propellant thermodynamics. The program was intended to establish a technology focus and provide for the design and demonstration of a resistojet with significantly improved performance using storable propellants.

At the beginning of the program there was a performance goal to produce a resistojet design capable of producing 50 to 100 mlbf of thrust with an Isp of 380 seconds utilizing no more than 1500 watts of electrical power. The total delivered impulse was to have been 70,000 lbf-sec. In the original program concept there were to have been four phases in which one development and two proto-flight thrusters were to have been built.

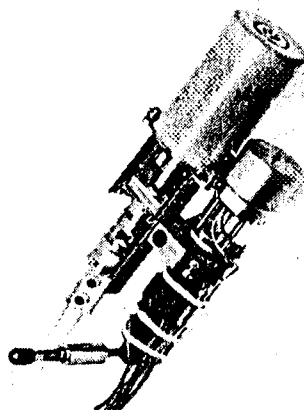
To meet these goals RRC originally planned a four phase program to investigate the fundamental physical limits to resistojet performance, identify and test key components necessary to overcome existing performance limits, and build and test an advanced resistojet thruster incorporating the design features identified.

Figure 1-2 provides a graphical summary of the original program plan. As will be discussed, this plan was changed significantly as results were obtained and user community requirements changed. These changes are also illustrated in Figure 1-2.

There were 3 key decision points during the program. First, at the end of Phase I an assessment was made of whether to proceed to conceptual design of the candidate thruster. The material data were inadequate to support commitment to a design. It was judged that

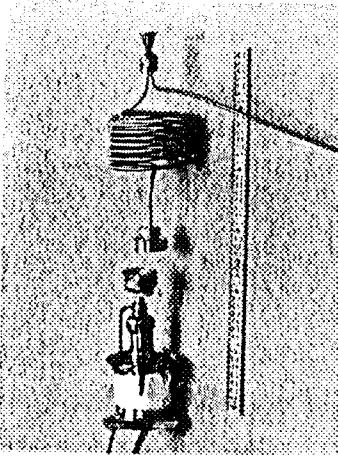
UNPARALLELED TECHNFOLDOUT FRAME

## AUGMENTED CATALYTIC THRUSTER



- 2635
- RCA-FUNDED CONTRACT
  - EMPLOYS RADIATIVE HEAT EXCHANGER
  - DEMONSTRATED PERFORMANCE:
    - MISSION AVG  $I_{sp}$  = 293 sec
    - TOTAL IMPULSE = 76,000 lbf-sec
  - 64 FLIGHT ENGINES DELIVERED

## HIGH TEMPERATURE AUGMENTED HYDRAZINE THRUSTER



- 2658
- AFRPL-FUNDED CONTRACT
  - COILED TUBE HEATER
  - DEMONSTRATED PERFORMANCE
    - 430 HRS OF LIFE
    - AVERAGE  $I_{sp}$  OF 293 sec

## ADVANCED HEATER DEVELOPMENT



## LONG-LIFE GAS GENERATOR DEVELOPMENT

- RRC IRAD PROGRAM
- DEMONSTRATED CATALYTIC GAS GENERATOR LIFE IN EXCESS OF 500 HRS

## HIGH TEMPERATURE REFRACTORY METAL CHARACTERIZATION

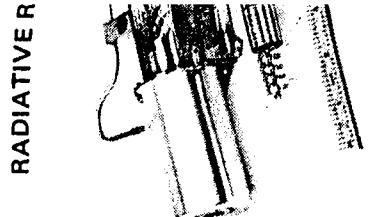
- RRC IRAD PROGRAM
- TESTING AT BATTELLE
- RRC METHODOLOGY

## ACT APPLICATION TO SBS-1A SATELLITE

- 2793-1
- HUGHES-FUNDED CONTRACT
  - DEMONSTRATED OVER 500 HRS OF AUGMENTATION HEATER OPERATION IN A 2.6 "g" FIELD

## EHT SCALING STUDY

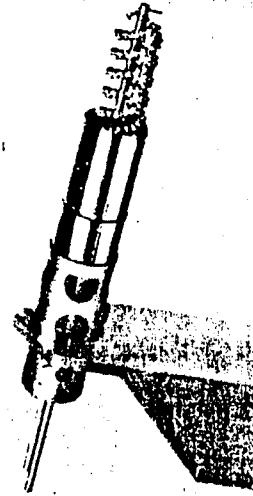
- NASA/LERC-FUNDED
- MEASURED PERFORMANCE LEVELS UP TO 1 kW
- DEMONSTRATED T UP TO 0.2 lbf
- EXCELLENT PERFORMANCE DELIVERED FOR EHT SCALING STUDY



## RADIATIVE HEAT

## AC IMPRINT

- RRC IRAD PROGRAM
- DEMONSTRATES CAPABILITY OF HYDRAULIC GAS GENERATOR OF HYDRAULIC GAS GENERATOR
- EMPLOYED ENVIRONMENT
- 120°F PROP
- STEADY-STATE DUTY CYCLE

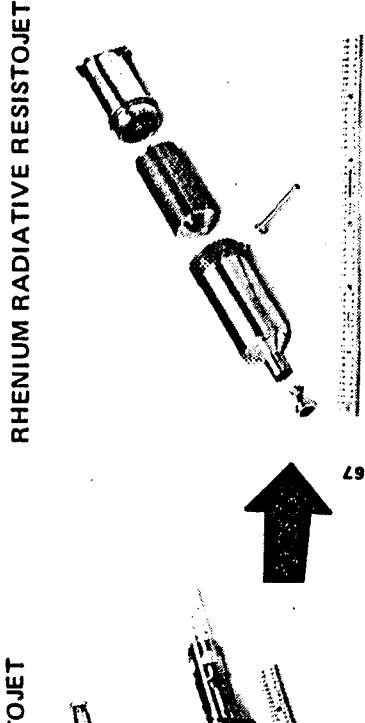


- RRC IRAD PROGRAM
- 90-g rms RANDOM VIB TEST
- EMPLOYED SAG-RESISTANT COIL MATERIAL
- TESTING INCLUDED 500-HR LIFE TEST



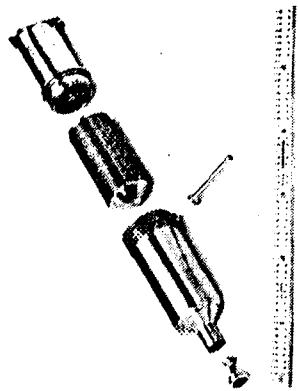
## Company Resistojet Experience

### ROCKET JET AND FLIGHT EXPERIENCE



ROCKET JET

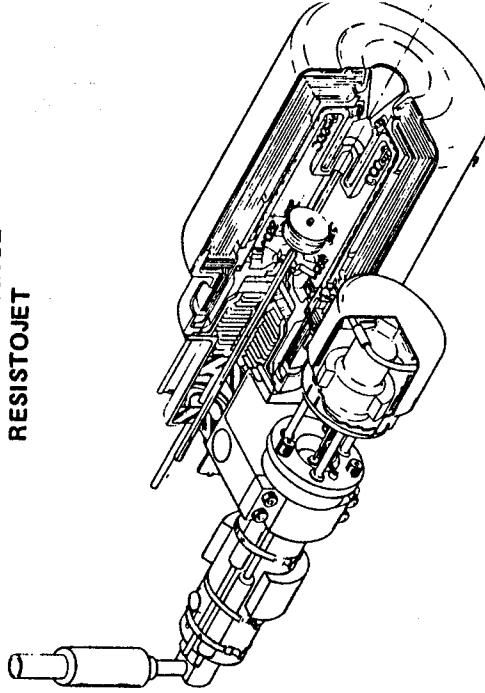
RHENIUM RADIATIVE RESISTOJET



- NASA/LaRC-FUNDED CONTRACT OBTAINING PERF MAP OF RHENIUM RESISTOJET USING GASEOUS DECOMPOSITION PRODUCTS OF HYDRAZINE
- RHENIUM FABRICATION EXPERIENCE

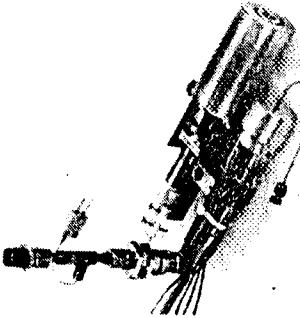
PERFORMANCE LEVELS

### HIGH PERFORMANCE RESISTOJET



- FLIGHT PROVEN DESIGN HERITAGE
- SPECIFIC IMPULSE CAPABILITY IN EXCESS OF 400 lbf-sec/lbm
- MINIMUM IMPACT TO USER SPACECRAFT
- LOGICAL EXTENSION OF CONTINUING EFFORTS

### TEST



TEST

TOJET THERMAL

- RRC IRAD PROGRAM
- DEMONSTRATED  $I_{sp}$  IN EXCESS OF 310 sec IN THE 0.04- TO 0.20-lbf THRUST RANGE
- EXTENSIVE PERF MAPPING

- RRC IRAD PROGRAM
- DEMONSTRATED HIGHER PER OF ACT W/INCREASED HEATER POWER (600 W)
- REFINED THERMAL & THERMO-MECHANICAL MODELS
- CHARACTERIZED GROUND TEST EFFECTS

2748-2

TEST

TOJET THERMAL

1-3

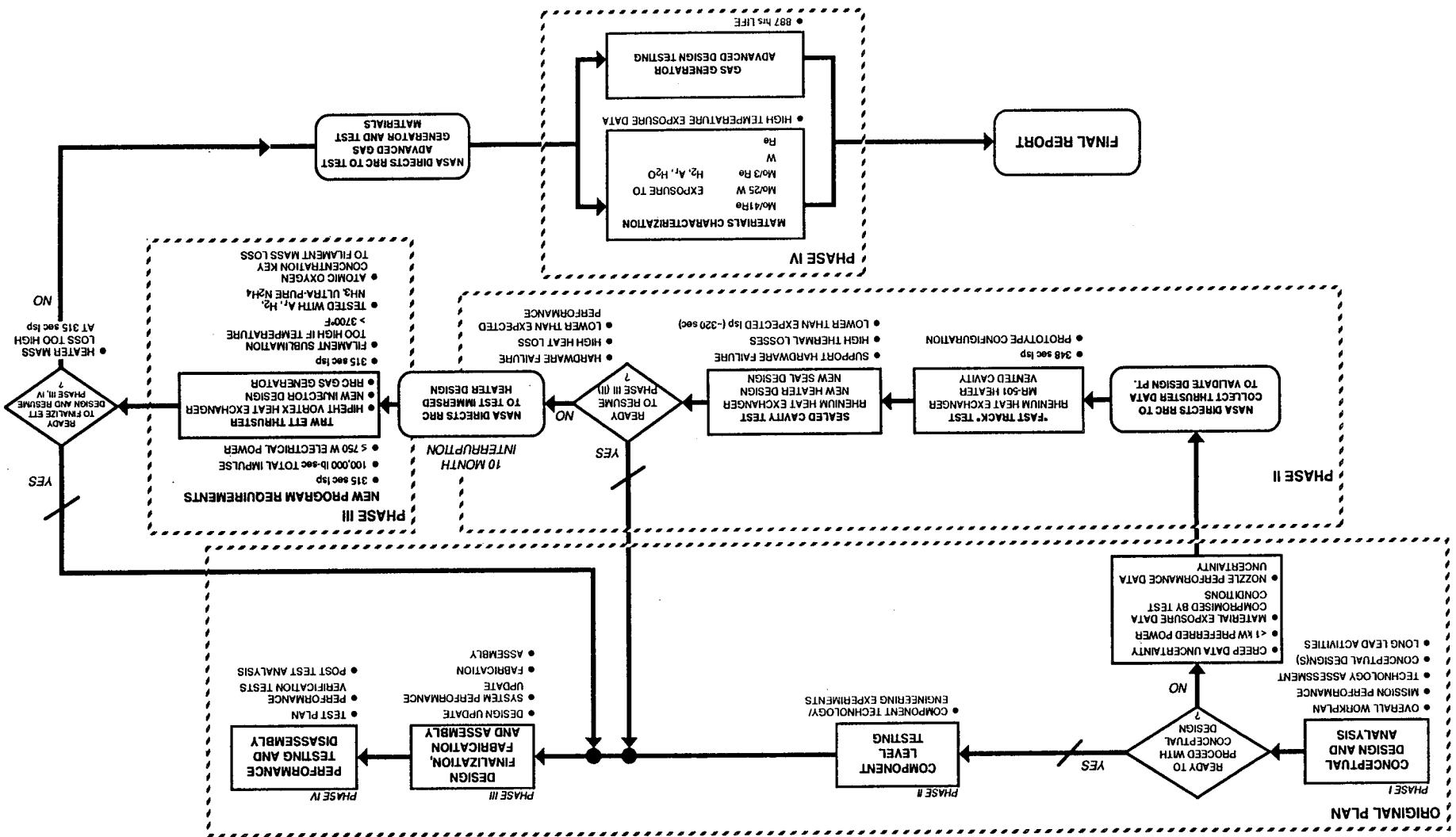
Figure 1-1



**Figure 1-2**

14

## HPSRJ OVERALL PROGRAM FLOW





collection of data directly from thrusters was the best way to clarify the issues. Second, following the thruster testing a decision was made that the performance did not merit a detailed design. Following an interval of 10 months the program was redirected to investigate an immersed heater design. The final decision was whether to finalize the proto-flight design based on the immersed heater configuration. Again, the performance did not merit a final design. Additional material data were determined to be more useful to the user community.

### 1.3 OUTLINE OF PROGRAM HISTORY

The following is a summary of the four phases of the program.

#### 1.3.1 Phase I — Program Initiation

This phase of the program was conducted as planned. Materials characterizations proved to be more difficult than originally thought. Both the creep characterization of heat exchanger materials and the wire compatibility tests were unable to achieve temperature ranges beyond 4500°F due to facility limitations. Thermal and structural modelling that depended on material data required extrapolations and therefore were subject to some uncertainty.

There were three specific design problems. First, the creep behavior of heat exchanger bodies was excessive for the higher temperatures required for higher performance. Second, a sealed cavity resulted in exposing the heater wire to N<sub>2</sub>H<sub>4</sub> decomposition products and there was an inadequate materials data base. A sealed cavity was proposed as a means of increasing life and performance by decreasing the filament mass loss rates. Finally, there was no seal design to implement a sealed cavity

#### 1.3.2 Phase II — Radiative Heater Thruster Experiments

Due to the lack of a test facility to collect creep data over the desired temperature range and the presence of water in the wire exposure tests there was considerable uncertainty at the end of Phase I regarding whether available materials would support the design features needed for performance improvements. To alleviate the uncertainty, Phase II was restructured to proceed immediately to thruster testing. Direct measurement of actual thruster performance promised to provide creep data on heat exchanger materials, wire compatibility data, nozzle performance data, and verification of thermal and structural models. In addition, thruster testing provided direct verification of performance estimates for component improvements identified in Phase I.

Two radiative heat exchanger thruster configurations were tested. The first was a vented cavity heat exchanger (filament in a vacuum) operating at a higher temperature than existing designs. The second was a sealed cavity thruster using the same heat exchanger design with the volume surrounding the filament pressurized with decomposition gases.

#### 1.3.3 Phase III — Immersed Heater Thruster Experiments

Phase II results indicated great difficulty in limiting performance losses due to heat conduction away from the thruster. There was also a support hardware failure in the second of the two tests. Requirements of the potential user community were also changing. Because of these factors the program was restructured for a second time starting in Phase III. The new

requirement was for a thruster delivering 315 sec Isp using 750 W of electrical power and providing 100,000 lbf-sec total delivered impulse. An immersed heater thruster based on the HiPEHT thruster was tested. A catalytic gas generator was substituted to obviate the feed tube nonvolatile residue problem previously encountered with thermal decomposition, and the vortex injector was redesigned to improve heat transfer. Two of these thrusters were tested on the program.

#### **1.3.4 Phase IV — Materials and Gas Generator Experiments**

Following Phase III, the program embarked on a study of advanced materials and gas generator designs that would be required to extend the life of resistojets. Coupon and wire samples of a range of refractory metals were tested with N<sub>2</sub>H<sub>4</sub> decomposition gases and other gases. A life test of an advanced gas generator was performed.

The following sections provide a detailed report of the program activities.

## 2.0 HARDWARE DEVELOPMENT REPORT

### 2.1 PHASE 1 — PROGRAM INITIATION

The objectives of this phase were to assess technologies to improve resistojet performance. The leading candidate design at the beginning of the program was a sealed cavity radiative heater concept with a super heater just upstream of the nozzle illustrated in Figure 2-1. Preliminary analysis indicated the sealed cavity heater filament could be operated at higher temperature with no increase in mass loss rate. A super heater could further enhance performance by adding more enthalpy to the flow. There were three technology areas that were of particular concern for this candidate thruster.

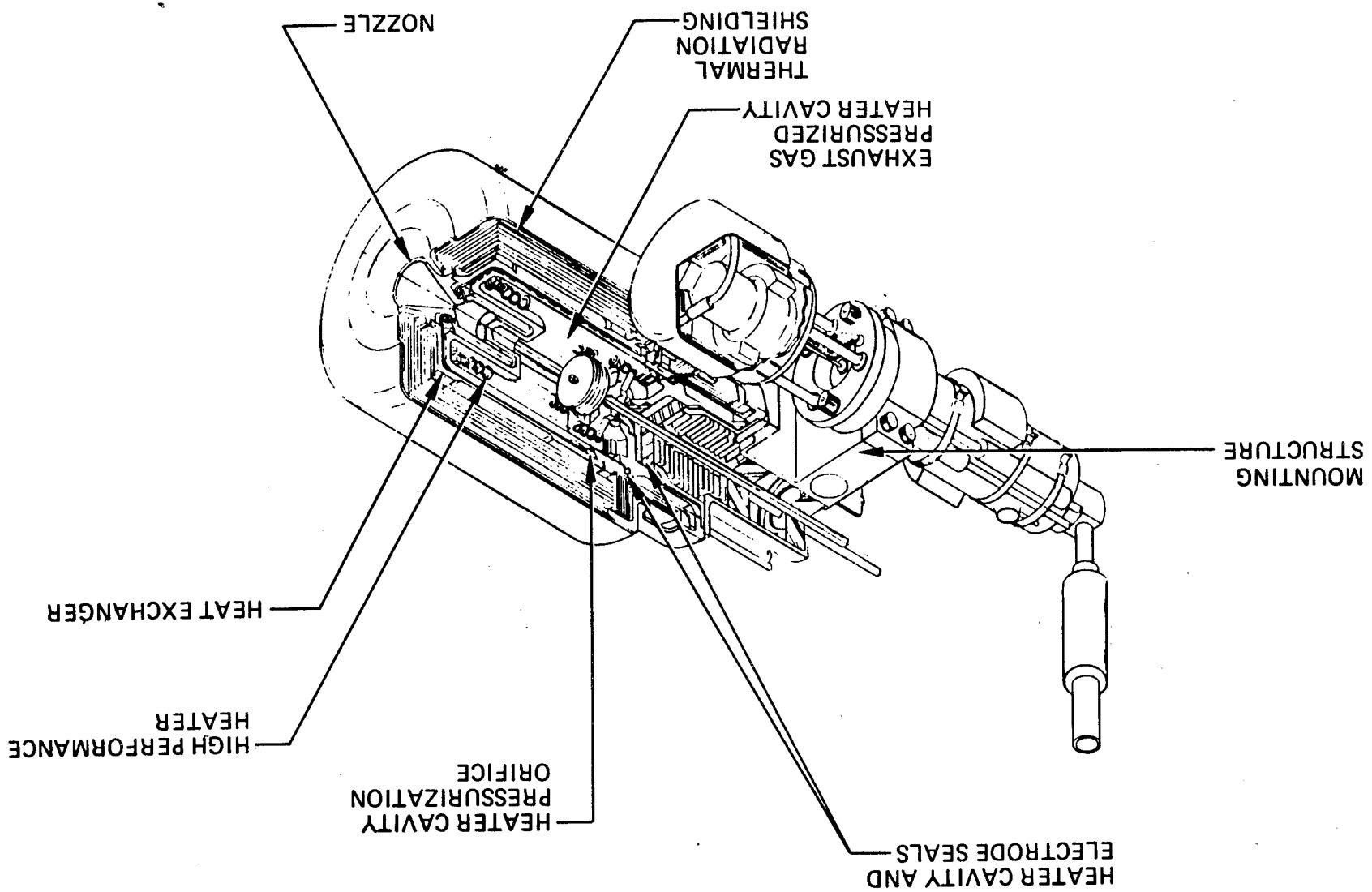
First, the creep behavior of heat exchanger bodies was excessive for the higher temperatures required for higher performance. The creep of the Mo/41Re heat exchanger in the MR-501 was measurable over the life of the thruster. Excessive creep would result in ballooning leading to rupture of the heat exchanger outer wall, or in shorting out the heater filament against the heat exchanger due to pressure from the outside forcing the heat exchanger in toward the filament. Engineering judgement was that there was no margin for increased temperature. A rhodium heat exchanger with superior creep life had been fabricated in a previous program. There were limited creep data for Mo/41Re and rhenium. The maximum temperature of the available data was 4100°F. Extrapolation was required for heat exchanger temperatures of >5000°F.

Second, a sealed cavity resulted in exposing the heater wire to N<sub>2</sub>H<sub>4</sub> decomposition products at temperatures for which there was an inadequate data base. Previous vented cavity designs (i.e. MR-501) only exposed the filament to vacuum. Sublimation of tungsten into a vacuum was reasonably well characterized by incandescent lamp industry data. The decomposition products of N<sub>2</sub>H<sub>4</sub> included H<sub>2</sub>, N<sub>2</sub>, and NH<sub>3</sub>, as well as trace quantities of H<sub>2</sub>O, CO<sub>2</sub> (as carboxic acid), and O<sub>2</sub>. The effects of these species on the hot filament were totally unknown. In addition, the expected wire temperature of up to 5300°F was higher than previous designs and was close to the thermal failure limit of the stabilized tungsten wire material.

Finally, there was no seal design to implement a sealed cavity. Such a seal must endure the temperature of the thruster, withstand thermal cycling over the multiple year life of the thruster, form an interface between ceramic insulating materials and refractory metals, and seal decomposition gases within the thruster body.

Another objective of this phase was to determine the needs of the user community for low thrust moderate power thrusters. Spacecraft power availability, Isp and thrust requirements, life, and thruster physical characteristics (weight and envelope) were needed to allow thruster optimization. A preliminary performance specification was to be developed using input obtained from spacecraft prime contractors.

Candidate design solutions were identified and evaluated. These and the mission survey results are presented in the following sections.



BASELINE DESIGN IMPROVEMENT FEATURES  
HIGH PERFORMANCE RESISTOJET

This page contains RRC proprietary information

## 2.1.1 Mission Analysis — Performance Specification

A mission analysis questionnaire was prepared and sent to numerous spacecraft prime contractors. Table 2-1 presents the questions that were included in the mission questionnaire. Table 2-2 presents the responses received from the prime contractors. The responses were reviewed and an attempt was made to summarize a consensus position. This effort resulted in the top level performance specification presented in Table 2-3.

**Table 2-1  
HIGH PERFORMANCE RESISTOJET  
APPLICATION SURVEY QUESTIONNAIRE**

- Near-term mission parameter questions:
  1. On-orbit spacecraft dry mass
  2. Stationkeeping maneuver  $\Delta V$
  3. Simultaneous stationkeeping and attitude control (Y/N)
  4. Off-pulsing for attitude control (Y/N)
  5. Stationkeeping propellant load
  6. Initial on-orbit feed pressure
  7. Spacecraft positional or pointing accuracy
  8. Desired thrust level or maximum maneuver duration
  9. Electric power availability
  10. Peak battery voltage and bus voltage letdown characteristics
  11. Special thruster weight, size or interface requirements
  12. Nominal mission duration

## 2.1.2 Thruster Analysis

Using the performance specification and starting with the baseline design a series of analyses were completed to form the basis of the conceptual design of the thruster to be carried into the next phase. The following sections summarize these analyses.

### 2.1.2.1 Heater Sizing

The determination of heater envelope and operating characteristics were key to achieving the desired level of performance. Sizing the heater involved a large number of variables and assumptions including:

- voltage
- wire diameter and length
- electrical power and resistance
- wire temperature
- major and minor filament diameter
- filament geometry
- filament view factors
- wire and cavity thermal emissivity
- cavity wall temperature
- cavity diameter and length
- heat losses
- life effects
- thermal expansion

1.	DRY SPACER CRAFT MASS (LBM)	12-15,000	5-10,000	3500	8000	1875	2200	5060	3200	2700	
2.	DELTA-V (FT/SEC/YR)	TBD	1000	0.26	TOTAL PER EVENT	5.1	1373	164	164	11.55	Per Manv.
3.	STATIONKEEPING & ATT.	YES	YES	YES	YES	YES	YES	YES	YES	YES	YES
4.	OFF PULSING?	YES	NO	YES	YES	NO	YES	YES	YES	YES	
5.	PROPELLANT LOAD (LBM)	2-300	1% of Mo	31	100	600	360	860	720	700	
6.	INITIAL FEED SYSTEM PRESSURE (PSIA)	3-400	300	125	300	350-400	220	125	260	220	
7.	POSITIONAL & POINTING ACCURACY (DEGREES)	N/A	+1 LONG.	N/A	N/A	0.1	0.1	0.06	0.06	.05-.07	.05-.07
8.	TOTAL THRUST LEVEL (LBF)	0.05-0.5	0.1	1 - 3	1	1	N/A	N/A	.4-.6	.4-.6	
9.	ELECTRICAL POWER	N/A	1 kW	DEPENDS ON BENEFITS	SOLAR ARRAY TRADES	1075-1275	PEAK 1500-1500	1500-2800	PEAK 3400	PEAK 3000	WATT WATT
10.	BATTERY VOLTAGE (VDC)	24-32	DESIGN DEPEND.	28 ± 6	30 R-HR	30 A-HR	2-46A-HR	28	4-60A-HR	2-27A-HR	100 100
11.	SPECIAL INTERFACES?	NOT AT THIS TIME	YES	NO	NO	NO	NO	NO	NO	NO	
12.	MISSION DURATION (YRS)	5-10	10	3-7 UP TO 10	8.86	+ 1 DRIFT	7-10	7-10	10	10	

## MISSION ANALYSIS SURVEY — QUESTIONNAIRE RESPONSES

Table 2-2

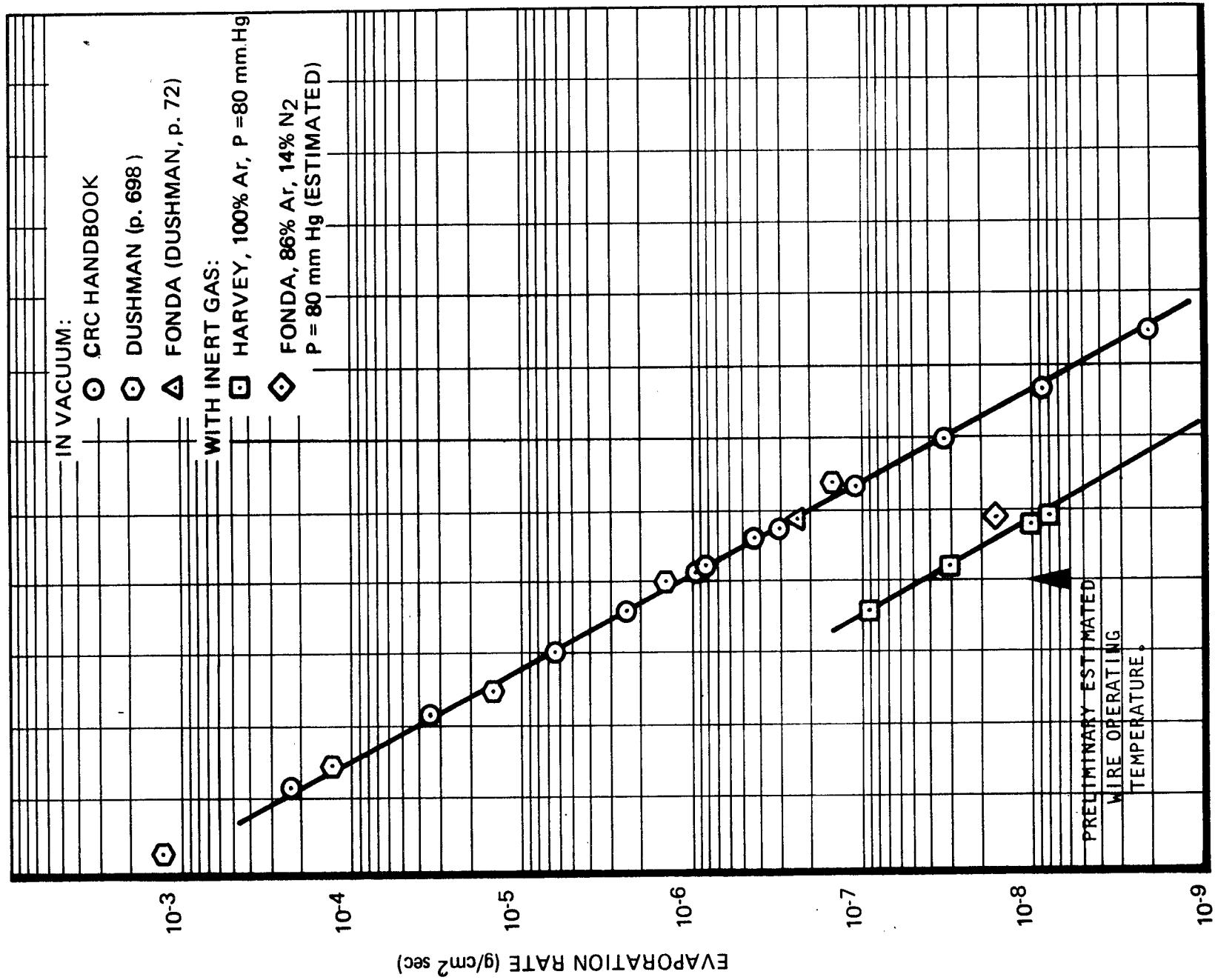
**Table 2-3**  
**HIGH PERFORMANCE RESISTOJET**  
**PRELIMINARY PERFORMANCE SPECIFICATION**

- Mission
  - N-S stationkeeping geosynchronous communication satellite
  - 10 years (250-hour total firing time)
  - 0.125-lbf nominal design point
  - 117,000 lbf-sec
  - 360 lbf-sec/lbm average over feed pressure blowdown range (300 — 100 psia)
- Thrust level
  - Direct operation from S/C bus. Simple current surge limiter circuit on start-up desirable but not required
  - Regulated or battery voltage let-down
  - 28 vdc nominal peak. adjustable as required
  - 1600 watt at  $P_f = 300$  psia; 1400 watt at 100 psia
- Total impulse
  - Blowdown 300 to 100 psia
  - Regulated 150 psia minimum
- Specific impulse
  - $\pm 5\%$
  - Spectrum  $0.2 \text{ g}^2/\text{Hz}$ , 20 — 20,000 Hz
  - Overall 13.0 g rms
- Power requirement
  - Spacecraft heat rejection
  - TBD
- Propellant pressure
  - Thrust repeatability
  - Vibration
  - Spacecraft heat rejection

Tungsten filament vaporization studies were initiated as part of the heater sizing effort. The need to reduce filament sublimation rate drives the design requirement to have heater cavity seals. If the vacuum sublimation rates were excessive at the desired element operating temperatures, a protective atmosphere would be required around the heater element to limit loss of tungsten from the wire. Figure 2-2 is a compilation of measured test results of tungsten filament sublimation rates as a function of temperature. The upper curve shows data in a vacuum, and the lower curve shows data obtained with a protective atmosphere of argon at a pressure of 80 torr.

Using a predicted temperature of 5000°F, the vacuum sublimation rate indicated is  $1.2 \times 10^{-6}$  grams/cm<sup>2</sup>-sec. With a filament 0.030 inch in diameter, the wire would be consumed in under 100 hours. With argon at 80 torr, the sublimation rate drops to  $4.5 \times 10^{-8}$  grams/cm<sup>2</sup>-sec resulting in about a 7% diameter decrease in 300 hours. This indicates that seals and a protective atmosphere are needed to sustain high performance operation for the desired lifetimes.

MEASURED TUNGSTEN FILAMENT EVAPORATION RATES



11176-87

$1/T \times 10^4 (\text{°K}^{-1})$

Figure 2-2

2-6

Mission analysis results suggested 125 milbf thrust as a nominal value. Preliminary thermal analyses were conducted to estimate heat exchanger cavity temperature profiles for various performance levels. The performance goals and temperature profiles were incorporated into an analysis to determine packaging configurations and heater wire temperatures for a range of heater surface areas. The results of the heater sizing are shown in Figure 2-3. Curves for three different performance levels are shown. Two curves are shown at each performance level to account for view factor uncertainty (F<sub>cc</sub>). Specific configurations that were optimized by the heater sizing program are shown as data points. Wire diameter, the number of filaments, and whether the configuration is parallel filament or double helix are noted. As shown, wire temperatures required for 388 seconds Isp are too close to the melting point of tungsten. The wire temperature could be decreased by increasing the heater cavity length and diameter, but thermal losses would increase with size and unreasonable amounts of power would be required. Wire temperatures around 5000°F associated with 359 sec Isp are more reasonable for an initial design point. However, wire creep in a 1-g environment may limit the amount of ground test time the thruster can be subjected to in one position.

The plot in Figure 2-4 was used to size the heater filament by defining the wire surface area required in the rhodium heat exchanger cavity to achieve a range of Isp values. This analysis assumes a wire diameter of 0.025 inch. The drawing in the figure illustrates two heater filament options that contain the desired wire surface area and fit in the envelope.

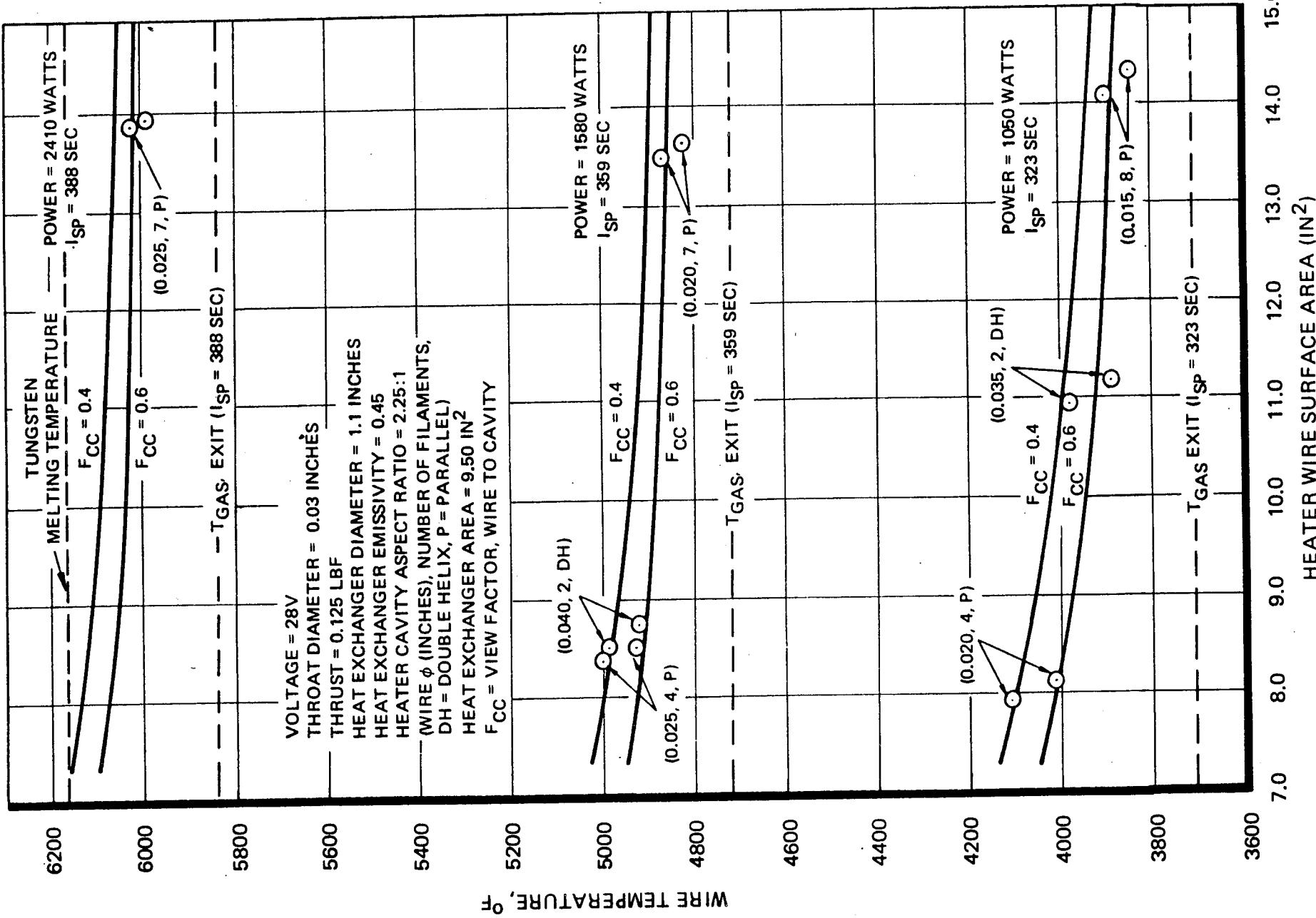
### 2.1.2.2 Nozzle Design

To support the selection of a thruster nozzle configuration, a study was performed that included analysis of several configurations validated with cold gas testing.

A trumpet nozzle and a MR-501 nozzle were modelled with the Two Dimensional Kinetics (TDK) code. The TDK code calculates a pressure profile in the chamber and throat using isentropic flow relations. Nozzle pressures are calculated with a numerical integrator. The nozzle pressure profile is subsequently used in a method of characteristics sub-routine that calculates divergence and viscous losses. Preliminary results indicated 6% better performance for the trumpet nozzle than for the MR-501 nozzle. A preliminary throat diameter of 0.035 inch was selected.

Four nozzle configurations were fabricated including two trumpet shaped, a conical, and one bell shaped design. Throat Reynolds numbers from 500 to 9000 were tested with H<sub>2</sub> and N<sub>2</sub> to validate the model predictions. The results of this test were reported separately (Grisnik, 1987). Figure 2-5 illustrates the nozzle configurations tested and Table 2-4 presents dimensions and performance values. Figure 2-6 is a combined analytical and experimental data plot of Isp efficiency as a function of Reynolds number. Isp efficiency is defined as the ratio of measured specific impulsed (thrust/flow) to theoretical maximum specified impulse (300 for H<sub>2</sub> and 78 for N<sub>2</sub> at room temperature). The Isp efficiency increases for increasing Reynolds number for all of the nozzles. All of the nozzles gave the same performance within experimental error. The analytical results proved to be a rather poor predictor of measured performance.

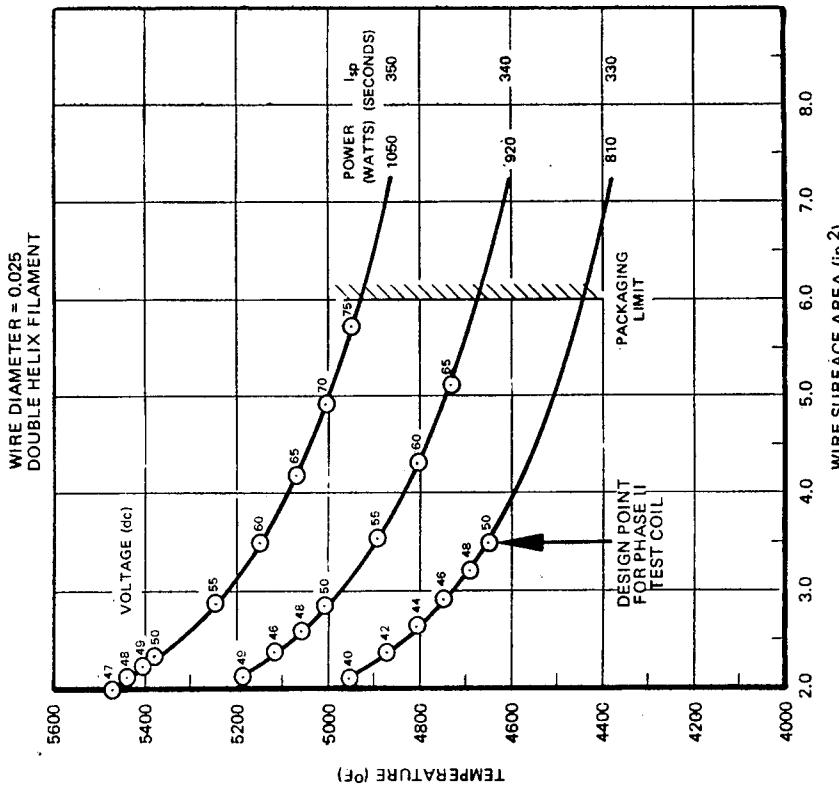
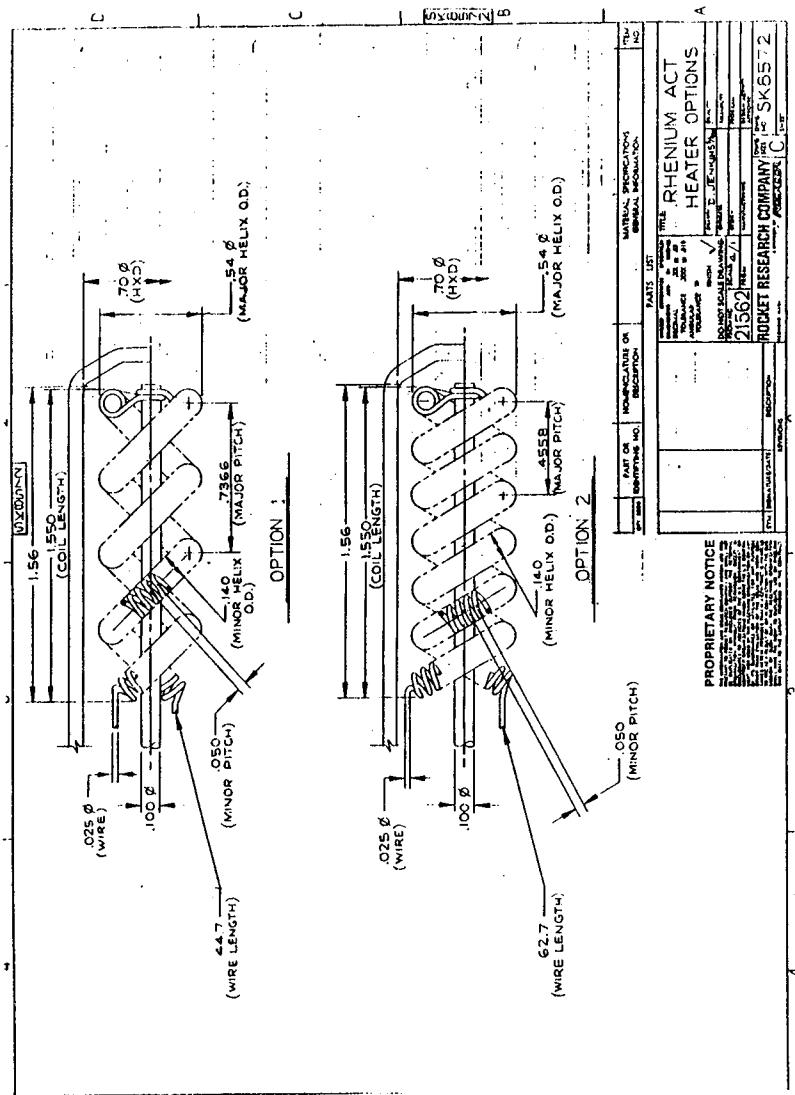
## HIGH PERFORMANCE RESISTOJET HEATER SIZING STUDIES



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Figure 2-3  
2-8

# RHENIUM ACT HEATER SIZING STUDY AND OPTIONS FOR FABRICATION



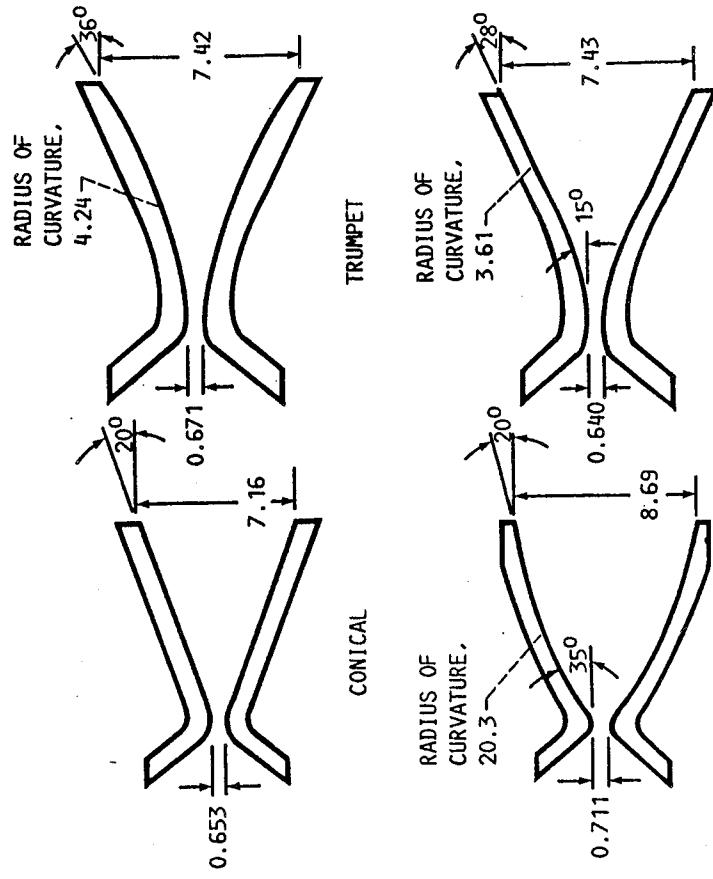
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2-9

Figure 2-4

## NOZZLE GEOMETRIES

**All Dimensions are in Millimeters**



11232-16

### 2.1.2.3 Configuration Trade Study

A structural and performance trade-study was conducted to determine a suitable thruster operating pressure ( $P_c$ ) range with acceptable heat exchanger creep margins. Figure 2-7 illustrates the effect of throat diameter on other performance parameters over a range of  $P_c$  values. The figure shows beginning-of-life (BOL), middle-of-life (MOL), and end-of-life (EOL) conditions for 3:1, 2:1, and 1.5:1 feed pressure blowdown schedules. The MOL design point is 125 mlbf thrust at 365 sec Isp and when 50% of the mission propellant has been consumed.

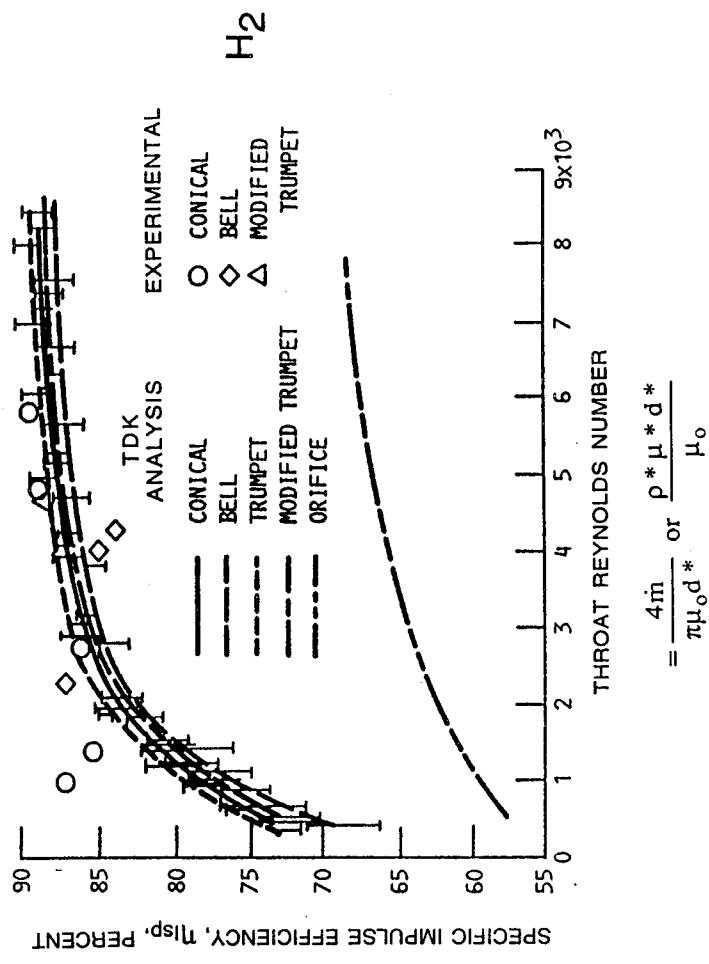
Figure 2-5

Creep effects drive operating conditions toward large throat sizes and lower pressures. The preliminary results predicted Isp values of up to 372 sec with thrust from 92 to 197 mlbf, gas temperatures from 4100° to 5000°F, and power from 1300 to 1650 W for a 250-hour mission. Creep of the outer heat exchanger body was the primary performance limiting feature. Thermal analysis at an Isp of 369 sec (exhaust gas almost 5000°F) yielded the following predicted temperatures: heater filament, nearly 5300°F; outer body, greater than 4800°F; metal to ceramic seal, about 1600°F; and throat, almost 3500°F. These temperatures were compared to the best available creep data using worst-case pressures. Rhenium appeared acceptable in terms of creep, rupture, and margin on rupture. Lower creep values are obtained

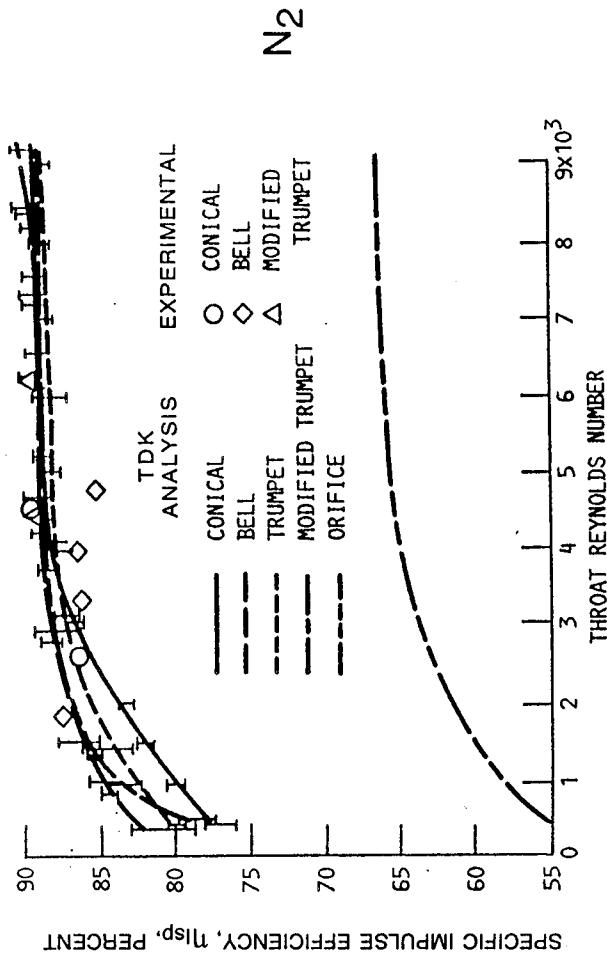
**NOZZLE PARAMETERS AND CHARACTERISTICS AT A REYNOLDS NUMBER OF 1000 FOR NITROGEN AND HYDROGEN**  
**Table 2-4**

Nozzle	Shape	Exit Half-Angle	Throat Diameter (mm)	Area Ratio (N <sub>sp</sub> )	Specific Impulse Efficiency (CD)	Discharge Coefficient	Specific Impulse Efficiency (N <sub>sp</sub> )	Thrust Coefficient (CF)	Nitrogen	Hydrogen	Nitrogen	Hydrogen	Nitrogen	Hydrogen
									Throat	Exit Plane	Nitrogen	Hydrogen	Nitrogen	Hydrogen
1	Cornical	20°	20°	0.653	120:1	0.87	0.88	0.80	0.78	1.18	1.17	1.17	1.14	0.94
2	Bell	35°	20°	0.711	150:1	0.91	0.92	0.85	0.75	1.31	1.31	1.31	1.24	1.20
3	Trumpet	0°	36°	0.671	125:1	0.88	0.86	0.83	0.78	1.24	1.24	1.24	1.14	0.97
4	Modified Trumpet	15°	28°	0.640	135:1	0.93	0.93	0.84	0.76	1.33	1.33	1.33	1.24	0.97
	Office plate	—	—	—	—	0.98	0.94	0.58	0.59	0.97	0.97	0.97	0.94	—

SPECIFIC IMPULSE EFFICIENCY AS FUNCTION OF  
THROAT REYNOLDS NUMBER FOR  $gH_2$  &  $gN_2$ . EXPERIMENTAL  
DATA AND TWO-DIMENSIONAL KINETICS (TDK) NOZZLE ANALYSIS



$$= \frac{4\dot{m}}{\pi\mu_0 d^*} \text{ or } \frac{\rho * \mu * d^*}{\mu_0}$$



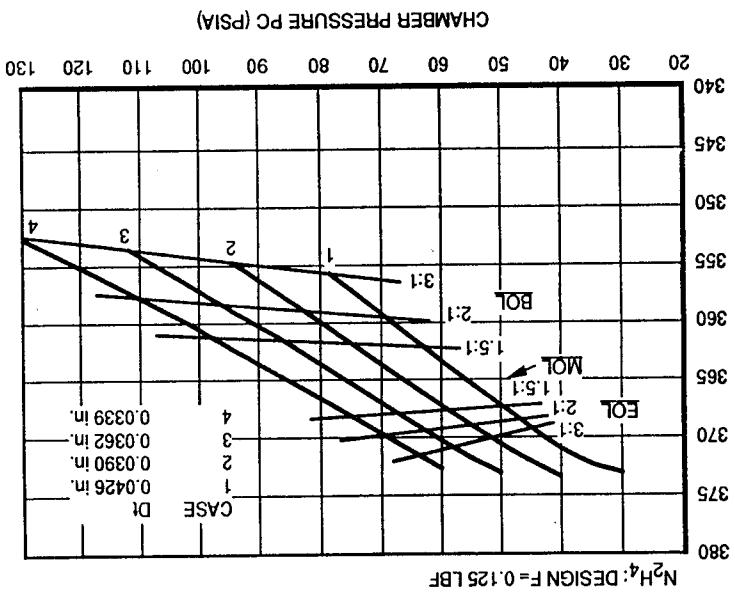
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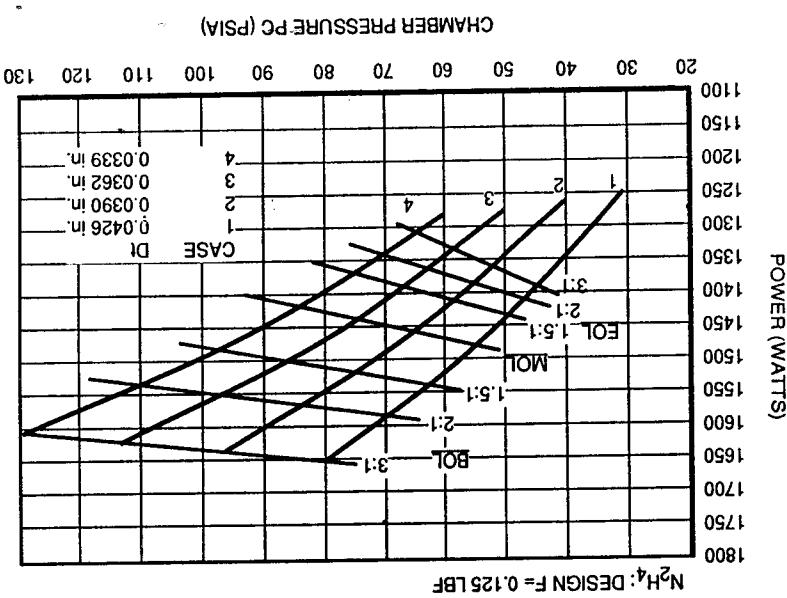
Figure 2-6

## THRUSTER OPTIMIZATION TRADE STUDY RESULTS

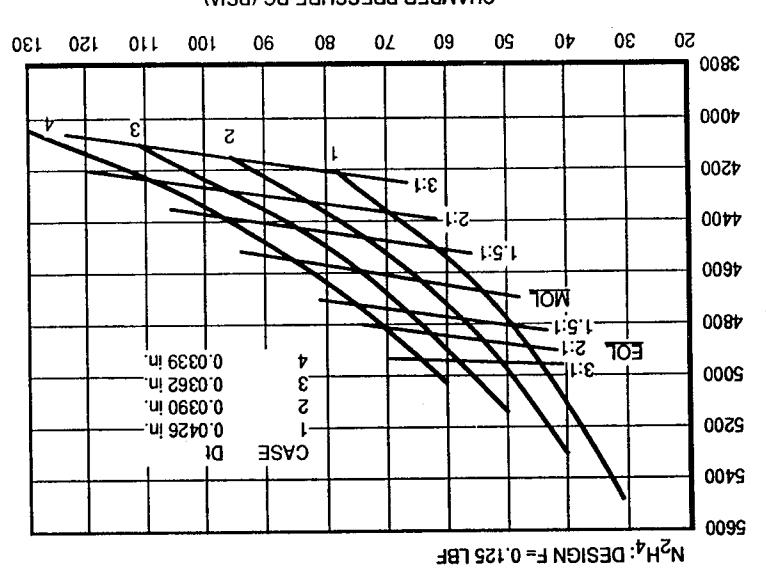
**ISP VS CHAMBER PRESSURE**



**HEATER POWER VS CHAMBER PRESSURE**



**GAS TEMPERATURE VS CHAMBER PRESSURE**



**Figure 2-7**

for W/25Re, but rupture, and margin on rupture are unacceptable. Tungsten was acceptable for creep, rupture and margin on rupture, but there were concerns regarding manufacturability..

An option considered was a super-heater which is shown in Figure 2-8 along with a conventional configuration. Thermal results also presented in the figure show no significant advantage for the super-heater. The super-heater requires more power due to increased thermal losses caused by greater HX diameter. Due to higher losses and greater complexity the super-heater concept was rejected.

### 2.1.3 Material Characterization

Several test series were needed to obtain data to complete the conceptual design. The approaches taken are presented below.

#### 2.1.3.1 Creep

Testing to confirm creep characteristics was planned for samples of hot isostatic pressed (HIP) rhenium, CVD rhenium, and W/25Re. Several laboratories were approached to conduct high temperature material characterization including:

Battelle Laboratories  
Westinghouse  
Oak Ridge National Laboratory  
Lawrence Livermore Laboratory

Los Alamos Laboratory  
Olin Materials Research Laboratory  
NASA-LeRC

There were few responses to the solicitation to perform this testing. Battelle's maximum temperature capability for material testing was reported to be 4700°F. However, long-term creep testing could only be conducted at 4000°F. The minimum temperature acceptable for testing was 3850°F. Battelle's furnace turned out to only be capable of 3450°F. This eventually led to termination of this test effort.

Without additional creep data, rhenium remained the material of choice for the heat exchanger due to superior joinability as compared to tungsten. The rhenium heat exchanger, fabricated in a previous program, was made available for testing.

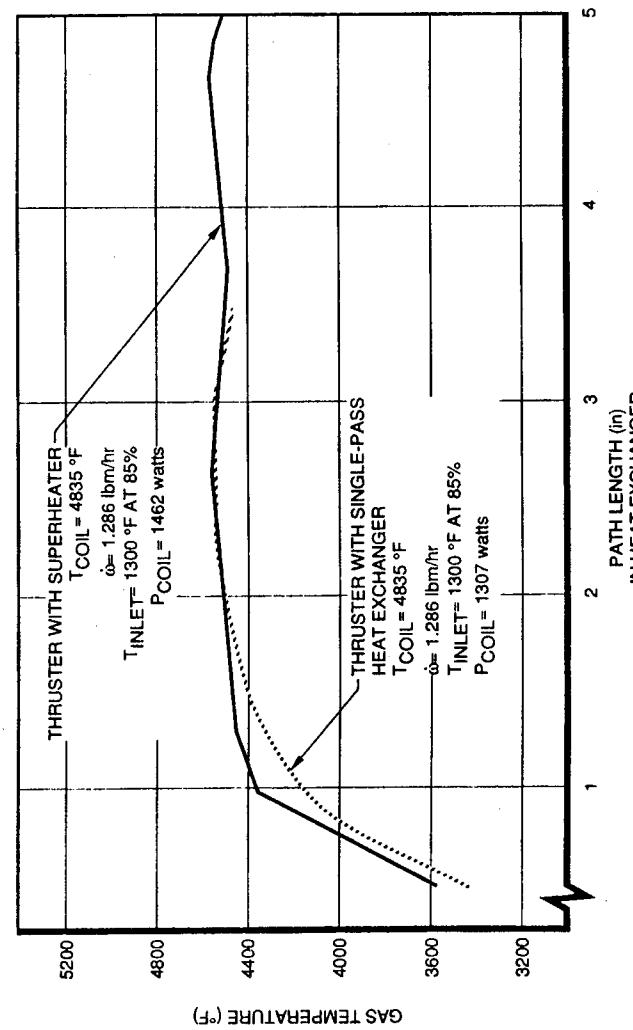
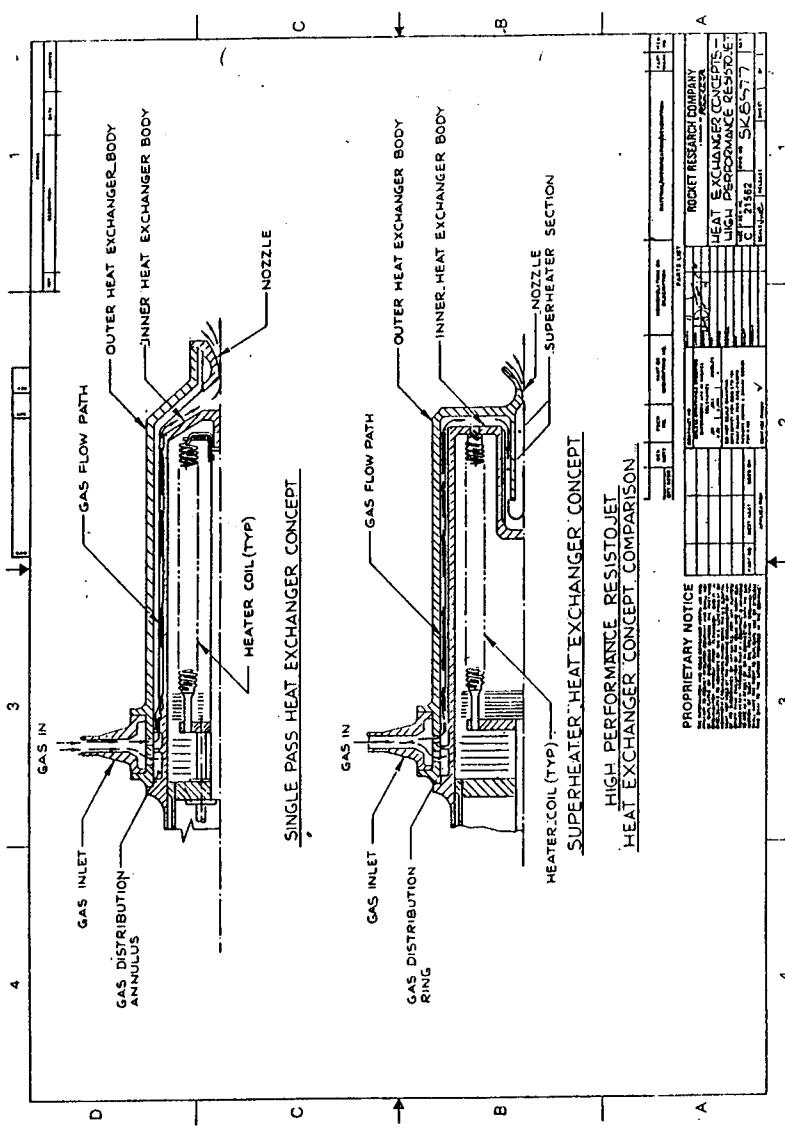
#### 2.1.3.2 High Temperature Filament Wire Exposure

Figure 2-9 illustrates the fixture fabricated to expose filament wire at high temperature to N<sub>2</sub>H<sub>4</sub> decomposition gases. The wire was tested using 1 hour on periods with 10 minute cooling periods between cycles. The temperature and pressure conditions in the chamber simulating thruster operation during life are presented in Table 2-5. Fifty cycles were performed at each test condition. Figure 2-10 shows the overall test apparatus.

**Table 2-5**  
**HEATER WIRE EXPOSURE TEST CONDITIONS**

Hours	ALL FAILURES				
	1-50	51-100	101-150	151-200	201-250
Wire Temp °F	4640	4815	4990	5135	5280
Chamber Pressure (psia)	80	72	63	54	46

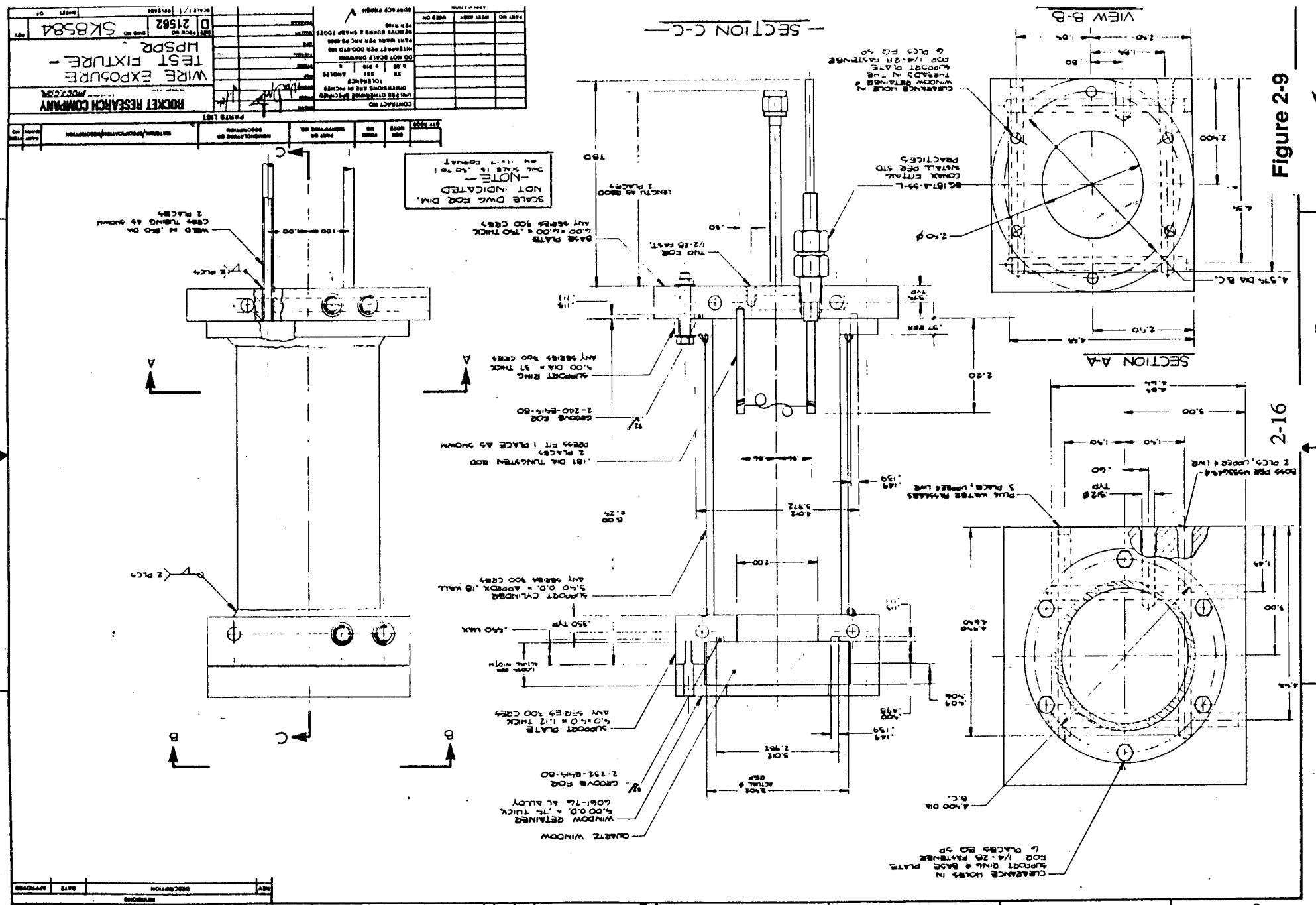
# HIGH PERFORMANCE RESISTOJET COMPARISON OF TEMPERATURE PROFILES FOR SINGLE-PASS AND SUPERHEATER HEAT EXCHANGERS



11230-20

2-15

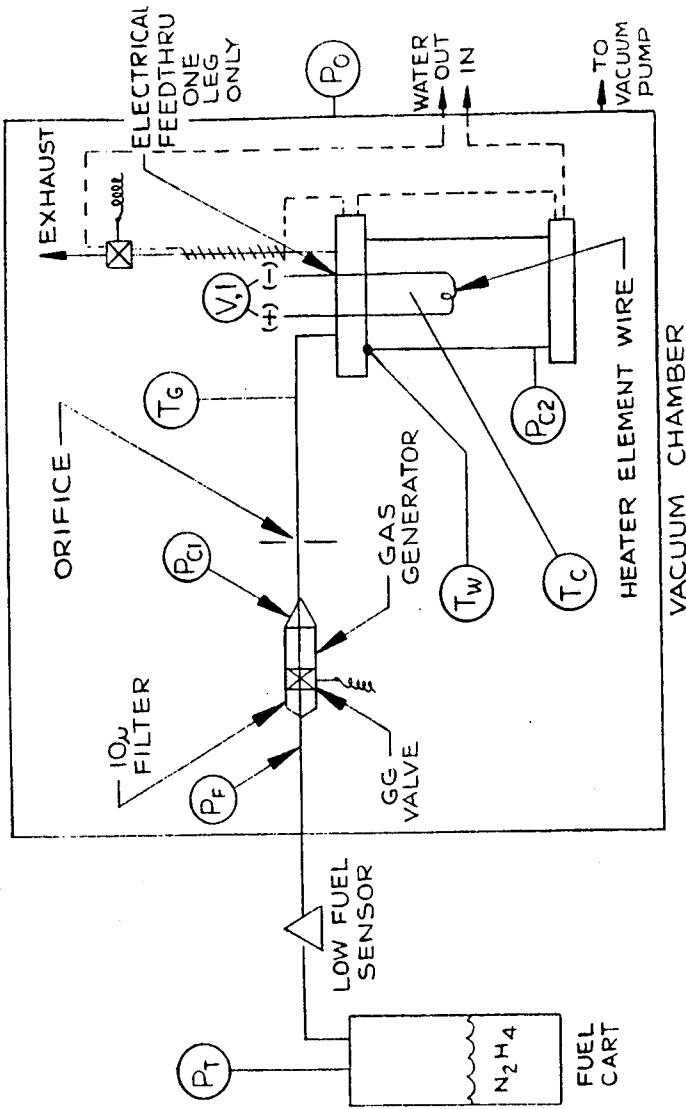
Figure 2-8



**Figure 2-9**

2-16

## HEATER WIRE EXPOSURE TEST SCHEMATIC



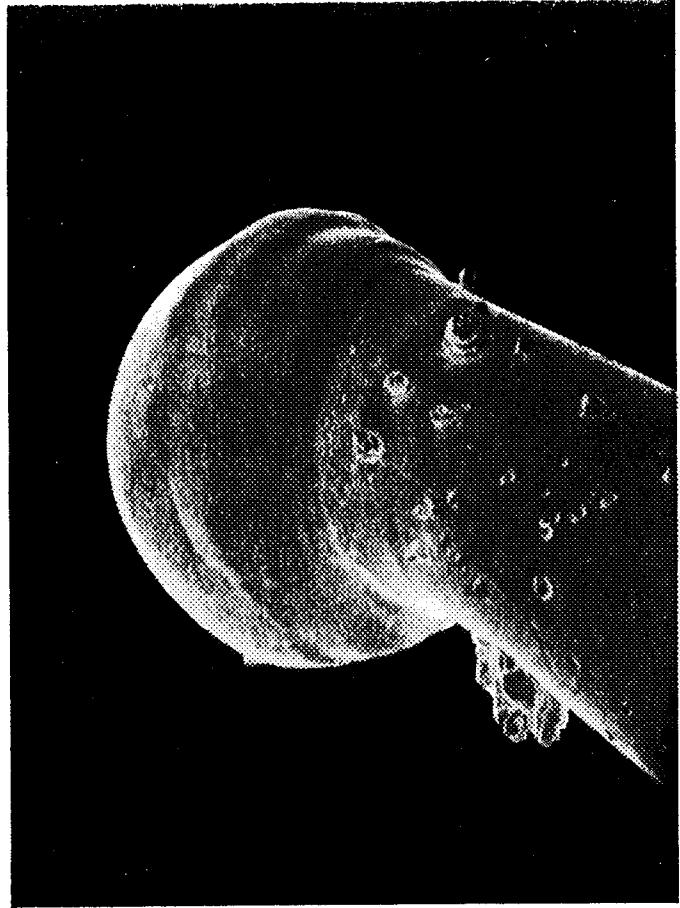
**11186-70A**

**Figure 2-10**

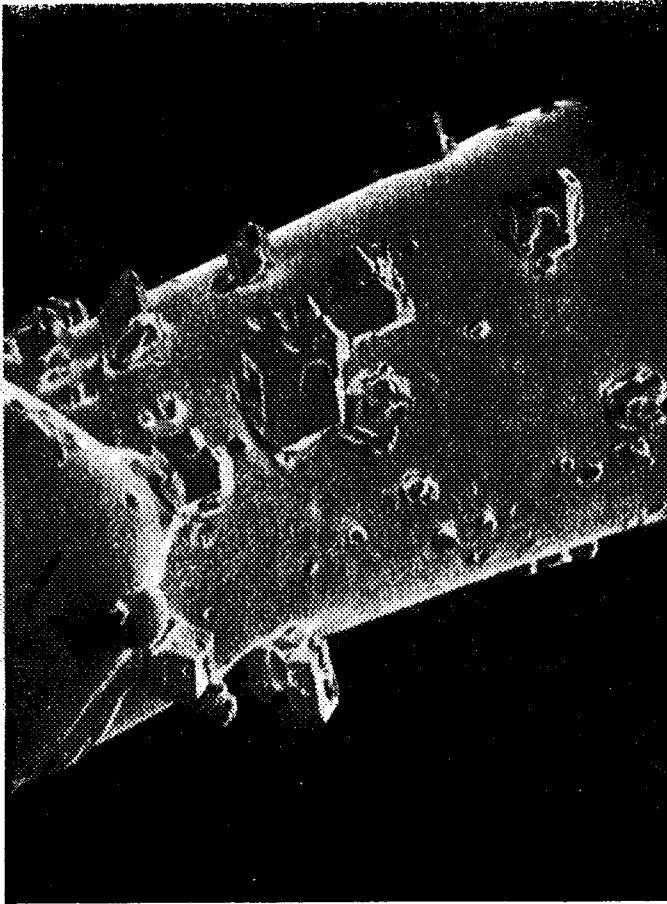
Three separate identical wire samples were tested. All three tests were terminated after approximately 120 cycles, short of the 250 cycle goal. In the first sample a visible kink developed in the wire near one of the two support posts and the test was terminated after 119 cycles. The second sample test was terminated when the wire failed open after 116 cycles. The third sample, which had a thermal expansion loop incorporated, was terminated when the wire failed open after 100 cycles. There was water condensation within the test apparatus in all cases.

Two wires were inspected using a Scanning Electron Microscope (SEM). Figure 2-11 shows one of the failed ends from each of the two samples. No evidence of chemical attack of the wires was detected during SEM examination. The primary failure mode appeared to be related to thermal stresses which severed some of the fibrous tungsten strands locally reducing the cross-sectional area and finally melting the wire. Chemical transport of tungsten atoms may have had a small secondary role as the temperature of the wire reached the melting point. Tungsten crystals renucleated from the vapor phase can be seen deposited on each of the wires in cooler regions away from the melted zone.

The conclusion drawn from this test was that heater filaments with the configuration tested will exhibit lifetimes of approximately 100 hours when operated in  $N_2H_4$  decomposition environments up to temperatures of  $5000^{\circ}F$ . This was only 40% of the lifetime goal. Figure 2-3 indicates an Isp value of  $<360$  sec associated with this temperature. The water condensate may have also played a role in accelerating these failures.



WIRE EXPOSURE TEST NO. 3  
SEM PHOTO OF FAILED TUNGSTEN WIRE AFTER 100 HOURS AT  
TEMPERATURES TO 5000°F



WIRE EXPOSURE TEST NO. 2  
SEM PHOTO OF FAILED TUNGSTEN WIRE AFTER 116 HOURS AT  
TEMPERATURES TO 5000°F

### 2.1.3.3 Heater Filament Design Support Tests

A wire sag test at 5300°F was performed. Figure 2-12 illustrates the configuration of the test fixture and one result. Test filaments were briefly flashed to 5400°F to recrystallize the microstructure. A theodolite was used from outside the test chamber to measure the amount of wire sag. The pointed tungsten spike in the middle of the test assembly serves as a spatial reference. Tests were conducted horizontally and vertically. Tests of 25 minutes at 5300°F resulted in a maximum deflection at the center of the filament of 0.060 inch. This amount of heater filament distortion was acceptable for the heater and heat exchanger configuration chosen.

Another critical design feature was the heater element wire transition joint. A sketch of the transition between the wire and the tungsten support post is shown in Figure 2-13. The transition joint was mounted to a hot swaged joint fixture illustrated in Figure 2-14. This assembly was fabricated, heated in a furnace simulating an acceptance test firing to cause diffusion bonding, vibrated to simulate launch, and checked for electrical continuity. The simulated joint was sectioned and the microstructure examined. No problem areas were observed.

### 2.1.3.4 Heat Exchanger Emissivity Samples

A trade study indicated performance benefits could be derived by increasing the efficiency of the heat exchanger. One approach considered was to increase the inner surface emissivity to enhance radiative heat transfer from the heater filament. Fabrication and evaluation of emissivity samples was performed.

Flat wafer refractory metal samples were machined with grooves 0.009 inch deep, 0.006 wide at the top and 0.012 inch apart to increase surface area. Other samples were machined and etched or coated with CVD tungsten. Based on NASA-LeRC reflectivity measurements for wavelengths from 0.2 to 2.5 microns the CVD tungsten surface performed best, the V-groove surface next, and the machined and etched surface performed worst. SEM analysis of a grooved and CVD coated heat exchanger indicated the coating process produced acceptable results with an even distribution of fine tungsten grains.

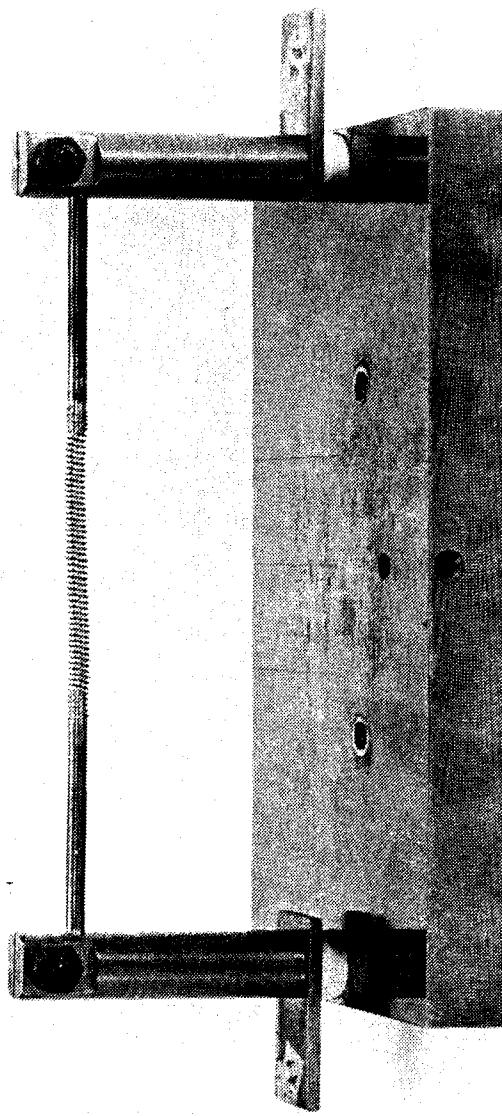
This activity was completed at the end of Phase I. Figure 2-15 presents the results of this effort.

### 2.1.3.5 Cavity Seal Tests

The primary cavity seal approach was compression of boron nitride (BN) powder. The BN powder was compressed between two ceramic pistons around the electrode post. The seal is to contain heater cavity pressurant gases while maintaining an electrical potential difference between the electrode and support structure. A fixture was designed to test the concept prior to committing to thruster hardware, as illustrated in Figure 2-16. The fixture was designed to operate at 1600°F. The fixture was pressurized with gH<sub>2</sub> or gN<sub>2</sub>, heated for 1 hour, then depressurized and cooled for 10 minutes. Seal helium leak rates were measured before and after 250 cycles. Testing was not completed in Phase I, but this activity was carried forward into Phase II. Results are presented in subsequent sections.

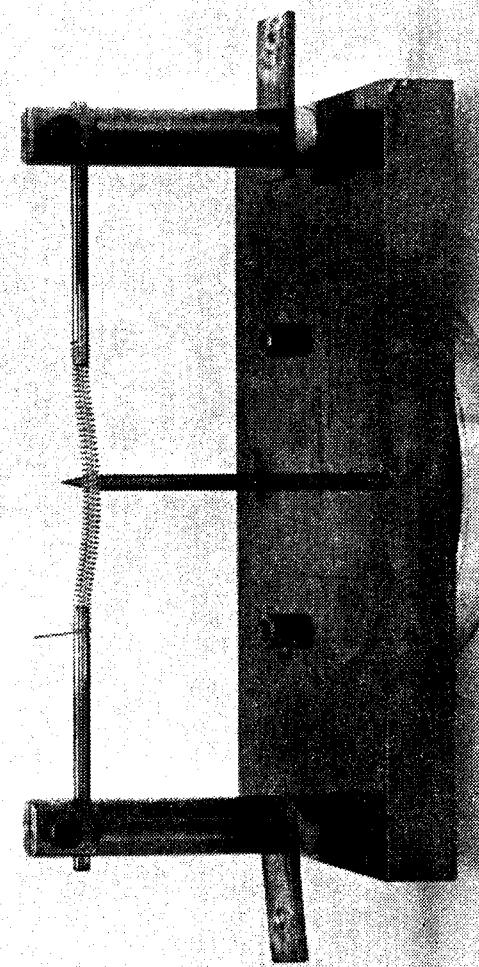
## 5300°F WIRE SAG TEST FIXTURE AND RESULTS

BEFORE



3037-3

AFTER



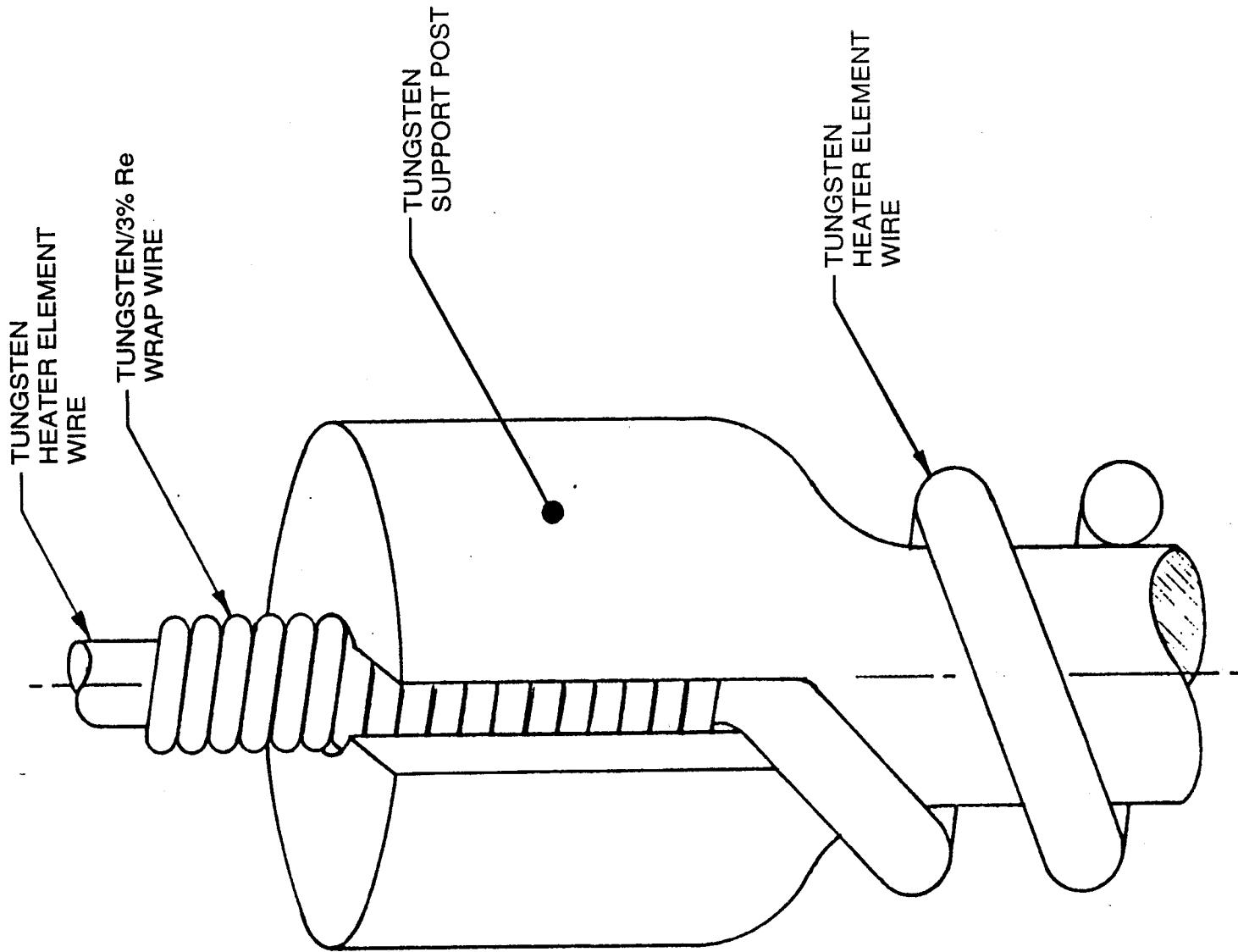
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2-20

Figure 2-12

**HEATER ELEMENT WIRE/SUPPORT POST HOT SQUEEZE JOINT**



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2-21

**Figure 2-13**

**Figure 2-14**

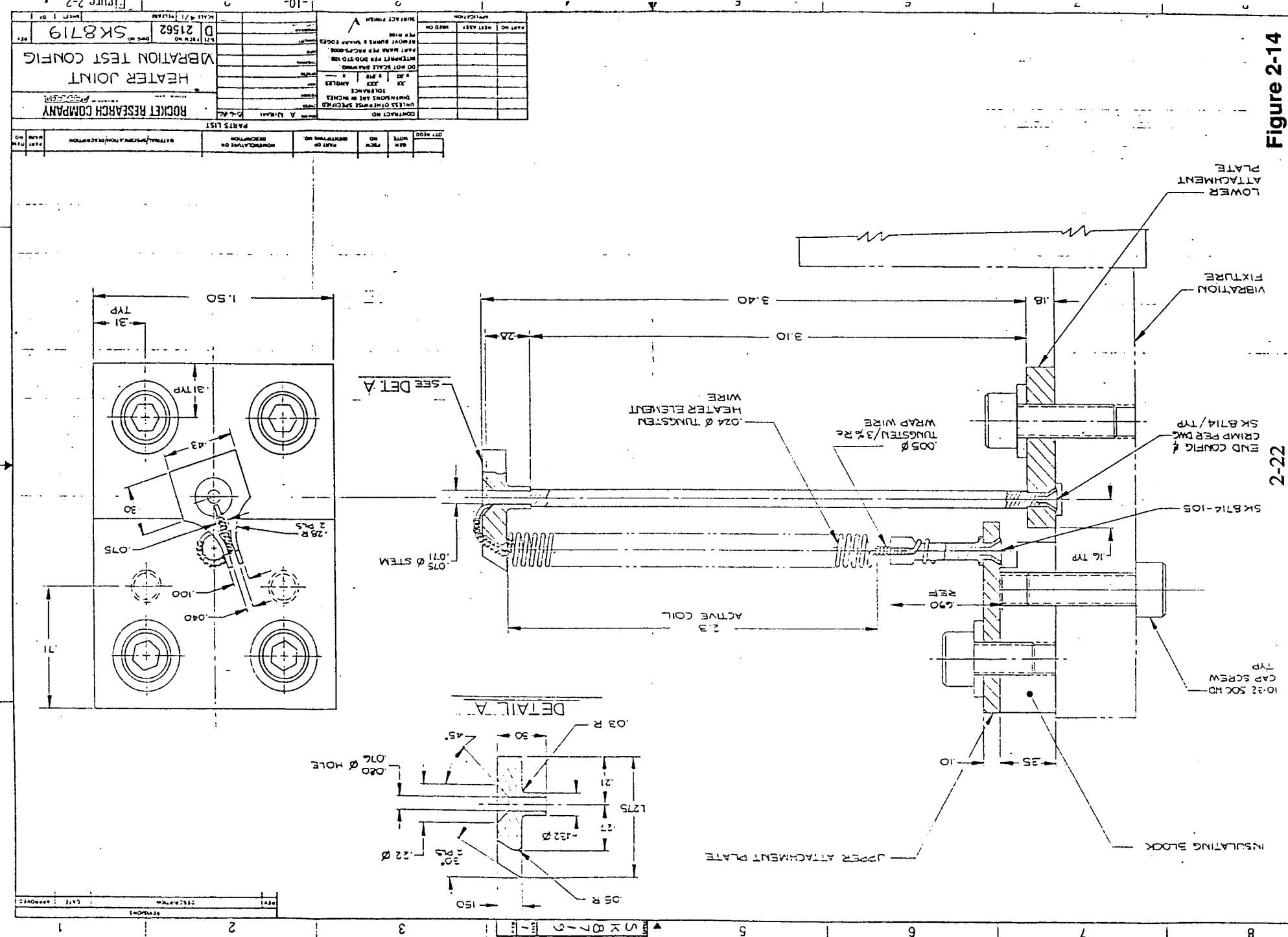


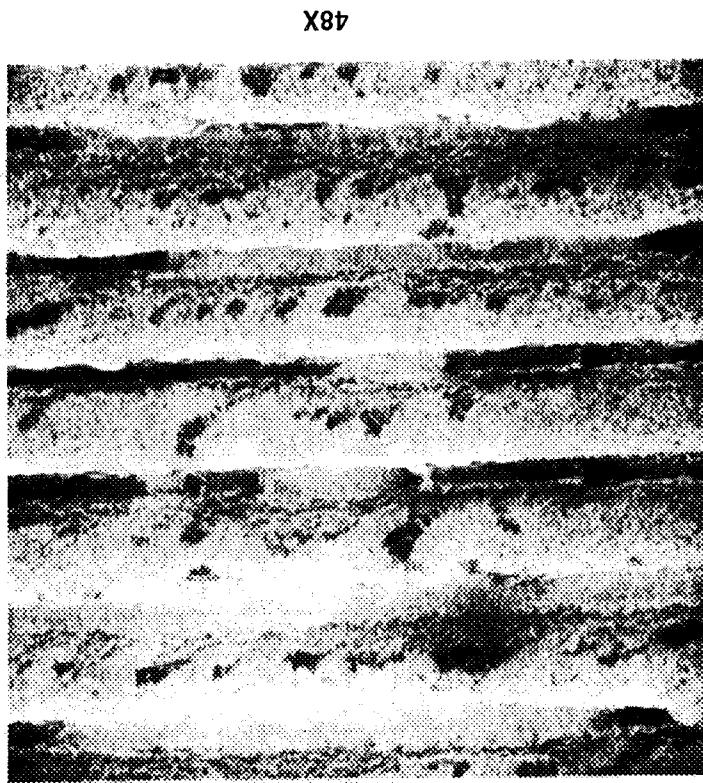
Figure 2-15

2-23

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HEATER CAVITY SURFACE ENHANCEMENT  
SAMPLE EMISSIVITY MEASUREMENT RESULTS

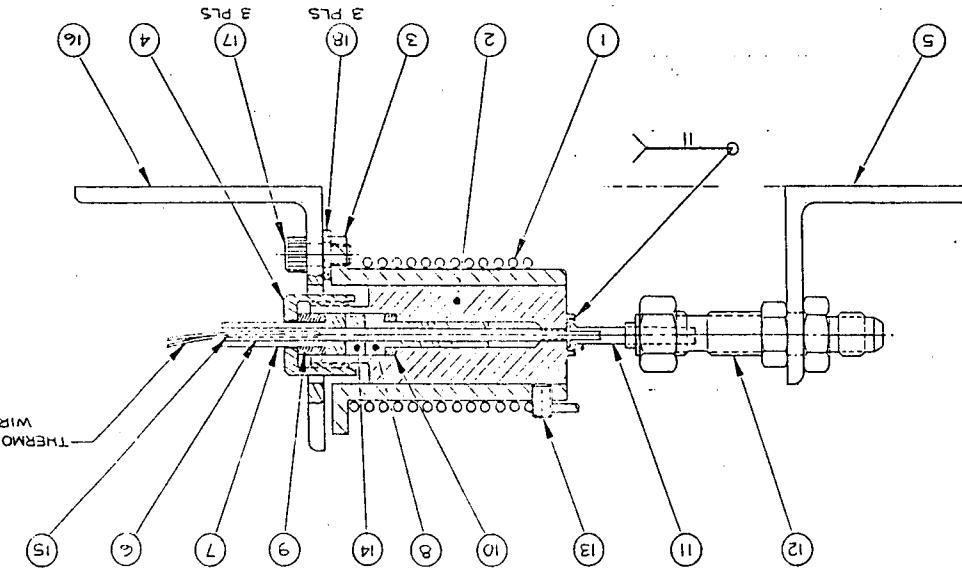
Sample No.	Surface Preparation	Emissivity
1	AS-MACHINED LIGHT GROOVES	0.32
2	AS-MACHINED SMOOTH	0.34
3	GROOVED	0.34
4	ETCHED	0.49



SEM PHOTOMICROGRAPH OF GROOVED AND CVD TUNGSTEN COATED CYLINDER INTERIOR SURFACE

Figure 2.16

2-24



### 2.1.3.6 Results and Conclusions from Phase I

At the end of Phase I an update to the baseline thruster design was completed. The overall configuration is illustrated in Figure 2-17. The heater design is illustrated in Figure 2-18. The heat exchanger design is illustrated in Figure 2-19. This configuration represented the best compromise of the design parameters developed in the configuration study. It was a sealed cavity design with a 1000 W nominal heater power and a rhodium heat exchanger. In most important respects it was very similar to the MR-501 configuration.

The gas generator and mount structure configuration, shown in Figure 2-17, was identical to the production design. The electrical passthrough area was different to assure that the cavity surrounding the heater was sealed. The details of these seals are discussed in subsequent sections. The nozzle shown is a trumpet configuration. A conical nozzle was actually built.

The heater configuration, shown in detail in Figure 2-18, was different than production designs. The heater filament design was the result of an optimization of wire surface area, wire-to-cavity view factor, and heat exchanger cavity size. Four parallel filaments with a grain-stabilized tungsten wire diameter of 0.025 inch was determined to be optimum. The support posts are shown hot swaged into place. More details of the post attachment will be discussed in subsequent sections. Ceramic insulators were included to allow the positive heater lead to pass through the support assembly which was at ground potential.

The sealed cavity heat exchanger, shown in Figure 2-19, was similar to production designs. The material was rhodium. The insulation system was improved to accommodate the higher temperature operation. The insulation is multi-layer molybdenum foil.

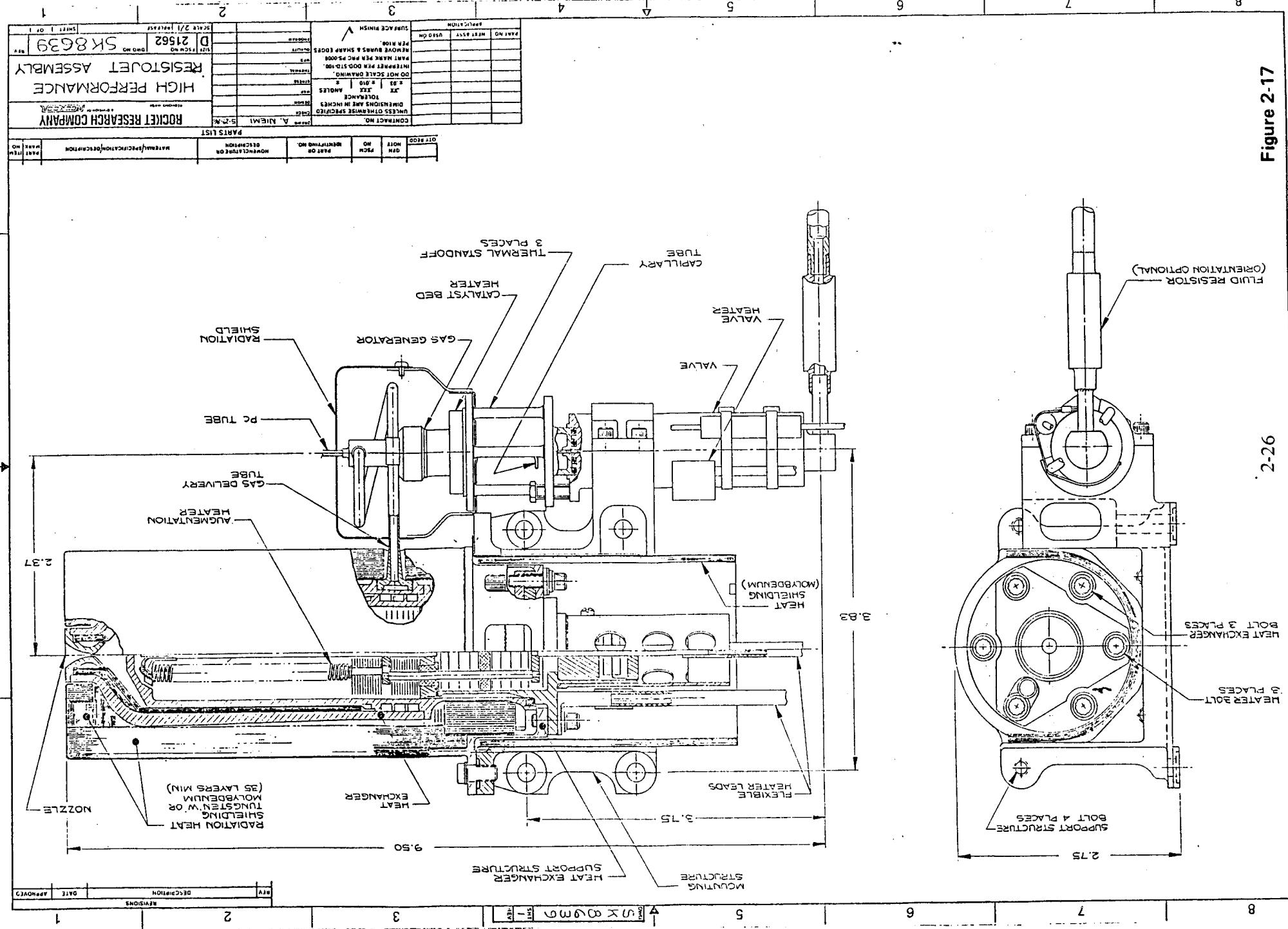
Two major areas of risk were left unresolved at this point in the program. First, the high temperature filament wire exposure tests suggested that an Isp of 360 seconds would be difficult to obtain with lifetimes greater than 100 hours. The test results had possible alternate explanations, such as water in the test fixture. However, the SEM analysis suggested that a thermal failure mechanism was most likely. Second, the inability to obtain creep data introduced risk that the heat exchanger would deform sufficiently to short out the wire.

## 2.2 PHASE II — RADIATIVE HEATER THRUSTER EXPERIMENTS

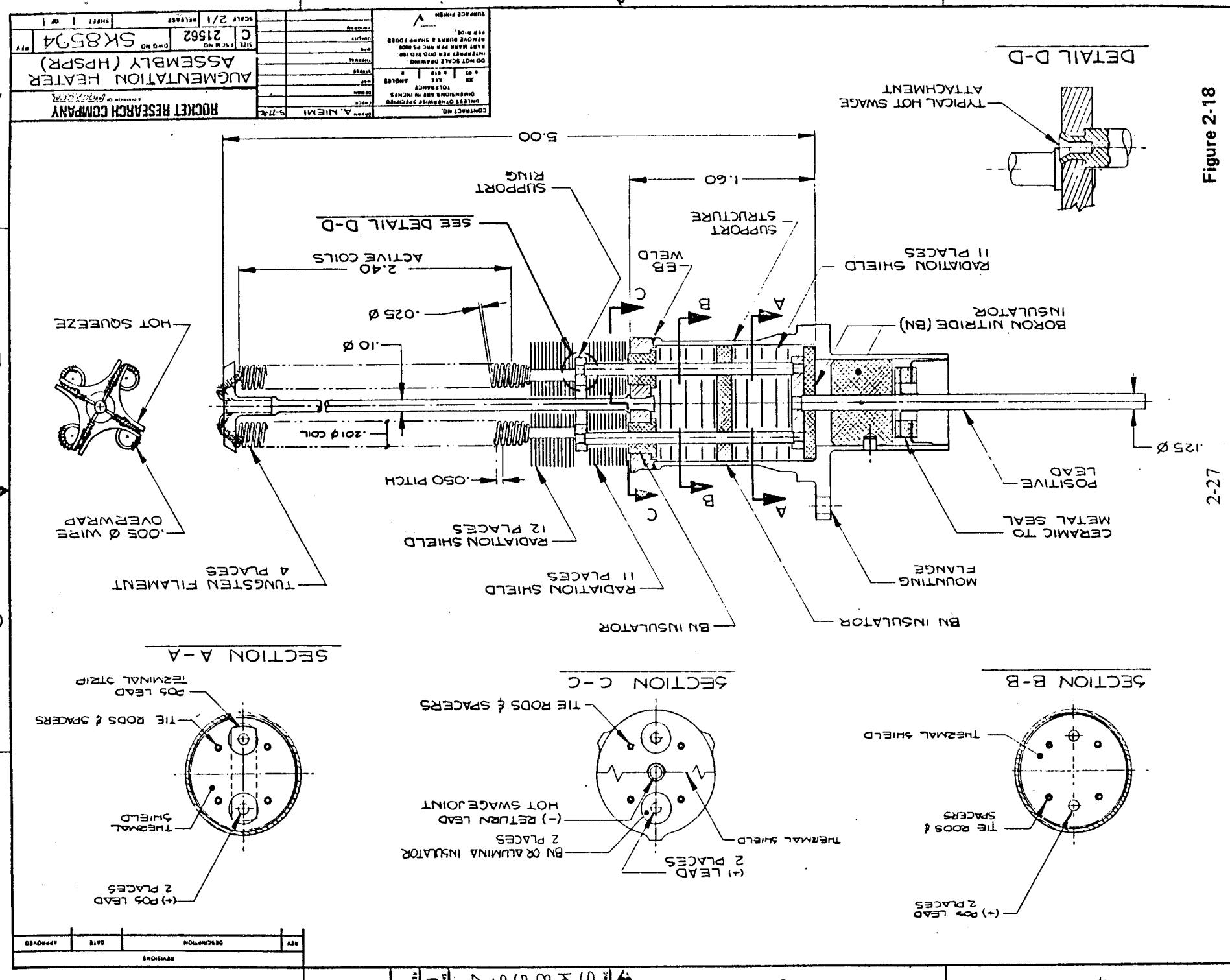
The original objective of Phase II was to design and conduct critical component level tests to assure reliable performance of the thrusters fabricated and tested subsequently during Phases III and IV.

Due to the limitations in material data following Phase I, Phase II was restructured to proceed immediately to thruster testing. Direct measurement of actual thruster performance promised to provide creep data on heat exchanger materials, wire compatibility data, nozzle performance data, and verification of thermal and structural models. In addition, thruster testing provided direct verification of performance estimates for component improvements identified in Phase I.

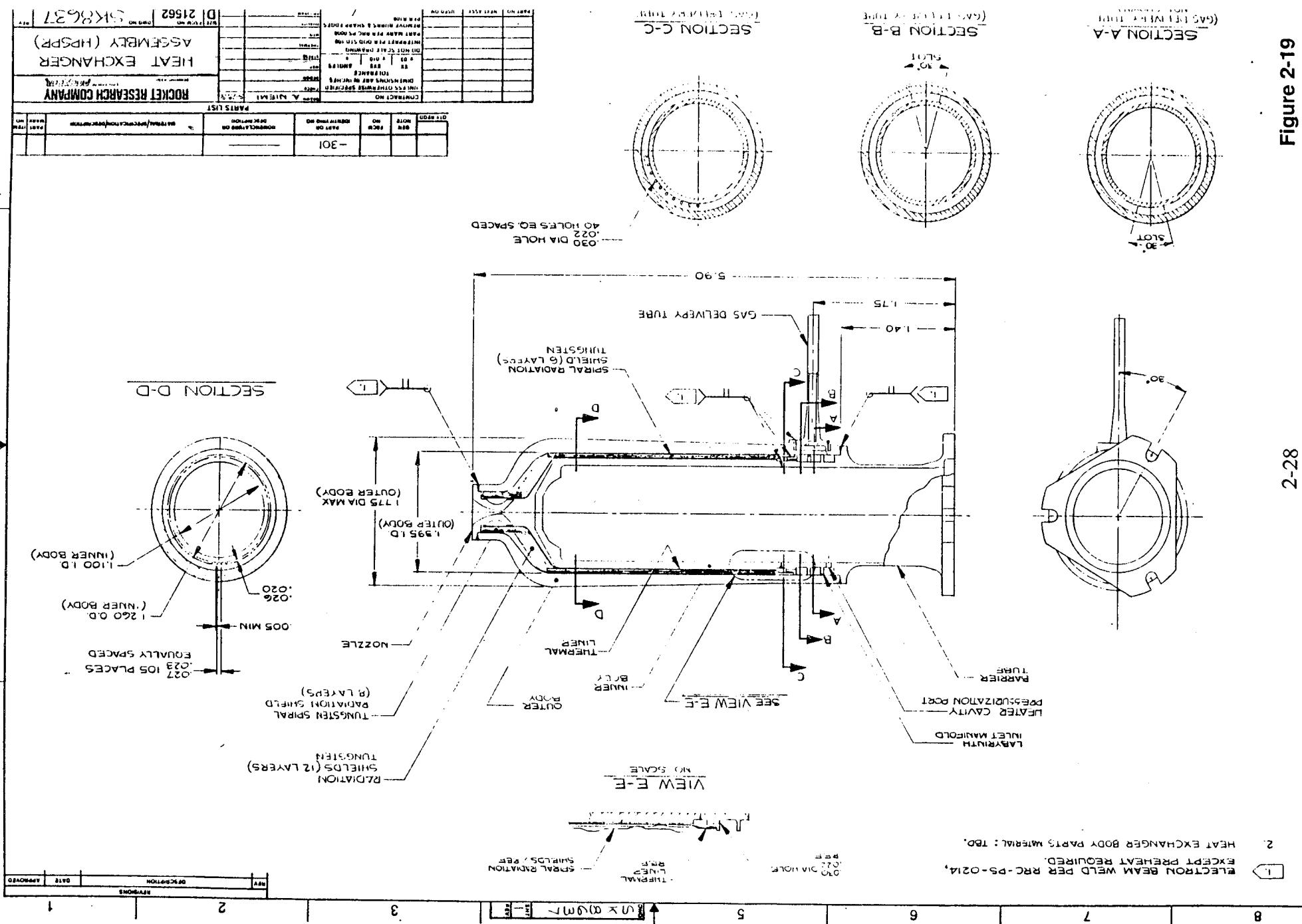
**Figure 2-17**



**Figure 2-18**



**Figure 2-19**



Several considerations went into making this choice:

1. The lack of material creep data at temperatures of interest introduced considerable uncertainty into the structural analyses. Since gathering the necessary data by conventional means proved to be unworkable (i.e., direct measurement from heated, stressed samples), it was decided to measure the deformation of an actual heat exchanger under representative conditions. From the measured deformation estimates could be made of the creep rate for use in further analysis.
2. Uncertainty in the actual nozzle efficiency made all performance estimates based on thermal analyses highly suspect. Actual performance measurements at conditions of interest were necessary.
3. The results of the heater wire exposure tests were inconclusive. An actual test of a filament of the appropriate configuration in a representative environment was the best available option.
4. A rhodium heat exchanger of the MR-501 design was available for use. Adaptation of this hardware to function with a sealed cavity seemed logical, even if the designs were not thermally optimal in order to learn more about the items above.

Two radiative heat exchanger thruster configurations were tested. The first was a vented cavity heat exchanger operating at a higher temperature than existing designs. The second was a sealed cavity thruster using the same heat exchanger design but modified to allow heater pressurization with decomposition gases.

## 2.2 Phase II — Hardware Support Activities

### 2.2.1 Seal Test

The heater cavity seal test is the only test from the original Phase II plan that was continued into the restructured program. Under this task seal designs developed for the sealed cavity heater were tested in the laboratory at the component level.

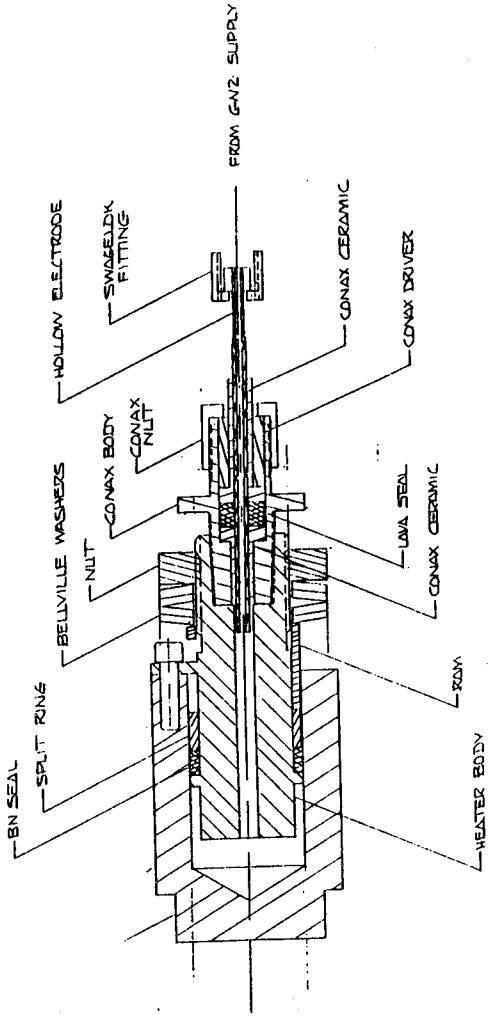
Three seals were required in the heater design:

1. Forward cavity seal
2. Conax-type electrode seal — denoted "Lava Seal"
3. Swagelok seal, connecting the hollow electrode to the pressurant gas feed system.

Of these, the most challenging was the forward cavity seal due to the severe thermal environment predicted by the thermal model. The electrode seal was located in a less severe thermal environment. Moreover, essentially identical seals had been used successfully at RRC on a number of low and high power arcjet programs, so that their viability is much less in doubt. Finally, the Swagelok-type seal was located in a still less severe thermal environment.

For these reasons, the design of the initial cavity seal tests and test fixtures concentrated on evaluation of the forward cavity seal. The seal test fixture is depicted in Figure 2-20.

## SEAL TEST RIG



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The seal tests were conducted in a tube furnace. The following steps were performed:

1. Pressurize test cavity to 60 psig at ambient temperature
2. Close valve and monitor pressure decay for 10 minutes
3. Raise test fixture temperature to 1800°F with valve open
4. Close valve and monitor pressure decay for 10 minutes
5. Cool fixture below 200°F with valve open
6. Close valve and monitor pressure decay for 10 minutes
7. Repeat 3—6 for 10 cycles.

The baseline seal material was powdered boron nitride (BN). Other seal materials were also evaluated:

1. *Boron Nitride (BN) Split Ring:* A split ring was machined from solid HP grade BN. It was hoped that the ring could be crushed as the axial sealing force was applied to form essentially a powder seal. Formation of the powder after assembly of the seal piece parts is attractive because it will minimize contamination by BN powder of other parts of the heater. Several unsuccessful tries resulted in the conclusion that it is not possible to crush the solid BN to a fine enough powder within allowable axial force limits. Solid BN was, therefore, rejected as a potential seal material.
2.  *$Al_2O_3$  Powder:* All seals produced with  $Al_2O_3$  powder exhibited unacceptably high leak rates, probably due to the large initial particle size, and the failure of the axial sealing force to produce smaller particle sizes by attrition. In addition,  $Al_2O_3$  is chemically less stable than BN at temperatures expected in the heater. For these reasons  $Al_2O_3$  was rejected as a seal material.
3.  *$MgO$  Powder:* Unacceptable leakage occurred in all tests with this material, although this material performed better than  $Al_2O_3$ .

Figure 2-20

4. *Iron Powder:* Finely divided iron powder was tried in one test. Gross leakage occurred and testing with this material was discontinued. Also, the seal material needed to be electrically insulating.
5. *Grafoil:* A Grafoil split ring cut to the size of the test fixture cavity was tested. Leak rates were unacceptable. The problem with this approach was apparently due to the split ring, which is necessary because a complete Grafoil ring would not fit over larger diameter features at either end of the barrier tube. The majority of the leakage apparently occurred at the interface between the two ring halves. Intimate, gas-tight contact between the halves was not achievable.
6. *BN Pellets:* In order to reduce the likelihood of contamination of other parts of the heater, BN pellets were tried as an alternative to BN powder, which is rather difficult to insert into the seal cavity.

### Sealing Procedure

Preliminary testing indicated that an ultimate axial seal thickness of 0.1-inch was optimal for this application, and that a 10/1 linear compression of free BN powder was required for adequate sealing. Use of the BN pellets described above allowed adequate sealing to be achieved with a manageable linear compression ratio.

After insertion of the BN pellets into the seal activity, a flat washer was installed in place of the Belleville washers, and approximately 2000 lb of axial force was applied to the pellets by tightening of the nut. It was found that this high axial force was necessary to properly crush the BN and form a tight seal. The flat washer was used because the Belleville washers are not strong enough to transmit such a high force. After seal formation the flat washer was replaced by the Belleville washers, and an ambient temperature leak test performed.

Initial testing at 1800°F was discontinued due to excessive leakage. Disassembly and inspection of the test fixture revealed that the Belleville washers, fabrication from Inconel 600, had relaxed at the test temperature. The resulting decrease in axial force had allowed the seal to leak. A second set of Belleville washers was then fabricated from TZM alloy and used in all subsequent testing. These TZM washers exhibited little or no tendency to creep in testing, so that a constant axial force of approximately 360 lbs (calibrated using a Carver press) was maintained on the BN seal. Acceptable seal performance was obtained.

Test results for the 16 cycles carried out using the TZM Belleville washers are summarized in Table 2-6. The first 11 cycles were carried out with an axial sealing force of approximately 360 lbs. The leak rate at ambient temperature and 1800°F was less than 0.01% of the expected thruster mass flow rate for all cycles.

While this leak rate was acceptable for sealing the cavity, results of thermal analyses ongoing during the early cycles of the seal test showed that the 360 lb sealing force may cause unacceptable creep deformation of the barrier tube, which forms the seal outer boundary. The axial sealing force was, therefore, incrementally reduced during the remaining 5 thermal cycles of the test, with encouraging results. Even at the final value of 50 lb the seal was effective. The leak rate was less than 0.01% of the nominal thruster mass flow rate.

Preliminary creep analysis indicated that this final sealing force, even if transmitted "hydrostatically" in the radial direction, would result in an acceptable creep deformation over a 250-hour life test of less than 3%.

**Table 2-6**  
**RESULTS OF CAVITY SEAL TEST**

Cycle No.	Sealing Force (lbf)	Ambient (scc/hr)	Leakage Rate — 1800°F	
		(scc/hr)	scc/hr gN <sub>2</sub>	%*
1	360	78	19	0.005
2	360	106	20	0.005
3	360	132	22	0.006
4	360	121	22	0.006
5	360	115	23	0.006
6	360	115	24	0.006
7	360	115	22	0.006
8	360	121	23	0.006
9	360	127	23	0.006
10	360	144	28	0.007
11	360	150	22	0.006
12	200	115	23	0.006
13	200	115	21	0.005
14	100	141	27	0.007
15	50	144	26	0.007
16	50	130	30	0.008

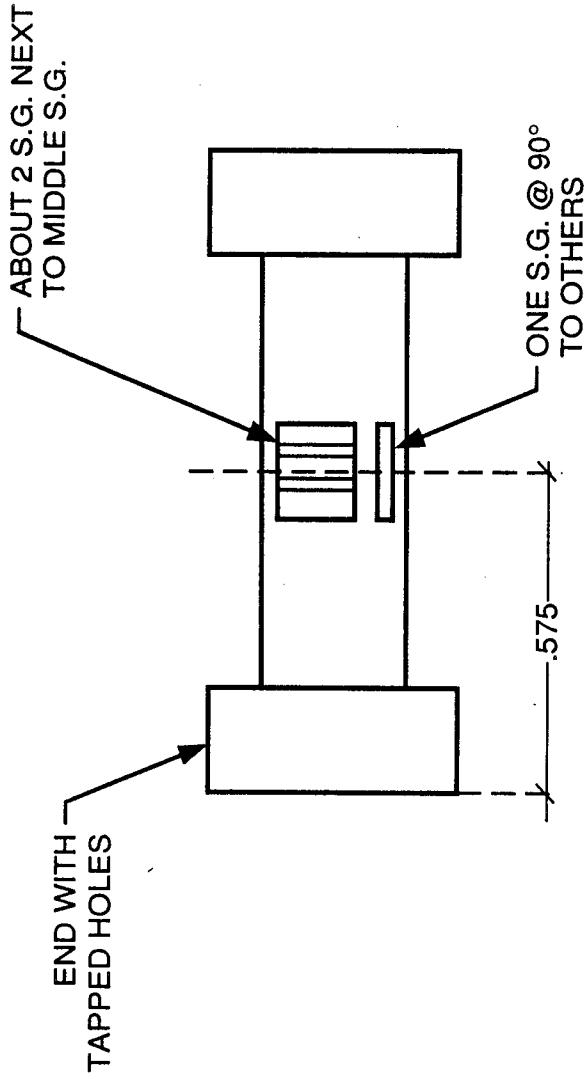
\*Defined as wt% of nominal thruster flow rate of 0.45 lbm/hr.

After the seal test was completed, an attempt was made to determine whether the transfer of the sealing force through the BN seal is "hydrostatic", i.e., whether the axial pressure in the seal is equal to the radial pressure. With regard to the creep problem, "hydrostatic" force transfer was an extreme worst-case.

The seal test rig was modified slightly to measure directly the hoop stress resulting from the BN seal force on the barrier tube. The wall of the outer body of the fixture was machined to a thickness of 0.022 inch to approximate the Re barrier tube. Five strain gages were installed as shown in Figure 2-21. A BN seal was formed from BN pellets with the strain gage output monitored during and after seal formation. The maximum hoop stress during seal formation was found to be approximately 30,000 psi, substantially less than the 200,000 psi predicted for "hydrostatic" transmission of the 2000 lb axial force estimated.

The heritage of boron nitride usage at RRC suggests that the BN should remain chemically stable as well as electrically insulative during the 250-hour life test. BN insulators used in the MR-501 resistojet typically see 2300°F during continuous operation, with no apparent degradation in dielectric strength or physical integrity. Moreover, testing at RRC has taken BN samples over 3000°F for extended periods of time. A BN plug was used as a support for a tungsten heater filament during extended testing of the filament at temperatures above 3000°F. During these tests, the BN plug was in direct contact with the resistively heated tungsten filament. No apparent degradation was observed after 100 hours at 3200°F, while

## STRAIN GAGE PLACEMENT ON CAVITY SEAL TEST FIXTURE



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Figure 2-21

significant degradation apparently occurred after an additional 40 hours at 3400°F. All of the thermal analyses carried out to date indicate that the temperature of the key BN parts in the sealed cavity heater will not exceed 3000°F.

### 2.2.1.2 Sealed Cavity Heater Design and Fabrication.

The redirected phase included tasks to design and fabricate a sealed cavity heater, install the heater into the existing rhodium heat exchange, and complete the assembly of a thruster for performance and life testing.

The new heater design was based on the following assumptions:

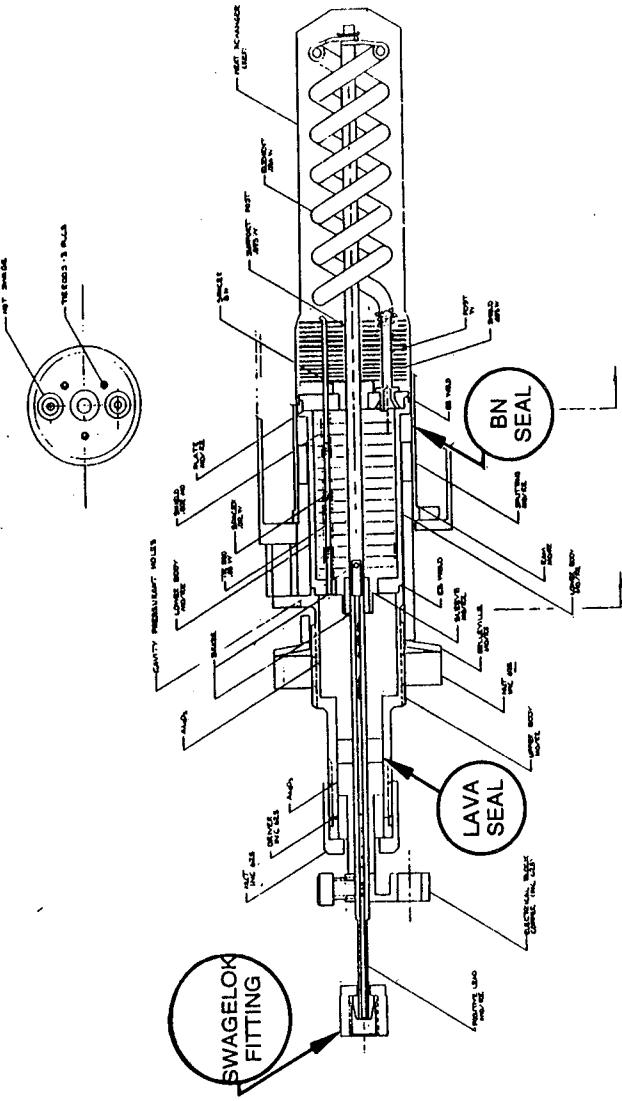
- A recrystallized heater filament fabricated in Phase I would be used. It was dimensionally inspected and the room temperature resistance was measured.
- The heater design would allow cavity pressurization with a variety of gases.

Performance and thermal analyses were conducted to determine Isp and thrust levels as a function of propellant mass flow and power. Critical component temperatures (e.g. heater braze joints) were determined over the operating range. Component level testing of designs for electrical isolation of the cavity pressurization tube as well as testing of hot swage heater support post joint techniques were also completed.

Prototype piece-parts were used to establish hot swage joint parameters prior to swaging the central rod to the heater support structure closure.

The original conceptual design for the sealed cavity heater is presented in Figure 2-22. Several notable features of this design are discussed below:

### CONCEPTUAL DESIGN OF SEALED CAVITY HEATER



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Figure 2-22

1. *Hollow Electrode Cavity Pressurization* — The cavity was to be pressurized during operation through a hollow tube, fabricated from Mo/41Re, which also serves as the positive electrode. The return electrical path is through the heater body and mounting structure. This approach has the advantage that only one pass-through at the rear of the heater is required, rather than two. Originally, it was intended to pressurize the cavity through a small hole in the inner body of the heat exchanger. The approach used had the advantage that it made no irreversible changes to the rhenium heat exchanger. An additional advantage to the approach was that it allowed pressurization of the cavity by various gases, including:

- Hydrazine decomposition products from the thruster gas generator.
  - Inert gases (e.g. argon, krypton) from a separate pressurization system.
  - Hydrazine decomposition products from a second gas generator (for test only).
- Moreover, vented cavity operation was possible simply by allowing the cavity to vent to the vacuum chamber through the hollow electrode.
- The option of pressurization with inert gases as well as hydrazine decomposition products was attractive for two reasons:
- Evaluation of the effect of a pressurized cavity on heater wire sublimation rates was possible, exclusive of any potential chemical interaction between the heater wire and hydrazine decomposition products.

- b. Flight designs of sealed cavity thrusters will be hermetically sealed by welding or brazing, so that seal failure will not be pertinent. The cavity seals discussed here were required for the Phase II life test only.
2. *Electrode Seal* — The electrode seal was designed as a Conax-type seal using a Conax BN pellet as the packing material.
3. *Cavity Seal* — Heater cavity seal testing was retained as part of the restructured Phase II program. The proposed design used Belleville washers to compress boron nitride powder firmly against the heater wall and the barrier tube with a split ring to effect a seal. Temperatures in the vicinity of the seal approached 3000°F.
4. *Belleville Washer* — The BN cavity seal was pressurized axially by one or more Belleville washers, fabricated from a refractory such as Mo/41Re or TZM. The axial force was necessary to allow for the effects of differential thermal expansion over multiple thermal cycles. The integrity of the washers was tested in a more severe environment than expected in the life test, since they will be brought to the same temperature as the seal itself.
5. *Connection of the Hollow Electrode to The Coil Support Post* — The electrode and support post were brazed together using a Mo/Re sleeve. A number of vent holes were incorporated into the connection to allow rapid evacuation of the cavity when the gas generator was turned on or off, so that pressure gradients across the heat exchanger inner body wall could be minimized when the heat exchanger was at high temperature.
6. *Use of Al<sub>2</sub>O<sub>3</sub>* — The spacers on either side of the BN electrode seal were fabricated from Al<sub>2</sub>O<sub>3</sub>. This material was chosen because solid BN was thought to be too soft, and therefore not suitable to withstand the sealing pressure applied by the Inconel nut. Al<sub>2</sub>O<sub>3</sub> was the next logical choice for an insulating material.
7. *Electrical Isolation of Hollow Electrode From Gas Generator* — Since the heater body acted as the return electrode, and was in electrical contact with the gas generator, the hollow electrode was isolated electrically from the gas generator (GG). This isolation was accomplished by a nonconductive fitting about two inches downstream from the GG.

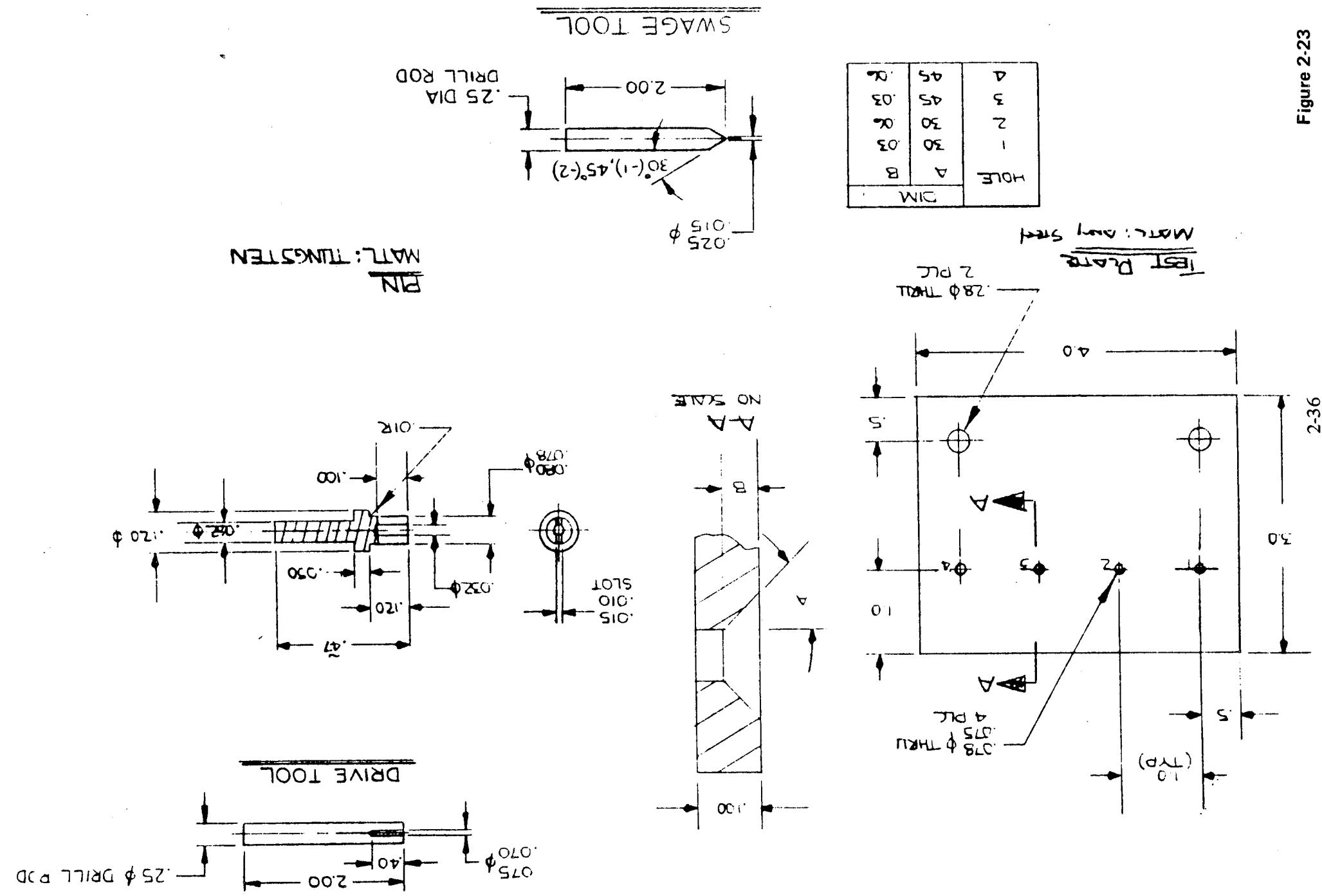
### **Hot Swage Test (Heater Support Post)**

The hot swage test fixture and test samples shown in Figure 2-23 were fabricated and hot swage testing completed. The test procedure included the following steps:

1. Tungsten pin was inserted into one of the four holes in the test plate. The drive tool was used because the pins and plate holes were designed for an interference fit.
2. The plate/pin assembly was heated to the swaging temperature with an air heat gun.
3. Swaging was attempted with an impact load using the swaging tool.

The swage tests were conducted at ambient temperature (1 pin), 350°F (2 pins), 750°F (1 pin). The results of the swage tests were disappointing at all temperatures. In all cases, severe cracking or splitting of the tungsten pins occurred at impact forces high enough to

**Figure 2-23**



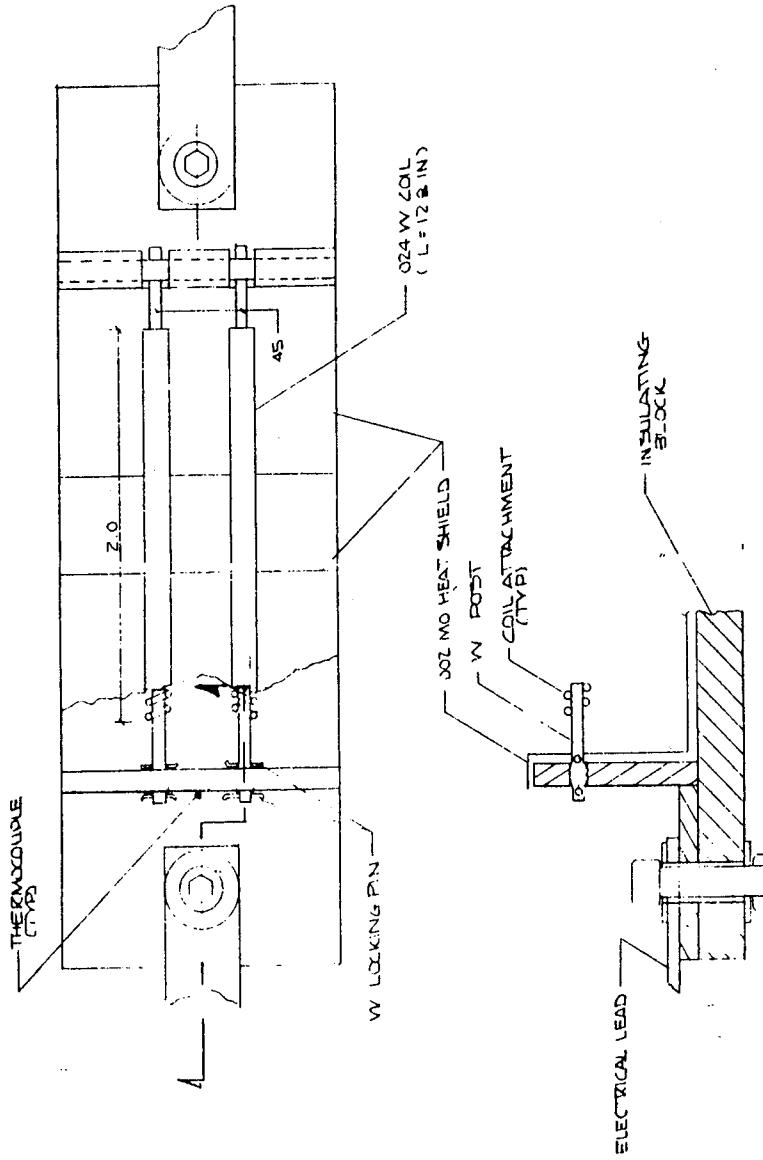
cause any bending of the "ears". The problem is the particular alloy used (tungsten/2% thorium). An alternate alloy with a specialized grain structure would be required for the swage joints to work as designed. Moreover, a specific swaging procedure would have to be developed by trial and error once the proper alloy was obtained.

Choosing the proper tungsten alloy and developing the proper swaging procedure would involve a significant effort. Therefore, design alternatives for attaching the tungsten filament support posts to the support plate were examined. Threading the posts and plate was identified as an attractive alternative and incorporated into the baseline design.

### Coil Attachment Test

To evaluate methods for attaching the heater filament to the support posts, and for attaching the support posts to the support plate, a filament attachment test was conducted. The test apparatus, depicted in Figure 2-24 incorporates the threaded filament support posts as well as the filament attachment scheme depicted in Figure 2-25. The straight sections of filamented wire were to be made from 0.024-inch, sag-resistant, W/3% Re heater wire developed by RRC under an IRAD program. These sections were connected electrically in parallel, so that a poor electrical connection between a filament and post caused a readily observable difference in temperature between the two filaments.

### COIL ATTACHMENT TEST – 3/10/87



2-37

Figure 2-24

This test fixture was primarily designed to test the interference fit between the filament and support post at high temperature. Different thermal expansion rates could cause the fit to loosen at temperature, increasing the electrical and thermal resistance at the interface. With a tight fit, it was expected that diffusion bonding of the filament to the post would occur.

The fabrication and assembly of the test apparatus was accomplished without problems. Single-point machining of the threads in the tungsten support posts proved feasible, and the tight diametrical tolerances were maintained. After assembly, the test fixture was installed in a vacuum chamber such that the unit was visible through a quartz window. After the appropriate vacuum level was achieved, power was applied to the filaments at a low voltage and gradually increased while the filament temperature was monitored with an optical pyrometer. The applied voltage was stabilized when the observed filament temperature reached 4600°F, and the filaments allowed to remain at the temperature for 2 hours.

The apparatus operated as expected, with no discernable difference in brightness temperature between the support posts, indicating good electrical and thermal contact between the two filament spans. Those filaments through which support posts passed were considerably cooler than those between the filaments and posts. This contact was apparent throughout the duration of the test, indicating that little or no squirm occurred. Upon disassembly, the filaments were found to be tightly attached to the support posts. Specific examination for diffusion bonding was not undertaken due to the short test duration.

The conclusion of the filament attachment test was that the proposed filament attachment design was less complex than previous methods with no performance loss.

### **Heater Design Update**

The heater design was updated to the design concept depicted in Figure 2-25. This update was performed to incorporate design improvement resulting from hot swage and filament attachment tests.

1. *Attachment of the Heater Coil to Support Posts:* The conical feature shown previously at the end of the support posts was removed in the updated design. The filament support posts were smooth cylinders at the filament end, with the attachment of the filament to the post occurring by interference fit. Test samples of the filament and post indicated that secure attachment could be achieved by this method at 70°F, and the assembly procedure is quite simple.
2. *Attachment of Coil Support Posts to Support Plate:* Due to the failure of the hot swage tests described above, the attachment problem was revisited, with the result that a superior attachment method was identified. The new design incorporates male and female threads on the post and support plate, respectively. Once in place, the posts were held securely by a short tungsten wire which passes through a hole in the post. Preliminary machining of samples at RRC indicated that the threads could be single-point machined into the tungsten posts. Tapping of the holes in Mo/41Re support plate was not expected to present any fabrication problems.

## Thermal Model

A thermal model of the sealed cavity heater was completed. This model used the RRC TMG thermal analysis program, and was based on the baseline heater design shown in Figure 2-25. Preliminary runs using boundary conditions typical of test data and results of thermal models of similar engines indicated that the model predicts accurate temperature profiles at lower power levels (450 to 550 W). Runs at higher power levels were difficult to evaluate due to lack of test data. These runs did predict steady-state temperatures in excess of 3000°F at the BN cavity seal and the BN spacer at the 750 W power level.

It became evident that the sealed cavity heater model which had been developed was not accurate as a stand-alone model with arbitrary assignment of thermal boundary conditions. A model of the heat exchanger and mounting bracket surrounding the heater was required so that temperature profiles within the heater could be more accurately assessed.

It was possible to adapt an existing MR-501 thermal model for use in the thermal model of the proposed life test thruster. The previously developed thermal model of the sealed-cavity heater was inserted into an existing model of the MR-501 thruster in place of the MR-501 heater. A few minor modifications were made to the MR-501 model to more closely represent the life test thruster. For example, the heat exchanger and barrier tube thermal properties were changed to those of rhodium, the throat diameter was changed to that of the existing rhodium heat exchanger, and the radiation shield configuration was changed slightly. The model was checked against test data at 500 watts input heater power, and agreement was acceptable.

The inputs to the thermal model were flow rate and heater power. The outputs of the model included temperatures and estimates of Isp and thrust. Table 2-7 presents results of the thermal model for three cases taken as first approximations of BOL, MOL, and EOL thruster operation for a typical feed pressure blowdown cycle. Heater input power was assumed to be a nominal 750 watts. The values of feed pressure ( $P_f$ ) and chamber pressure ( $P_c$ ) are not used explicitly by the thermal model, but are the values necessary to give the specified flow rate.

The results presented in Table 2-7 illustrate the central performance problem encountered in the design effort. The Reynold's numbers for the flow conditions of interest were so low that lower predicted Isp efficiencies offset potential Isp gains expected for lower flow rates. In lower power resistojet designs, as flow rate decreases with decreasing feed pressure, Isp, and the internal thruster temperatures increase. These thrusters operate at Reynold's number regimes in which Isp efficiency remains essentially constant. In the present case, however, as feed pressure, and thus flow rate, decrease, the resulting chamber temperature increase did not produce the expected increase in actual Isp. The thermal model predicts that Isp values above approximately 335 sec are not attainable with the existing heat exchanger, as shown in Figure 2-26.

The performance predictions in Figure 2-26 illustrate for several power levels the decrease in performance with decreasing flow rate caused by the reduction in Isp with decreasing

PREDICTION OF SEALED CAVITY THRUSTER  
SPECIFIC IMPULSE AS A FUNCTION OF PROPELLANT  
MASS FLOW RATE FOR SEVERAL POWER LEVELS

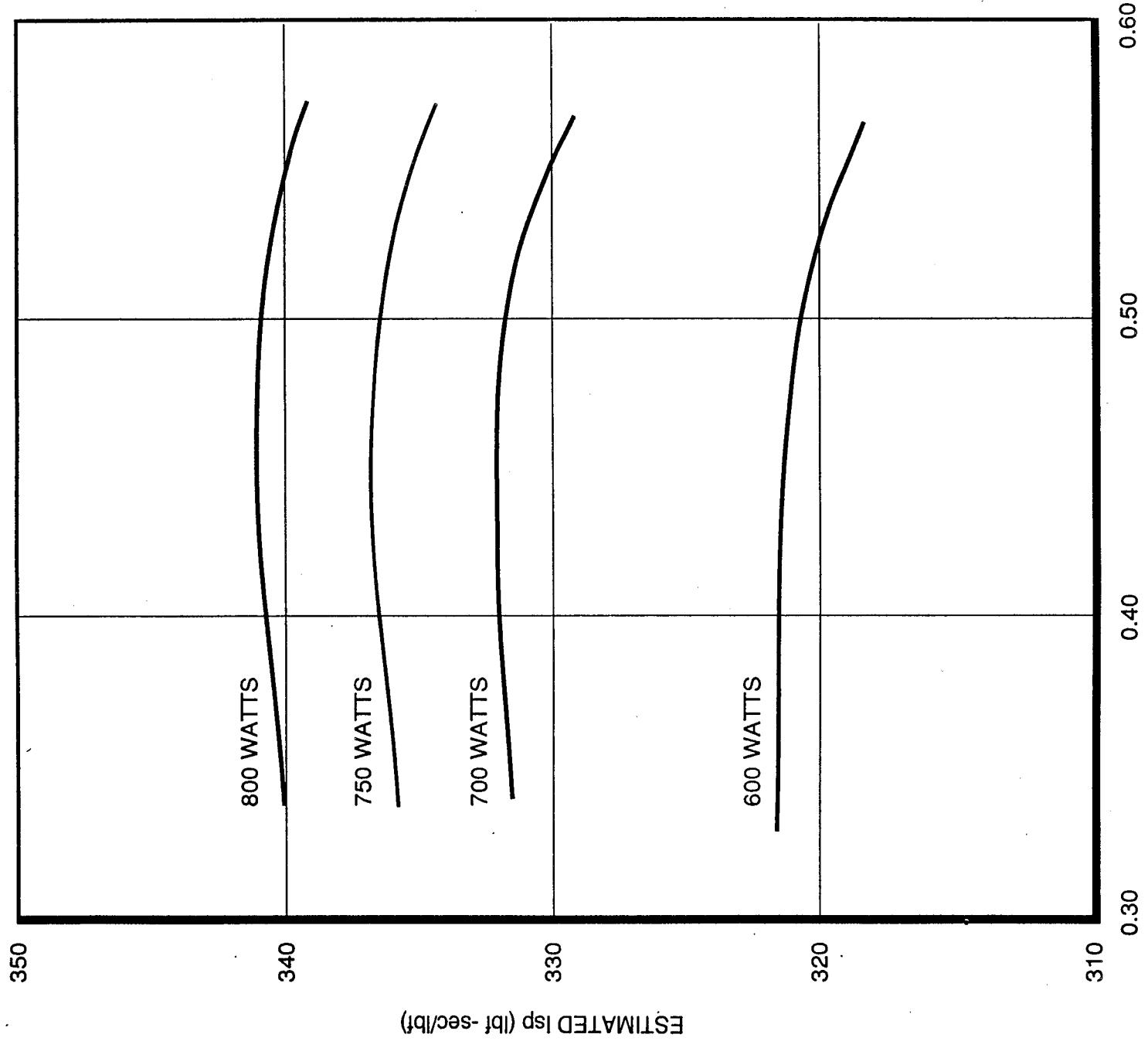


Figure 2-26

2-41

Table 2-7  
PREDICTED NOMINAL THRUSTER OPERATING CONDITIONS  
Typical Blowdown Cycle

Parameter	BOL	MOL	EOL
Power (watts)	763	739	724
Flow rate (lbm/hr)	0.68	0.45	0.35
Isp (lbf-sec/lbm)	337	336	335
Thrust (lbf)	0.064	0.042	0.032
Feed pressure (psia)	300	150	100
P <sub>c</sub> (psia)	78	54	43
T <sub>c</sub> (°F)	4250	4475	4635

Reynold's number. As the flow rate is increased above approximately 0.45 lbm/hr (at 750 watts), the predicted Isp also decreases slightly. This is due to the decrease in chamber temperature which is not overcome by an increase in efficiency.

Figures 2-27, 2-28 and 2-29 present temperature profiles within the sealed cavity heater for the three cases characterized in Table 2-7. The node number column is denoted by a "#", and the temperature column by an °F. Of particular interest in these profiles are the temperatures of the BN parts, the heater filament, and the heat exchanger outer body.

As with all parts of the heater, the BN spacer temperature (node 7031) increases with feed pressure blowdown. The spacer temperature starts at approximately 2850°F, and increases to approximately 3150°F at the end of life. The spacer average temperature was predicted to be approximately 3050°F. Thus during the entire life test, the spacer temperature was predicted to be below the temperature at which the successful operation of BN in contact with tungsten was demonstrated for 100 hours. The predicted temperatures of the BN cavity seal near node 7044 range from 2500° to 2760°F, well within the range of apparent material stability.

Predicted heater filament temperatures at the nozzle end (node 1100) range from 4560° to 5050°F, with an average of approximately 4920°F. These temperatures are well into the range at which tungsten sublimation retardation would be required to prevent premature heater failure.

Heat exchanger outer body temperatures (nodes 2222 and 2224) ranged from approximately 4100° to 4600°F. Combined with the chamber pressures necessary to achieve the required flow rates, these temperatures could cause unacceptable creep during the 250-hour life test.

In cases that matched performance predictions to previous data, the thermal analyses discussed above show that the sealed-cavity heater design is less efficient than a flight-type design would be, primarily due to the more massive components necessary to effect an

CASE 36

Heater Power (watts) 72.4  
 Fluerate (lb/hr) 0.346.  
 Ext air temp ( $^{\circ}\text{F}$ ) 46.47  
 Head Loss (watt) 46.6  
 Maximum Temp (sec) 333.2

$P_F = 100 \text{ psia}$

#  $\cdot F$ 

7007 1175

7025 1640

7025 1731

7013 1747

7021 2110

7011 2117

2051 2334

2040 2080s

7041 2763

3 2752

2141 3130

167 3114

229 4634

2197 3013

1078 49.95

2204 4754

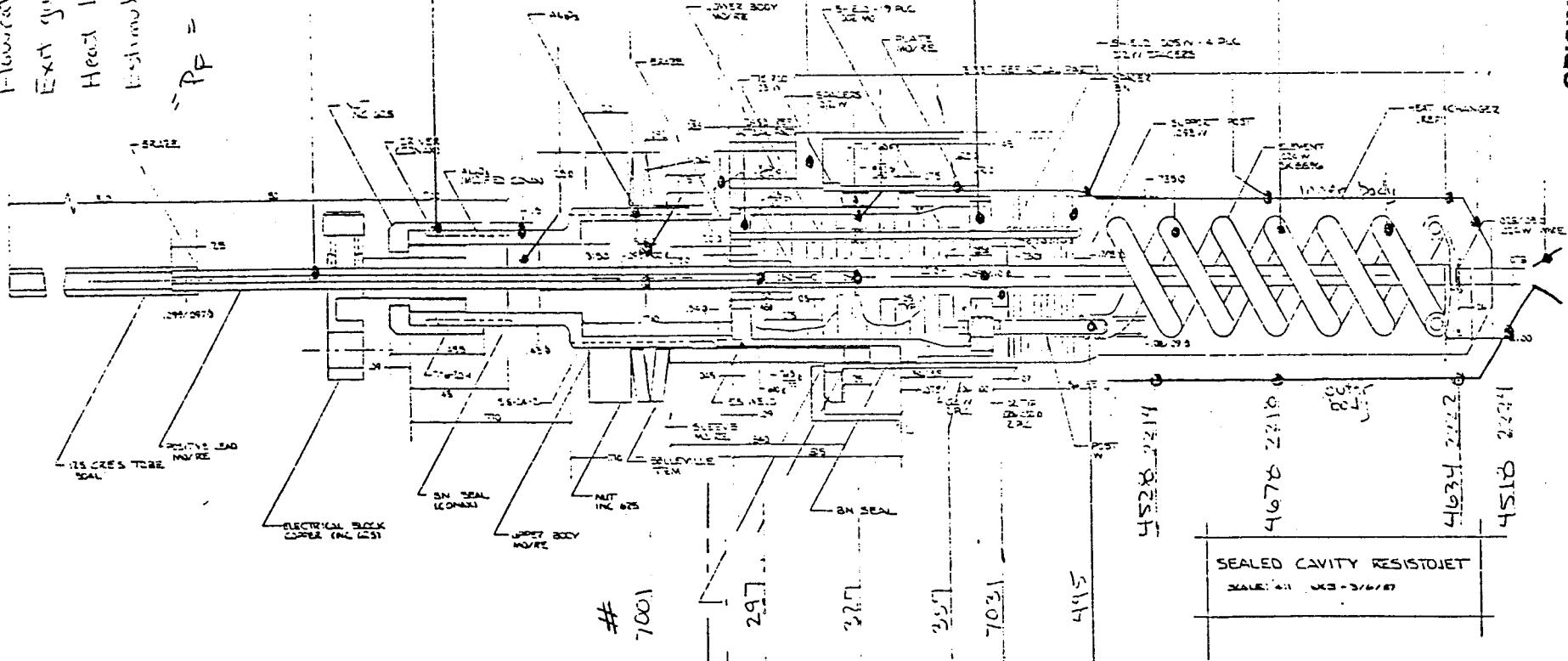
10591 5034

1100 5052

2205 4774

2211 4670

2222 3626

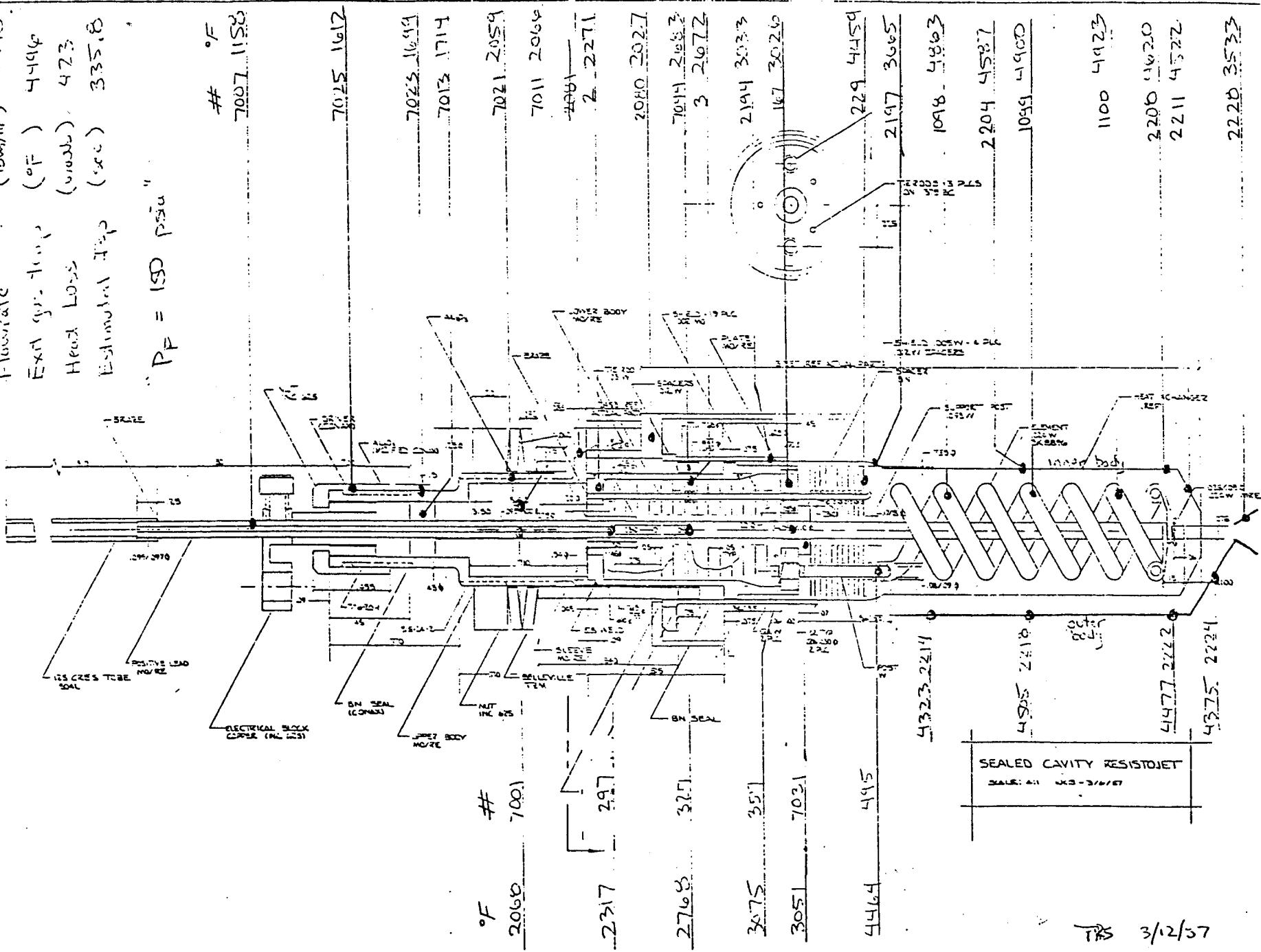


2-43

ORIGINAL PAGE IS  
OF POOR QUALITY

Figure 2-27

Figure 2-28

ORIGINAL PAGE NO. 244  
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**OPTIONAL PAGE 18**

**Figure 2-29**

245

CASE 36

Heater Power (watts) 763  
Flowrate (lb/mhr) 0.671  
Exit gas Temp (°F) 416.6  
Head Loss (in.) 33.1  
Estimational Disp (sec) 333.2

$$P_F = 3000 \text{ psi}$$

# °F

7007. 1118

7045. 1550

# °F

1954 7001

-2182 - 297

2594 327

2874 357

2854 7031

4120 495

3870 2214

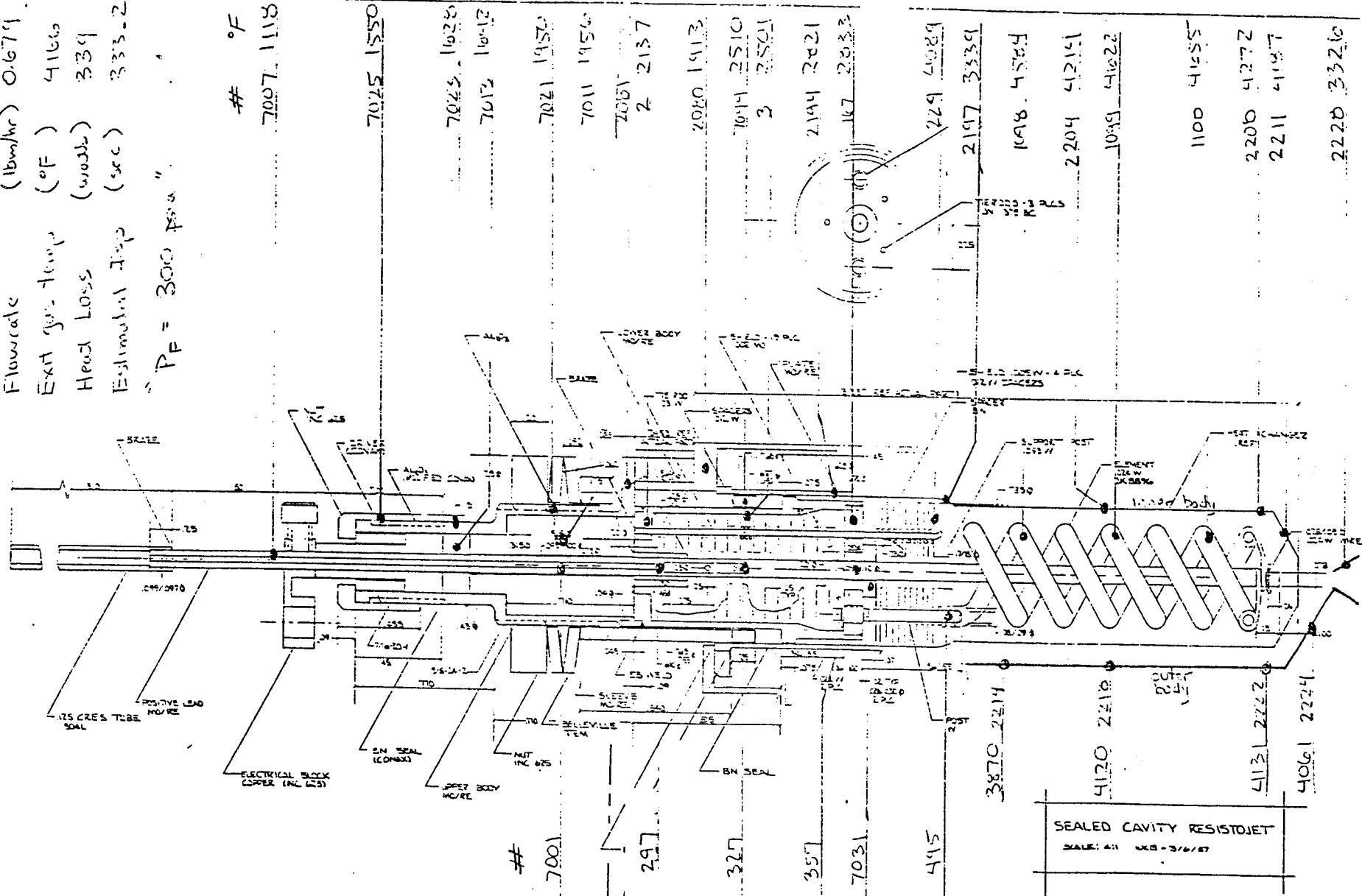
4120 2410

4061 2244

4131 2212

2220 3320

2200 4127.2  
2211 4147



TBS 3/12/57

adequate seal to the existing heat exchanger. More heat would be conducted to the rear of the heater than in a flight thruster for which the cavity seal would most likely be welded or brazed.

### Structural Analysis

In the absence of a vibration requirements for the life test thruster, the most pressing structural concern was creep life of the heat exchanger outer body. Since the cavity was to be pressurized, the pressure differential across the heat exchanger inner body would be negligible during steady-state operation, and creep was not a concern for this component. The outer body of the rhodium heat exchanger, however, was not designed to withstand the combination of temperature and pressure predicted by the analyses described above.

The empirically derived algorithm used at RRC to predict creep rates in rhodium may be written, for constant stress and temperature, as:

$$\epsilon = 177.8 [e^{-4.96 E4/T}] [\tau^{1.727}] \times t$$

where:

- $\epsilon$  = strain, in./in.
- T = temperature, °F
- $\tau$  = stress, ksi
- t = time, hr

The outer body creep predicted for the 250-hour life test using the above algorithm is given in Table 2-8. Typical margins for flight-qualified resistojects are less than 5% creep over the component life. The predicted 250-hour total creep for the heat exchanger outer body far exceeds this guideline.

Table 2-8  
PREDICTED 250-HOUR CREEP  
HEAT EXCHANGER OUTER BODY

Pressure (psi)	Temperature (°F)	Strain Rate (%/hr)	Total Strain (250 hrs) (%)
78	4200	0.09	22
54	4500	0.10	25
43	4650	0.10	24

Even considering the uncertainty in the creep algorithm, these creep predictions were cause for concern. A further examination of this problem was conducted. An estimate was made of the performance degradation necessary to remain within creep margins during the life test. Further evaluation of the Isp efficiency correlation used in the thermal model was conducted. Small changes in the predicted outer body temperatures can result in substantial changes in the predicted outer body temperatures which are driving the predicted creep rates to unacceptable levels. The test of this design is described in paragraph 2.2.3, but the duration of the test was not sufficient to validate creep performance.

## 2.2.2 "Fast Track" Test

The so-called "fast track" test was added to the program to provide testing of important design elements in a vented cavity design, as well as to provide vented cavity heater performance data as a benchmark for comparison with sealed cavity heater performance data.

The fast-track test was undertaken to supplement the performance predictions of the thermal model. Questions regarding the use of the Isp predictions in the thermal model remained. These predictions were based on cold-gas experiments, and their applicability at high temperatures was uncertain.

The configuration on which the rhenium heat exchanger was based had been tested at RRC at power levels up to 600 watts but not at the low flow rates, and thus low Reynold's numbers anticipated for the life test unit. RRC therefore proposed to NASA LeRC that vented cavity testing at power levels and flow rates typical of the life test unit be undertaken.

The rhenium heat exchanger assembly, including the mounting bracket, valve, and viscojet, were used in the fast-track test, along with a standard design EHT heater borrowed from an in-house program. This heater had been recently rebuilt, and the filament had not had power applied except for initial checkout during assembly. After assembly, the thruster was mounted on RRC's pendulum balance thrust stand in a vacuum test chamber.

Each test at power levels >500 W typically lasted only 15 minutes in order to minimize filament sag and to avoid unnecessary stress on the rhenium heat exchanger. After an initial thrust stand calibration, the thruster was run for approximately 15 minutes at a power setting of 500 watts for warm-up to minimize the operating time at high power. The thruster was run at high power only until the measured performance stabilized, at which time a 2-minute data acquisition sequence was initiated. After data acquisition was complete, the heater was shut down. The gas generator was run for an addition 2 minutes to avoid damage to the valve due to thermal soakback. After the gas generator was shut down, the thrust stand was recalibrated and automatic data reduction of the HP controller commenced.

A total of 33 runs were carried out during the fast-track test program. The results of these runs are summarized in Table 2-9. Figure 2-30 presents a plot of measured Isp as a function of heater power. Included in this figure for comparison are the results of previous testing of the same heat exchanger at lower power levels.

Much performance data were obtained in these tests at power levels higher than any previously performed with resistojets. Specific impulse levels near 350 seconds were demonstrated.

## 2.2.3 Sealed Cavity Test

The most important technology demonstration planned for the restructured Phase II effort was a 250-hour life test of a sealed cavity resistojet. This test was planned to utilize the existing rhenium heat exchanger and the sealed cavity heater designed and fabricated during Phase II.

**Table 2-9**  
**FAST-TRACK TEST RESULTS**

Run No.	Power (watts)	Flow Rate (lbm/hr)	Thrust (mlbf)	T-Heater (°F)	Isp (lbf-s/lbm)
7	500	0.475	40	4265	303
2	500	0.477	41	4287	307
12	500	0.477	40	4223	304
15	500	0.467	39	4252	301
17	500	0.468	40	4241	313
20	500	0.651	58	4099	319
21	500	0.474	40	4262	305
29	500	0.465	39	4225	302
3	600	0.470	41	4551	317
4	650	0.460	41	4668	323
5	656	0.531	47	4532	322
6	750	0.445	40	4836	324
8	750	0.445	50	4787	336
11	751	0.431	39	4862	330
14	751	0.449	41	4850	328
18	751	0.547	49	4789	328
22	750	0.636	58	4711	328
24	750	0.633	60	4697	342*
26	751	0.632	57	4701	328
30	750	0.541	50	4724	330
33	747	0.569	53	4768	337
9	800	0.542	50	4881	333
10	801	0.431	40	4895	336
13	800	0.456	42	4906	334
19	800	0.537	50	4555	336
23	800	0.650	61	4792	338
25	800	0.641	60	4796	340
27	855	0.660	62	4751	339
28	850	0.652	61	4799	338
32	848	0.542	52	4975	348

\*Probably spurious

## SPECIFIC IMPULSE AS A FUNCTION OF POWER

### RHENIUM RESISTOJET BASELINE PERFORMANCE

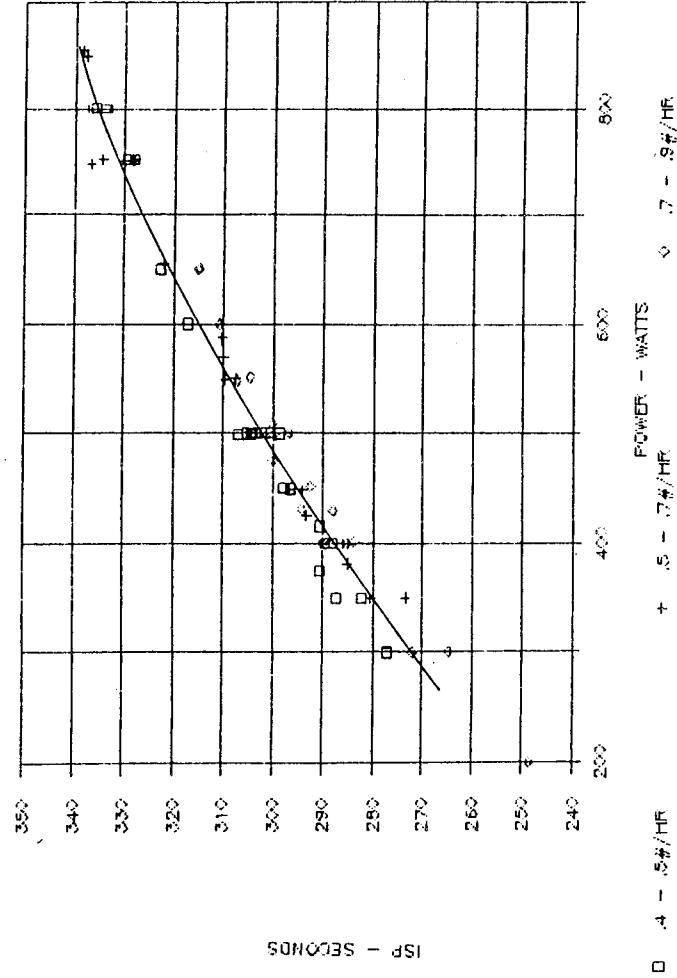


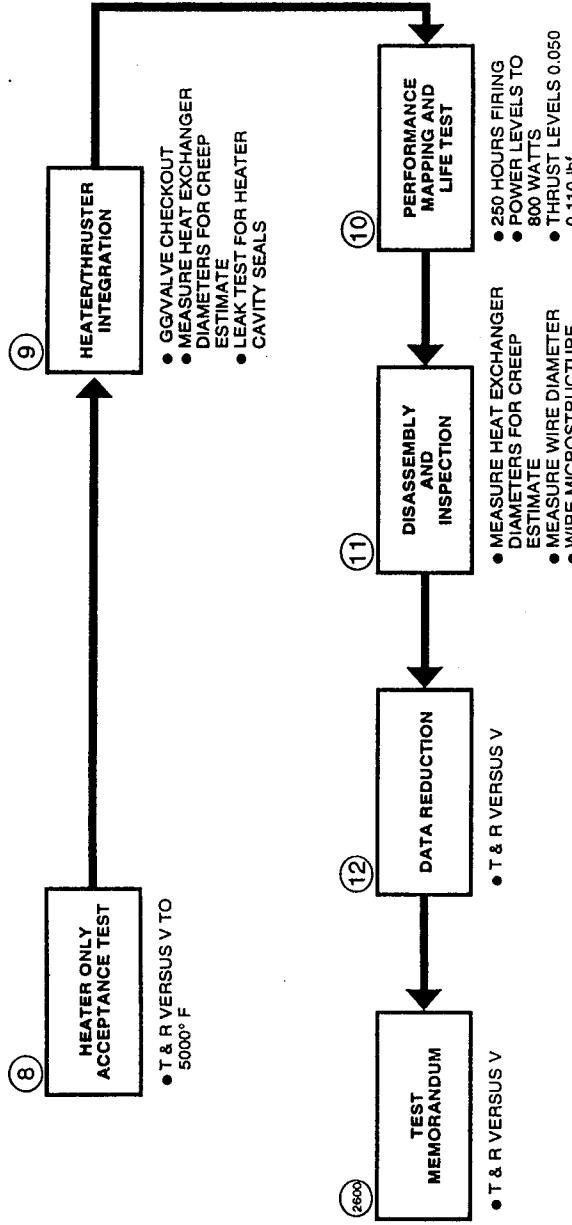
Figure 2-30

The objective was to test the sealed cavity heater in the rhenium heat exchanger for 250 hours at flow and power levels typical of spacecraft operation. High-purity hydrazine propellant was utilized. Wire temperatures necessary to achieve 330 to 340 seconds Isp were expected to cause significant sag, even with sag-resistant wire. An electric motor was used to rotate the test bed 180 degrees after each run was used during life testing to counteract the effects of sag. The test flow plan is given in Figure 2-31. The thruster was to be tested on a thrust stand during pretest and post-test performance mapping as well as during health checks after 100 and 200 hours. During the remainder of the life test, the thruster was to be attached to the rotator and not the thrust stand. Chamber pressure and flow rate were measured to calculate characteristic velocity for performance assessment during the periods the thruster was not on the thrust stand.

A check-out test of the heater was performed prior to installation into the rhenium heat exchanger. An existing fixture with cooled walls and a view port to measure the wire temperature was used. Power, resistance, voltage and wire temperature measured over the operating range were within acceptable limits.

**PHASE II SEALED CAVITY HEATER,  
RHENIUM HEAT EXCHANGER TEST FLOW PLAN**

**TASK 2900**

**C11232-30**

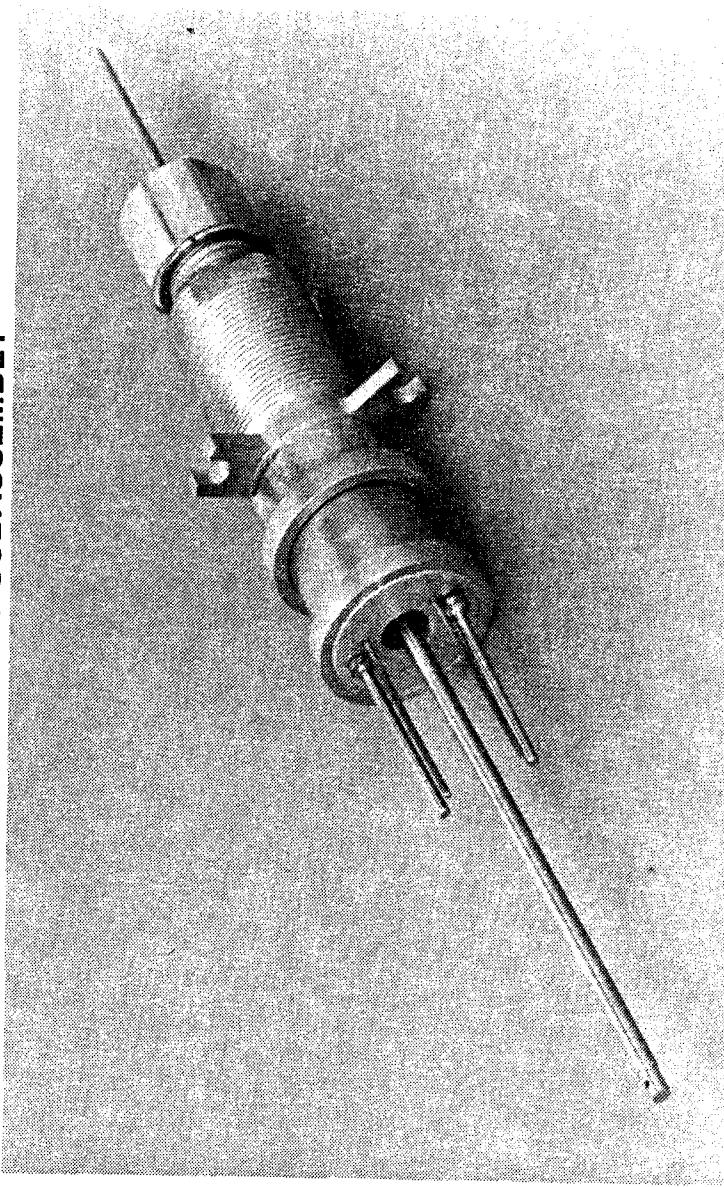
### 2.2.3.1 Thruster Assembly

The sealed cavity heater fabrication and assembly was completed, and the heater was installed in the rhodium heat exchanger to complete the life test thruster assembly. Key areas of activity included the following:

1. The modified designs of the lower body housing parts were fabricated along with suitable weld samples. After successful testing of the weld sample, the lower body subassembly was successfully EB welded to the upper body subassembly. The resulting subassembly is shown in Figure 2-32 and 2-33, after installation of the inlet tube subassembly, Conax seal, and nut.
2. The decision was made to use 0.002-in. radiation shields instead of 0.005 in. shields that proved to be difficult to fabricate. These parts were on hand. The filament ends slipped readily over the tungsten support posts and the center post. The completed assembly is shown in Figures 2-34 and 2-35. Two minor flaws are apparent from the views in these figures: misalignment of the tungsten radiation shields due to a hand operation, and a slightly missshapen configuration of the installed heater filament.
3. The heater filament was recrystallized in situ by resistance heating. The heater, as shown in Figures 2-34 and 2-35, was mounted vertically in the vacuum chamber, and the recrystallization process carried out with the filament restrained only by the support posts and the center post. Minimal squirm was observed. A flexible filament was required for mounting on the support posts.

**Figure 2-31**

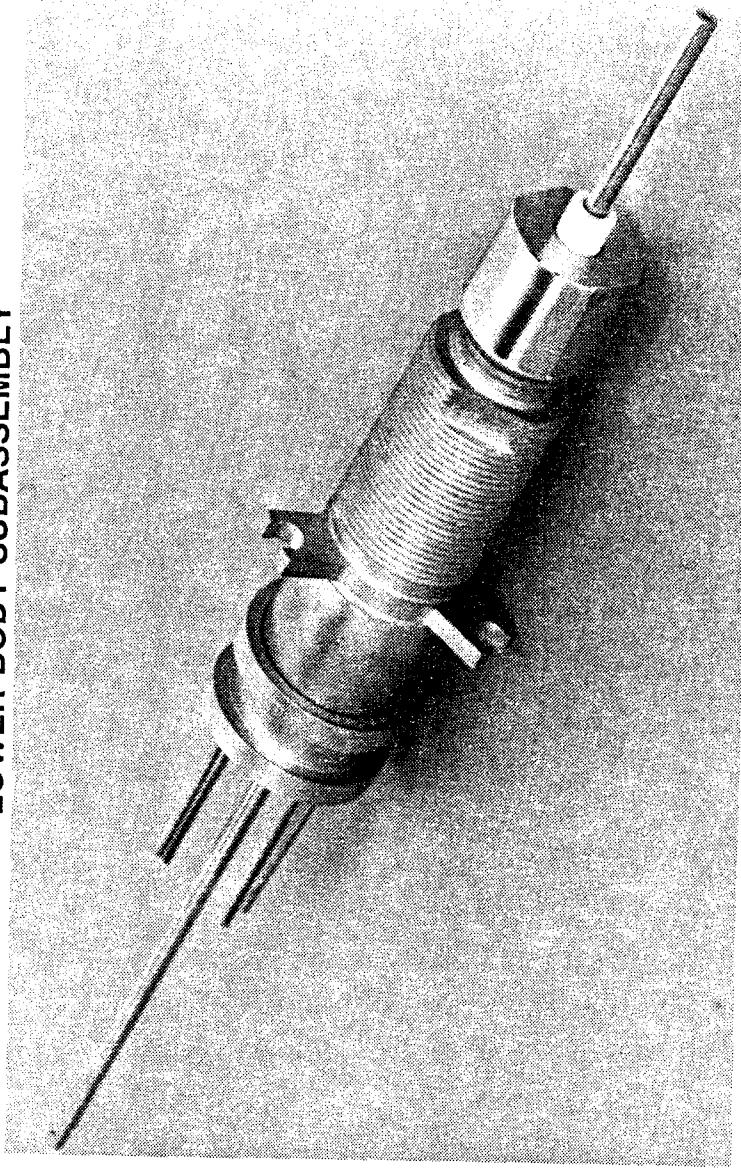
**LOWER BODY SUBASSEMBLY**



11195-82 3244-2

Figure 2-32

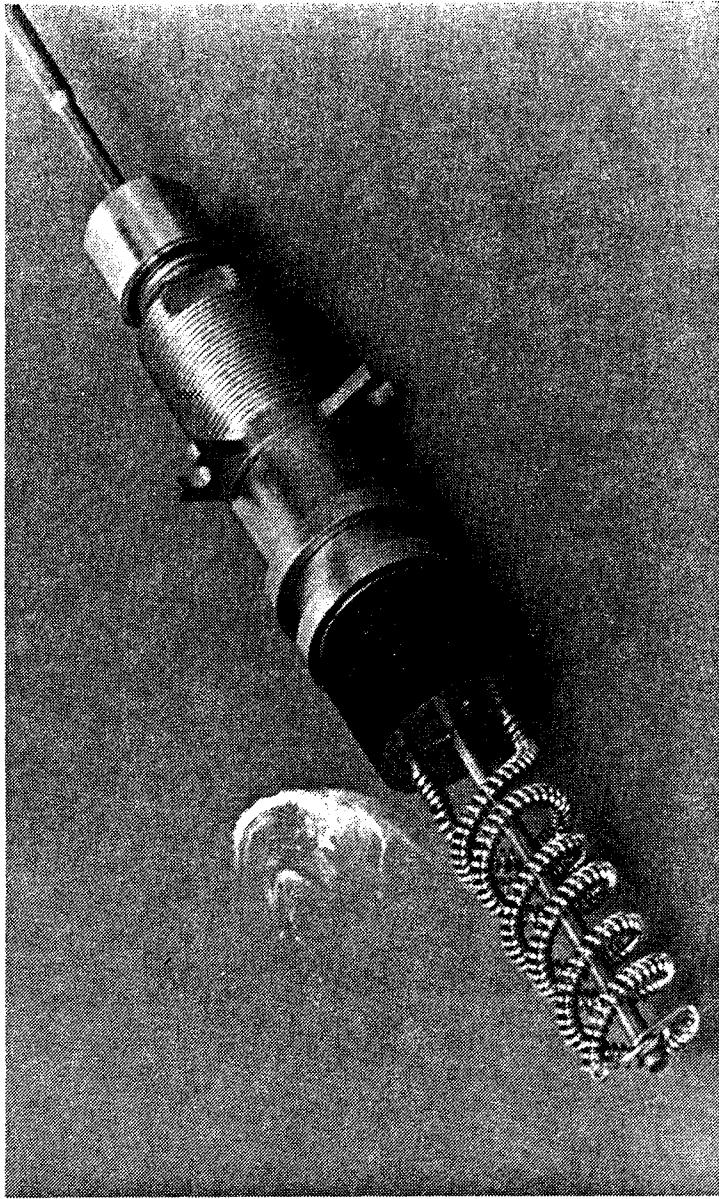
**LOWER BODY SUBASSEMBLY**



11195-83 3244-1

Figure 2-33

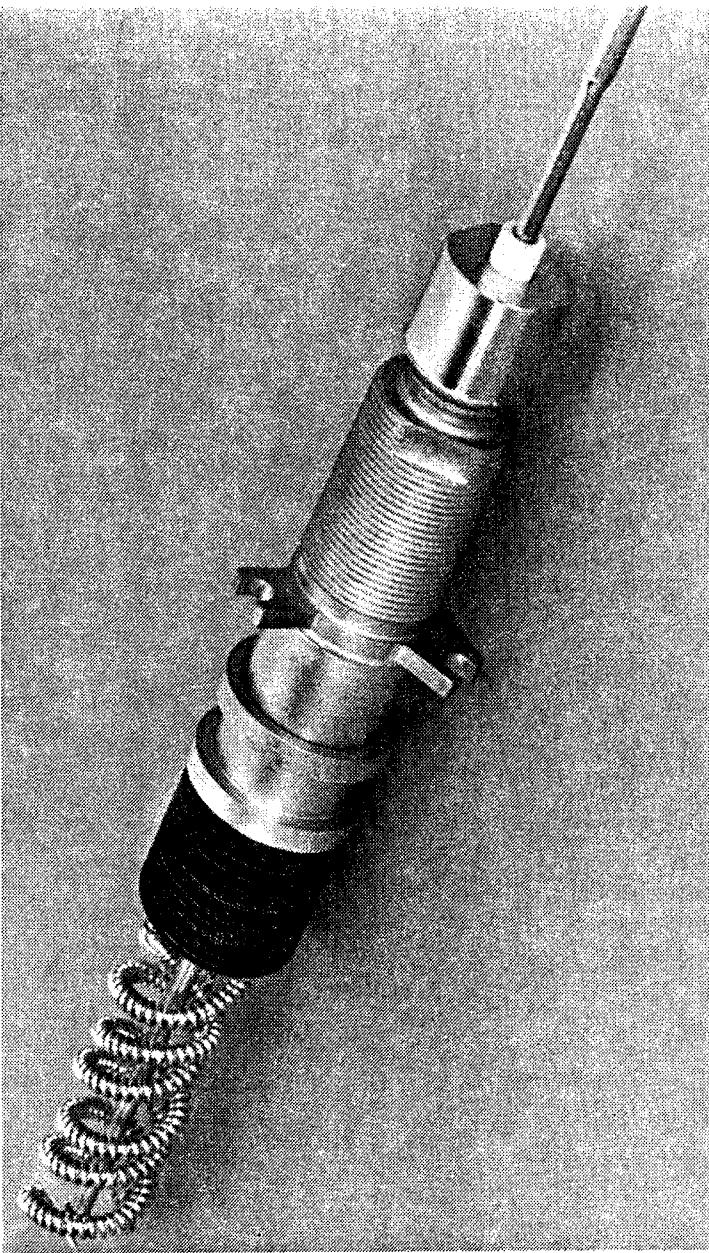
HEATER SUBASSEMBLY



11195-82 3244-5

Figure 2-34

HEATER SUBASSEMBLY



11195-84 3244-4

Figure 2-35

4. The heater was installed into the rhodium heat exchanger to complete the thruster assembly. The formation of the seal was completed without incident using the procedure developed during seal testing. After connection of the  $P_c$  tap to the hollow electrode, a leak check of the sealed cavity indicated a leak rate similar to those measured during the seal testing. Use of BN pellets allowed the seal to be formed with only trace amounts of BN contamination on other parts of the heater. This completed thruster assembly is shown in Figures 2-36 and 2-37.

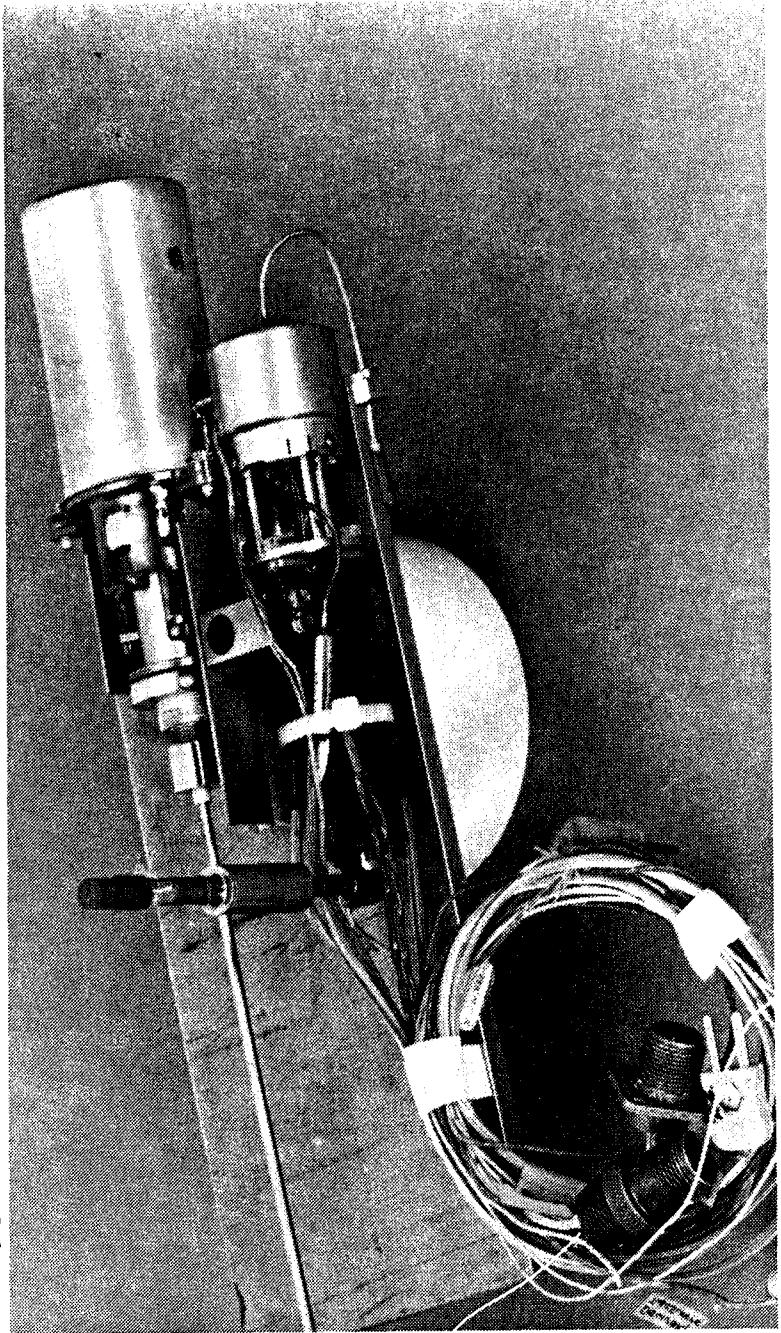
### 2.2.3.2 Sealed Cavity Thruster Life Test

Preparation of Cell 10 was completed and operation of the thruster at low power was begun. Activities under this task included the following:

1. Fabrication, assembly, and testing of the rotator was completed, and the rotator was installed on the thrust stand.
2. A water-cooled thruster mounting bracket was fabricated and installed on the rotator.
3. The thruster was mounted on the bracket, and the propellant and water lines routed onto the thrust stand as required.
4. Modifications to the controller software were completed and tested.
5. Thermocouples were installed on the thruster assembly.
6. Support brackets for the W3Re/W25Re thermocouple probes were designed, fabricated, and installed. The thermocouples were then mounted in place so that their tips made firm contact with the heat exchanger outer body wall.
7. A tool for measurement of heat exchanger OD was designed and fabricated. This tool was a "C" shaped frame which supports diametrically opposed bolts. When positioned properly around the nozzle end of the heat exchanger, the bolts could be screwed through the frame and through the thermocouple access holes until each seats firmly on the heat exchanger outer body in "micrometer-like" fashion. Figure 2-38 shows one of the thermocouple access holes.
8. An unaugmented firing was conducted successfully, followed by a powered firing at 550 W for approximately 30 minutes. This powered run was aborted due to a short from a power cable to the thrust stand. Preliminary results obtained prior to shutdown indicated thruster performance was lower than was expected from thermal analysis.

The sealed thruster was operated for approximately 10 hours total, during which performance mapping in both the sealed cavity and vented-cavity modes was carried out. Operation in both modes was successful, although heat exchanger and gas temperatures, and thus thruster performance, were lower than that expected based on thermal model predictions. In an attempt to raise heat exchanger and temperatures, radiation shielding was installed around the rear end of the heater after the performance mapping was complete. During the first run after the radiation shields were installed, the thruster failed due to the melting of three bolts and a nut plate which attached the heater to the heater support structure.

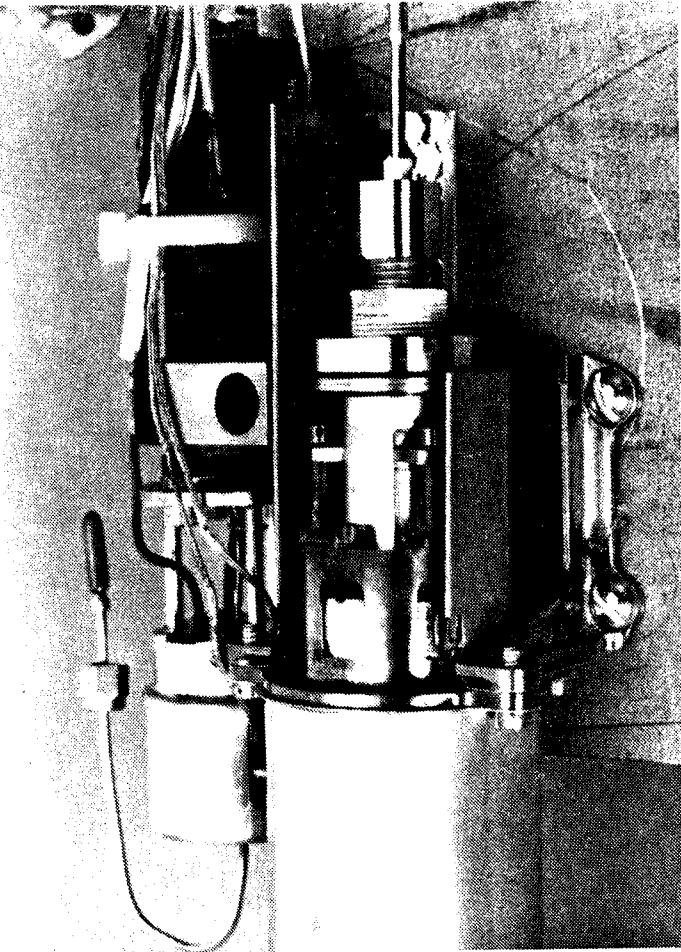
**THRUSTER ASSEMBLY**



11195-85 3150-2

Figure 2-36

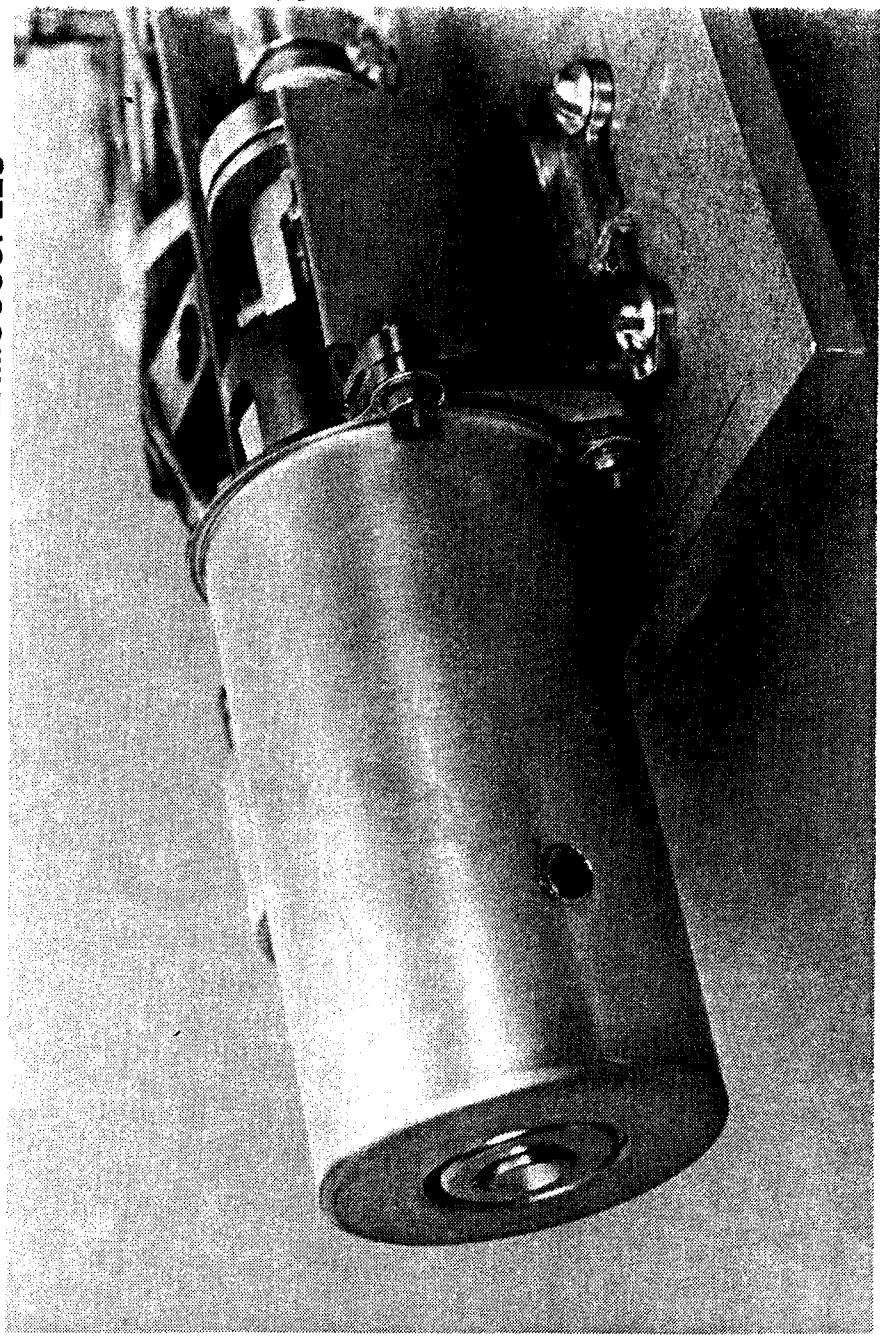
**THRUSTER ASSEMBLY**



11195-87 3250-5

Figure 2-37

## DETAIL OF OPENING ON THRUSTER FOR THERMOCOUPLES



11195-86 3250-4

### 2.2.3.3 Sealed-Cavity Thruster Test Results

A total of 18 sealed-cavity runs were made after the heater lead short was fixed. The tests were run at various power levels up to 850 W, and flow rates at or near 0.45, 0.55, or 0.65 lbm/hr. These flow rates were used in most of the thermal modelling carried out prior to the testing.

The thruster operated for 30 minutes before performance data were recorded. This time period was sufficient to allow the thruster to reach thermal equilibrium, but short enough to avoid heater shorting due to filament sag.

Results of the sealed-cavity tests and subsequent vented cavity tests are summarized in Table 2-10. Performance at 750 W was lower than expected, based on thermal model runs which assumed a hard vacuum. This result was attributed to heat losses from the rear (of the thruster) which were higher than thermal model predictions. This additional heat loss was attributed to:

1. Greater than expected heat conduction to the rear of the heater.
2. Greater than expected convection losses from the thruster due to chamber back pressure.

Figure 2-38

**SEALED CAVITY THRUSTER PERFORMANCE MAPPING TEST RESULTS**

Run No.	Cavity Status	Power (W)	Flow Rate (lbm/hr)	Thrust (mlbf)	T-Heater* (°F)	Isp (lbf-sec/lbm)
2	Sealed	0	0.512	25.9	—	182
3	"	295	0.468	34.6	3200	267
5	"	500	0.547	44.0	3675	290
6	"	500	0.645	51.6	3625	288
7	"	500	0.554	44.5	3675	290
8	"	650	0.542	45.7	4000	304
9	"	650	0.445	37.1	4075	300
10	"	649	0.641	53.8	3960	303
11	"	0	0.611	31.3	—	186
12	"	650	0.643	54.2	3975	304
13	"	750	0.532	46.1	4190	312
14	"	750	0.444	38.0	4275	309
15	"	751	0.629	54.5	4220	312
16	"	751	0.530	46.0	4180	313
17	"	850	0.625	55.9	4300	322
18	"	850	0.526	46.4	4340	318
19	Vented	500	0.553	44.5	3625	290
20	"	500	0.457	36.8	3650	290
21	"	500	0.647	52.0	3540	289
22	"	750	0.529	45.9	3875	312
23	"	750	0.435	37.4	3840	310
24	"	750	0.628	54.5	3850	312
25	Sealed	751	0.528	46.1	4340	315

\*Based on pyrometric temperature calibration conducted after test completion (Fig. 2-31).

The filament temperatures inferred from measured filament resistances were also lower than expected. Filament temperatures and, by inference, heat exchanger temperatures, were not high enough for filament evaporation evaluation or heat exchanger creep evaluation planned for the life test.

The cavity seal appeared to work flawlessly during the sealed-cavity testing. Upon completion of run 18, the test cell was opened and the thruster was inspected. A pressure test on the heater cavity indicated no change in the seal leak rate compared to the leak rate at the start of testing. Heat exchanger measurements also indicated no change in OD as measured through the thermocouple access holes. Creep in the rhenium heat exchanger was negligible.

Upon completion of the thruster inspection described above, the cavity pressurization line was disconnected and the test cell closed in preparation for the vented-cavity tests.

## 2.2.3.4 Vented-Cavity Thruster Test Results

A total of 6 test runs were carried out with the thruster in the vented-cavity configuration. Results of these runs are summarized in Table 2-10 (Runs 19—24). Performance measurements were made at 500 and 750 W and at flow rates 0.45, 0.55, and 0.65 lbm/hr. No change in performance between the sealed and vented-cavity configurations was observed. The only difference between the two sets of runs was the apparently lower vented-cavity filament temperatures at 750 W and at all flow rates. In contrast, the filament temperatures at 500 W were nearly identical for the two configurations.

Upon completion of Run 24, the test cell was opened and the thruster was inspected. Measurement of the heat exchanger OD through the thermocouple access holes indicated no measurable creep. Throughout the preceding 24 runs, the tantalum shielded, Re/W25Re thermocouple probes had given obviously spurious readings. For this reason, and because the access holes were needed for pyrometric temperature readings, these thermocouple probes were not reinstalled after the heat exchanger OD measurements. Observation of previous runs had also indicated that the probes constituted significant heat conduction paths. It was, therefore, expected that removal of the probes would result in a slight performance improvement.

Data sheets for all valid test runs have been included in Appendix B. Note that flow rates based on sightglass readings have been handwritten on the data sheets. These values were used to calculate the handwritten Isp values on the sheets.

## 2.2.3.5 Further Pressurized-Cavity Runs

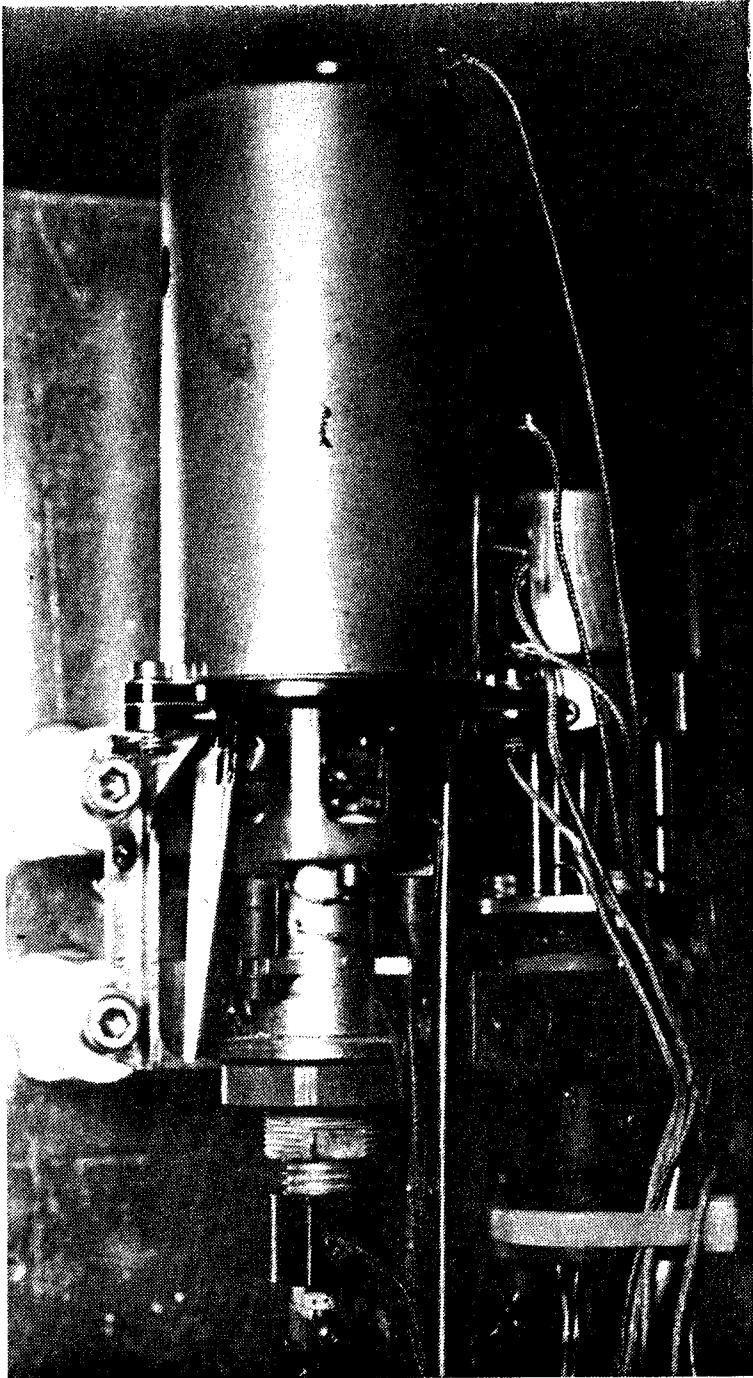
After the heat exchanger inspection, the cavity pressurization line was reconnected. A cavity leak check indicated that the cavity seal was still performing as in the first 19 runs. One sealed-cavity test (Run 25) was then carried out at 750 W and 0.528 lbm/hr. Results were consistent with previous sealed-cavity runs (Runs 13 and 16), with Isp and filament temperatures slightly higher. These slightly higher values may have been due to the removal of the thermocouple probes.

The low thruster efficiency observed in all tests was attributed to high heat conduction to the rear of the heater, and to high heat losses from the hot rear heater surfaces due to thermal conduction by test cell gases. In an attempt to reduce the heat loss from the rear of the heater, and thus increase the overall thruster efficiency, 13 wraps of moly foil radiation shielding were installed around the rear of the heater and the heater support structure. The thruster failed 1172 seconds into the next run, which was conducted at 700 W and approximately 0.52 lbm/hr. The computer which controlled the experiment had not yet recorded performance data when the failure occurred.

Complete disassembly revealed failures in the three nuts and the nut plate by which the heater was attached to the heater support structure (Figures 2-39, 2-40 and 2-41). When the nuts had failed, the pressure inside the heater cavity had momentarily driven the heat exchanger forward causing misalignment of the heat exchanger with the gas generator, thereby bending the rhenium gas delivery tube.

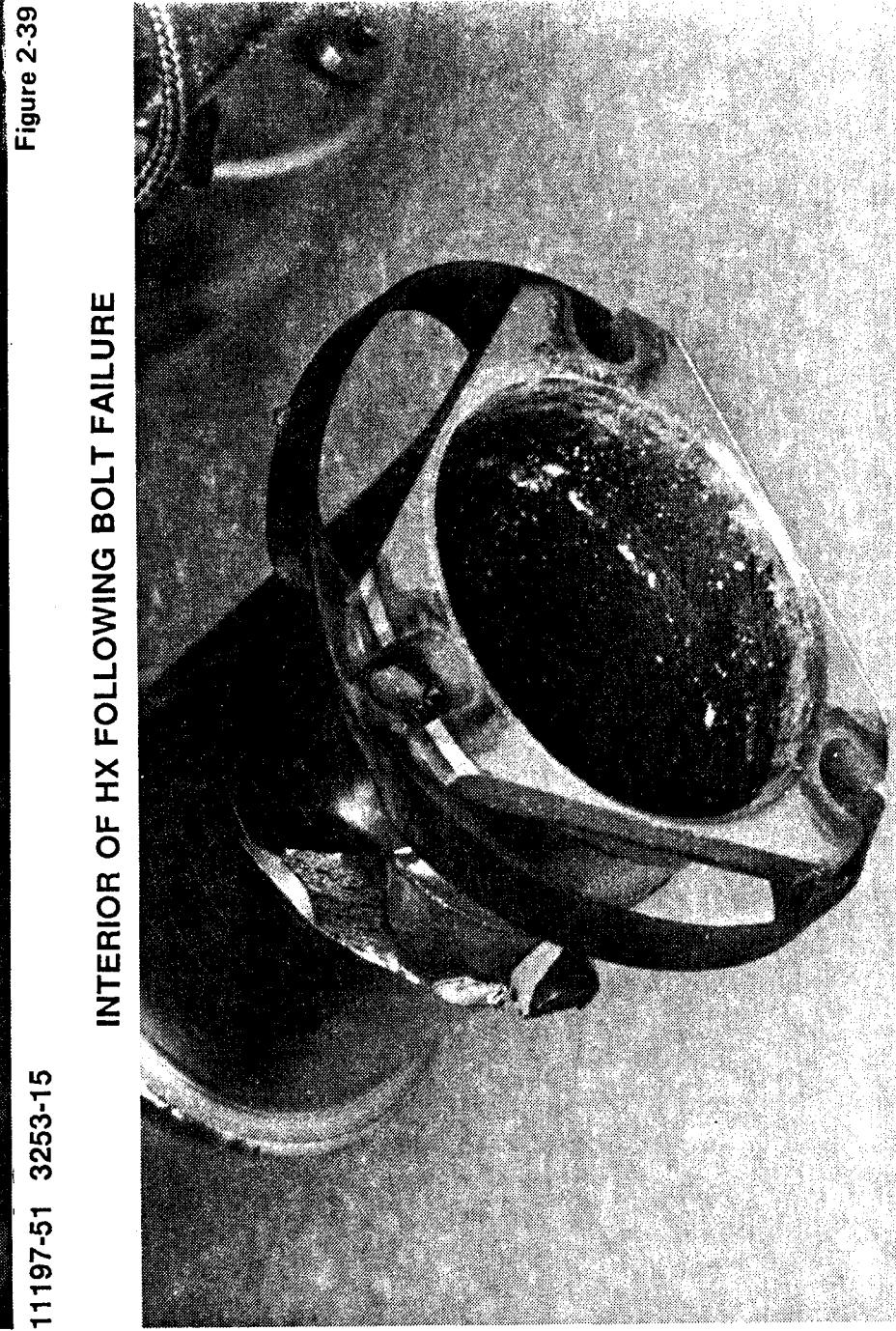
91-R-1553

THRUSTER ASSEMBLY FOLLOWING BOLT FAILURE



11197-51 3253-15

INTERIOR OF HX FOLLOWING BOLT FAILURE



11197-50 3253-5

Figure 2-40

The bolts and nut plate were fabricated from Waspalloy, a nickel based alloy containing chromium (19%), cobalt (13.5%), and molybdenum (4%). The presence of the radiation shields caused the temperature in the region of the heater/support bracket attachment point to approach the melting point of Waspalloy. After the nut failed, a portion of the nut plate came in contact with the barrier tube and melted, spreading out in the process to cover approximately 1/5 of that part.

The most critical damage was done to the rhenium heat exchanger in three areas:

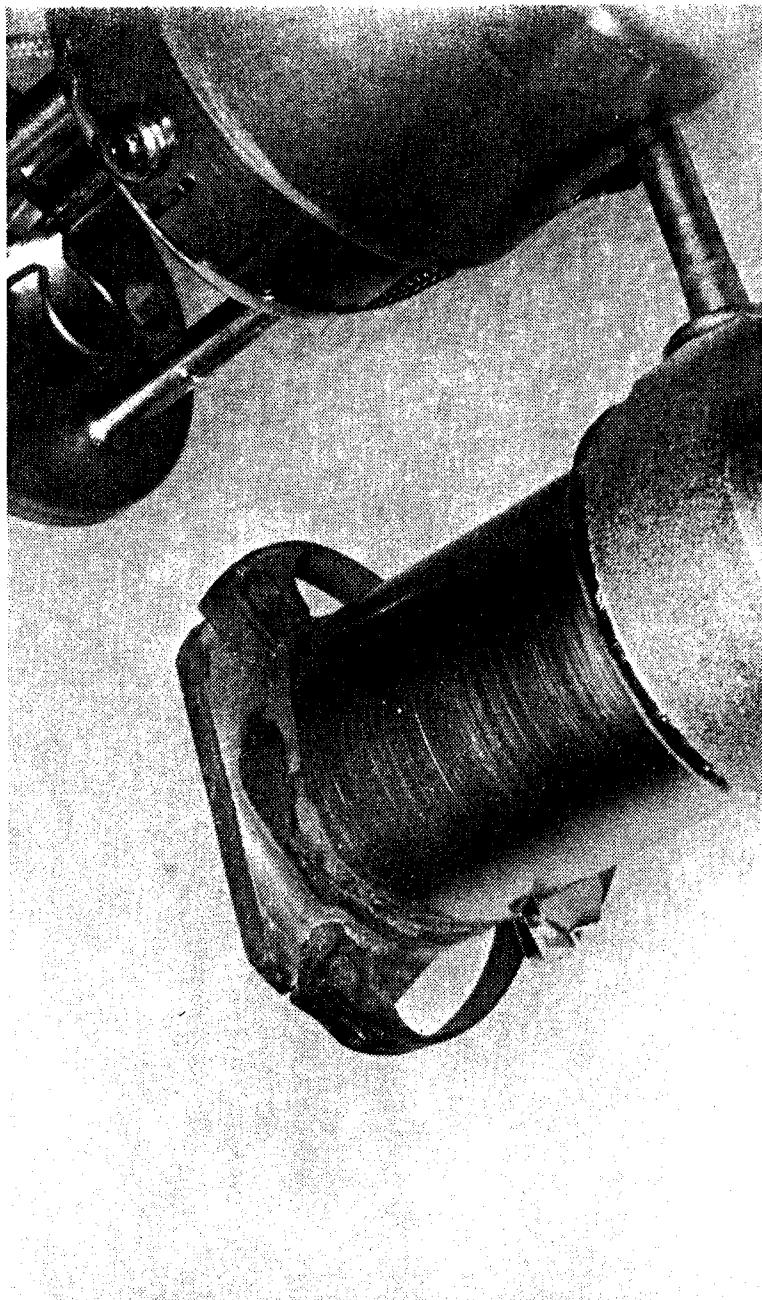
1. A thin layer of molten Waspalloy had spread over the rear end of the barrier tube OD (Figure 2-42). Inspection of the damaged part suggested it was salvageable. The nickel in the Waspalloy can irreparably damage pure rhenium by quickly diffusing into the rhenium metal structure and weakening it such that its high temperature strength would be substantially reduced. The contaminant Waspalloy was completely removed. After the Waspalloy had been completely removed, the barrier tube was inspected. The results of this inspection were encouraging. All dimensions were within specification.
2. A small crack was found in the rhenium gas delivery tube. This crack was probably caused by the movement of the heat exchanger when it was driven forward after the failure. The crack was repaired with an electron beam weld. Subsequent pressure testing revealed no leakage.
3. The same event which caused the crack in the gas delivery tube also caused the tube to be bent at the point where it is welded to the heat exchanger (Figure 2-43). Since rhenium work hardens readily, straightening of the bent tube was deemed too risky and it was left as it was.
4. The molybdenum foil radiation shields which surrounded the heat exchanger were damaged beyond repair during disassembly. Such damage is normal for resistojet disassembly. No abnormal degradation of the shields was noted.
5. The sealed cavity heater itself was undamaged except for possible diffusion bonding of the ram to the seal ring. Figure 2-44 shows the heater, with the BN seal intact, after removal from the thruster.

### **2.2.3.6 Heater Resistance — Temperature Correlation**

For Runs 24—26, a two-color pyrometer was loaned to RRC by NASA LeRC. After the thruster failure and subsequent disassembly, a correlation of the heater filament resistance and temperature was established using the NASA pyrometer. The correlation was established in the same test cell as the thruster testing, and approximately 20 data points were taken.

The correlation is shown in Figure 2-45 along with the original correlation and corresponding values from the Handbook of Chemistry and Physics. The agreement between the slope of this correlation and that of the literature values is acceptable, with the shift to the left being attributable to uncertainty in the length of wire in the heater filament.

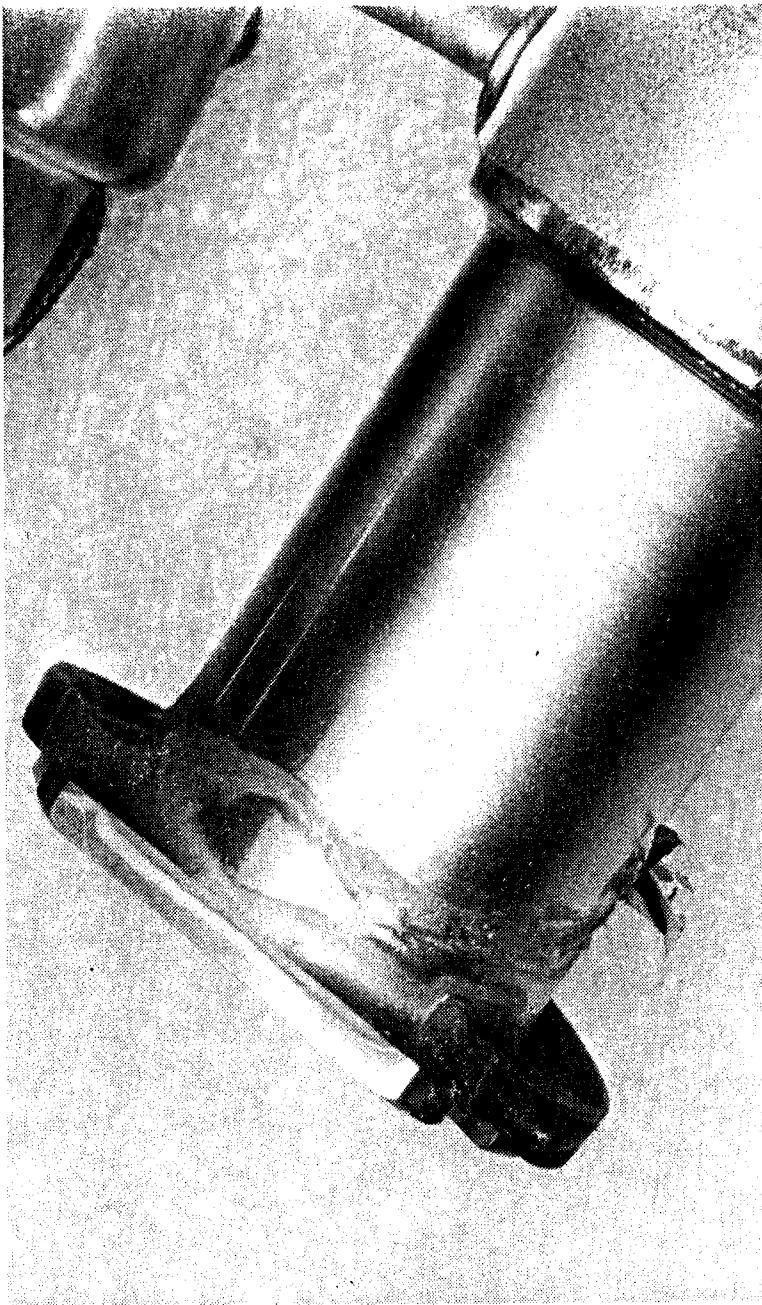
BENT RHENIUM GAS FEED TUBE



11197-53 3253-17

Figure 2-41

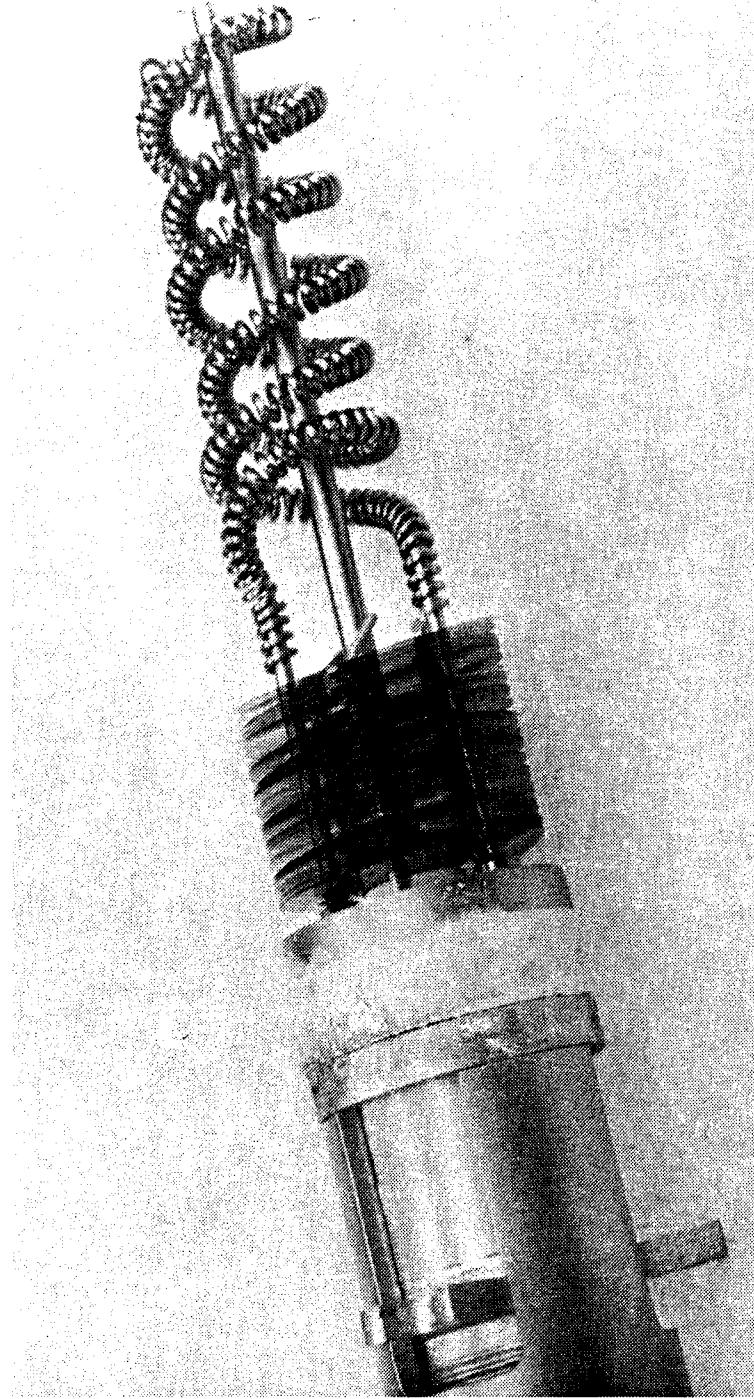
MOLTEN METAL SOLIDIFIED ON BARRIER TUBE



11197-52 3253-16

Figure 2-42

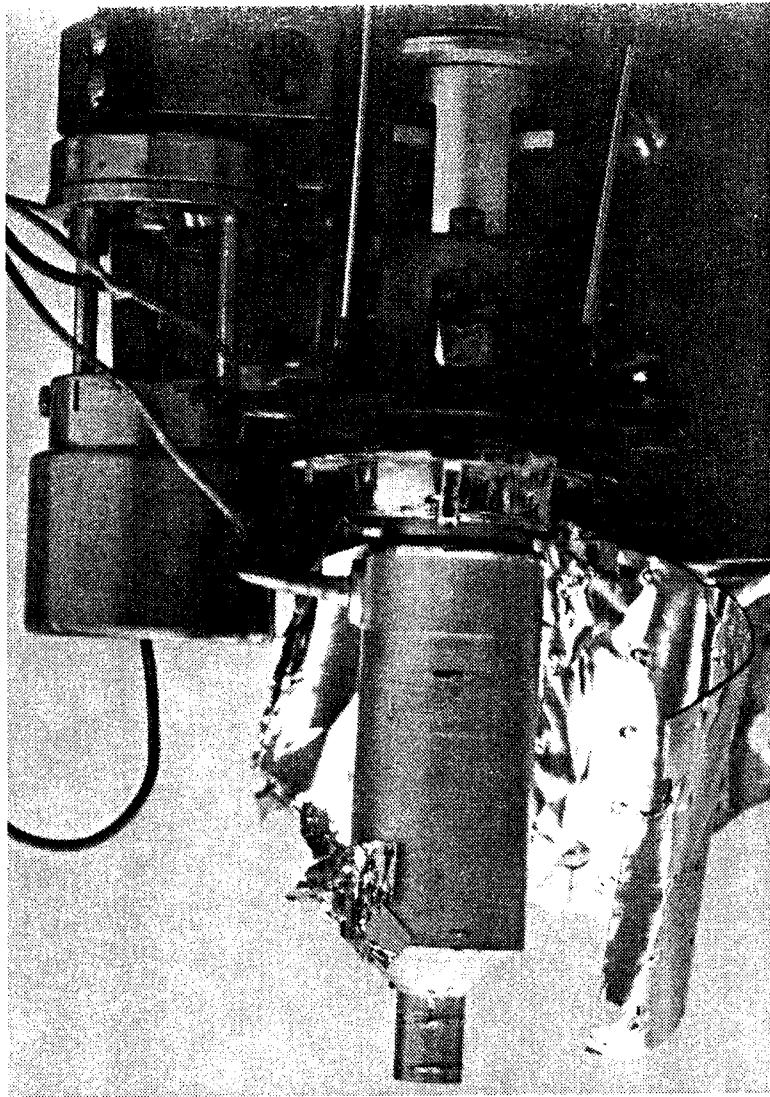
**HEATER SUBASSEMBLY FOLLOWING TEST**



11197-55 3253-12

Figure 2-43

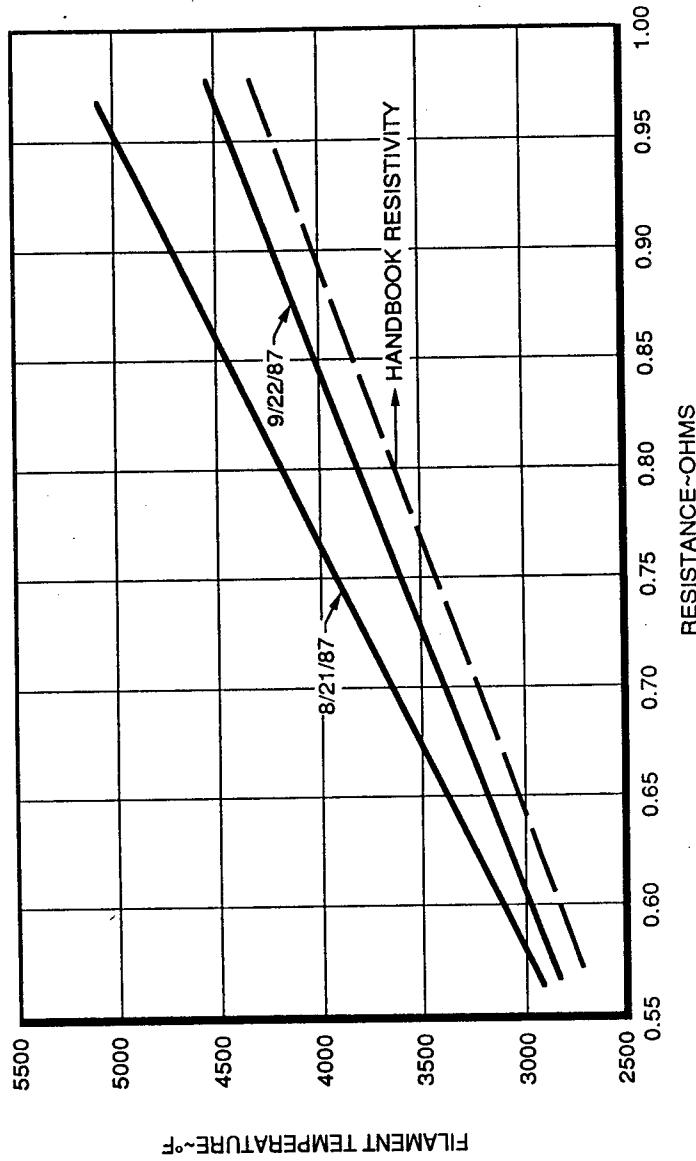
**RHENIUM HX FOLLOWING PARTIAL DISASSEMBLY**



11197-54 3253-17

Figure 2-44

## HIGH PERFORMANCE RESISTOJET FILAMENT RESISTANCE VS. TEMPERATURE



C11232-31

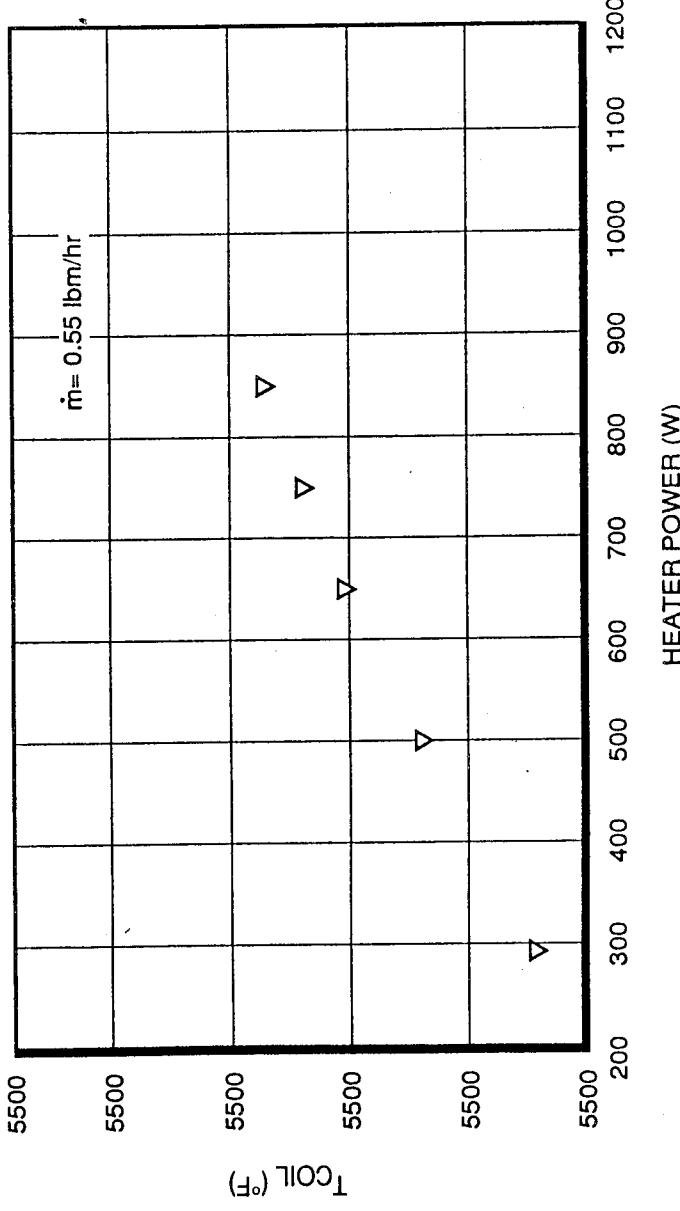
This correlation was used to calculate the filament temperatures in Table 2-10. Filament temperatures calculated are considerably lower than those from the original correlation, which was used in the data sheets of Appendix B. Even at 850 W, the filament temperatures do not approach the temperatures needed to test the sealed cavity concept for evaporation retardation.

### 2.2.3.7 Review of Test Results

The performance data gathered on the 25 successful runs were disappointing. Measured specific impulse values at 750 W were in the range of 310 to 315 sec, rather than the 330 to 335 sec values anticipated. This poor performance was attributed to energy losses from the thruster which were much higher than expected from thermal analysis at hard vacuum. Predicted performance was derated to near the measured values when an empirical model for conductive heat losses was included in the thermal model.

Figure 2-46 presents measured filament temperature plotted against measured heater power. From these results, it is clear that neither the performance nor material temperature goals of the original test program were realized with the present apparatus. These limitations led to a revision of the test goals described below.

## HIGH PERFORMANCE RESISTOJET SEALED-CAVITY TEST RESULTS



C11232-32

Figure 2-46

### 2.2.4 Results and Conclusions from Phase II

At this point in the program NASA-LeRC agreed to conduct an evaluation of material creep at high temperature. The task was eliminated from the HPSPR program. In addition, NASA-LeRC agreed to conduct experiments on N<sub>2</sub>H<sub>4</sub>/wire compatibility and this task was also eliminated from the HPSPR program.

Since the data from these tasks were critical for continuing work on radiative heater concepts, a decision was made to suspend all other program activities for a period of approximately 10 months during the collection of this data.

During this period of time members of the user community provided input indicating that greater thrust efficiency at moderate power was needed by near term spacecraft for N-S stationkeeping missions. Immersed heater designs are inherently better in this regard. This input to NASA-LeRC led to the next phase of the program. The creep and wire compatibility data were not incorporated into the HPSPR program and is outside the scope of this report.

#### 2.2.4.1 Review of Test Goals

The original goals of the sealed-cavity thruster tests, as defined by NASA and RRC during the restructuring of the Phase II program plan are listed below, in order of importance:

1. Demonstration of 750 W sealed-cavity operation at enhanced performance levels (greater than 330 sec Isp) for an extended period of time, arbitrarily set at 250 hours.

2. Evaluation of the creep performance of HIP'ed rhenium at the heat exchanger temperatures necessary for 330 sec Isp and above. Examination of limited rhenium creep data available in the literature indicates that a steady-state outer body temperature of 4200°F or higher will be required to induce measurable creep with a reasonable test duration (100 to 200 hours).
3. Evaluation of the effect of a pressurized cavity on heater filament evaporation rates. Examination of the sometimes conflicting tungsten evaporation rates available in the literature indicates that a steady-state filament temperature of 5000°F or higher will be required to demonstrate significant evaporation retardation with a reasonable test duration.
4. Evaluation of long-term compatibility of rhenium with hydrazine decomposition products at temperature.
5. Evaluation of other effects of the pressurized cavity, such as axial heat conduction and heat exchanger inner-body creep.

It was clear from the test results that goal number 1 was not possible with the sealed-cavity heater design. It was also clear from the results of the earlier "fast-track" tests with a vented cavity heater that performance levels above 330 sec Isp at 750 W was achievable when the constraint to use an existing heat exchanger was removed.

A review of test goals eliminated goal number 1 above, and retained the remaining goals in the order listed. However, test results indicated that heat exchanger outer body temperatures and heater filament temperatures necessary to achieve the remaining goals were not achievable with the sealed cavity thruster. A parametric study using an updated thermal model was undertaken to determine whether minor improvements to the thruster design could improve performance enough to meet the test goals.

#### 2.2.4.2 Thermal Analysis

The wide discrepancy between the sealed-cavity test results and the thermal model predictions pointed out the need for an examination of the thermal model to determine the cause of the discrepancy. The goals of this study were two-fold:

1. Bring thermal model predictions into better agreement with the sealed-cavity test results.
2. Suggest hardware or test modifications which would allow required temperatures to be achieved in future tests.

To address the first goal, three tasks were completed:

1. Comparison of the actual hardware drawings and model to verify the conductive interface used between the mounting bracket and the thrust stand.
2. Recalculation of the conductance network within the sealed-cavity heater sub-assembly per the hardware drawings as built.
3. Recheck the radiative connections between the heat exchanger outer surfaces and the ambient environment.

Although minor modifications were made to the model as a result of these investigations, no significant changes in predicted temperatures or performance resulted.

The only way the model could be made to predict performance near the sealed-cavity test results was by invoking assumptions which led to gas conduction/convection losses significantly higher than had been thought reasonable. This high-conduction assumption (Case 1) was consistent with thermal models used on several previous programs at RRC. However, it was not consistent with the results of the fast-track tests. Fast-track test results correlated well with a similar thermal model in which very low gas conduction losses were assumed. This dilemma remains unresolved.

An effort was made to increase surface emissivities assumed by the model in order to reduce various surface temperatures. Somewhat better agreement was achieved by this method, although unrealistically high emissivities must be assumed in some cases to get reasonable agreement with test results. Without more definitive emissivity data for specific thruster surfaces, further attempts at better agreement with measured surface temperatures would be little more than an exercise in curve fitting. This effort was therefore discontinued.

In general, the model underpredicted performance and overpredicted most external temperatures. Surface emissivities were pushed as high as was justifiable in light of literature data, and the large amounts of heat rejected from the thruster via ambient gas conduction. Predicted performance can be adjusted by assuming a different nozzle efficiency. However, reducing the predicted external temperatures without unrealistic assumptions concerning surface emissivities was not possible. Surface emissivity data specific to the Mo/41Re heater assembly and the rhodium plated heat exchanger outer can would be very helpful in this regard. The largest unknown in this entire analysis was the validity of the assumptions used for gas conduction heat losses.

One aspect of the model that warrants revisiting is the correlation between the radiation view factors used and the actual hardware in the region of the heater subassembly. Due to the complexity of the geometry in this region and the various materials used, it is possible that the radiative losses from this region were considerably higher than predicted by the model.

### 2.3 PHASE III — IMMERSED HEATER THRUSTER EXPERIMENTS

In the original Phase III plan the resistojet design was to be updated to incorporate Phase II results. Following approval of a design review, two thrusters and associated tooling and test equipment were to have been fabricated and assembled.

Based on the results of the sealed cavity tests in Phase II the HPSPR program was redirected to pursue an immersed heater resistojet (similar to the TRW HiPEHT design) concept following a joint decision between NASA-LeRC and RRC. The immersed heater design was expected to have an advantage over the radiative design by means of a higher electrical-to-thrust conversion efficiency. If higher efficiencies were obtained, higher Isp levels (315 to 325 seconds) were expected at relatively low power levels (450 to 600 Watts). The combination of higher performance at lower power would provide a unique capability with a high potential for the thruster to be utilized in a flight program. For this phase, the program had an Isp goal of 315 seconds and a life goal of 100,000 lb-sec. total delivered impulse. The performance and life goals were to be achieved with a power input of no more than 750 W.

The primary technical challenge was the reduction or elimination of heat exchanger (HX) degradation with life noted in IR&D testing during 1988 (Roberts, 1988). Material loss from the tungsten heater filament, and subsequent redeposition of that material within the HX cavity caused unacceptable degradation of the HX at I<sub>sp</sub> levels near 315 seconds. Analysis indicated that operation at filament temperatures exceeding 4200°F resulted in excessive filament mass loss. The design effort modified the HX injector and close-coupled the HX to an MR-501 type gas generator (GG).

Drawings for the immersed heater thruster are included in Appendix D.

### 2.3.1 Gas Generator to HX Coupling

The close-coupled HX/GG design concept was based on test and manufacturing experience at RRC, TRW and NASA-LeRC. The design incorporated flight-type structural, thermal, and interface features where possible. Limited thermal analyses were performed to ensure that seal temperatures were within limits, and that the support structure did not increase thermal conductive losses.

The MR-501 design GG outlet was modified to incorporate a face-seal interface with the HX to allow easy replacement of the HX. A gas tight seal was required which would withstand thermal cycling from ambient to 1400°F. A metal "V-ring" type of seal was selected over a grafoil seal to avoid reactions between N<sub>2</sub>H<sub>4</sub> decomposition gases and carbon. The V-seal material selected was Hastalloy X with a soft nickel coating which is compatible with N<sub>2</sub>H<sub>4</sub> decomposition products and has adequate yield strength at temperatures well above 1400°F.

A filter to prevent particulates from reaching the heater filament was incorporated just downstream of the GG exit. A chamber pressure (P<sub>c</sub>) tap was incorporated between the GG and the HX.

The primary structural support of the HX was via the GG flange interface. Therefore the GG support was designed to be structurally adequate for this purpose and to minimize the thermal soak-back from the GG and the HX to the propellant valve.

### 2.3.2 Heat Exchanger Description

The immersed heater HX consists of three main parts:

1. Heater
2. Nozzle body
3. Adaptor tube interface

The heater included the filament, support post, and electrical pass-through. Mo/50Re tubing was used for the electrical pass-through assembly. The nozzle body encloses the heater filament, injects the propellant into the HX cavity, and expands the flow out the nozzle. The adaptor tube feeds decomposed N<sub>2</sub>H<sub>4</sub> from the gas generator to the nozzle body injector, and also provides structural support for the HX assembly. One HX and four heaters using the TRW HiPEHT design were procured.



TRW applied extensive immersed heater design experience to the identification of HX design modifications focused on altering the injector flow field to enhance heat transfer from the wire into the exhaust gas. Injector port location, diameter, and orientation relative to the heater filament, as well as the number of injection ports were possible variables. A redesign of the HX body cross-section and a reduction in the HX body diameter were also considered and rejected. Details of the injector redesign analysis are included in Appendix F.

The principal change from the HiPEHT configuration was replacing the injector slot with an array of holes. The nozzle body and injector design are shown in Figure 2-47. The injector ports were sized to a nominal diameter of 0.010 in. to closely match the discharge coefficient of the HiPEHT.

### 2.3.3 Gas Generator

RRC supplied a standard MR-501 type GG for use in the program. For the thrust levels likely with the immersed HX, the standard MR-501 gas generator life expectancy was sufficient to meet a 100,000 lb-sec life requirement. A mating flange was added to the design. The modified GG is shown in Figure 2-48. The support hardware obtained from RRC for use during this Phase included:

- Gas generator
- Mounting structure
- Shielding
- Tooling
- Fluid resistor
- Propellant valve

The overall assembly is shown in Figure 2-49. An adaptor plate was made to support the input electrical leads for attachment to the heat exchanger. To accommodate the MR-501 mount structure geometry, the HX gas inlet tube was lengthened from 0.81 to 1.25 inches. Immediately downstream of the GG bed plate three 100 X 100 mesh screens formed the GG outlet filter. The outlet/adaptor was welded to the GG. A pressure tap was brazed in place.

### 2.3.4 Immersed Heater Thruster Testing with N<sub>2</sub>H<sub>4</sub>

The test using the new HX was planned to meet the following objectives:

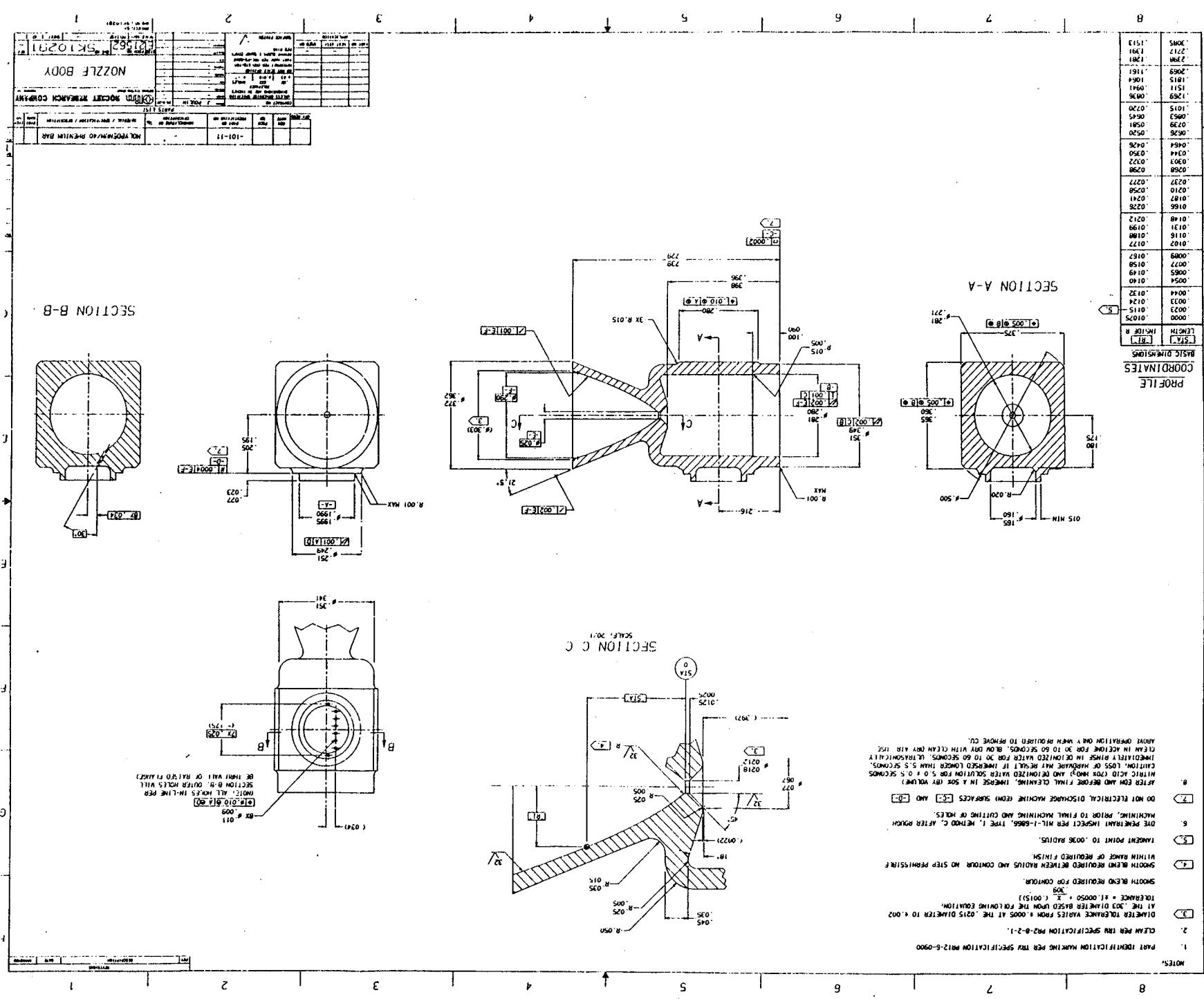
- a. Determine the maximum allowable heater temperature to minimize HX degradation.
- b. Determine the effects of the injector modification. Data from previous IR&D testing using the unmodified HiPEHT HX design were compared to the data from this test.

The basic flow of testing planned is shown in Figure 2-50. A performance map of the IR&D thruster was completed prior to installation of the immersed heater thruster to verify operating points and validate the test set-up. The IR&D thruster had a slot injector and a standard hot gas transfer tube from the gas generator.

#### 2.3.4.1 Fuel and Pump/Chamber System

The testing was performed in RRC's altitude chamber number 6, which is shown schematically in Figure 2-51. The altitude chamber is a single wall, 98-ft<sup>3</sup> vacuum tank connected to two high-capacity mechanical vacuum pumps with 3,850 and 2,630 cfm capacity. Mechanical pumps maintained the cell pressure to less than 150 mtorr throughout testing.

91-R-1553



## IMMERSED HEATER NOZZLE BODY DESIGN

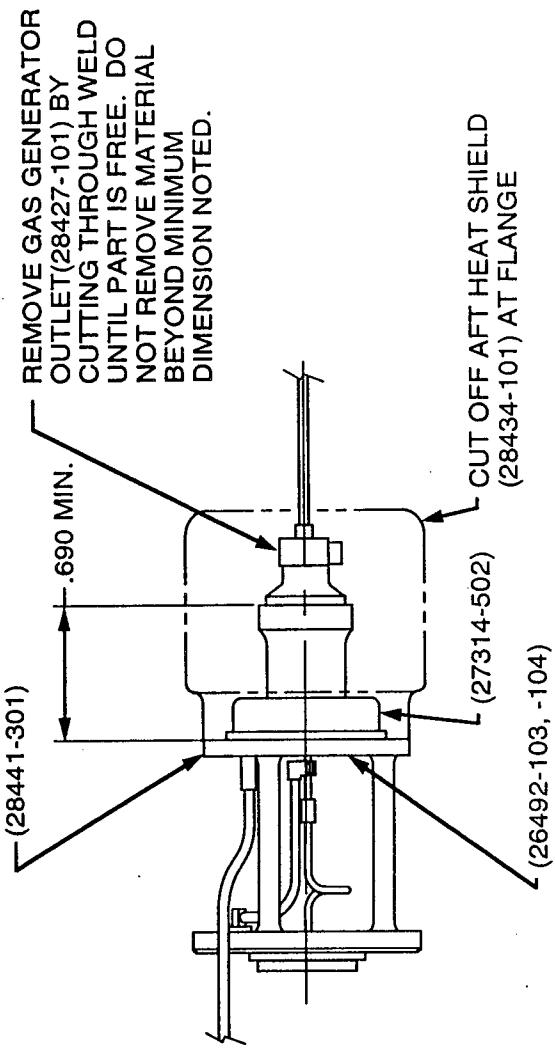
**Figure 2-47**

Figure 2-48

2-69

C11235-68

## GAS GENERATOR MODIFICATION SEQUENCE



-101-11

MAKE FROM ASSEMBLY OBTAINED  
BY REMOVING GAS GENERATOR FROM  
EHT ASSEMBLY (28420-305)

Figure 2-49

2-70

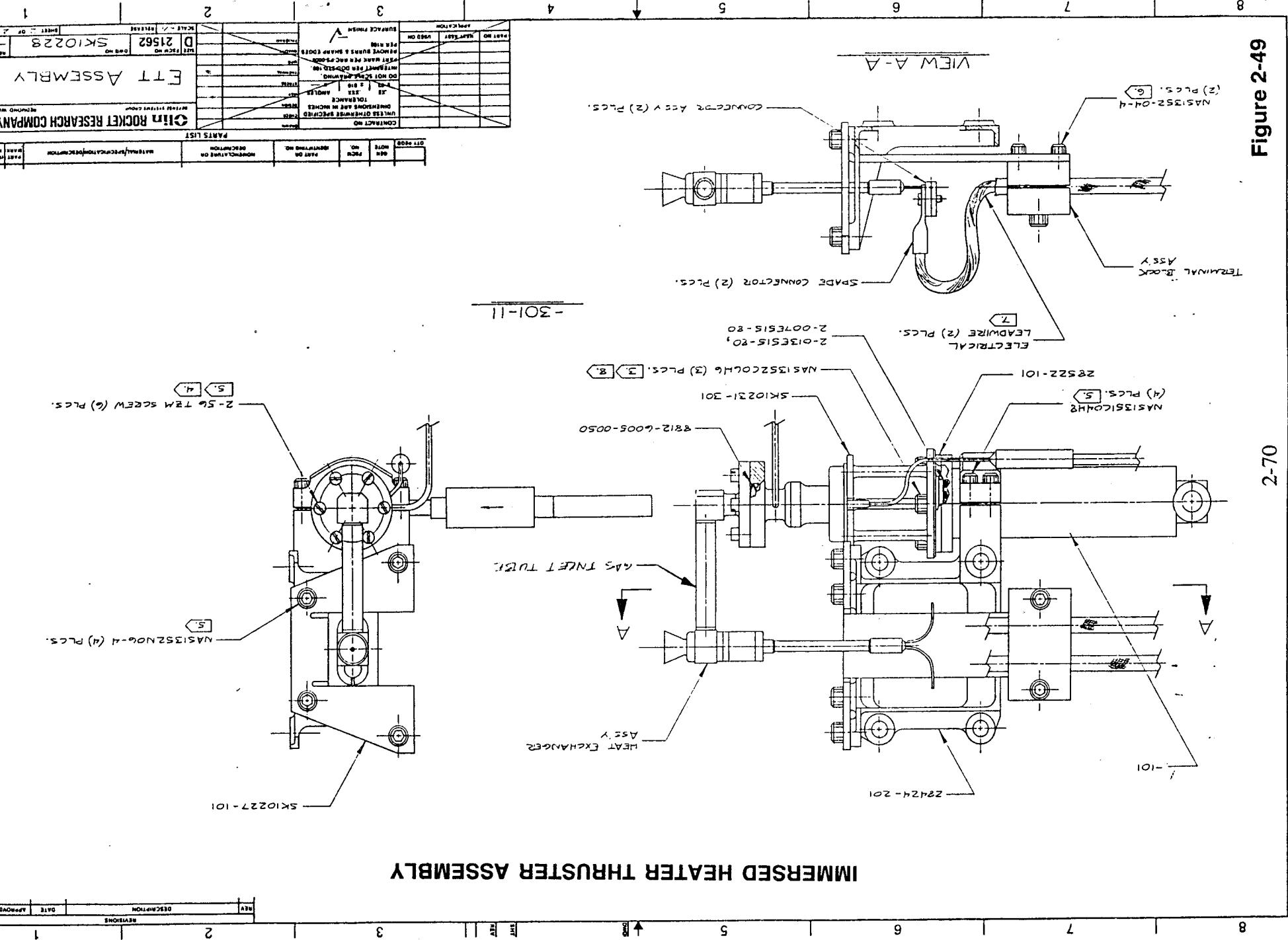
C

D

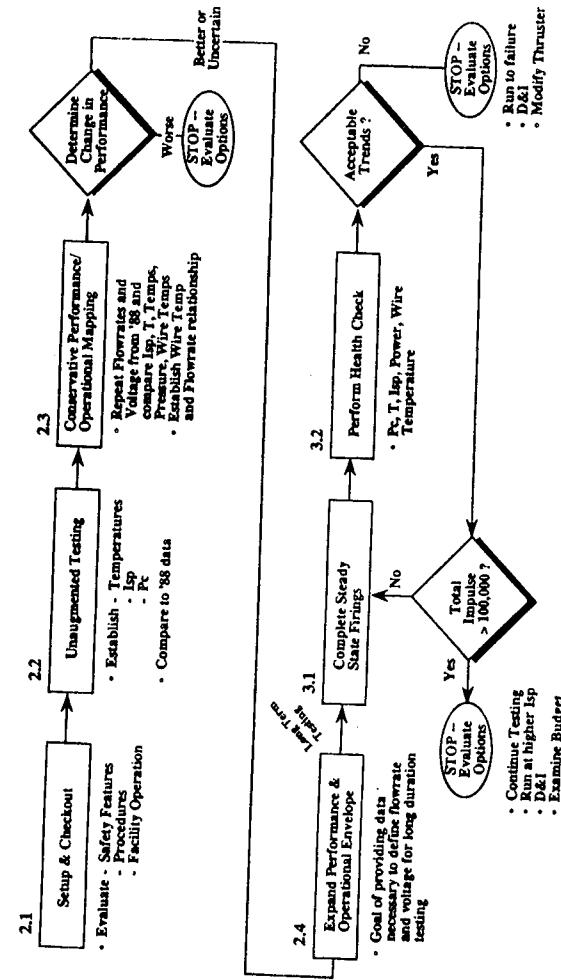
A

B

### IMMERSED HEATER THRUSTER ASSEMBLY



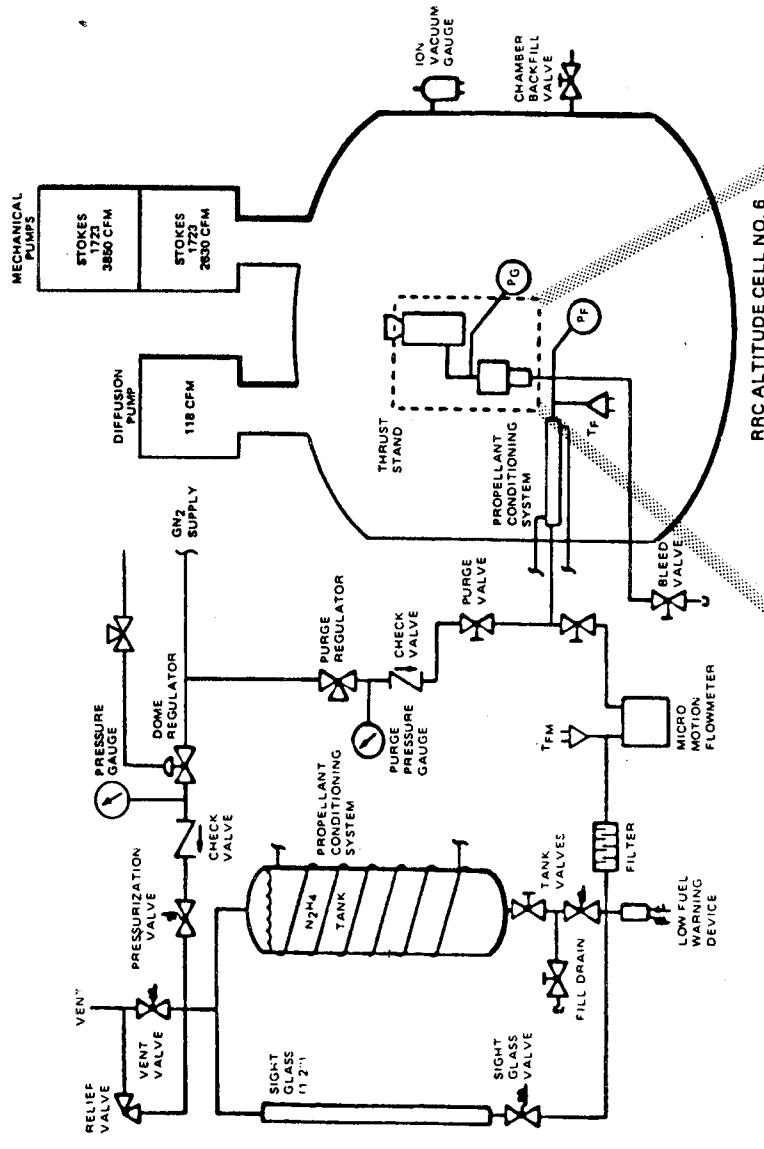
## IMMERSED HEATED THRUSTER TEST FLOW PLAN



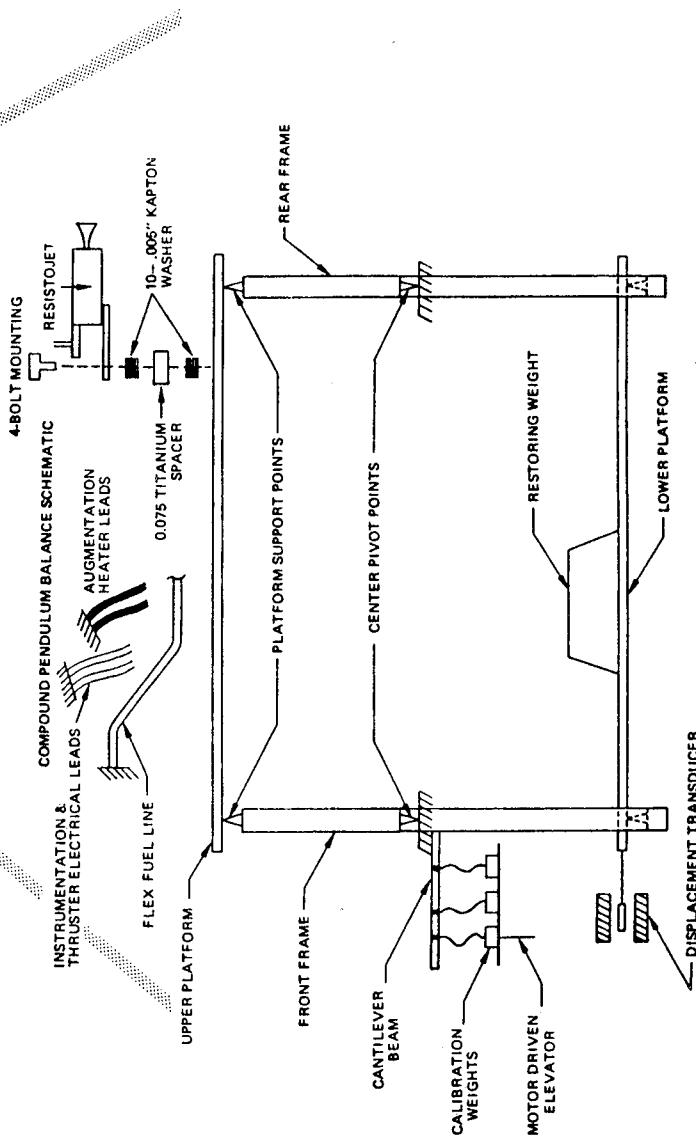
2-71

Figure 2-50

## N2H4 PROPELLANT SYSTEM THRUST STAND AND TEST CELL



11150-68B



11038-34

Prior to connecting the inlet line to the thruster, propellant samples were taken for chemical and particulate analysis to certify the propellant system cleanliness. All testing was completed using hydrazine meeting MIL-P-26536C, Amendment 2, High Purity Grade.

### 2.3.4.2 Performance Mapping

Tests were conducted to establish a trend of  $I_{sp}$  versus wire resistance (temperature) at various flow rates. Each test lasted for approximately 30 minutes to ensure that thermal equilibrium was reached. The first testing objective was to repeat the low filament temperature (higher flow rate, lower voltage) operating points obtained in previous IR&D testing. Additional tests including a broader range of feed pressures and voltages were completed to determine the performance envelope of the redesigned thruster. The specific objective was to determine filament temperatures at an  $I_{sp}$  of 315 seconds and greater. These data were used to determine the life test conditions.

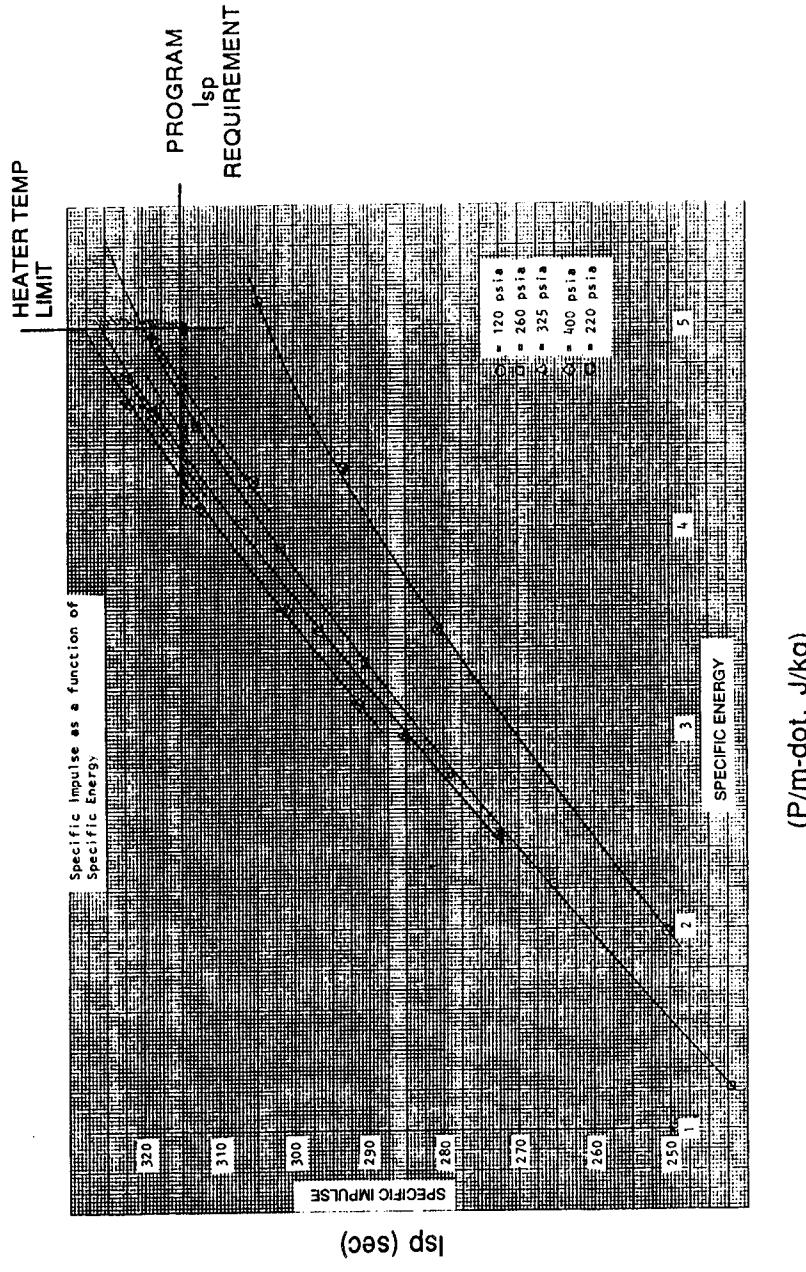
The IR&D version of the immersed heater thruster was tested to validate the performance measurements of the test cell setup. Good agreement was achieved with 1988 data. Additional performance mapping was performed. Trends of  $I_{sp}$  as a function of specific energy (power/mass flow,  $P/m\cdot\text{dot}$ ) were established for three mass flow rates. Performance testing of the new immersed heater thruster established  $I_{sp}$  trends as a function of  $P/m\cdot\text{dot}$  for several mass flow rates as shown in Figure 2-52. The results indicated a possibility of completing a life test using a shallow blow-down (300 to 200 psia) of 100,000 lbf-sec at an  $I_{sp}$  greater than or equal to 315. The peak power would be about 720 watts. Performance of the new thruster and the IR&D thruster were virtually identical. No performance improvement was obtained from the design changes.

Testing prior to the beginning of the life test resulted in a test anomaly characterized by increased  $P_c$  and decreased mass flow rate. A crystalline metallic material was observed in the throat, shown in Figure 2-53. Testing was stopped pending identification of the material. The thruster was removed from the test cell, decontaminated and the crystalline material in the throat was subjected to SEM analysis. There was also streaky black material deposited in the nozzle. The SEM analysis showed that the material in the throat was tungsten. Trace quantities of titanium and calcium were also found. The source of these materials was unknown.

Following the hardware failure a meeting was held with TRW to review the test results and form a consensus for future action. Five options were considered:

1. Resume parametric or life characterization testing using the immersed heater thruster to fully characterize the failure (i.e. determine damage thresholds, test anhydrous propellant, test with Argon, put witness plates in the plume).
2. Lower the wire temperature to a "safe" level and continue the life test at a lower performance level (e.g. 306 sec  $I_{sp}$ ).
3. Modify the injector to improve the Reynolds number and/or change the filament wire to tungsten/rhenium wire and determine the performance of the new thruster.

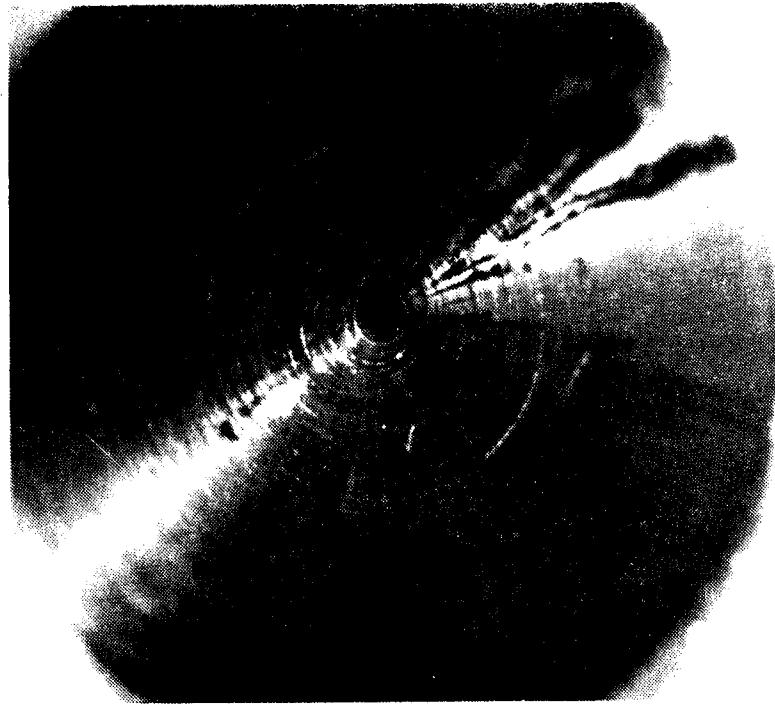
## IMMersed HEATER THRUSTER SPECIFIC IMPULSE AS A FUNCTION OF SPECIFIC ENERGY



2-74

Figure 2-52

CLOSEUP OF IMMERSSED HEATER THRUSTER NOZZLE  
SHOWING CRYSTALLINE MATERIAL IN THROAT



4. Complete a low power, high mass flow performance test of the rhenium heat exchanger vented radiative wire (i.e. "fast track") thruster. Possibly enter a life test.
5. Perform a material characterization test (e.g. tungsten, tungsten/rhenium, and other wire over a range of temperatures in decomposition products for several hundred hours).

The recommendations of the meeting were the following:

1. Summarize power, thrust, efficiency, and  $I_{sp}$  data for the immersed heater thruster, the vented radiative wire thruster and other low power EP devices. Present a summary to spacecraft managers and assess mission sensitivities to these parameters.
2. Based on the outcome of the above assessment assemble and test either a further improved version of the immersed heater thruster or the vented radiative wire EHT.

Figure 2-54 compares  $I_{sp}$  as a function of specific energy (p/m-dot) for the "fast track" and immersed heater thrusters. Specific energy for the immersed heater thruster (immersed heater thruster) is limited by the allowable wire temperature. Figure 2-55 shows that the immersed wire thruster is capable of greater thrust levels than the radiative wire thruster. Without flow over the filament the radiative wire thruster can tolerate higher wire temperatures. Figure 2-56 illustrates the relationship of thrust efficiency to specific impulse. The radiative wire offers high specific impulse but produces less thrust for equivalent power than the immersed wire thruster.

## HIGH PERFORMANCE STORABLE PROPELLANT RESISTOJET PROGRAM

### Specific Impulse as a Function of Specific Power

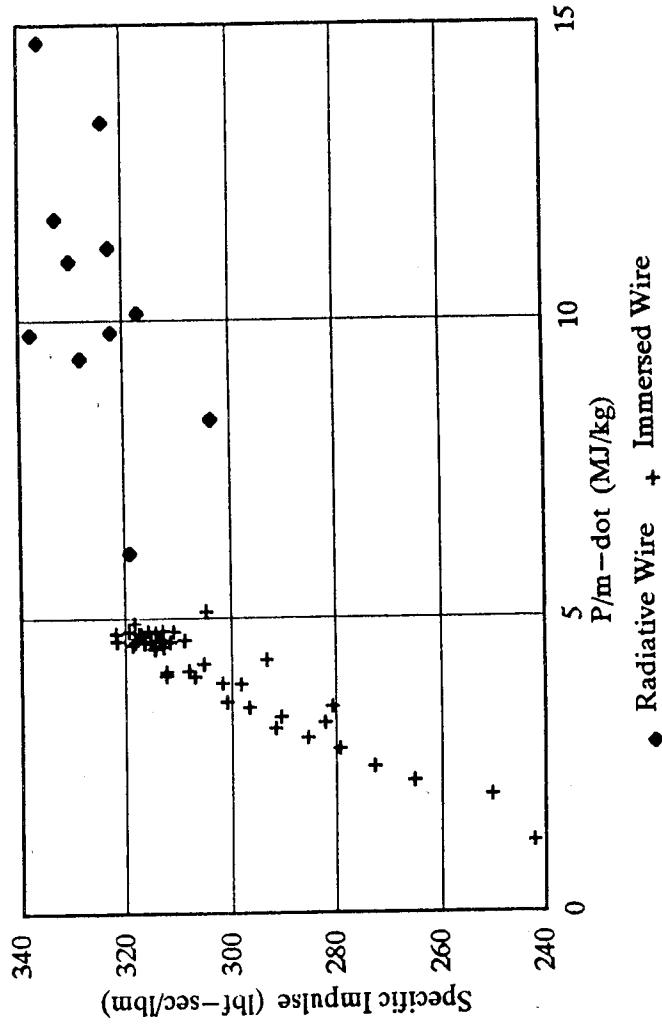


Figure 2-54

## HIGH PERFORMANCE STORABLE PROPELLANT RESISTOJET PROGRAM

### Thrust as a Function of Heater Temperature

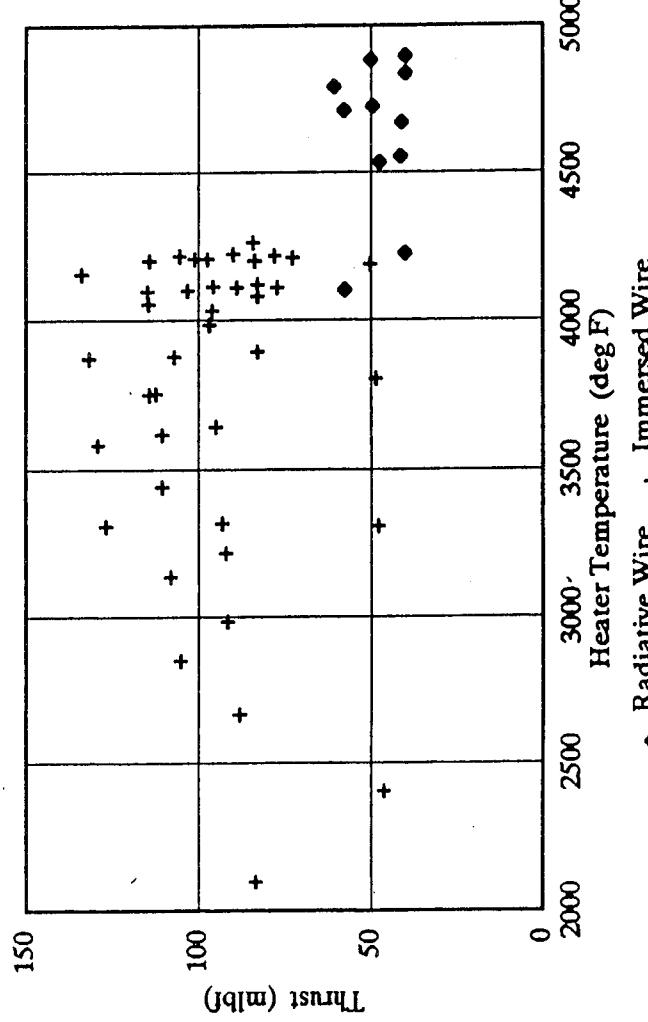
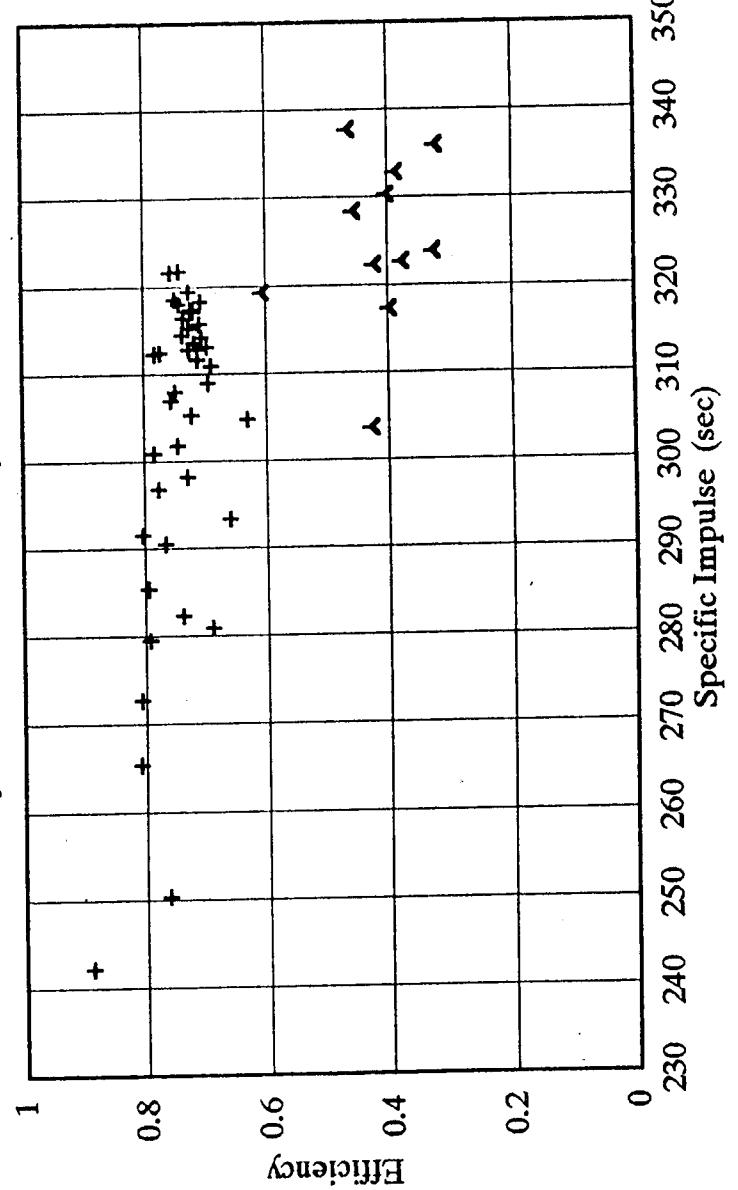


Figure 2-55

## HIGH PERFORMANCE STORABLE PROPELLANT RESISTOJET PROGRAM

### Efficiency as a Function of Specific Power

Figure 2-56  
2-77

### 2.3.5 Immersed Heater Follow-up Testing

After reviewing the above results the recommendations were amended as follows:

1. Resume parametric or life characterization using the immersed heater thruster and fully characterize the failure.
2. Lower the wire temperature to a "safe" level (e.g. 3900°F and continue the life test at a lower performance level (e.g. 310 sec I<sub>sp</sub>). Continue this test until program resources are depleted.

The immersed heater thruster was reinstalled in the test cell. Argon testing was attempted first. Five tests were completed with filament temperatures ranging from 3800° to 4200°F. At 4200°F, there was a gradual downward drift of the heater current. Following the test the cold resistance had changed from 0.343 ohms to 0.395 ohms. No crystalline material was observed in the throat or elsewhere. The thruster was blue indicating that gO<sub>2</sub> may have reacted with the molybdenum in the Mo/41Re nozzle body. The witness plates shattered so that data were not obtained as planned.

The original plan was to complete the argon analysis and start N<sub>2</sub>/H<sub>2</sub> testing. However, the results of the argon testing showed that testing with N<sub>2</sub> would introduce unacceptable risk of additional damage to the hardware. A decision was made to proceed with H<sub>2</sub> testing. Test software was modified to compensate for heater resistance changes. The slope of the resistance vs. temperature curve was modified to compensate for mass loss in previous testing. Two H<sub>2</sub> test series were completed at 250 and 500 W with filament temperatures up to 4300°F with no heater degradation. An NH<sub>3</sub> test series was completed and there was no heater degradation.

A special Ultra-Pure N<sub>2</sub>H<sub>4</sub> propellant sample was obtained with lower than usual trace quantities of CO<sub>2</sub> and H<sub>2</sub>O. This propellant minimized sources of atomic oxygen that was suspected as a mechanism causing filament degradation.

A separate feed system for the Ultra-Pure N<sub>2</sub>H<sub>4</sub> propellant was constructed to assure that the specially prepared anhydrous propellant absorbed no water from the ambient atmosphere. The feed system was cleaned, dried, and purged with dry GN<sub>2</sub>. The special propellant was shipped in reagent bottles. For propellant loading the bottles were tapped in a GN<sub>2</sub> atmosphere with a dual hole stopper. Propellant was vacuum transferred into the feed system. In this way the only material the propellant was exposed to was dry GN<sub>2</sub>.

An unaugmented burn-in using the special N<sub>2</sub>H<sub>4</sub> was completed. The unaugmented N<sub>2</sub>H<sub>4</sub> P<sub>c</sub> had returned to the value recorded prior to the Ar tests (179 vs 186 psia). This suggested that testing with reducing gases (i.e. H<sub>2</sub> and NH<sub>3</sub>) had restored the hardware to its original condition. Most likely a small amount of material deposited around the throat was eliminated by testing with these gases.

The Ultra-Pure N<sub>2</sub>H<sub>4</sub> propellant test was performed at filament temperatures of 4000°F and 4100°F. A decline in current at constant voltage indicated further heater mass loss. The conclusion drawn for N<sub>2</sub>H<sub>4</sub> was that any heater temperature that would produce an

acceptable I<sub>sp</sub> resulted in unacceptable filament mass loss rates. A cold resistance check revealed the resistance was still 0.395 ohms. Evidently, the declining current at constant voltage was a more sensitive indication of heater mass loss than the cold resistance.

### 2.3.6 Analysis of Filament Mass Loss Mechanism

Using a chemical kinetics program the quantities of oxygen bearing species in the N<sub>2</sub>H<sub>4</sub> decomposition products were estimated. Atomic oxygen was of particular concern since it is the most reactive species. Argon with 1 ppm oxygen produces an O-atom concentration 3 orders of magnitude higher than hydrazine with 0.5 percent water at the typical filament temperature. Hydrogen with 0.5 ppm oxygen produces an O-atom concentration 3 orders of magnitude lower than hydrazine with 0.5 percent water at the filament temperature. Figure 2-57 presents the results of this analysis for compositions of the gases used in the test sequence. Pressure had only a minor effect on the concentration level.

A close examination of the argon test data revealed that the heater degradation began to appear at a 4000°F filament temperature. A complete set of data plots for the top 3 temperatures were generated and are presented in Figure 2-58. The negative slopes clearly seen for the 4100° and 4200°F cases correspond to heater mass loss. The H<sub>2</sub> test data do not show current declines. Cold resistance measurements also showed that there was no additional mass loss.

Testing of the immersed heater thruster performance with argon, H<sub>2</sub> at 230 W, H<sub>2</sub> at 500 W, and NH<sub>3</sub> are summarized as follows:

I<sub>sp</sub> for Resistojet Gas Test Cases (sec)

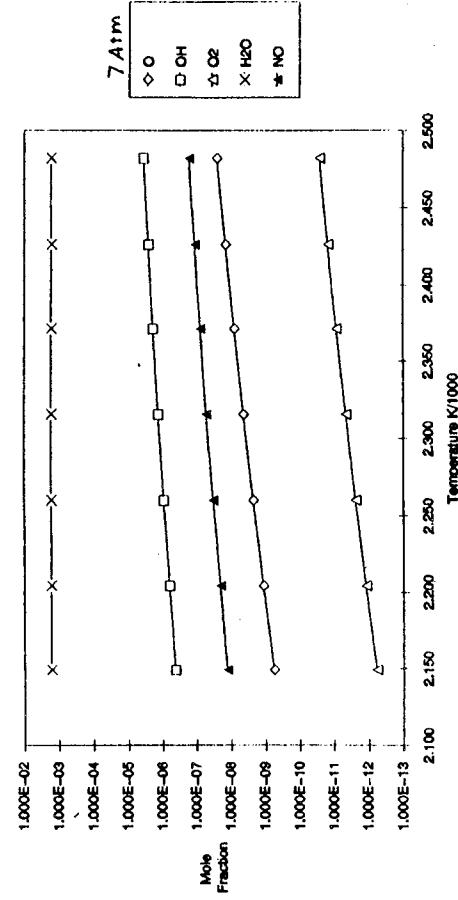
Case	Filament Temperature (°F)				
	3800	3900	4000	4100	4200
Argon	123	124	126	126	123
H <sub>2</sub> 250 W	461	472	474	482	490
H <sub>2</sub> 500 W	—	—	527	537	536
NH <sub>3</sub>	246	241	240	242	252

Time at Temperature (min)

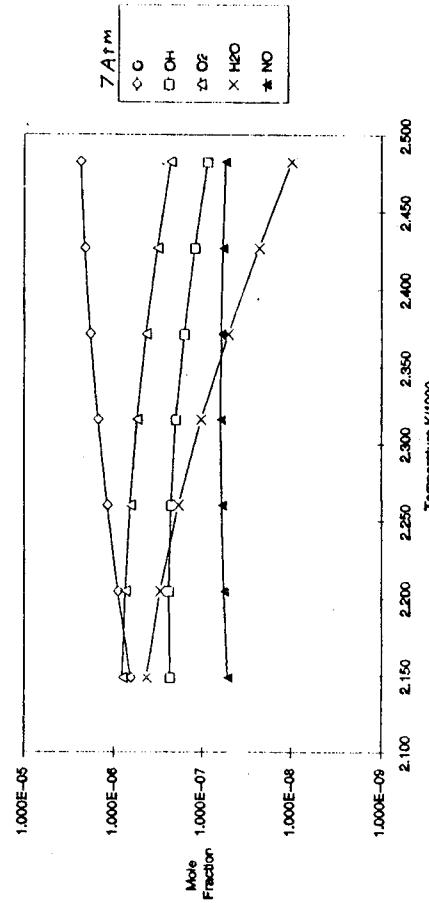
Case	Filament Temperature (°F)				
	3800	3900	4000	4100	4200
Argon	33	46	44	41	178
H <sub>2</sub> 250 W	80	78	68	70	131
H <sub>2</sub> 500 W	0	0	70	71	71
NH <sub>3</sub>	106	67	77	68	72
Total (hrs)	3.7	3.2	4.3	4.2	7.5
					1.2

## CONCENTRATION OF OXYGEN BEARING SPECIES FOR THREE PROPELLANTS

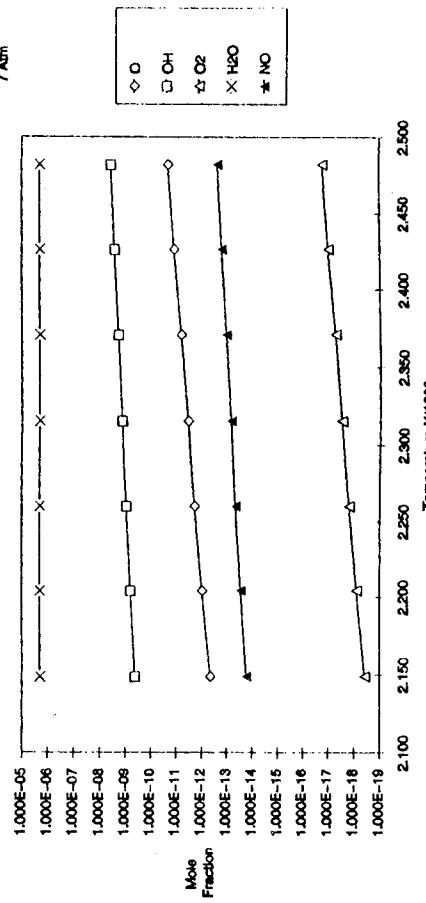
Equilibrium Composition of 0.5% Water in N<sub>2</sub>H<sub>4</sub>



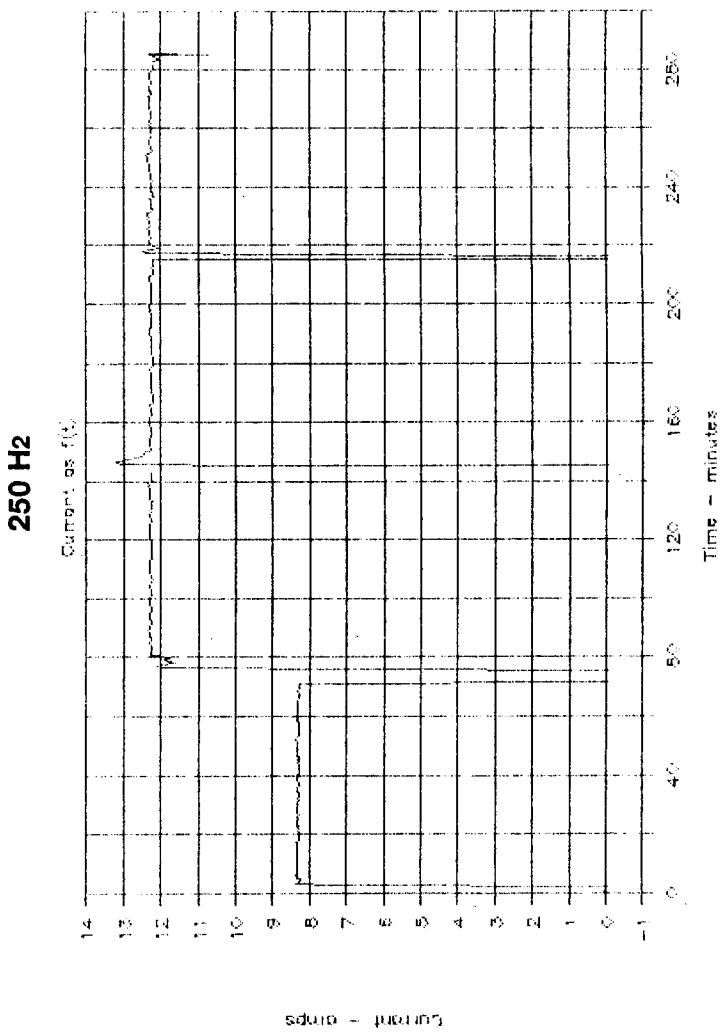
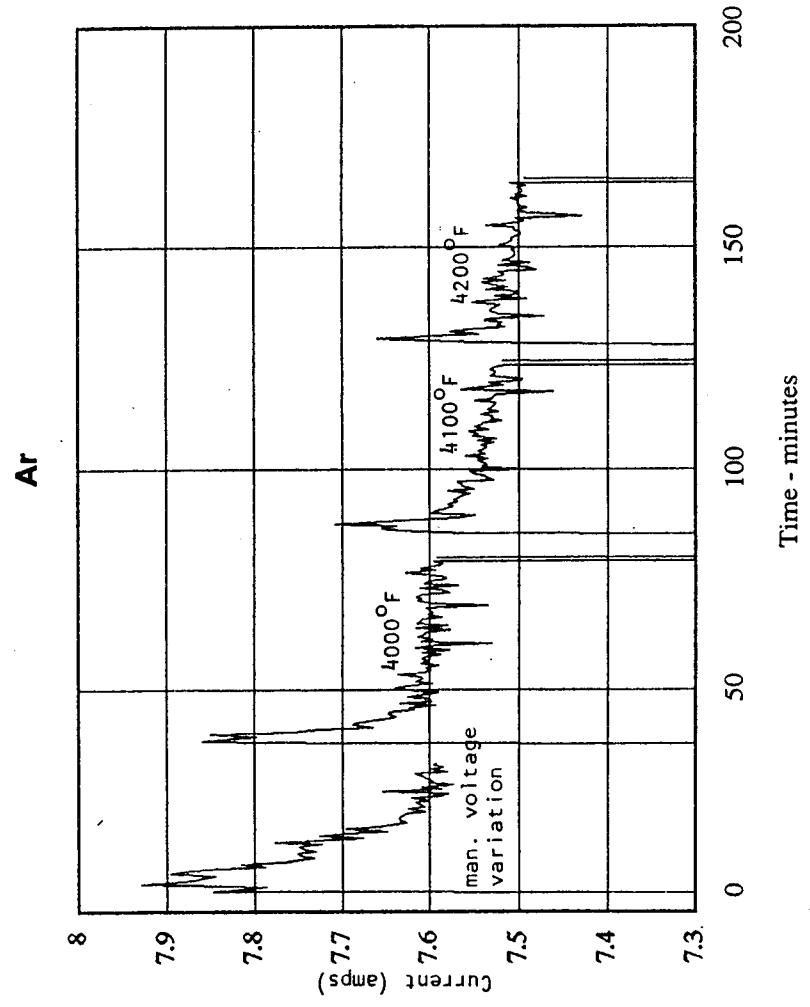
Equilibrium Composition of 1PPM Water & 1PPM O<sub>2</sub> in Argon



Equilibrium Composition of 1PPM Water & 0.5 PPM O<sub>2</sub> in Hydrogen



**CURRENT AS A FUNCTION OF TIME WITH HEATER DEGRADATION OCCURRING (Ar) AND WITHOUT SIGNIFICANT HEATER DEGRADATION (250W H<sub>2</sub>)**



**Figure 2-58**

There was a grand total of 24.1 hours of testing at temperature in the gas test cases. The first 5.7 hours with argon resulted in a 15% heater degradation. The remaining total of 18.4 hours resulted in no additional measurable heater degradation.

### 2.3.7 Results and Conclusions from Phase III

The key measured variable for the N<sub>2</sub>H<sub>4</sub>, H<sub>2</sub>, NH<sub>3</sub>, and Ar tests turned out to be current at constant voltage. Decreases in current were proportional to the loss of mass from the heater filament wire. During constant power testing, the voltage was continuously adjusted to compensate for heater variation. This masked the current decline effect. During the argon test series voltage was held constant and current was allowed to vary.

Figure 2-59 illustrates the current traces for each of the gas and dry N<sub>2</sub>H<sub>4</sub> tests at 4100°F (4000° and 4200°F are similar). The current decay is not visually apparent due to the scale. However, minor changes in a one hour test have major implications for a 100,000 lb-sec impulse requirement. Some of the variation noted is caused by mass flow variation due to regulator cycling when using lower pressure gas cylinders.

Regression analysis of these curves was performed. The start and stop transients were eliminated by calculating regression equations only for "constant" voltage data points. One hypothesis was that atomic oxygen was related to heater degradation. Using values of 1 ppm for oxygen, 0.5 ppm for water, and 0.5 ppm for carbon dioxide for the gases tested, the pressure for test cases, and a temperature of 4100°F, a chemical kinetics code was used to determine the O-atom concentration. The results were as follows:

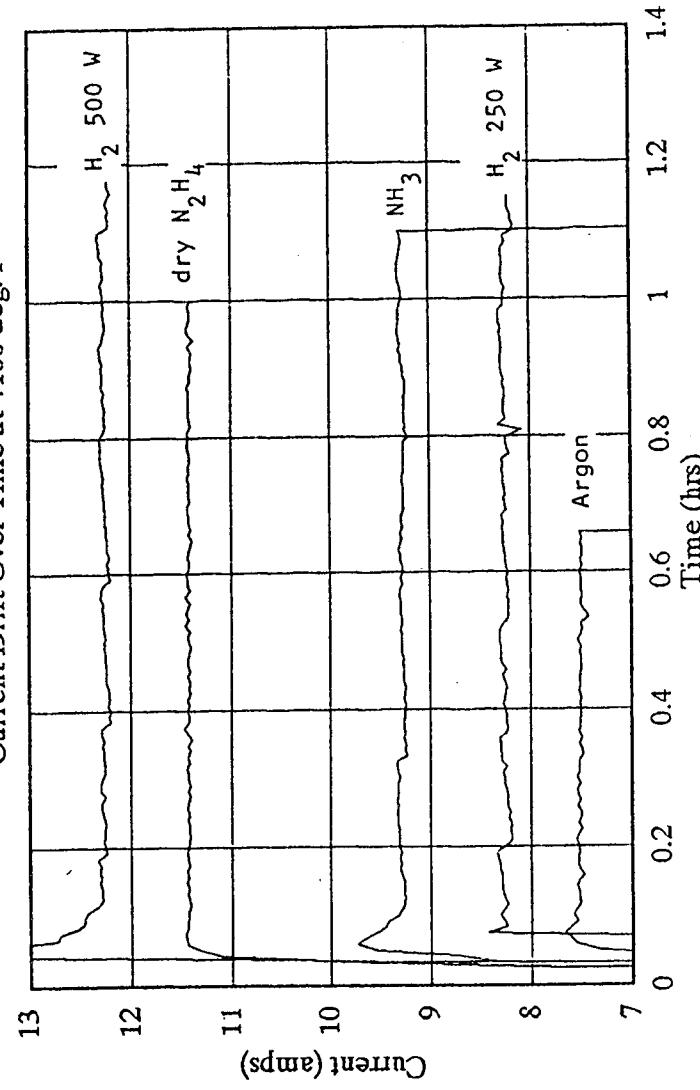
Test condition	O-Atom mole Fraction
Argon	2.74 x 10 <sup>-6</sup>
H <sub>2</sub> at 250 W	1.08 x 10 <sup>-10</sup>
H <sub>2</sub> at 500 W	5.29 x 10 <sup>-11</sup>
NH <sub>3</sub>	5.17 x 10 <sup>-11</sup>
Dry N <sub>2</sub> H <sub>4</sub>	1.15 x 10 <sup>-8</sup>

The second plot in Figure 2-59 presents the heater degradation rates derived from the regression analysis plotted as a function of O-atom concentration, all 4100°F. The data confirm the hypothesis that increases in O-atom concentration result in greater heater degradation rates. The figure also indicates that at 4100°F no propellant would be acceptable for a life test. For instance, dry hydrazine (0.15% water by weight) would have a 78% current decline in a 400-hour test. Clearly, the heater element would fail long before the end of the test. Even the minimum degradation rate would result in a 17% current decay in 400 hours.

In conclusion, an I<sub>sp</sub> of 315 cannot be attained with the immersed heater thruster unless the filament temperature is greater than 4100°F. However, at that temperature there are unacceptable rates of heater degradation.

## ATOMIC OXYGEN EFFECTS ON IMMERSED HEATER FILAMENTS

High Performance Storable Propellant Resistojet  
Current Drift Over Time at 4100 deg. F



High Performance Storable Propellant Resistojet  
 $dI/dt$  vs. [O] for Various Gases

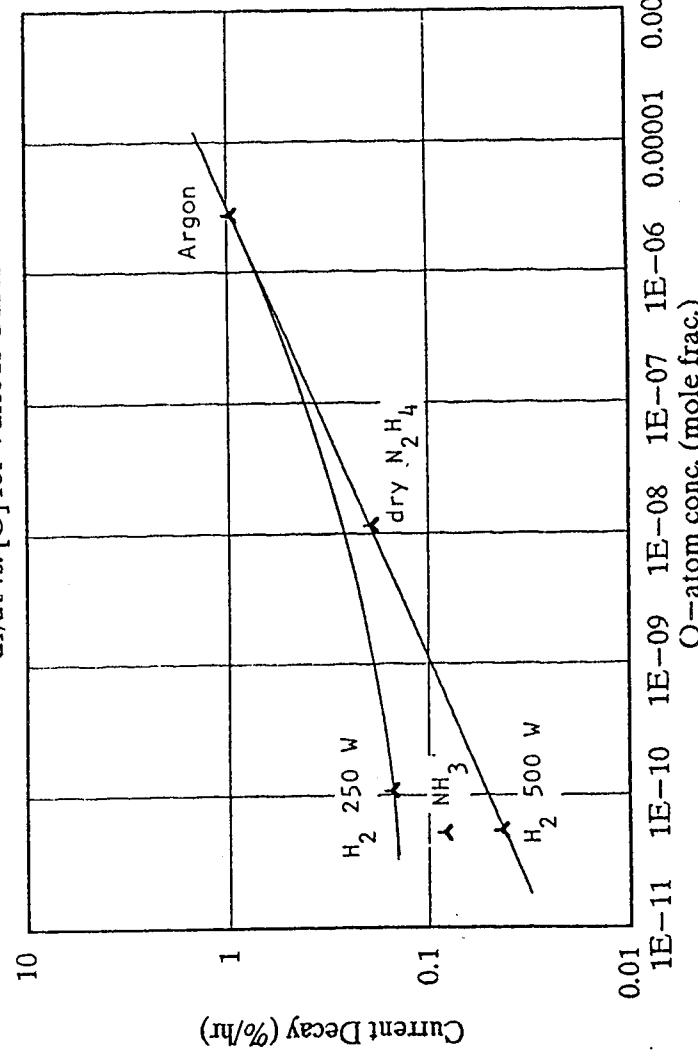


Figure 2-59

## 2.4 PHASE IV — MATERIALS AND GAS GENERATOR EXPERIMENTS

At the completion of the immersed heater test series it was apparent that the thruster was not suitable for a life test due to unacceptable rates of heater mass loss at acceptable performance levels. A number of new options to conclude the technical effort on the program were considered including:

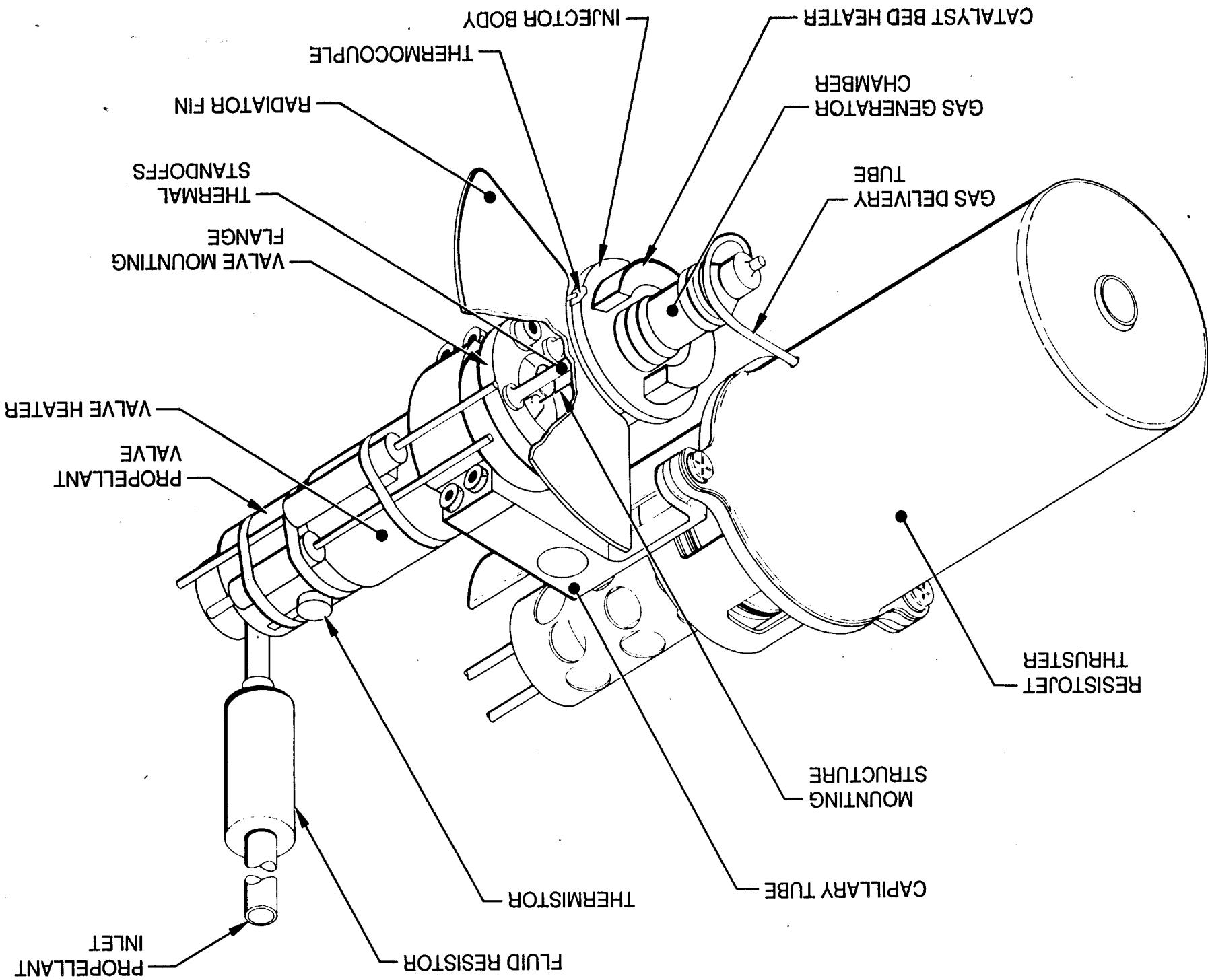
1. Evaluation of materials.
  - a. One proposed materials effort was to evaluate materials selected to avoid O-atom degradation at high temperature in N<sub>2</sub>H<sub>4</sub> decomposition products. For instance, rhenium additions to tungsten were predicted to help resist O-atom attack. Iridium or other refractory metal coatings were another approach.
  - b. Throughout the immersed heater thruster test, the leak performance of the heater passthrough seal was suspect. Another materials test program proposed was to characterize joining and sealing techniques for refractory metals and ceramics (perhaps with braze seals) at moderate temperature in N<sub>2</sub>H<sub>4</sub> decomposition products.
  - c. An effort was proposed to develop the application of high emissivity coatings to refractory metals (i.e. tungsten).
2. Low flow gas generators exhibit combustion instability due to minimal injector pressure drop and limited life due to feed-tube blockage. Low-flow resistojects needed improvements to existing gas generator designs to extend life and improve stability, and an effort was proposed to test a candidate design.
3. An ammonia performance characterization test of the TRW/EIT was proposed.
4. Another proposal was made to rebuild and test the rhodium heat exchanger EHT in an attempt to meet the program performance requirements.
5. Finally, a proposal was made to build and test an immersed heater thruster with W/3Re wire and a higher Reynolds number injector.

The HPSPR program was redirected to demonstrate an enhanced hydrazine catalyst bed for electrothermal thruster applications and investigate thruster material/hydrazine decomposition product interactions.

### 2.4.1 Gas Generator Demonstration

RRC provided a proprietary design gas generator, illustrated in Figure 2-60, at no cost to the HPSPRP program. This gas generator had a number of design enhancements to increase its life. The gas generator was transferred to the HPSPRP following assembly, vibration and functional tests. The HPSPRP performed a life test with a goal of 850 hours of hot firing time over a blow-down range of 300 to 220 psia. An outlet heater with an orifice sized to control flow simulated an electrothermal thruster during the life test.

The gas generator was installed in RRC Cell #6. The test configuration consisted of the gas generator (GG), valve/fluid resistor, and a resistance heater assembled to an MR-501 mounting structure. The heater, mounted in place of the thruster, was sized to provide electrothermal thruster heat loads simulating electrothermal thruster firing thermal environment.



RRC ADVANCED GAS GENERATOR

### 2.4.1.1 Gas Generator Test Results

Initial firing performance mapping consisted of 30 and 60-minute steady-state burns to characterize gas generator performance. Approximately 12 hours of total firing time were accumulated. Initial runs were conducted to correctly size an orifice for the GG outlet. Since the GG was not coupled to a thruster for this testing, orifices were installed in incremental sizes until representative values of GG chamber pressure ( $P_c$ ) over the desired flow rate range were achieved. An outlet orifice diameter of 0.014-in. was selected. Baseline data were acquired in this configuration.

GG steady-state performance was consistently stable throughout these runs with smooth chamber pressure and flow characteristics. The life test was on a 24-hour-per-day basis using computer control. The feed pressure blowdown schedule shown in Table 2-11 was followed.

Table 2-11  
LIFE TEST FEED PRESSURE BLOWDOWN SCHEDULE

	$P_f$ (psia)	Duty Cycle (hrs on/off)	Firings	Hours
Performance	220—300	0.5/AR	17	12
BOL Block	300	1-1.25/0.5-0.25	AR	178
MOL Block	257	1.25/0.25	344	430
EOL Block	220	1.25/0.25	184	230

As noted in Table 2-11, an adjustment in the duty cycle was made during the beginning-of-life block from 1 hour on/0.5 hour off to 1.25 hour on/0.25 hour off. This change was made after the temperature profiles on cool down were evaluated during the initial cycles of the life test. Within an 0.25-hour off period, the catalytic chamber temperature cooled below the minimum limit (200°F) required to restart the GG. Based on this evidence, the duty cycle was adjusted slightly. The total run time in a 24-hour period was therefore increased from 16 to 20 hours.

Life test data are shown in Figures 2-61 through 2-63. Figure 2-61 shows the chamber pressure and flow rate levels which correspond to the BOL feed pressure block. Small excursions in the data are due to differences in feed pressure levels established when the propellant tank is pressurized following refueling or test shutdowns. Figure 2-62 shows gas generator catalytic chamber temperature. The large drop in temperature between 36 and 41 hours was traced to a faulty thermocouple attachment which was repaired. The measured chamber temperatures between 900° and 950°F were consistent with expected values.

The quantity  $P_f - P_c/m^2$  shown plotted in Figure 2-63 is useful for observing changes in pressure losses in the test hardware. Of specific interest is the pressure drop through the GG injector, which is typically very small at the arcjet flow rates. If the pressure losses through both the fluid resistor and valve are assumed to remain constant, an increase or decrease in  $P_f - P_c/m^2$  can be associated with a change in GG pressure drop. For example, life-limiting

Figure 2-61

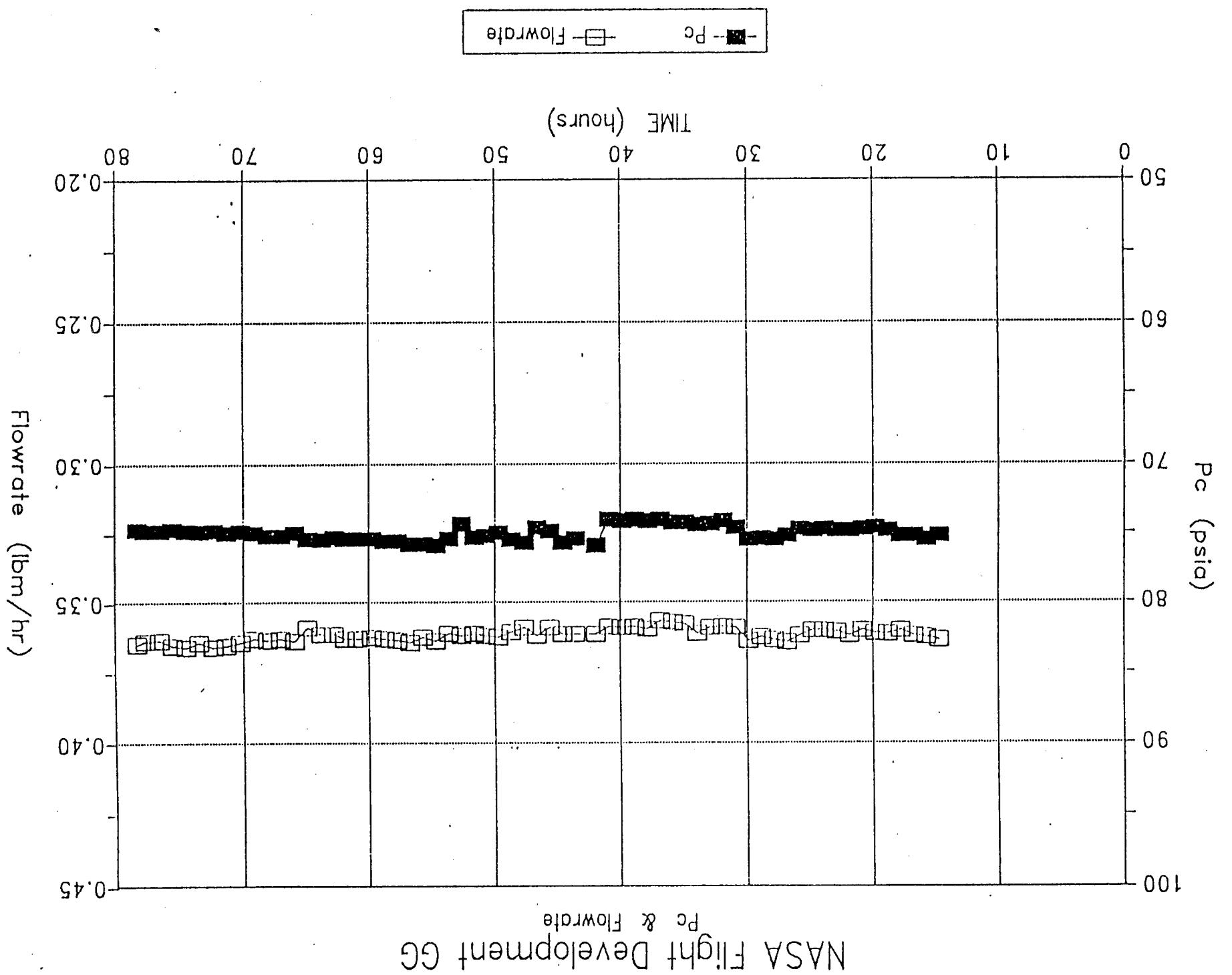


Figure 2-62

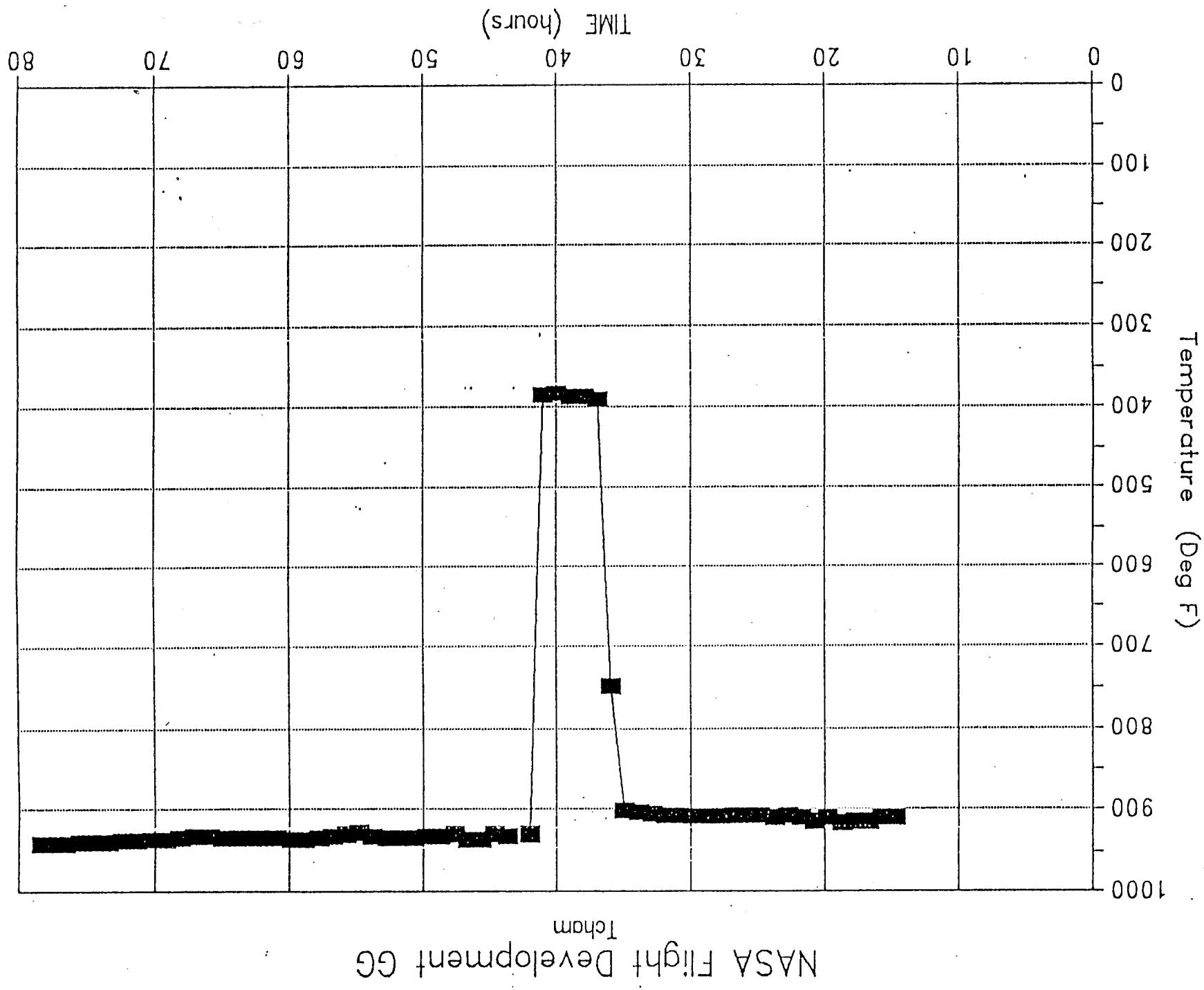
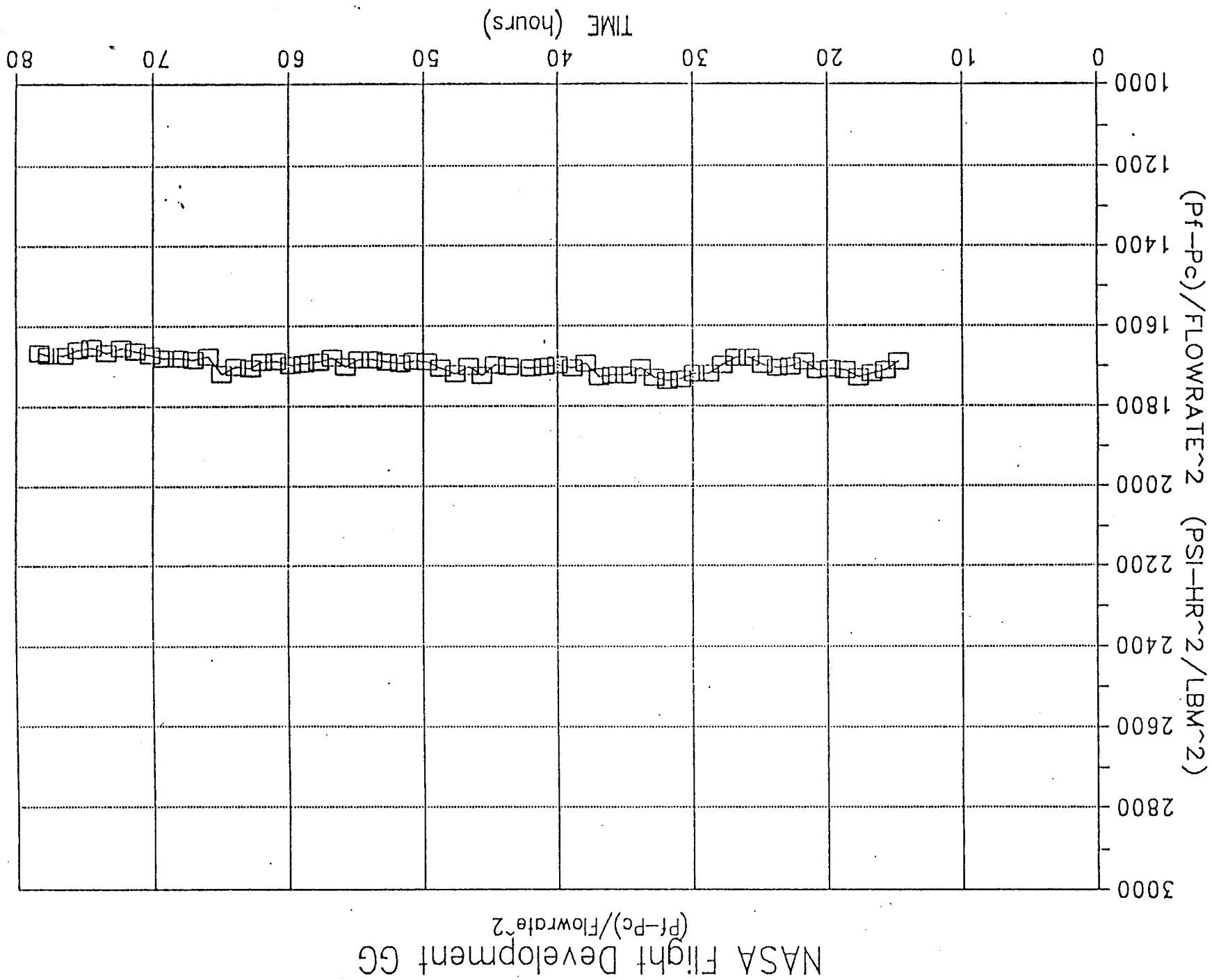


Figure 2-63

2-89



clogging of injector flow manifests itself through a substantial increase in this parameter. Figure 2-63 indicates that consistent flow conditions through the GG were maintained with no detectable changes in injector pressure drop. No other measured data indicated degradation in GG performance.

Additional data are included in Appendix G. The following is a summary of the hot fire testing completed:

1. Demonstrated Capability
 

Total lifetime (hrs)	887
Total propellant throughput (lbm)	293
Total number of starts	730
Flow rate (lbm/hr)	0.360 to 0.305 in simulated blowdown
2. End-of-life gas generator condition
  - No detectable loss in performance
  - Flow rate vs. feed pressure characteristic agrees with beginning of life measurement.
  - Pressure drop through GG injector (i.e.  $(P_f - P_c/m^2)$ ) remains constant; no restrictive clogging is evident.
  - No chamber pressure roughness or oscillations.
  - Injector temperature maintained within expected design limit.
3. Conclusions
  - Significant additional life margin appears to exist.
  - High level of confidence has been established in this GG design to meet near term qualification requirements in the tested flow rate range and thermal environment.
  - Extended life testing and comprehensive disassembly and inspection are recommended.

## 2.4.2 Material/Hydrazine Decomposition Products Interaction Study

In electrothermal thrusters a variety of metals are exposed to  $N_2H_4$  decomposition products at elevated temperatures. Testing with resistojets indicated that atomic oxygen (AO) in  $N_2H_4$  decomposition products attacks tungsten filaments. Several candidate materials and coatings were targeted for test evaluation to address various parts affected by AO attack in  $N_2H_4$  decomposition products. Coupon and filament tests were conducted to evaluate various material problems. Table 2-12 is a summary of these parts and materials.

### 2.4.2.1 Coupon Test Results

Coupons needed to complete the materials testing were fabricated (1 by 1 cm each). For coupon tests  $N_2H_4$  decomposition products with AO were simulated using  $NH_3$  saturated with  $H_2O$ . This was accomplished by bubbling the  $NH_3$  through liquid  $H_2O$ . The saturated gas continuously passed over the coupon samples in a tube furnace that heated the coupons to temperatures close to electrothermal thruster internal temperatures ( $2100^{\circ}F$ ). Samples of

**Table 2-12  
MATERIALS AND APPLICABILITY IN ELECTROTHERMAL THRUSTERS**

Material	Problem	Type of Part	Type of Test
Mo/41	Attack Creep	Thruster body Injectors Heat exchanger body	Coupon
W 100	Evaporation AO attack	Heater Anodes	Filament coupon
W/3Re	New material	Heater	Filament
W/25Re	New material	Thruster body Anode	Coupon
Ir coating (on W)	New process	Heat exchanger body	Coupon
Ir coating (on Mo41Re)	New process	Injectors Anodes	Coupon
Rhenium	Machining Availability Cost	Heat exchanger body Injectors	Coupon

tungsten (W100), tungsten-rhenium (W/25Re), molybdenum-rhenium (Mo/41Re), rhenium (Re), and iridium chemically vapor deposited on tungsten (Ir CVD W) that had previously been exposed to the same conditions under a commercial arcjet program effort were processed for an additional 390 hours during this test. All samples were exposed to 493 hours at 2100°F 25°F in moist ammonia containing 1% water. Table 2-13 provides the test results.

All samples held up quite well, except the Mo/41Re which lost considerably more weight and had a serious fracture problem. The iridium coating on the tungsten began to flake off at about 150 hours. Iridium coating of on Mo41Re was not tested.

The tungsten sample with a 0.001-in. thick chemically vapor deposited iridium was exposed to thermal cycling between 2100° and 200°F. Testing was accomplished by cycling the part on a shuttle into and out of a tube furnace held at 2125°F. A nitrogen atmosphere was provided to protect the part at high temperature and an increased flow over the part during cooling to help accelerate cooling. Table 2-14 lists the results of testing on these samples:

**Table 2-13  
TESTING @ 2100°F WITH 1% MOISTURE IN AMMONIA**

Sample	103 Hrs % Wt Gain/Loss	493 Hrs % Wt Gain/Loss
IrCVDW	+0.0078	+0.026*
W/25Re	+0.0087	0
Mo/41Re	-0.205	-0.366
Re	+0.0038	+0.015
W100	-0.022	-0.026

- | Sample  | Appearance                                                                                                                                                                                                                         |
|---------|------------------------------------------------------------------------------------------------------------------------------------------------------------------------------------------------------------------------------------|
| IrCVDW  | <ul style="list-style-type: none"> <li>Ir coating unchanged. Bare W showed dull greying. Fractured surface was black. No visible cracks.</li> </ul>                                                                                |
| W/25Re  | <ul style="list-style-type: none"> <li>Polished surface still had some reflective properties. No visible cracks.</li> </ul>                                                                                                        |
| Mo/41Re | <ul style="list-style-type: none"> <li>Polished surface still had some reflectivity. Sample had cracks that appeared to penetrate through the piece, i.e. cracks continued from one surface into the adjoining surface.</li> </ul> |
| Re      | <ul style="list-style-type: none"> <li>Strip had original appearance and is reflective. No visible cracks.</li> </ul>                                                                                                              |
| W100    | <ul style="list-style-type: none"> <li>Polished surface was totally nonreflective, dull, grey. No visible cracks.</li> </ul>                                                                                                       |

**Table 2-14  
CYCLING BETWEEN 2100° AND 200°F UNDER A NITROGEN ATMOSPHERE**

Cycles	Wt (g)	% Wt Gain	Appearance
0	42.0164	—	<ul style="list-style-type: none"> <li>Typical "white" Ir color</li> </ul>
55	42.0153	-0.0026	<ul style="list-style-type: none"> <li>Reddish brown on top OD</li> </ul>
103	42.0149	-0.0036	<ul style="list-style-type: none"> <li>Reddish brown increasing.</li> <li>Upstream 40% ID has smoother appearance.</li> </ul>
205	42.0163 (42.0148 - chips)	-0.0002	<ul style="list-style-type: none"> <li>Coating flaking and chipping on upstream end. Reddish brown area increasing.</li> </ul>
302	42.0103	-0.0145*	<ul style="list-style-type: none"> <li>Severe flaking. Upstream bore coating is loosened. Reddish brown surface on all exposed W substrates.</li> </ul>

\* This value includes change due to lost flakes

Earlier IR&D testing of Ir CVD W and Re CVD W showed that the Re CVD W was not satisfactory while the Ir CVD W showed crack formation thought to be due to a heavy coating of iridium. A thinner coating sample was also tested. This sample showed promise until a reddish brown coloration started to form. The nitrogen flow was at a lower level than on an earlier test, and air could enter the apparatus where the thermocouple/pushrod exited the apparatus. The sample was only about three inches away from the thermocouple exit port into which air could possibly back flow during the entire cooling cycle. The nitrogen flow was controlled by boil-off of LN<sub>2</sub>. This flow apparently was not adequate to purge air from the T/C port. The type of coating, cracking noted on the earlier tested part was not found on the latest tested part. All test samples exhibited some form of failure.

#### 2.4.2.2 Filament Test Results

Tungsten and W3Re wire were selected for filament tests. Each filament was wound from a length of 20 mil wire 18 inches long. The wire was coiled around 1/16-inch diameter tungsten welding rod to maintain a constant filament diameter. A constant pitch of about 23 turns per inch was achieved by wrapping the wire beside a filler wire which was separated from the filament following the winding process. The pure tungsten wire required heating up to 400° to 500°F during winding to prevent splintering, whereas the tungsten/3% rhenium wire was wound at room temperature. The resulting filament length was 3 inches.

The test apparatus is illustrated schematically in Figure 2-64. The filaments were held at each end by a TZM clamp which provided a low resistance electrical connection. A short length of the 1/16-inch diameter tungsten welding rod was placed inside the filament at the clamps to provide additional clamping force. The TZM clamps were mounted to a common piece of alumina which provided mechanical support of the assembly and electrical isolation. The alumina was wrapped with a stainless steel radiation shield under 80% of the filament length to minimize possible cracking due to the high level of heat loading. Two separate pairs of copper lead wires were fastened to each of the filament assemblies to provide both power and voltage measurements across the filament. The filaments were placed inside a 2-inch diameter, 4-foot long quartz tube with water cooled aluminum end plates which provided a sealed interface for the electrical and gas feedthroughs. One end of the quartz tube was used as the gas outlet, and the other as the gas inlet. On the outlet end, the gas line was routed into a bucket of water to provide a slightly higher than atmospheric pressure within the quartz tube. Each filament was powered by a DC power supply operating in voltage control mode. The output current of each power supply was measured by the voltage drop across a low resistance shunt. The voltage drop across each filament and the two current shunts were recorded on a 4-channel strip chart recorder. An optical pyrometer was used to measure the filament temperature during testing. A correction of roughly 500°F for the emissivity of the tungsten and transmissivity of the quartz was applied to the pyrometer readings as follows:

$$(1/T_t) = (1/T_p) + (w/C)(\ln(\epsilon_t))$$

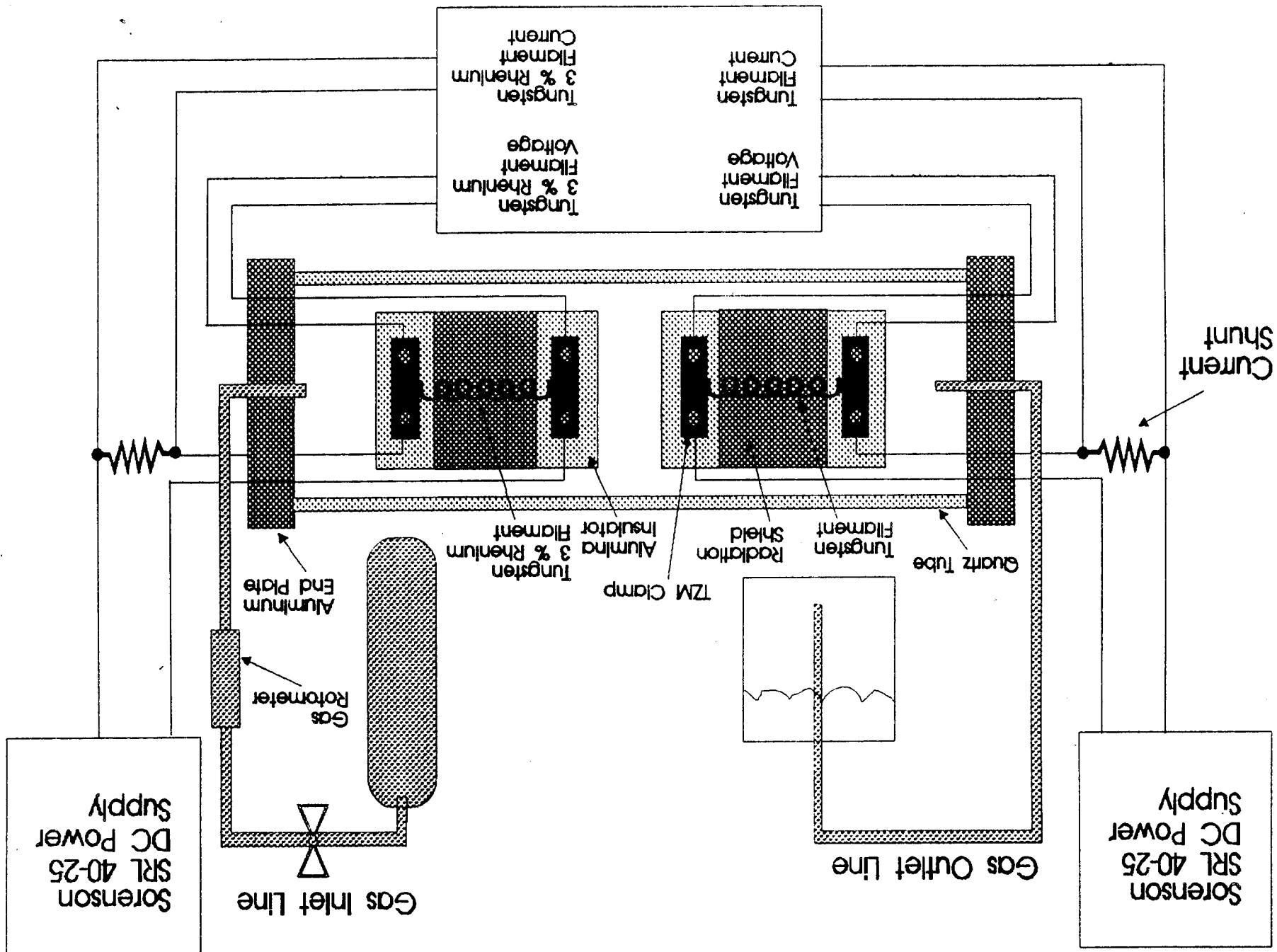
where:

$T_t$  = true temperature (K)

$T_p$  = pyrometer indicated temperature (K)

Figure 2-64

2-94



- w = operating wavelength of pyrometer =  $650 \times 10^{-9}$  m  
 C = constant =  $hc/k = 0.0144$  m-K  
 $\epsilon$  = emissivity of tungsten =  $0.42 \rightarrow 0.43$  ( $2600 \rightarrow 3200$  K)  
 t = transmissivity of quartz = assumed constant at 0.92

### Test Results

The filaments were first rapidly heated in a hydrogen environment with a gas flow rate of 1500 ml/min. The filament temperature was quickly raised to 4600°F and allowed to set for 3 minutes. The resulting electrical characteristics are tabulated below:

	Tungsten	Tungsten/3% Rhenium
Voltage (volts)	31.25	34.40
Current (amps)	24.40	25.00
Resistance (ohms)	1.281	1.376
Power (watts)	763	860

The filaments were then heated in an Argon environment (with 1.0 ppm O<sub>2</sub>) with a gas flow rate of 1,000 ml/min. Each filament was maintained at a temperature of 4200°F while the resistance was monitored. After 38 hours at 4200°F, the resistance change of the pure tungsten filament was about 2.3%, and the tungsten/3% rhenium filament change was about 1.1%. The pure tungsten filament had sagged about 1/2 of an inch. In an attempt to increase the rate of degradation, the filaments were raised in temperature by 150°F. After one hour at the higher temperature, there was no measurable increase in resistance so the temperature was raised another 150°F. The test was then terminated after 17 hours at the 4500° level. No measurable change in resistance was seen, although the middle portion of the tungsten/3% rhenium filament was in contact with the radiation shield covering the alumina insulator due to sagging. Table 2-15 shows the results.

Table 2.15  
TUNGSTEN AND TUNGSTEN/3% RHENIUM FILAMENT TEST RESULTS

Volts	Amps	Ohms	Watts	Temp	Hours	Volts	Amps	Ohms	Watts
21.6	17.8	1.213	385	4200	0	22.0	17.8	1.236	392
21.6	17.5	1.234	378	4200	6	22.0	17.75	1.239	391
21.6	17.4	1.241	376	4200	32	22.0	17.6	1.250	387
24.0	18.25	1.315	438	4350	1	22.9	18.0	1.272	412
25.3	18.94	1.336	479	4500	17	24.5	18.75	1.307	459

The total accumulated time was 56 hours. After about 38 hours, a buildup of tungsten residue on the inner surface of the quartz tube made accurate pyrometer measurements difficult. Following test shutdown, visual observations showed a colorful dendritic buildup of material

on the TZM clamps, steel mounting bolts, and tungsten weld rod tip. Samples were obtained for future material analysis. The filaments were observed under an optical comparator which showed a reduction in wire diameter only at the free ends closest to the clamps where the temperature gradient occurred. Measurements showed the pure tungsten filament experienced a wire diameter loss of 1 mil compared to a loss of 2 mil on the tungsten/3% rhenium filament.

The post-exposure wire examination was accomplished using dimensional wire analysis, scanning electron microscope analysis (SEM), and metallographic cross section analysis. The dimensional analysis was accomplished using an optical comparator to establish the wire diameter at five locations along the filaments length. The SEM analysis was performed on sections of the filaments removed from each end. This region of the filament was selected for SEM evaluation because dimensional data showed that the most mass loss occurred nearest the filament/test fixture interface.

With the completion of the SEM examination, the end segments of each filament were mounted for metallographic examination which allowed the correlation of SEM surface findings and observations with those obtained from the wires cross sections. All metallographic findings of interest were documented via photomicrographs.

### **Examination Results**

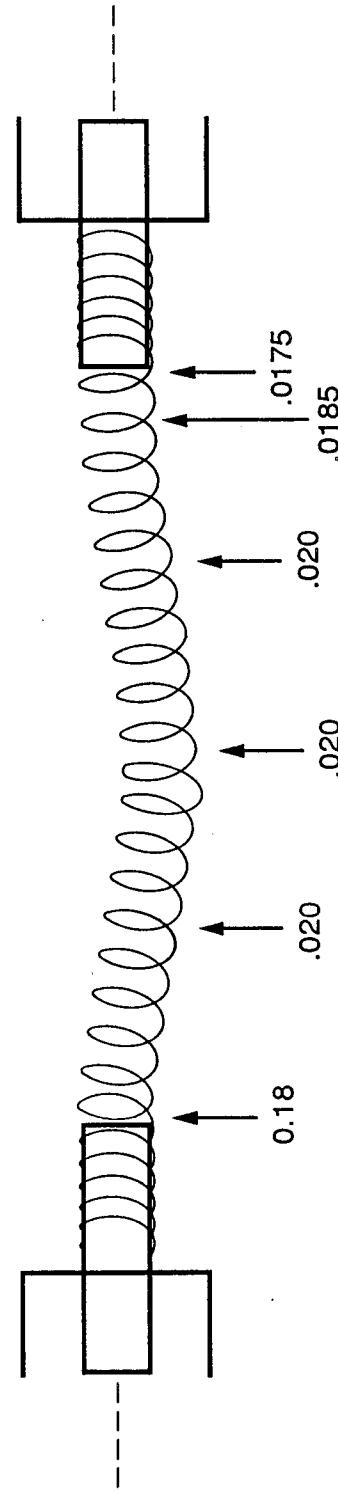
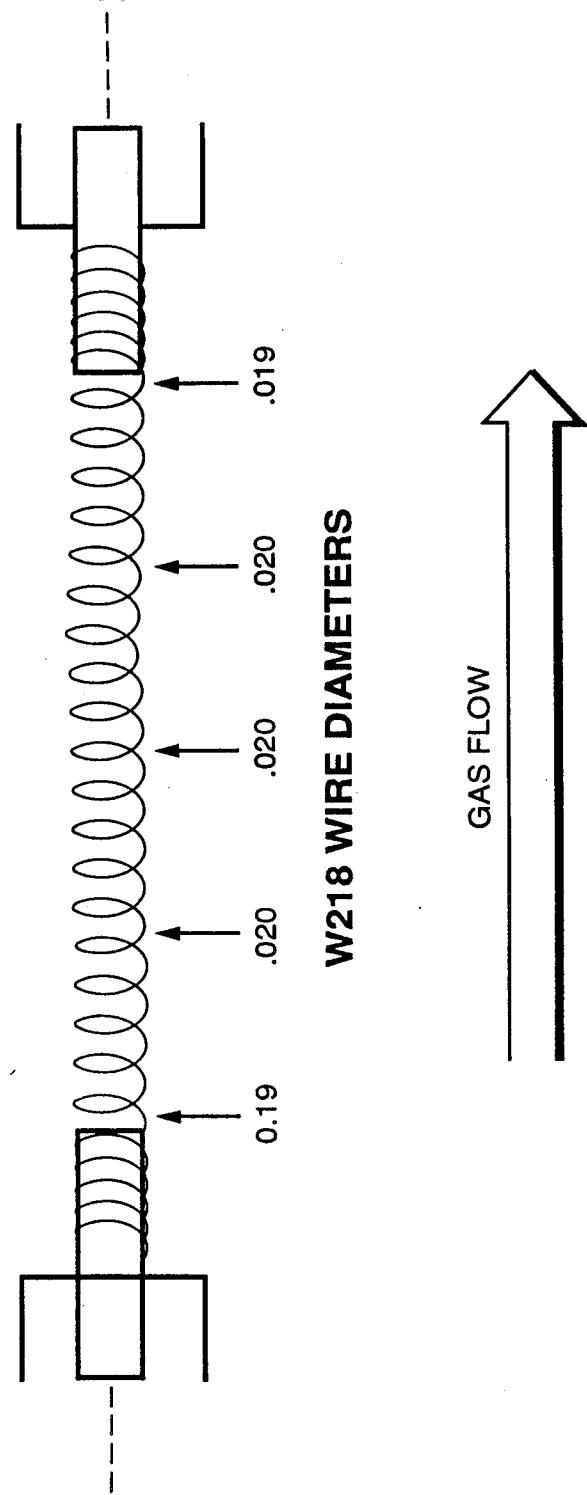
The W/3Re filament lost more in cross section than the W218 (GTE standard lamp filament alloy) wire filament, as documented in Figure 2-65. Both filaments lost the most cross section in the cooler transition areas at each end of the filaments where the test fixture interface reduced the operating test temperature. This condition is most likely caused by the instability of the wire oxide that forms at the higher temperatures.

Evidence of thermal etching was observed on both filaments and redeposition and tungsten crystals was detected on the W218 filament. Documentation of the SEM surface examination findings is presented in Figures 2-66 through 2-70. The W/3Re wire surface formed surface voids which is evidence of localized thermal corrosion/oxidation of the wire substrate. The depth of penetration of these surface voids was established through the metallographic cross sectioning of the exposed filament material.

The cross sections of the exposed filament material revealed that incomplete wire recrystallization was produced during the filament thermal setting procedure. This would explain the cause of the wire sagging seen on the W218 filament, where little to no sagging is normally observed under the same thermal exposure conditions and durations. The cross sections of the filament material also revealed that the W/3Re wire subsurface voids had a depth of penetration of 0.0015 inch. The evidence of this void formation is documented in Figures 2-71 and 2-72.

### **Conclusion**

Based on the observed post-test wire conditions, it can be stated that the W/3Re wire did not reveal any improved resistance to oxygen attack under the laboratory conditions used in this exposure test. The W/3Re wire was found to have lost more surface material as revealed by



C11232-71

2-97

Figure 2-65

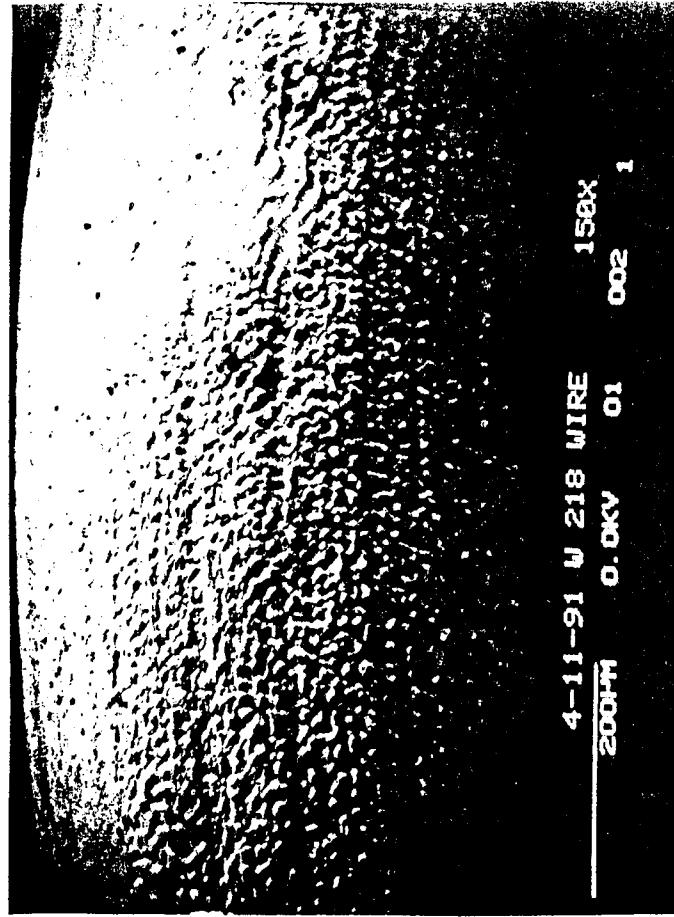


Figure 2, SEM photomicrograph of W218 tungsten wire at a location near the test fixture/wire interface. Photomicrograph documents the formation of tungsten crystals developing on the wire surface in the cooler regions of the coil. This condition was observed at both ends of the test coil.

Figure 2-67

2-99



Figure 3, SEM photomicrograph of the same area as figure 2, but at higher magnification. Photograph reveals details of tungsten crystal deposit topography. Crystals are formed by either vapor deposit or chemical vapor deposit mechanism produced by the laboratory test environment.

Figure 2-68

2-100

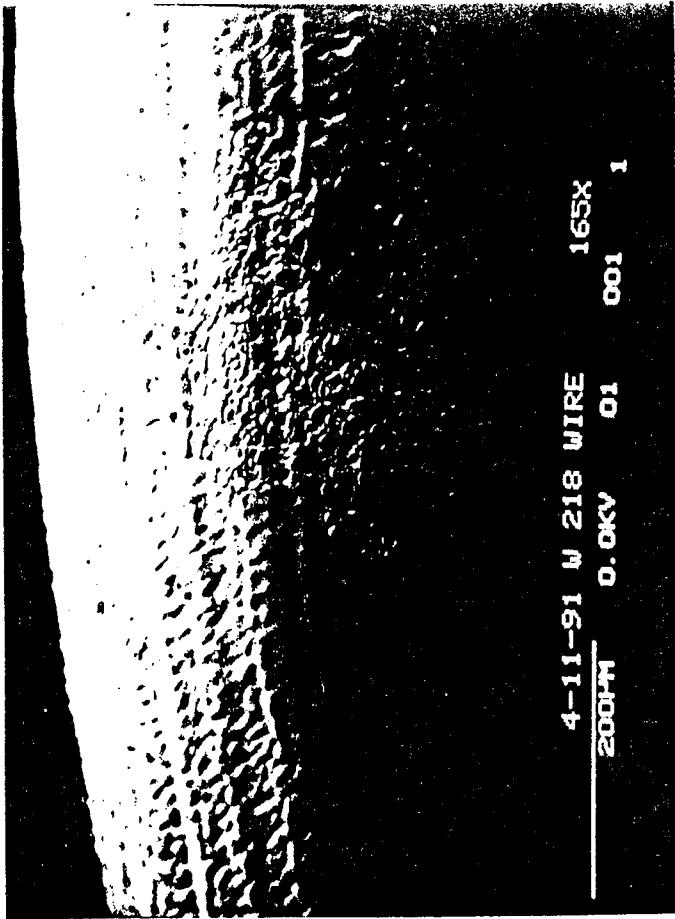


Figure 4. SEM photomicrograph of the W/3Re wire at a location near the test fixture/wire interface. The photograph documents the typical surface conditions and evidence of surface void formations.

Figure 2-69

2-101

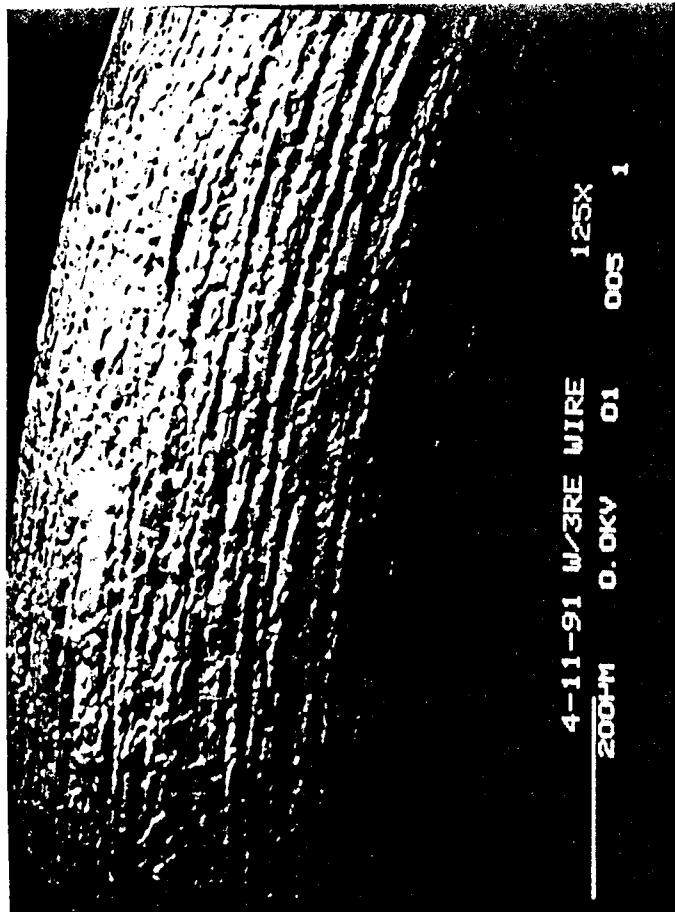


Figure 5. SEM photomicrograph of W/3Re wire at a location in the high temperature region of the coil. Photograph indicates evidence of thermal etching and extensive surface void formation.

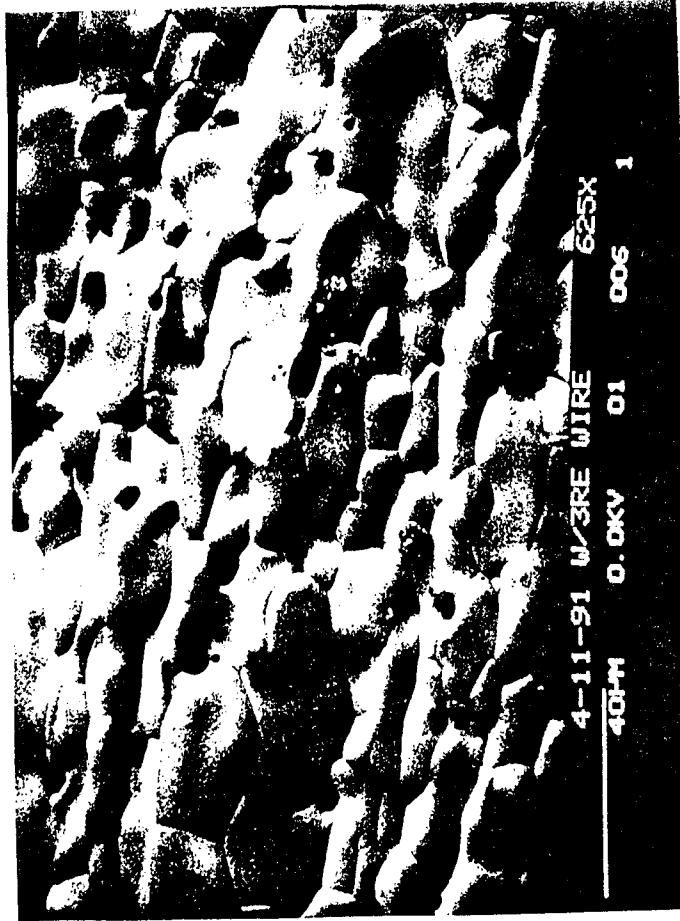


Figure 6, SEM photomicrograph of the same area as documented in figure 5 but at higher magnification. Photomicrograph reveals effects of thermal etching, surface wire grain size and surface void formation.

Figure 2-71

2-103

Etched, 200x

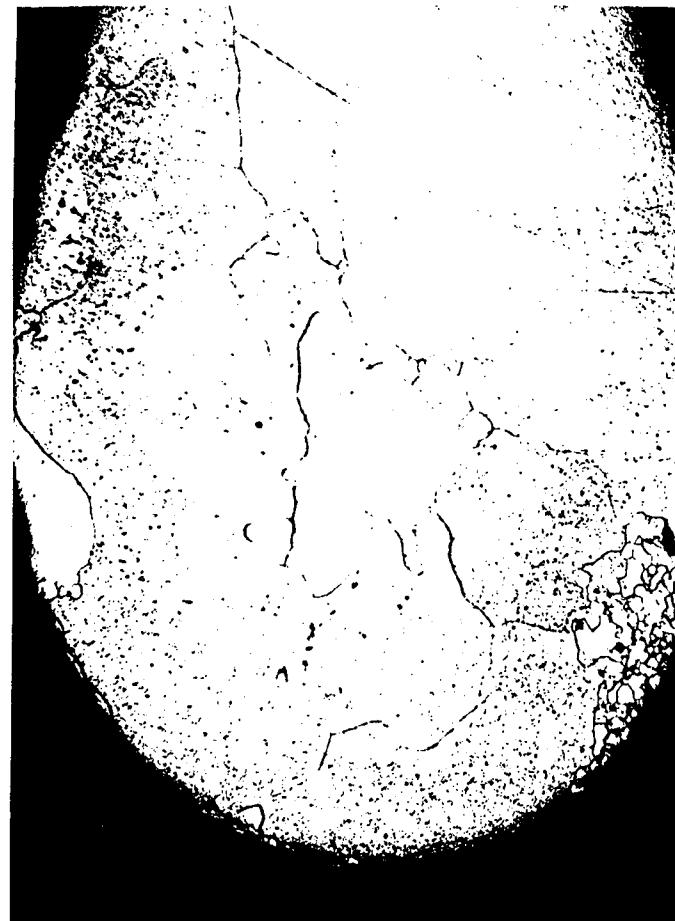
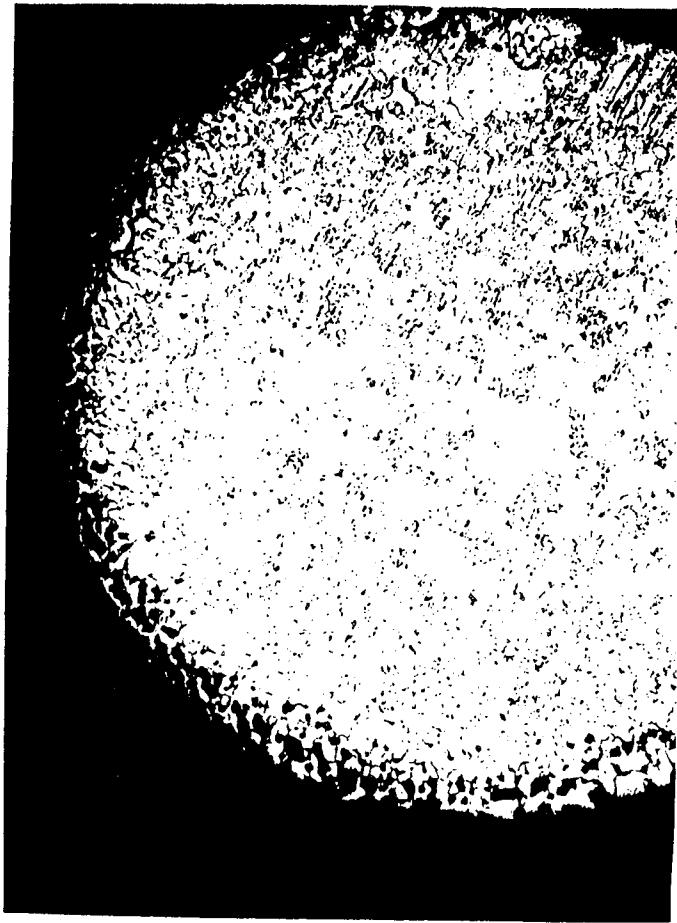


Figure 7. Photomicrograph of W218 wire cross section that reveals a duplex microstructure, with both large recrystallized grains and fine equiaxed non-recrystallized grains. This cross section was obtained near the test fixture/coil interface and is in the same location as that of the SEM surface examination.

**Figure 2-72**

2-104

Etched, 200x



**Figure 8.** Photomicrograph of the W/3Re wire cross section revealing surface and subsurface void formation typical of the exposed wire. Some grain recrystallization is evident, but most grain remain fine and equiaxed. Void formations would indicated a lack of thermal corrosion resistance in the argon test environment.

the post-exposure wire diameter examination. The W218 wire filament material was found to be more resistant to the laboratory test exposure environment, and produced the smallest wire diameter changes. One observation seen only on the W218 wire was the development of vapor deposited/chemical vapor deposited tungsten crystals on the cooler ends of the filaments near the fixture/filament interface.

Overall, no significant advances in material technology were achieved in these tests. Several coatings were shown not to have good resistance to attack. The filament wire currently used in production EHT thrusters was shown to be better than W/3Re wire.



## 3.0 CONCLUSIONS

The following are the notable accomplishments of the HPSPR program:

1. Advanced the understanding of low Reynolds number nozzles. Fabricated nozzles for NASA cold gas study and performed analysis in support of the study. Resulted in NASA TM-89858.
2. Comprehensive performance mapping of vented cavity, sealed cavity, and immersed filament resistojets were completed. Performance limits of each configuration were determined. An extensive data base in the moderate power (500 to 850 W) range for I<sub>sp</sub>, thrust efficiency, and thermal performance was obtained for the three thruster configurations.
3. High temperature tungsten filaments for three resistojet configurations were characterized. Material loss rates and mechanisms were investigated.
4. A better understanding of refractory metals was obtained. The mechanisms of attack of N<sub>2</sub>H<sub>4</sub> decomposition products were explored. The effects of high temperature atomic oxygen were investigated. Metals studied included rhenium, tungsten, Mo/41Re, and Mo/3Re.
5. Conducted a life test of an advanced low flow gas generator. The life test ultimately achieved 887 hours and the gas generator demonstrated reduced feed tube plugging.

## 3.1 BACKGROUND FOR KEY ACCOMPLISHMENTS

Results obtained in the HPSPR program are wide ranging and contribute to understanding within a number of propulsion disciplines.

Low Reynolds number nozzles continue to be used in resistojets, arcjets and low thrust chemical thrusters. The cold gas nozzle test results using H<sub>2</sub> and N<sub>2</sub> generated in association with the HPSPR program are useful data for thruster design development. Both the cold gas data and the thruster data collected during HPSPR program resistojet testing show that empirically derived data are preferable over available numerical models for the selection of nozzle designs. The study further illustrates that at low Reynolds number, nozzle configuration has a minimal effect on performance.

Comprehensive performance mapping of vented cavity, sealed cavity, and immersed filament resistojets completed in the HPSPR program will allow comparisons between these technologies in the future. This data base will also allow comparisons with arcjet thrusters and other electrothermal thrusters.

Filament material loss rates are the principal performance limit for resistojets. Sealed cavity heater designs can exhibit lower mass loss rates than vented cavity designs. Immersed heater filament mass loss rates are prohibitive at high performance temperatures.

The improved understanding of refractory metals obtained in the HPSPR program can potentially benefit any user of high temperature materials. The mechanisms of chemical attack

of  $\text{N}_2\text{H}_4$  decomposition products is of interest to arcjet design as well as resistojet design. The observations made during the HPSPR program testing led to a better understanding of the effects of high temperature atomic oxygen. These chemical reactions may account for a portion of the observed material loss in any high temperature media containing oxygen bearing species. Hydrazine generally has trace quantities of  $\text{H}_2\text{O}$ ,  $\text{CO}_2$ , and  $\text{O}_2$ .

The gas generator technology supported by the life test in the HPSPR program is already in use in a flight qualification program for a commercial spacecraft.

### 3.2 THEORY OF TUNGSTEN EVAPORATION AND DIFFUSION

Of the program key accomplishments discussed above none have a more direct effect on resistojet design than an increased understanding of the behavior of high temperature tungsten filaments.

Results of the program confirm the fundamental sublimation and diffusion behavior for tungsten wire presented in Figure 2-2. The resulting design limits presented in Figure 2-3 were confirmed as well. It is of interest to compare the three design approaches that were tested.

Table 3-1 presents a summary of the theoretical underpinnings for the three approaches. For the vented cavity approach the filament mass loss rate is moderate and is driven by simple sublimation and diffusion of tungsten away from the wire. The heat transfer model is also simple. However, the resulting thermal efficiency is the lowest of the three approaches. The stated performance approximates the flight thruster qualification values.

For the sealed cavity, the viscous boundary layer steepens the tungsten gas concentration gradient, thereby reducing sublimation. This should result in either longer filament life or higher filament temperatures and higher performance. The heat transfer from the filament to the heat exchanger wall is enhanced by the presence of the gas. This raises the heat exchanger wall temperature and promotes thermal efficiency. The negative effects of higher wall temperatures are lower creep life and higher thermal losses. During testing the thermal loss effect masked any potential performance benefit. Due to the thermal failure of support hardware the expected enhanced life was not demonstrated.

For the immersed heater, filament mass loss rates are driven by forced convective removal of the tungsten boundary layer. Higher mass loss rates were demonstrated with the immersed heater design than seen with the radiative cavity designs at equivalent filament temperatures. The heat transfer is considerably better than that produced with the heat exchanger approaches, resulting in a high thermal efficiency. Filament mass loss drives down both  $I_{sp}$  and life.

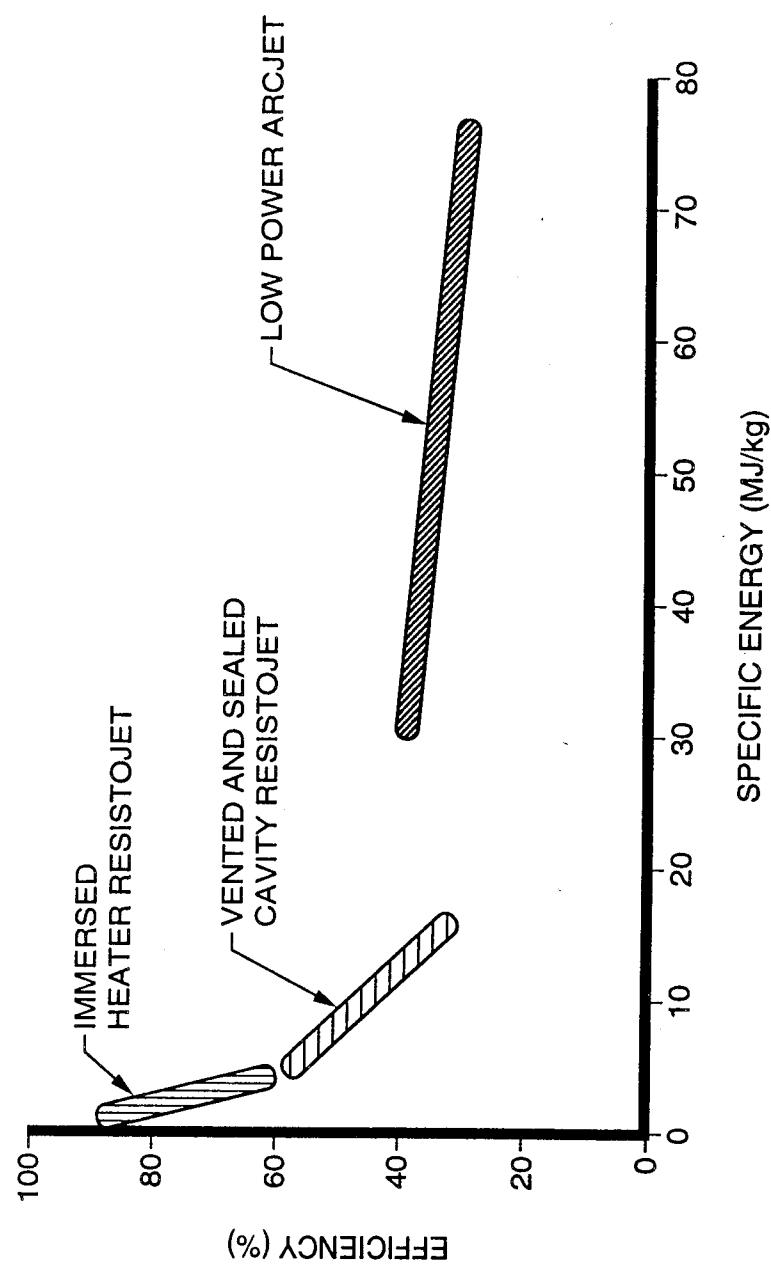
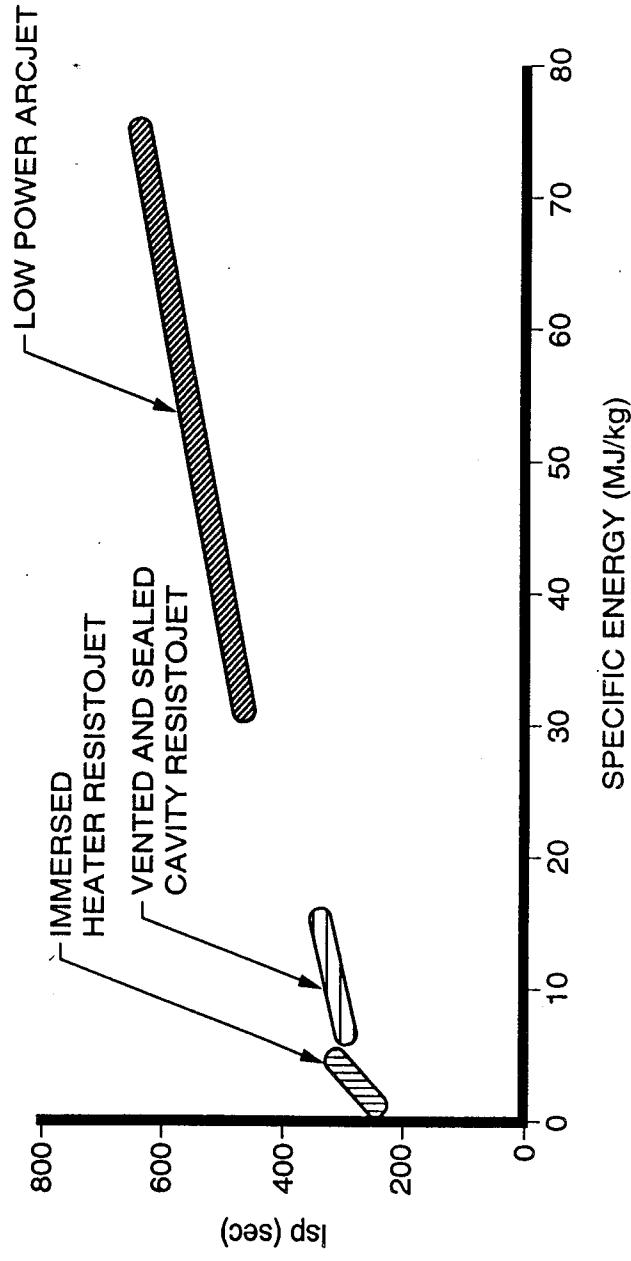
### 3.3 PERFORMANCE LIMITS OF RESISTOJETS

Figure 3-1 presents a graphical summary of performance for resistojets and arcjets. Both plots indicate that the specific energy of arcjets is considerably higher than for resistojets. This means that the power required to run an arcjet for equivalent mass flow rates is higher

DESIGN APPROACH	FILAMENT MASS TRANSPORT	HEAT TRANSFER	PERFORMANCE
VENTED CAVITY	ideal sublimation and diffusion to a vacuum.	<p>FILAMENT HT. EX. GAS</p> <p>RAD. COND. CONVECT.</p>	lsps >100,000 lbf-sec
SEALLED CAVITY	Viscous boundary layer limits diffusion. Increases sublimation rate.	<p>FILAMENT HT. EX. GAS</p> <p>RAD. COND. CONVECT.</p>	lsps 340 ? Life
IMMERSED HEATER	Forced convective removal of tungsten. Decreased tungsten partial pressure increases sublimation rate.	<p>FILAMENT GAS</p> <p>RAD. COND. CONVECT.</p>	lsps 300 50,000 lbf-sec

Table 3-1  
RESISTOJET DESIGN EFFECTS ON FILAMENT BEHAVIOR

## PERFORMANCE COMPARISONS OF RESISTOJETS AND ARCJETS



C11235-03

3-4

Figure 3-1

than for resistojets. One result, illustrated in the upper plot, is higher I<sub>sp</sub> since more total electrical power gets into the exhaust. Efficiencies of arcjets, shown in the lower plot, are on average lower than resistojets.

The conclusion to be reached is that the selection of either a resistojet or an arcjet is very application specific. If the spacecraft is power limited, and if the thruster efficiency is more important than I<sub>sp</sub>, the resistojet should be considered. If higher power levels are available, and higher I<sub>sp</sub>'s are required, the arcjet is favored.

### **3.4 SIGNIFICANCE OF THE CURRENT EFFORT**

This program provided a demonstration of the limits of resistojet technology development. Applications requiring I<sub>sp</sub>'s greater than 400 seconds must look to arcjets or other technology. It was shown that resistojets can achieve I<sub>sp</sub>'s in the range of 340 to 350 seconds. However, lifetimes associated with these performance levels were prohibitively short. Levels of I<sub>sp</sub> in the range of 310 to 320 seconds are feasible with acceptable lifetimes.

Solving the heat loss problems of the sealed cavity configuration appear to have more pay-off than further work on the immersed heater configuration.

### **3.5 POTENTIAL FOR FUTURE WORK**

Resistojets continue to be used in spacecraft applications, primarily for North-South station keeping of spacecraft in geosynchronous equatorial orbit. For high power spacecraft, arcjets appear to have a clear performance advantage and will likely replace resistojets. The advantage for arcjets is not as clear-cut for low power, small spacecraft. The relatively simple power conditioning, light weight, and low cost of resistojets provide advantages for these applications.

Further increases in performance with minor modification to existing resistojet designs appear feasible from the results of the HPSPR program. For instance, the vented cavity thruster tested in Phase 2 achieved performance 10% higher than existing qualified designs.

Space Station Freedom gas and liquid waste disposal may well rely on resistojets. Extended life, materials selection, compatibility with a range of waste fluids, power handling, and performance optimization are all areas that could benefit from future work in this technology area.

### **3.6 HARDWARE DISPOSITION**

The sealed cavity hardware was never reassembled. The rhenium heat exchanger, piece parts, components, and raw materials required for reassembly are in storage at RRC. The immersed heater thruster was removed from the gas generator. Piece parts, components, and raw materials required for reassembly are available. Four additional immersed heater assemblies are also available. All of this hardware will be shipped to NASA-LRC at program close-out.



#### 4.0 REFERENCES

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**APPENDIX A**

**"Fast Track" Data**

## UNREFINED DATA (volts)

	THRUST ZERO	STEP 1	STEP 2	STEP 3	STEP 4	STEP 5	STEP 6
PWR	+1.870	+1.810	+3.069	+4.288	+3.117	+1.671	+1.6
4 (TRUN)	+0.00012	+0.23956	+0.000001	+0.000045	+0.000000	+0.000000	+0.000000
+0.13790							

## RECORDED TEMPERATURES (degrees F)

	Tc	Tb	Tv	Tn
	+1120.2	+457.1	+1140.2	+133.9
TEMP	TEMP	TEMP	TEMP	TEMP
	+32.5	+32.0	+32.0	+0.0
+89.2				

## REDUCED DATA

	PLATE	CHAM	PP	PF	FE	FE	FE
in.							
.06228	.00692	.00022	.00022	.00022	.00022	.00022	.00022
HEATER							
Volts	Eng	Watts	Watts	Watts	Watts	Watts	Watts
.94	2.500	0.0	0.0	0.0	0.0	0.0	0.0

	WICKO	THRUST	THRUST	THRUST
PLATE	MOTION	MEAS	VAC	VAC
bs/hr	lbs/hr	lbs	lbs	lbs
0.00000	94107	47.851	48.253	48.253

	C*	REYNOLDS	GAS	PSP	ISF
	ft/sec	NUMBER	TEMP. deg. f	sec.	sec.
3466	1.4972	3966.4	6572.4	1071.1	0.000

## UNREDUCED DATA (cycles)

	THRUST RNU	STEP ZERO	STEP 1	STEP 2	STEP 3	STEP 4	STEP 5	STEP 6
F(run)	+.012000	+.591	+1.614	+3.072	+4.308	+3.089	+1.862	+.604
Ff(0)	-0.000118							
F(run)			Ff(0)		Ff(0)		Ff(0)	
			+.007250		+.000001		+.0000020	

## RECORDED TEMPERATURES (degrees F)

	T <sub>f</sub>	T <sub>c</sub>	T <sub>16</sub>	T <sub>2</sub>	T <sub>v</sub>	T <sub>b</sub>
T <sub>b</sub>	+77.2	+1148.7		+1171.2	+224.0	+581.9
T <sub>c</sub>						
T <sub>f</sub>						
T <sub>16</sub>	1263.1	422.0		+22.0	+0.0	+0.0

## REDUCED DATA

(	WATTAGE							
Volts	110	110	110	110	110	110	110	110
Watts	.06228	.000623	.000637	.96.7	45.5	0.0	0.0	0.0
Heater	Heater	Heater	Heater	Heater	Heater	Heater	Heater	Heater
Volts	25.47	45.616	499.5	1.2982	4257.4	0.00000	0.0	0.0

WEIGHT	WICKC	THRUST	THRUST	VAC
PLACE	MOTION	MEAS	MEAS	
1bs/hr	1bs/hr	lbs	lbs	
0.00000	47730	40.514	40.715	

CD	Cf	REYNOLDS	GAS	PSP	ISP
( 003	1.4365	6878.1	1687.3	3676.4	307.1

## UNREDUCED DATA (CONT'D)

	THRUST ZERO	STEP 1	STEP 2	THrust CALIBRATION	STEP 3	STEP 4	STEP 5	STEP 6
2.301	+ .562	+1.821	+3.053	+4.307	+3.093	+1.844	+1.45	
$f(\text{run})$	$\cdot \text{Pf}(0)$	$\text{Pf}(\text{run})$	$\text{Ps}(0)$	$\text{Ps}(\text{run})$	$\text{Ps}(\text{run})$	$\text{Ps}(\text{run})$	$\text{Ps}(\text{run})$	$\text{Ps}(\text{run})$
.012000	-.000187	+.007260	+.000001	+.000016	+.000016	+.000016	+.000016	+.000016

## RECORDED TEMPERATURES (DEGREES F)

	T <sub>C</sub>	T <sub>B</sub>	T <sub>E</sub>	T <sub>V</sub>	T <sub>F</sub>
+81.0	+1167.6	+1608.9	+2197.7	+242.4	+65
T <sub>C</sub>	TEMPER	TEMPER	TEMPER	TEMPER	TEMPER
1240.5	432.0	430.0	430.0	430.0	430.0

## REDUCED DATA

(unit)	ACCELERATION	ACCELERATION	FF	FF	FF	FF	FF	FF
g's, in.	sec. <sup>-2</sup>	in. sec. <sup>-2</sup>	sec. <sup>-2</sup>					
.66228	.000622	.000622	.000622	.000622	.000622	.000622	.000622	.000622
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER
Volts	amps	amps	amps	amps	amps	amps	amps	amps
28.80	20.842	600.02	1.7817	45EC.6	0.0000	0.0000	0.0000	0.0000

IGHT	MICRO MOTION	THRUST MEAS	THRUST VAC	REYNOLDS NUMBER	GAS TEMP. deg. f	ISP
0.00000	47048	1.275	414466			
CL	CL*	ft/sec	ft/sec			

3985	1.4547	7017.5	1622.0	3537.8	14.474	217.3

UNREDUCED DATA (volts)					
THRUST FRUN:	STEP 1	STEP 2	STEP 3	STEP 4	STEP 5
2.263	+ .556	+1.797	+3.025	+4.253	+3.042
F (frun)	F(0)	F(0)	F(0)	F(0)	F(0)
.012000	- .000273	+ .007215	+ .000001	+ .000013	- .000004

RECORDED TEMPERATURES (degrees F.)					
T <sub>c</sub>	T <sub>b</sub>	T <sub>16</sub>	T <sub>16</sub>	T <sub>b</sub>	T <sub>c</sub>
+64.2	+1176.3	+1572.5	+1209.4	+257.5	+690.6
T <sub>b</sub>	TEMPER	TEMPER	TEMPER	TEMPER	TEMPER
-1485.0	+32.0	+32.0	+32.0	+0.0	+0.0

REDUCED DATA					
(Revit)	ACCELERATION GOLF sec. in.	ACCELERATION GOLF sec. in.	FF	FE	FE
.06228	.000622	.000639	96.7	46.1	0.0
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER
Volts	amps	watts	amps	temp	wire resis
30.38	21.410	650.4	1.4189	4667.8	.00000
MICRO CLASS MOTION abs/hr	THRUST MEAS in/lbs	VAC in/lbs			
0.00000	46017	11058	238		
CD	C*	REYNOLDS NUMBER	GAS TEMP. des., f	PSF	ISF
3957	1.4365	7225.4	1531.7	4081.9	15.772

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UNREDUCED DATA (cycles)		THRUST CALIBRATION			STEP 5	
	THRUST	STEP 1	STEP 2	STEP 3	STEP 4	STEP 5
THRUST ZERO						
2.538	+ .578	+1.809	+3.056	+4.290	+3.073	+3.870
F(run)	-.000296	+ .009400	+ .000001	+ .0000020	+ .0000020	+ .0000020
F(run) - F(0)						
.013173						

## RECORDED TEMPERATURES (degrees F.)

	T <sub>C</sub>	T <sub>B</sub>	T <sub>V</sub>	T <sub>H</sub>
T <sub>C</sub>	+1176.7	+1630.2	+1201.5	+243.0
T <sub>B</sub>				+66.1
TEMPERATURE				
T <sub>C</sub>	+1176.7	+1630.2	+1201.5	+243.0
T <sub>B</sub>				
T <sub>V</sub>				
T <sub>H</sub>				

	PERIOD	DATE	PERIOD	DATE	PERIOD	DATE	
Volts	ACCUM.	100.0	Volts	100.0	Volts	100.0	
Amps	ACCUM.	0.000637	Amps	0.000637	Amps	0.000637	
Heater	HEATER	121.0	Heater	121.0	Heater	121.0	
Volts	AMPS	655.5	Volts	655.5	Volts	655.5	
30.04	21.036		30.04	21.036		30.04	

	REYNOLDS	GAS	PERF.	ISP
	NUMBER	TEMP., des. f		SEE
CP				
1051	1.5102	6844.2	1871.2	3704.1

## UNREDUCED DATA (volts)

THRUST ZERO	STEP 1	STEP 2	STEP 3	STEP 4	STEP 5	STEP 6
-2.507	+2.084	+3.308	+4.540	+3.339	+2.119	+.671
.012000	+0.000135	+0.006610	+0.000039	+0.000025	+0.000041	

f(run) =  $4.795 + 0.000135x + 0.006610x^2 + 0.000039x^3 + 0.000025x^4 + 0.000041x^5$ 

## RECORDED TEMPERATURES (degrees F)

Tf	Tc	T16	Tb	Tv	Tm
479.5	+1183.9	+1801.8	+1225.0	+253.3	+745.0
Tn	Temp	Temp	Temp	Temp	Temp
1606.9	+32.0	+32.0	+32.0	+6.0	+6.0

## REDUCED DATA

(exit) in.	Micro cold so. in.	Micro cold so. in.	Psi	Psi	Psi	Psi	Psi	Psi
.66228	.66228	.000640	0.0	45.6	0.0	0.0	0.0	0.0
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER
Volts	Vamps	Watts	Ohms	Temp	Temp	Temp	Temp	Temp
33.22	22.566	749.7	1.4723	4836.4	.00000	.00000	.00000	.00000

RIGHT  
GLASS  
bs/hr

MICRO  
ACTION  
bs/hr

THRUST  
MEAS  
lbs

VAC  
in.  
lbs

REYNOLDS  
NUMBER

GAS  
TEMP.  
deg. f

CD	D*	ft/sec	ft/sec	sec.
C 3928	1.4117	7380.0	1447.4	4260.8
				18.712

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## THERMIUM HEAT EXCHANGER - BASELINE HEATER

SN# 001 RUN4

UNREFINED DATA (volts)					
THRUST RUN	STEP 1	STEP 2	STEP 3	STEP 4	STEP 5
2.547	+2.150	+3.370	+4.581	+3.375	+2.155
f(run)	PF(0)	Pg(run)	Pa(0)	Palt(run)	Palt(0)
.011677	-.000231	+.007263	+.000039	-.000022	-.000006

RECORDED TEMPERATURES (degrees F)  
Tf      Tc      Tb      Tg

Th	Tsphere	Tsphere	Tsphere	Tsphere
+61.4	+1157.5	+1541.4	+1161.7	+225.7
1408.9	432.0	432.0	432.0	40.0

(exit)	Amperage	Ampere	PPF	PPF	PPF	PPF
s. in.	COLD	COR.	PPF	PPF	PPF	PPF
.06228	.000023	.000637	96.4	44.7	0.0	.0077
HEATER	HEATER	HEATER	HEATER	HEATER	WIRE	DELTA
volt	amps	watts	ohms	temp	res	PPF
25.41	19.676	499.9	1.2912	4265.2	.00000	0.0

MICRO MOTION bs/hr	THRUST MEAS lbs	VAC miles	REYNOLDS NUMBER	ISF
0.0000	647480	39.767	40.000	
.0008	1.4362	6794.3	1705.1	12.497

CD	Cf	REYNOLDS NUMBER	ISF
	ft/sec	des. f	sec.

## HEMIUM HEAT EXCHANGER - BASELINE HEATER

SN# 002 RUN# 5

## UNREDUCED DATA (volts)

THRUST ZERO RUN	STEP 1	STEP 2	THRUST CALIBRATION	STEP 4	STEP 5	STEP 6
-0.999	+2.160	+3.397	+4.627	+3.431	+2.211	+1.562
-0.000273	-0.000273	-0.00039	-0.00050	-0.00050	-0.00050	-0.00050
.014000						

## RECORDED TEMPERATURES (degrees F)

Tf	Tc	T16	Tb	Tn
-453.4	+1181.9	+1735.5	+1218.5	+240.8
Tn	Tspare	Tspare	Tspare	Tspare
-155E.4	-	-	-	-
	-	-	-	-
	-	-	-	-
	-	-	-	-
	-	-	-	-

## REDUCED DATA

(exit)	A(tubeout)	A(tubein)	Ff	Fg
ccf	ccf.	ccf.	Ff	Fg
sc. in.	sc. in.	sc. in.	Ff	Fg
.0622E	.000622	.000639	127.5	53.6
HEATER	HEATER	HEATER	HEATER	HEATER
volts	watts	watts	ohms	wire dile
33.06	22.692	756.2	1.4568	.00000
			4787.6	0.0

## MICRO THRUST MEAS VAC VIBS

LASS bs/hr	MEAS lbs/hr	VAC lbs	VIBS lbs
0.00000	54279	50502	50.433
0.9038	1.5107	7123.8	1819.3

CG	Cf	REYNOLDS NUMBER	GAS TEMP. deg. f	ISF

## UNREDUCED DATA (volts)

THRUST RUN	THRUST ZERO	STEP 1	STEP 2	STEP 3	STEP 4	STEP 5	STEP 6
3.022	4.976	+2.191	+3.428	44.633	+3.452	+2.219	+4.97
-f(run)	Ff(0)	Pf(run)	Ps(0)	Ps(run)	Falt(run)	Falt	
.014000	-.000291	4.010050	4.000039			-.000020	

## RECORDED TEMPERATURES (degrees F.)

Tf	Tc	T16	Tb	Tv	Tn
+85.0	+116.9	41789.6	+1222.9	4239.7	+721
Tb	Temper	Temper	Temper	Temper	Temp
159.0.7	430.0	430.0	432.0	432.0	431

## REDUCED DATA

(const)	Att(Percent)	Att(Percent)	Ff	Fg	Fg
S. in.	Cold	Cold	Fg1	Fg2	Fg3
	Sec. in.	Sec. in.	Fg1z	Fg2z	Fg3z
.06228	.000022	.000022	127.0	52.7	0.0
HEATER	HEATER	HEATER	HEATER	HEATER	Eff
Volts	amps	amps	amps	wire	Delta voltage
34.49	23.198	800.0	1.4866	4881.4	0.0000
					0.0

IGHT LASS bs/hr	MICRO MOTION lbs/hr	THRUST MEAS lbs	THRUST VAC lbs
0.00000	54222	449.830	50.136

CP	Cf	Cx	REYNOLDS NUMBER	GAS TEMP. deg. f	PSP	ISP
.9036	1.4552	7140.3	1812.2	4052.8	15.942	732.7

## CHENIUM HEAT EXCHANGER - BASELINE HEATER

SN# 001 RUN# 10

## UNREDUCED DATA (volts)

THRFUET RUN	STEP 1 ZERO	STEP 2	STEP 3	STEP 4	STEP 5	STEP 6
2.594	+ .947	+ 2.164	+ 3.397	+ 4.618	+ 5.411	+ 2.184
f(run)	Ff(0)	Pf(run)	Pf(0)	Pf(run)	Pf(run)	Pf(0)
.011737	.0000295	.006520	.000039	.000020	.000012	

## RECORDED TEMPERATURES (degrees F)

Tf	Tc	Tg	Tb	Tv	Tb	Tb
485.2	4189.0		+1972.3	+1236.6	+957.9	+764.5
Tf	TEMP	TEMP	TEMP	TEMP	TEMP	TEMP
164.0	432.0		432.0	432.0	40.0	40.0

## REDUCED DATA (degrees F)

Current	Portance	Attenuity	PF	FE	FE	FE
0.06226	0.00055	0.000642	0.000642	0.000642	0.000642	0.000642
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER
Volt	Volts	Watts	Other	Temp	Temp	Temp
34.54	23.172	800.5	1.4908	4894.6	.00000	0.0

IGHT MICRO THRUST MEAS VAC  
LASS MOTION 1bs/hr. 1bs  
bs/hr. 1bs

0.00000 43195 0.052 0.295

CF	C*	REYNOLDS NUMBER	GAS TEMP. deg. f	PSF	ISF
8899	1.4219	7546.1	1367.3	4462.6	19.865

## CHENIUM HEAT EXCHANGER - BASELINE HEATER

SN4 001 RUN4

## UNREDUCED DATA (Volts)

THEUST RUN	THRUST ZERO	THROST CALIBRATION				STEP 5,	STEP 2	STEP 3	STEP 4	STEP 5,	STEP 2
		STEP 1	STEP 2	STEP 3	STEP 4						
+2.56E	+4.954	+2.172	+3.398	+4.625	+3.412	42.179	42.179	42.179	42.179	+4.922	+4.922
df (run)	PF(0)	PS(run)	PS(0)	PS(0)	PS(run)	PS(run)	PS(run)	PS(run)	PS(run)	Falt(0)	Falt(0)
+.011017	-.000257.	+.006440	-.000039	-.000039	-.000022	-.000022	-.000022	-.000022	-.000022	-.000000	-.000000

## RECORDED TEMPERATURES (degrees F)

Tf	Tc	T16	Tb	Tv	POINT	
					TEMPER	TEMPER
+89.7	+1193.2	+1847.3	+1240.1	+269.2	+76.1	+76.1
Tb	Temper	Temper	Temper	Temper	Temp	Temp
1627.9	+32.0	+31.9	+32.0	+0.0	+0.0	+0.0

## REDUCED DATA

(exit)	A (thrust)	A (thrust)	Pf	Ps	POINT	
					CORR.	CORR.
g. in.	sc. in.	sc. in.	Pf1	Ps1	Ps1	Ps1
.06225	.00023	.00023	E7.1	42.3	0.0	0.0
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER
volt	watts	watts	ohms	temp	wire	delta
33.33	22.516	750.5	1.4803	4861.5	.00000	0.0

## REDUCED DATA

Cf	C*	REYNOLDS	THRUST	MEAS	VACUUM	TEMP.	ISP	POINT	
								DEG. F	SEC.
.8939	1.4969	7085.0	147.3	3678.3	19.012	329.6	329.6		

## RHENIUM HEAT EXCHANGER - BASELINE HEATER

## UNREDUCED DATA (volts)

THRUST RUN	THRUST ZERO	STEP 1	STEP 2	STEP 3	STEP 4	STEP 5
2.935	+1.298	+2.506	+3.732	+4.915	+43.737	+2.527
(run)	(ff(0))	(run)	(run)	(run)	(run)	(run)
+012000	-060065	+007168	+000016	+0000010	+0000030	

## RECORDED TEMPERATURES (degrees F)

Tf	Tc	T16	Tb	Tv	Tb	Tb
+79.0	+1145.1	+1553.4	+1170.3	+208.7	+576.3	
Th	TEPERE	TEPERE	TEPERE	TEPERE	TEPERE	
1402.8	+32.0	+32.4	+32.0	+0.0	+0.0	

## F11t

exit)	A(thrust)	A(thrust)	REDUCED DATE	F11t
g. in.	C(GF)	C(GF)	FF	FF
g. in.	sq. in.	sq. in.	F11t	F11t
.06228	.000623	.000623	96.0	96.0
HEATER	HEATER	HEATER	HEATER	HEATER
Volt	amps	watts	amps	watts
25.29	15.768	500.4	1.2776	42.23.0
0.00000				0.0000

LIGHT GLASS	THRUST	THRUST	REYNOLDS	GAS
bs/hr	MEAS	VAC	NUMBER	TEMP.
	lbs	lbs		deg. f
0.00000	0.033	0.245		
	447728			

CD	C*	ft/sec	ft/sec	ISF
.9006	1.4261	6848.4	1696.5	3641.8

## PHENIUM HEAT EXCHANGER - BASELINE HEATER

SMT 001 RUN

## UNREDUCED DATA (volts)

THRUST RUN	THRUST ZERO	STEP 1	STEP 2	STEP 3	STEP 4	STEP 5	CTFP
12.980	+1.249	+2.458	+3.691	+4.910	+3.711	+2.508	+1.24
.012000	.000187	.007073	.000016	.000010	.000000	.000000	

## RECORDED TEMPERATURES (degrees F)

Tf	Tc	Tb	Tv	Tm
+64.7	+1166.2	+1957.1	+1230.0	+763.2
Th	Tsfare	Tesare	Tspare	Tefare
+1642.8	+32.0	+32.0	+32.0	+0.0

## { PREDICTED ACTIVATED AIR DATA (F)

Entity	ACTIVATED COLD air in.	AIR COP.	Ff	FE	FE14
g. in.	sc. in.	sc. in.	FE12	FE12	FE12
.06228	.00622	.00622	.94.5	.45.8	.0522
HEATER	HEATER	HEATER	HEATER	HEATER	PF HEATER
VOLTS	AMPS	Watts	Ohms	WIRE diameter	HELTAP FE12
34.57	23.132	799.7	1.4945	.4906.2	.00000

## { PREDICTED THRUST DATA (lb's)

CH	Cf	C*	THRUST MEAS	THRUST VAC	ISP
.8952	1.4851	7234.2	1517.7	4088.8	18.890

UNREDUCED DATA (volts)					
THRUST	STEP 1	STEP 2	STEP 3	STEP 4	STEP 5
ZERO					
2.941	+1.258	+2.468	+3.719	+4.923	+3.739
f(run)	f(0)	P(run)	TPS(0)	PS1t(run)	PS1t(0)
.012000	.0000197	.0006990	.000016	.0000010	.0000025

## RECORDED TEMPERATURES (degrees F)

Tf	Tc	T15	Tb	Tv	Tb
+8E+7	+1188.5		+1819.2	+1230.5	+258.9
Th	Temp	Temp	Temp	Temp	Temp
1619.6	+22.0	+32.0	+32.0	+0.0	+0.0

## REDUCED DATA

(exit)	Altitude cold	Altitude COP	Pf	Fg	Felt
S. in.	sc. in.	sc. in.	psie	psie	psie
.06228	.000640	.000640	93.9	45.8	0.0
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER
Volts	amps	watts	ohms	temp	wire gig
33.29	22.544	750.5	1.4767	4850.1	.00000

LIGHT LOSS bs/hr	MICRO MOTION 1bs/hr	THRUST MEAS mlbs	THRUST VAC mlbs	GAS TEMP. deg. f	ISF
0.00000	.44859	.0.741	.40.900		

DP	Cf	REYNOLDS NUMBER	GAS TEMP. deg. f	PSP	ISF
.8933	1.4337	7346.0	1460.3	4246.6	18.345

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## UNREDUCED DATA (volts)

THRUST RUN	STEP 1 ZERO	STEP 2	STEP 3	STEP 4	STEP 5
.2.335	+1.752	+1.970	+3.213	+4.425	+3.213
f(run)	f(0)	Pg(run)	Pg(0)	Pg(run)	Pg(0)
.011707	-0.000039	+.006905	-0.000002	+4.000000	-0.0000

## RECORDED TEMPERATURES (degrees F)

Tf	Tc	T16	Tb	Tv	Tb
479.3	+1144.1	+1544.2	+1173.7	+210.4	+50.7
Th	Tspare	Tspare	Tspare	Tspare	Tspare
441.4	+30.0	+37.0	+37.0	+0.0	+0.0

## REDUCED DATA (ft/lb-sec)

Effit	Act. force	Eff. force	Ft/lb-sec	Ft/lb-sec	Ft/lb-sec
1.0	1.0	1.0	0.00000	0.00000	0.00000
.06228	.000023	.000023	92.8	44.1	0.0
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER
VOLTS	AMPS	WATTS	TEMP	WIRE TEMP	WIRE TEMP
25.37	19.710	500.0	1.2870	4252.1	.00000

IGHT GLASS	MICRO MOTION	THRUST MEAS	THRUST VAC
bs/hr	lbs/hr	lbs	lbs
0.00000	46648	38.794	38.942

CP	Cf	REYNOLDS NUMBER	GAS TEMP. deg. f	FSP	ISP
.6998	1.4175	6821.4	1668.8	3601.2	12.639
					300.5

Post = calc n. 6 - Data on F is questionable  
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## NICHNIUM HEAT EXCHANGER - BASELINE HEATER

EN# CO1 RUN# 14

## UNREDUCED DATA (VOLTS)

THRUST RUN	STEP 1 ZERO	STEP 2	STEP 3	STEP 4	STEP 5	STEP 6
-3.001	+ .017	+2.032	+3.284	+4.485	+3.299	+2.093
op (run)	Pf (C)	Pf (run)	Pf (run)	Pf (run)	Palt (run)	Palt (C)
.014233	-.000069	+ .010101	-.000001	-.000001	-.000010	-.000031

## RECORDED TEMPERATURES (degrees F)

Tf	To	Ti6	Tb	Tv	Tn
+84.9	+116.5	+176.2	+1209.6	+217.0	+714.3
Tf	TECNE	TEMPER	TEMPER	TEMPER	TEMPER
1627.3	+32.0	+32.0	+32.0	+0.0	+0.0

## REDUCED DATA

Volts	Amperes	Atmosphere	Eff.	Ff	Fg	Fh
34.40	23.270	800.6	1.4785	4855.9	.00000	0.0
0.00000	0.5464	53.094	53.017	53.017	53.017	53.017
3.001	0.010101	-.000001	-.000001	-.000001	-.000001	-.000001

## MICRO THRUST MEAS VAC

WEIGHT GLASS bs/hr	MOTION Amps/hr	VAC inches
0.00000	0.5464	53.094

## REYNOLDS GAS PSP ISP

ICF	Cx	NUMBER	GAS TEMP. deg. f	PSP	ISP
.9040	1.5938	7114.3	1626.1	4022.1	15.016

Thrust Stand Mac Functions

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UNREDUCED DATA (volts)						
THRUST RUN	THRFST ZERO	STEP 1	STEP 2	STEP 3	STEP 4	STEP 5
2.759	+1.090	+2.310	+3.561	+4.791	+3.581	+2.356
f(run)	Ff(0)	Ps(run)	Pf(0)	Pe(run)	Palt(run)	Palt(0)
.012000						
	- .000069		+ .007176		+ .0000010	- .0000030

RECORDED TEMPERATURES (degrees F)			
T <sub>f</sub>	T <sub>c</sub>	T <sub>b</sub>	T <sub>v</sub>
+67.6	+1147.9	+1550.2	+1177.8
T <sub>b</sub>	TEMP	TEMP	TEMP
143.6	+32.0	+32.0	+6.0

REDUCED DATA			
level)	Actuator)	(thrust)	PF
S. In.	COLP	ENR.	PF
.06228	SG. In.	SC. In.	PF
.000623	.000623	.000623	.000623
HEATER	HEATER	HEATER	HEATER
volt	watts	others	temp
25.32	19.726	499.5	1.2838
EIGHT	MICRO	THRUST	THRUST
CLASS	MOTION	HEAS	VAC
bs/hr	lbs/hr	lbs	lbs
0.00000	8.67E0	40.42E	40.701

REDUCED DATA			
level)	Actuator)	(thrust)	PF
S. In.	COLP	ENR.	PF
.06228	SG. In.	SC. In.	PF
.000623	.000623	.000623	.000623
HEATER	HEATER	HEATER	HEATER
volt	watts	others	temp
25.32	19.726	499.5	1.2838
EIGHT	MICRO	THRUST	THRUST
CLASS	MOTION	HEAS	VAC
bs/hr	lbs/hr	lbs	lbs
0.00000	8.67E0	40.42E	40.701
CP	C*	REYNOLDS	GAS
		NUMBER	TEMP.
.8984	1.441E	ft/sec	deg. f
			sec.

REDUCED DATA			
level)	Actuator)	(thrust)	PF
S. In.	COLP	ENR.	PF
.06228	SG. In.	SC. In.	PF
.000623	.000623	.000623	.000623
HEATER	HEATER	HEATER	HEATER
volt	watts	others	temp
25.32	19.726	499.5	1.2838
EIGHT	MICRO	THRUST	THRUST
CLASS	MOTION	HEAS	VAC
bs/hr	lbs/hr	lbs	lbs
0.00000	8.67E0	40.42E	40.701
CP	C*	REYNOLDS	GAS
		NUMBER	TEMP.
.8984	1.441E	ft/sec	deg. f
			sec.

UNREFERENCED DATA (CONT'D)

THRUST RUN	f (run)	Ff (0)	Psi (run)	Psi(0)	Psi(trun)	Psi(t0)	Psi(trun)	Falt (0)	Falt (t0)
THRUST CALIBRATION									
STEP 1									
ZERO									
STEP 2									
STEP 3									
STEP 4									
STEP 5									
3.305	+1.257	+2.479	+3.707	+4.981	+3.781	+2.550			
3.14000	-0.000318	-0.000001	-0.000001	+0.009993	+0.000001	-0.000010	-0.000026		

## **REC'D BY TELEX-EARTHINES (DEGREES F)**

EFFECTIVE

**MICRO THREESTEP**

CLASSE	MOTION ips/hr	MEAS lbs	VAC lbs
0.00000	54250	49.247	49.437

C\* REYNOLDS 64 FGF IGF

ft/sec	NUMBER	TEMP. deg. f	SEC.
1000	2	4000	15 100
1000	3	4000	15 100
1000	4	4000	15 100

## HENIUM HEAT EXCHANGER - BASELINE HEATER

SN# 001 RUN#

## UNREDUCED DATA (volts)

THRUST RUN	THRUST ZERO	STEP 1	STEP 2	STEP 3	STEP 4	STEP 5.	STEP 6.	STEP 7.
3.342	+1.277	+2.502	+3.733	+5.000	+3.614	+2.570	+1.314	
f(run)	Ff(0)	Pf(run)	Pf(0)	Pf(run)	Pf(0)	Pf(run)	Pf(0)	
.014000	.000280	+.009978	+.000001	+.000018	+.000003	+.000003	+.000003	

## RECORDED TEMPERATURES (degrees F)

Tf	Tc	T16	Tb	Tv	Tf	Tc	Tb	Tv
494.9	+1108.7	+1794.5	+1225.9	+243.0				
Th	Temp	Temp	Temp	Temp				
1646.9	432.0	432.0	432.0	40.0				

## REDUCED DATA

Item	P(Thruster) CCF	A(Thruster) CCF	Ff	Pg	Pg14	Pg15	Pg16	Pg17
g, in.	sec. in.	sec. in.	Pg14	Pg15	Pg16	Pg17	Pg18	Pg19
.06228	.000623	.000620	127.5	83.6	0.0	.0024		
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	Ff
volt	amps	watts	ohms	temp	temp	temp	temp	temp
33.26	24.648	799.8	1.3831	4554.9	+00000	+00000	+00000	0.0

IGHT LASS bs/hr	THRUST MEAS lbs/hr	THRUST VAC lbs
0.0000	53739	9.971

TR	Cf	C*	REYNOLDS NUMBER	GAS TEMP. deg. f	PSP	ISP
.5028	1.5006	7198.9	1778.7	4173.0	15.959	335.2

Run terminated early because of  
 Heater shorting  
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## RHENIUM HEAT EXCHANGER - BASELINE HEATER

SN# 001 FNU# 21

## UNREDUCED DATA (volts)

THRUST FLN	THRUST ZERO	STEP 1	STEP 2	STEP 3	STEP 4	STEP 5	STEP 6
-3.516	+1.134	+2.353	+3.622	+4.870	+3.648	+2.442	+1.174
(fln run)	(fln C)	Pf (run)	Pf (0)	Pd1 (run)	Pd1 (0)	Felt(0)	Felt(0)
+ .015680	-.000059	+ .013540	-.000003	+ .000019	-.000037		

## RECORDED TEMPERATURES (degrees F)

Tf	Tc	T16	Tb	Tv	Th
+80.1	+1159.7	+1404.1	+1184.5	+192.5	+504.5
Th	Tempe	Tempe	Tempe	Tempe	Tempe
+309.2	+32.0	+32.0	+32.0	+0.0	+0.0

(exit)	Attachment	Distance	CO2	PF	FE	Felt
1 fl.	CGT	SC. 1in.	PS12	PS12	PS12	PS12
.62228	.000622	.000636	167.9	56.1	0.0	+0.021
HEATER	HEATER	HEATER	HEATER	HEATER	WIRE	DELTA
volt	amps	watt	ohms	temp	dia	Fe
24.96	20.100	500.4	1.2387	4099.5	.00000	0.0

IGHT	MICRO	THRUST	THRUST
LASS	MOTION	MEAS	VAC
bs/hr	lbs/hr	lbs	lbs
0.00000	.65161	57.582	57.372

CD	Cf	Cx	REYNOLDS	GAS	ISP
.9157	1.5693	6543.6	2408.4	3404.2	8.662
					319.2

## UNREDUCED DATA (cycles)

THRUST RUN	THRUST ZERO	STEP 1	STEP 2	STEP 3	STEP 4	STEP 5	STEP 6
2.764	+1.136	+2.359	+3.584	+4.810	+43.599	+42.387	+1.130
f(run)	Ff(0)	Fs(run)	Fs(0)	Fs(run)	Fs(0)	Felt(0)	Felt(0)
.011997	-.000134	+.007249	-.000003	-0.000020	-.0000035	-.0000020	-.0000035

## RECORDED TEMPERATURES (degrees F)

TR	TC	T16	Tb	Tv	Tn	Tr
+63.2	+1149.3	+1525.1	+1174.6	+211.8	+572.0	
Tb	TEMPER	TEMPER	TEMPER	TEMPER	TEMPER	
1410.1	432.0	432.0	432.0	40.0	40.0	

## REDUCED DATA

ENVIT	ACTHANE	ACTHANE	PP	PP	PP	PP
3. IN.	sec. in.	sec. in.	PSIG	PSIG	PSIG	PSIG
.06226	.000622	.000622	96.7	45.6	0.0	0.0
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER
voitie	ENPE	watt	ohms	temp	ohms	temp
25.39	19.678	499.6	1.2901	4261.7	0.0000	0.0

MICRO LESS	THRUST	THRUST	VAC
bs/hr	MEAS	MEAS	inches
0.00000	47391	40.025	40.184

CD	Cf	C*	REYNOLDS NUMBER	GAS TEMP. deg. f	ISP
.8995	1.4163	6934.5	1657.7	3742.1	12.432

UNEFI-EMBEDDED HETEROGENEITY

THRUST RUN	THrust ZERO	STEP 1	STEP 2	STEP 3	STEP 4	STEP 5	STEP 6
3.675	41.285	42.485	+3.737	+4.960	+3.768	+2.545	+1.256
f(run)	f(0)	Fs(run)	Fs(0)	Ft(run)	Ft(0)	Falt(run)	Falt(0)
014000	-0000246	+013590	-0000803	-0000020	-0000037		

## RECORDED TEMPERATURES (degrees F)

Th	Tf	Tc	T16	Tb	Tv	Tn
+85.6	+1191.2	+1460.2	+1219.8	+229.2	+680.7	
Th	TEPERE	TEPERE	TEPERE	TEPERE	TEPERE	TEPERE
+825.0	+37.0	+37.0	+32.0	+0.0	+0.0	+0.0
REDUCED DATA						
(enit)	W(hance)	A(treant)	Ff	Ff	Ff	Ff
g. in.	g. in.	g. in.	psia	psia	psia	psia
+0.225	.00062	.00062	165.4	61.0	0.0	+0.0
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER
voltE	ampE	watE	chme	temp	wire	delta
-20.7	-20.7	-20.7	-1.175	-210.7	-20000.0	-20.0

CD	Cr	C*	REYNOLDS NUMBER	GAS TEMP. deg. f	PSP	ISP
1.9118			6922.0	2188.5	3848.2	15.930

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## HEMIUM HEAT EXCHANGER - BASELINE HEATER

SN# 001 FURN#

## UNREFINED DATA (volts)

THRUST RUN	THRUST ZERO	STEP 1	STEP 2	STEP 3	STEP 4	STEP 5	STFF
3.827	+1.314	+2.532	+3.759	+5.016	+3.829	+2.573	+1.314
f(run)	Ff(0)	FE(0)	FE(run)	PSlt(run)	PSlt	Pelt(0)	
.016980	-.000307	+.014098	-.000003	-.000020	-.000020	-.000020	

## RECORDED TEMPERATURES (degrees F)

T <sub>C</sub>	T <sub>16</sub>	T <sub>4</sub>	T <sub>8</sub>	T <sub>12</sub>	T <sub>16</sub>	T <sub>4</sub>	T <sub>8</sub>
+025.5	+1195.8	+1692.7	+1227.8	+236.4	+1227.8	+1195.8	+025.5
T <sub>H</sub>	Temperature	Temperature	Temperature	Temperature	Temperature	Temperature	Temperature
1555.6	+32.0	+32.0	+32.0	+32.0	+32.0	+32.0	+0.0

## REDUCED DATA

Entity	Attenuation	Attenuation	Eff.	Eff.	Eff.	Eff.	Eff.
g. in.	g. in.	g. in.	PS1E	PS1E	PS1E	PS1E	PS1E
.06228	.000623	.000640	174.2	64.9	64.9	64.9	64.9
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER
volt	amps	watts	ohms	temp	temp	temp	temp
34.16	23.424	800.1	1.1.4583	4792.2	4792.2	4792.2	4792.2
0.00000				.00000	.00000	.00000	.00000

## IGHT MICRO THRUST MEAS VAC

LASS	MOTION	MEAS	VAC
bs/hr	bs/hr	lbs	lbs
0.00000	0.00000	60.795	80.964

CD	C*	REYNOLDS	GAS TEMP	PSP	ISP
.9106	1.5078	7204.6	2124.8	4215.9	13.125
					33.6

## CHENIUM HEAT EXCHANGER - BASELINE HEATER

S/N# 0612 Run# 2

UNREDUCED DATA (cycles)			
	THRUST CALIBRATION		
	STEP 1	STEP 2	STEP 3
THRUST ZERO RUN	+1.2e1	+2.6e2	+3.839
-3.647	+45.045	+43.852	+42.672
ff(run)	(P0)	(P10)	(P101)
-0.013222	+0.000010	+0.016013	+0.000045
			+0.000062

## RECORDED TEMPERATURES (degrees F)

Tf	Tc	T16	Tb	Tv	Tn
+76.9	+1175.6	+1656.7	+1209.5	+210.6	+654.4
Tf	TEMP	TEMP	TEMP	TEMP	TEMP
+22.3	+32.3	+30.0	+32.0	+0.0	+0.0

Test	Altitude ft	Altitude sec.	Altitude deg	Reduced Data	Test	Test
1. in.	CCLP	CCLP	0.0000	FE	FE	FE
2. in.	SC. in.	SC. in.	0.0000	FE	FE	FE
3. in.	.000022	.000022	0.000022	FE	FE	FE
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER
volts	amps	watts	ohms	temp	wire	delta
32.74	22.922	750.4	1.1e1	4282	4697.2	0.00000

WEIGHT lb/s	THRUST MEAS lb/s	THRUST VAC lb/s	REYNOLDS NUMBER	GAS TEMP. deg. f	PSF	ISG Sec.
0.0000	1.63260	1.539.922	30,123	3811.4	12.4E1	342.1

## HEMIUM HEAT EXCHANGER - BASELINE HEATER

SH# 002 RUN#

UNREDUCED DATA (volts)					
	THRUST	STEP 1	STEP 2	STEP 4	STEP 5
RUN	ZERO	STEP 1	STEP 2	STEP 4	STEP 5
-3.801	+1.324	+2.547	+3.773	+4.986	+3.793
(run)	F4 (C)	F5 (run)	F6 (run)	F7 (run)	F8 (run)
.613720	.40000010	.4016953	.4000223	.4000048	.4000056

## RECORDED TEMPERATURES (degrees F)

	T <sub>C</sub>	T <sub>B</sub>	T <sub>A</sub>	T <sub>M</sub>	T <sub>R</sub>
+80.6	+118C.1	+168E.6	+1221.4	+225.2	+791.1
T <sub>b</sub>	TEMPER	TEMPB	TEMPA	TEMP	TEMP
155.7	432.6	432.0	432.0	+0.0	+6.0

## REDUCED DATA

entity	efficiency	current (amps)	power (watts)	temp (ohms)	temp (deg F)	heat (deg F)	heat (deg C)	heat (deg F)	heat (deg C)	heat (deg F)	heat (deg C)
SH. IN.	sec. in.	sec. in.	sec. in.	sec. in.	sec. in.	sec. in.	sec. in.	sec. in.	sec. in.	sec. in.	sec. in.
+622E	.000002	.0000041	.0000041	.0000041	169.6	64.5	0.0	.0016	.0016	.0016	.0016
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER
VOLTS	EMFE	WATTS	OHMS	TEMP	TEMP	TEMP	TEMP	TEMP	TEMP	TEMP	TEMP
34.16	23.406	799.5	1.4594	4795.6	.00000	.00000	.00000	.00000	.00000	.00000	.00000

LIGHT	MICRO	THRUST	THRUST	MEAS	VAC						
1.506	1.506	1.506	1.506	1.506	1.506	1.506	1.506	1.506	1.506	1.506	1.506
6.00000	6.00000	6.00000	6.00000	6.00000	6.00000	6.00000	6.00000	6.00000	6.00000	6.00000	6.00000

EF	E1	C*	REYNOLDS	GAS	PSF	TEM
			NUMBER	TEMP.		SEC.
			des.	des.		
.9096	1.506	7261.5	2674.2	4287.5	13.207	340.1

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## HENIUM HEAT EXCHANGER - BASELINE HEATER

SN# 0001 Run# 2

## UNREDUCED DATA (volts)

THRUST RUN	STEP 1 ZERO	STEP 2	STEP 3	STEP 4	STEP 5	STEP 6
3.690	+1.335	+2.557	+3.777	+5.004	+3.800	+2.574
f(run)	-ppf(0)	-Pf(run)	-Ps(0)	-Pst(run)	-Psit(0)	-Pstt(run)
.013338	+.000010	+.016000	+.000287	+.000046	+.000041	+.000041

## RECORDED TEMPERATURES (degrees F)

T <sub>f</sub>	T <sub>c</sub>	T <sub>16</sub>	T <sub>b</sub>	T <sub>v</sub>	T <sub>n</sub>
+83.5	+1185.1	+1655.6	+1218.7	+227.2	+680.2
Th	Tspare	Tspare	Tspare	Tspare	Tspare
1532.9	430.0	+32.0	+32.0	+0.0	+0.0

## REDUCED DATA

exit	A(ccretion) cold	A(ccretion) hot	Ff	Ff	Ff	Ff
g. in.	sc. in.	sc. in.	Fse1e	Fse1e	Fse1e	Fse1e
.06228	.000622	.000639	165.4	61.2	6.0	.0025
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER
Volt	Temp	Watts	Ohms	temp	wire	delta
32.75	22.512	750.4	1.4295	4701.3	.00000	0.0

LIGHT LOSS bs/hr	MICRO MOTION inches/ hr	THRUST MEAS lbs	THRUST VAC lbs	REYNOLDS NUMBER	GAS TEMP. deg. f	PSP	ISP
0.00000	\$8163	37.380	157.518				

CD	C*	REYNOLDS NUMBER	GAS TEMP. deg. f	PSP	ISP
.9111	1.5104	6985.7	2150.3	3923.6	13,042
					3274

PHENIUM HEAT EXCHANGER - BASELINE HEATER

SNT OCC RUN#

UNREDUCED DATA (volts)					
THRUST RUN	THRUST ZERO	STEP 1	STEP 2	STEP 3	STEP 4
-3.954	+1.430	+2.641	+3.847	+5.077	+2.666
f(run)	f(0)	P(run)	P(0)	Palt(run)	Palt(0)
+.014310		+0.017000		-.000205	-.000041

RECORDED TEMPERATURES (degrees F)					
Tf	Tc	T16	Tb	Tc	Tf
+82.7	+1150.0				+1224.6
		+1725.7			+222.8
Tb	Temper	Temper	Temper	Temper	Temper
1586.2	427.0	+32.0	+32.0	+0.0	+0.0

REDUCED DATA					
(exit)	Air (exit)	Air (front)	CCP.	FEA	FEA
.5. in.	50. in.	5. in.	FEA	FEA	FEA
.66225	.000623	.000639	176.4	64.6	C. C
HEATER	HEATER	HEATER	HEATER	HEATER	FEA
volt	watts	amps	temp	wire	delta
25.16	24.326	855.3	1.4453	4751.2	.00000
					0. C

MIGHT					
CLASS	THRUST	MEAS	VAC	GAS	FSP
lbs/hr	lbs/hr	inches	inches	temp.	
0.00000	31.924	62.104	13.772		

CR	Cf	C*	REYNOLDS	GAS	FSP	ISP
	ft/sec		NUMBER	TEMP.		SEC.
.9123	1.5432	7057.4	2213.8	4032.0	13.772	358.5

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## HEMIUM HEAT EXCHANGER - BASELINE HEATER

SN# 001 RUN# 25

THRUST RUN	THRUST ZERO	STEP 1	STEP 2	STEP 3	STEP 4	STEP 5	STEP 6
3.679	+1.381	+2.588	+3.810	+5.039	+3.834	+2.610	+1.354
f(run)	f(0)	Pg(0)	Pg(tur)	Pg(tur)	Palt(run)	Palt(tur)	
.014308	4.000010	+0.017000	+0.003117	-0.000041	-0.000056		

UNREDUCED DATA (volts)							
THRUST	ZERO	STEP 1	STEP 2	STEP 3	STEP 4	STEP 5	STEP 6
3.679	+1.381	+2.588	+3.810	+5.039	+3.834	+2.610	+1.354
f(run)	f(0)	Pg(0)	Pg(tur)	Pg(tur)	Palt(run)	Palt(tur)	
.014308	4.000010	+0.017000	+0.003117	-0.000041	-0.000056		

REFINED TEMPERATURES (degrees F)							
Tf	Tc	Tt6	Tb	Tv	Tt	Tb	Tt6
+86.5	+1200.5	+1730.2	+1236.7	+244.0	+775.8		
Tf	Temp	Temp	Temp	Temp	Temp		
1564.5	+62.0	+32.0	+32.0	+0.0	+0.0	+0.0	+0.0

REFUSED DATA							
Event	Action	Action	Action	Ff	Ff	Ff	Ff
5. init.	CCLI	CCLI	CCLI	PEIE	PEIE	PEIE	PEIE
	sc. in.	sc. in.	sc. in.	PEIE	PEIE	PEIE	PEIE
.065228	.000622	.000640	.000640	172.4	65.0	65.0	65.0
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER
Volt	amps	watts	ohms	temp	temp	temp	temp
35.20	24.156	850.2	1.4570	4786.2	0.0000	0.0000	0.0000

SIGHT							
THRUST	MEAS	VAC	TEMP.	REYNOLDS	GAS	PSP	ISP
CLASS	inches	inches	deg. f	NUMBER	TEMP.	SEC.	SEC.
bs/hr	inches	inches					
0.00000	365265.	31.064	31.244				
9105	1.5121	7125.3	2138.9	4196.3	13.882	337.8	

## RHENIUM HEAT EXCHANGER - FASELINE HEATER

SN# 001 PSP

## UNREFINED DATA (volts)

THRUST RUN	THEST ZERO	STEP 1	STEP 2	STEP 3	STEP 4	STEP 5	STEP 6
12.8E6	+1.2E5	+2.510	+3.744	+4.947	+3.756	+2.576	+1.751
PF (run)	PF (0)	PF(run)	PF(0)	PF(run)	PF(0)	PF(run)	PF(0)
+.007088	+.000010	+0.011000	+0.000303	+0.000050	+0.000000	+0.000000	+0.000000

## RECORDED TEMPERATURES (degrees F)

Tf	Tc	Td	Tb	Tv	Tu	Tn	Tm
487.7	+1159.4	+1533.4	+1184.8	+297.4	+582		
Th	TEMP	TEMP	TEMP	TEMP	TEMP	TEMP	TEMP
1427.6	+32.0	+32.0	+32.0	+0.0	+0.0	+0.0	+0.0

## REDUCED DATA

(event)	ATTACHMENT	DETACHMENT	PF	PF	PF	PF	PF
sec. in.	sec. in.	sec. in.	sec.	sec.	sec.	sec.	sec.
.06228	.000622	.000435	.04.8	.04.8	.42.4	.6.0	.00000
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER
VOLTS	EMFS	MATS	SHMS	SHMS	SHMS	SHMS	SHMS
25.28	19.774	499.9	1.2786	4.225.3	.00000	.0.0	.0.0

WEIGHT CLASS	THRUST IN MOTION	THRUST VAC	REYNOLDS NUMBER	GAS TEMP. deg. f	ISP
bs/hr.	lbs/hr.	lbs	ft/sec	sec.	
0.00000	46559	38.966	1.4767	1745.7	3323.4
		39.993			12.788

CD	C*	C*	REYNOLDS NUMBER	GAS TEMP. deg. f	ISP
.9019	1.4767	6576.9			302.7

## HENIUM HEAT EXCHANGER - BASELINE HEATER

SIN# 001 RUN# 30

## UNREDUCED DATA (volts)

THRUST RUN	THRUST ZERO	STEP 1	STEP 2	STEP 3	STEP 4	STEP 5	STEP 6
3.365	+1.334	+2.550	+3.773	+4.994	+3.786	+2.560	+1.306
.010221	.000010	.014000	.006319	.000050	.000000	.000063	.000000

## RECORDED TEMPERATURES (degrees F)

Tf	Tc	T16	Tb	Td	Tg	Th	Ti
+68.5	41206.0	41715.9	41225.2	4246.7	4717.8		
Tf	Temperature	Temperature	Temperature	Temperature	Temperature		
1562.0	432.0	432.0	432.0	432.0	432.0		

## FEDUCED

Volts	Amperes	Resistance	FEDUCED DATE	Ff	Fe	Ff1	Fe1
32.82	22.846	749.9	1.4367	4723.9	.000000	0.0	.000000
0.00000	.54147	49.525	49.663	49.663	.000000	0.0	.000000
.9034	1.462E	7164.2	1802.3	4082.3	15.067	330.2	330.2

Cf	C*	REYNOLDS NUMBER	GAS TEMP. deg. f	PSP	TSF

## PHENYL HEAT EXCHANGER - BASELINE HEATER

SN# 002 RUN#

THRUST RUN	THRUST ZERO	STEP 1	STEP 2	STEP 3	STEP 4	STEP 5
+5.556	+1.891	+3.111	+4.320	+5.526	+6.332	+3.116
+ (run)	(off (C))	P(run)	P(0)	Palt(run)	Palt(0)	
.007140	.006307	.014983	.000031	.000009	.000009	.000009

## UNREDUCED DATA (volts)

THRUST RUN	THRUST CALIBRATION	STEP 2	STEP 3	STEP 4	STEP 5	STEP 6
+5.556	+1.891	+3.111	+4.320	+5.526	+6.332	+3.116
+ (run)	(off (C))	P(run)	P(0)	Palt(run)	Palt(0)	
.007140	.006307	.014983	.000031	.000009	.000009	.000009

## RECORDED TEMPERATURES (degrees F)

TF	TG	T16	Tb	Tv	Tp
47E+2			41521.3	41172.5	+207.5
			+1244.6		+674
TB	TEMP	TEMP	TEMP	TEMP	TEMP
1440.1	432.6	224.6	152.6	50.0	0.0

REDUCE TEMP	REFUGEE TEMP	REF	REF	REF	REF
HEATER	HEATER	SWIRE	SWIRE	SWIRE	SWIRE
Watts	Watts	Ohms	Ohms	Ohms	Ohms
2E+37	1E+666	4E9.6	1.2E80	4.25E+01	+00000
GOL	GOL	deg. A	deg. B	deg. C	deg. D
5E+20.4	5E+20.4	5E+20.4	5E+20.4	5E+20.4	5E+20.4
0.0001	0.0001	0.0001	0.0001	0.0001	0.0001

THRUST	THRUST	VAC	VAC	VAC	VAC
MICRO	MEAS	WIDS	WIDS	WIDS	WIDS
SHOTS	INJECTION	1bs/hr	1bs/hr	1bs/hr	1bs/hr
bs/hr	1bs/hr				
0,00000	3.6563	40.529	40.577	40.577	40.577

REYNOLDS	GAS	FSS	TEM
CF	NUMBER	TEMP.	TEM
E992	1.4475	6530.8	1646.1

## NENIUM HEAT EXCHANGER - BASELINE HEATER

SN# 001 FURN 2

## UNREDUCED DATA (volts)

THRUST RUN	STEP 1	STEP 2	STEP 3	STEP 4	STEP 5	STEP 6
ZERO						
4.029	+1.062	+3.094	+4.345	+5.562	+4.364	+1.894
f(run)	F1(0)	F2(0)	F3(0)	F4(0)	F5(0)	F6(0)
.010371	.0000007	.014373	.000208	.00005	.000005	.000016

## RECORDED TEMPERATURES (degrees F)

TR	Tc	T16	Tb	Tv
+84.2	+1193.9	+1807.7	+1235.1	+251.3
Tb	Temp	Temp	Temp	Temp
+672.7	+32.0	+32.0	+32.0	+0.0

GENETIC	Atmosph.	REDUCED DATA	REDUCED DATA	REDUCED DATA
COLI	CO2,	PF	PF	PF
in.	sc. in.	sc. in.	sc. in.	sc. in.
.06228	.000227	.000640	.131.6	.54.8
HEATER	HEATER	HEATER	HEATER	HEATER
Volts	Watts	Watts	ohms	wire
35.85	23.644	647.6	1.5163	.975.0
0.00000	0.542E1	52.089	52.305	.00000

RIGHT LASS	THRUST lb/hr	MEAS. lb/s	THRUST lb	GAS TEMP. deg. f	PSF	ISP
4.925	1.533E	7281.5	1769.0	4233.8	16.206	347.1

## THERMIUM HEAT EXCHANGER - BASELINE HEATER

SN# 001 RUN#

## UNREDUCED DATA (volts)

THRUST RUN	THRUST ZERO	STEP 1	STEP 2	STEP 3	STEP 4	STEP 5	STEP 6
4.226	+2.065	+3.264	+4.468	+5.720	+4.527	+3.309	+2.066
f(run)	Ff(0)	Pf(run)	Pf(0)	Pf(run)	Pf(0)	Pf(run)	Pf(0)
.010610	.000007	.014997	.000023	.000024	.000024	.000024	.000024

## RECORDED TEMPERATURES (degrees F.)

Tf	Tc	T16	Tb	Tv
475.9	4170.5	41712.4	41207.8	4214.8
Tb	TEMPER	TEMPER	TEMPER	TEMPER
4596.2	452.0	432.0	432.0	40.0

## REDUCED DATA

current	Amperes	Resistance	FF	FS	PEL1
0.6228	.000623	.000623	134.6	56.4	0.070
HEATER	HEATER	HEATER	HEATER	HEATER	FF
voltage	amps	watts	ohms	temp	WIRE RES
32.91	22.666	746.7	1.4506	4767.9	.00000

## MICRO THRUST

LOSS	MOTION	MEAS	VAC	1/16
0.00000	2.56864	253.968	53.295	1.4506
CP	Cf	REYNOLDS	GAS	FSP

ft/sec	ft/sec	NUMBER	TEMP.	FSP
.9055	1.5142	7155.7	1689.8	14.037

**APPENDIX B**  
**Sealed Cavity Data**

ROCKET RESEARCH COMPANY

DATE: 31 Aug 1987 15:

## RHENIUM HEAT EXCHANGER - BASELINE HEATER

SN# 001 RL

		UNREDUCED DATA (volts)							
		THRUST ZERO		STEP 1		THRUST CALIBRATION			
				STEP 2		STEP 4		STEP 5	
THRUST RUN		+0.331	+0.000	+0.000	+0.000	+0.000	+0.000	+0.000	+0.000
Pf(run)	Ff(0)			Fg(run)	Fg(0)	Falt(run)	Falt		+0.0
+.002082	-.000650			+.008187	+.001019	+.002019			-.002

		RECORDED TEMPERATURES (degrees F)							
		Tc		Tb		Tv		Th	
								Te	
+1104.1		+439.9			+1107.8			+562.2	+60
Ts1	Ts2			Ts3		Tcon		Th1	Th
+248.9		+282.6			+327.2		+489.3	+269.2	+27

		REDUCED DATA							
		A(exit)		A(throat)		Pg			
		COLD	COR.	sq. in.	sq. in.	Psia	Psia		
ss. in.									
.062228		.000623	.000626		88.5	28.8	0.0		+0.004
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	WIRE	Pf DELTA Psia
Volts	Amps	Watts	ohms	temp				dis	
-0.00	-0.004	0.0	0.0000	0.0				0.0000	0.0
SIGHT	MICRO	THRUST	THRUST	VAC					
GLASS	MOTION	MEAS	MEAS	mbs					
lbs/hr	lbs/hr								
0.00000	.51260	25.872	25.899						
CD	Cf	C*	REYNOLDS	GAS				PSF	
		ft/sec	NUMBER	TEMP.					
				deg. f					sec.
.9296	1.4463	4046.3	3550.6	1076.2	0.000				181.9
			B-2						

UNREDUCED DATA (volts)					
	THRUST ZERO	STEP 1	STEP 2	THRUST CALIBRATION	STEP 5
THRUST RUN	+.329	+0.000	+0.000	+0.000	+0.000
Pf(run)	Pf(0)	Pf(run)	Pf(0)	Falt(run)	Falt(0)
+.002080	-.000650	.010623	.000962	-.002024	-.002041

RECORDED TEMPERATURES (degrees F)					
	T <sub>c</sub>	T <sub>16</sub>	T <sub>b</sub>	T <sub>v</sub>	T <sub>b</sub>
+1119.4			+1006.0	+1135.8	+156.6
T <sub>s1</sub>			T <sub>s2</sub>	T <sub>s3</sub>	Tcon
+785.0			+650.7	+792.6	+1020.1
					+1270.7
					+2956.0

REDUCED DATA					
A(exit)	A(throat)	A(throat)	Pf	Fg	Falt
ss. in.	COLD sq. in.	COR. sq. in.	Psia	Psia	Psia
.062228	.000623	.000632	88.4	38.8	0.0
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER
volt	amps	watts	ohms	temp	wire dis
13.83	21.340	295.2	.6482	3376.7	.00000

SIGHT GLASS	MICRO MOTION 1bs/hr	THRUST MEAS mlbs	THRUST VAC mlbs	REYNOLDS NUMBER	GAS TEMP. des. f	ISP
0.00000	.46796	34.659	34.682			
.9075	1.4369	5974.0	1978.9	2699.4	8.512	266.8

		UNREDUCED DATA (volts)					
		STEP 1		STEP 2		STEP 3 CALIBRATION	
THRUST RUN	THRUST ZERO	+0.000	+0.000	+0.000	+0.000	STEP 4	STEP 5
+ .557	+ .338					- .444	- .444
Ff(run)	Ff(0)	Fs(run)	Fs(0)	Fs(run)	Fs(0)	Falt(run)	Falt
+ .003418	- .000640	+ .013054	+ .000908	+ .0002051	- .002051	- .00	- .00

		RECORDED TEMPERATURES (degrees F)					
		T <sub>c</sub>	T <sub>16</sub>	T <sub>b</sub>	T <sub>v</sub>	Thb	T
+1179.7	+672.7		+1173.0		+173.0	+1539.3	+16
T <sub>s1</sub>	T <sub>s2</sub>		T <sub>s3</sub>	T <sub>can</sub>	T <sub>can</sub>	Th1	T
+1025.5	+836.1		+1014.9		+1140.4	+1719.5	+34

		REDUCED DATA					
A(exit)	A(throat)	A(cold)	A(throat)	F <sub>f</sub>	F <sub>s</sub>	Falt	Falt
sq. in.	sq. in.	COLD COR.	sq. in.	psia	psia	psia	psia
.06228	.000623	.000635	.000635	124.3	48.7	0.0	.0004
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER
volts	amps	watts	ohms	temp	wire	wire	delta
19.54	25.568	499.6	.7643	3988.4	.00000	0.0	psia
SIGHT GLASS	MICRO MOTION	THRUST MEAS	VAC	REYNOLDS NUMBER	GAS TEMP.	ISF	ISF
lbs/hr	1bs/hr	mlbs	m lbs	des. f	des. f	sec.	sec.
0.00000	.54212	44.032	44.055				
5474							

CD	C <sub>f</sub>	C*	ft/sec	REYNOLDS NUMBER	GAS TEMP.	FSP	FSP
.9091	1.4518	6483.5	2050.2	3274.3	11.341	292.1	289.1

UNREDUCED DATA (volts)					
THRUST RUN	THRUST ZERO	STEP 1	STEP 2	THRUST CALIBRATION	STEP 5
- +.600	+ .343	+0.000	+0.000	+0.000	+0.000
Pf(run)	Ff(0)	Pg(run)	Pg(0)	Palt(run)	Palt(0)
+.004828	-.000640	+.014890	+.000922	-.002019	-.002042

RECORDED TEMPERATURES (degrees F)					
Tc	T16	Tb	Tv	Thb	Ts
+11190.1	+633.5	+1183.1	+160.5	+1496.4	+1627.8
Ts1	Ts2	Ts3	Tcom	Th1	Th2
+965.7	+783.8	+952.8	+1114.0	+2055.9	+0.0

REDUCED DATA					
A(exit)	A(throat)	A(throat)	Pf	Ps	Palt
COLD	COR.	COR.			psia
.06228	.000623	.000635	162.3	56.0	0.0
HEATER	HEATER	HEATER	HEATER	HEATER	Pf
volt	amps	watts	ohms	temp	delta psia
19.42	25.732	499.8	.7549	3938.9	.00000

SIGHT					
GAS	MICRO	THRUST	THRUST	GAS	ISP
1bs/hr	MOTION	MEAS	VAC	TEMP.	sec.
0.00000	.62444	51.567	51.597	3308.3	297.5
.6457					
.9146	1.4786	6472.8	2347.5	9.687	287.7

## RHENIUM HEAT EXCHANGER - BASELINE HEATER

SN# 001 RU

UNREDUCED DATA (volts)						
THRUST RUN	THRUST ZERO	STEP 1	STEP 2	STEP 3	STEP 4	STEP 5
-	-	-	-	-	-	-
+ .543	+ .321	+ 0.000	+ 0.000	+ 0.000	+ 0.000	+ 0.000
Pf(run)	Pf(0)	Pg(run)	Pg(0)	Pg(run)	Pg(0)	Pg(run)
-	-	-	-	-	-	-
+ .003384	- .000633	+ .013139	-	+ .000980	-	- .002140
-	-	-	-	-	-	-

RECORDED TEMPERATURES (degrees F)						
Tc	T16	Tb	Tv	Thb	Te	Tt
-	-	-	-	-	-	-
+1180.7	+665.6	+1175.4	+1327.2	+1539.0	+164	
Ts1	Ts2	Ts3	Tcan	Th1	Tt	
-	-	-	-	-	-	-
+1026.7	+834.6	+1017.9	+1137.4	+1962.9	+394	
-	-	-	-	-	-	-

REDUCED DATA						
A(exit)	A(throat)	A(throat)	Ff	Fg	Fgt	Fgt
sq. in.	COLD sq. in.	COR. sq. in.	psia	psia	psia	psia
.06228	.000623	.000633	123.2	48.8	0.0	.0013
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	F <sup>f</sup> DELTA psia
volt	amps	watts	ohms	temp	wire dis	
19.56	25.536	499.4	.7659	3997.0	.00000	0.0
SIGHT GLASS	MICRO MOTION	THRUST MEAS	THRUST VAC			
1bs/hr	1bs/hr	mils	mils			
0.00000	.56569	44.513	44.596			
.5541						
CD	Cf	C*	REYNOLDS NUMBER	GAS TEMP. deg. f	PSP	ISF
-	-	ft/sec	-	-	-	sec.
+ 9160	1.4680	6007.6	2427.8	2794.2	11.199	224.1 281.7

## UNREDUCED DATA (volts)

THRUST RUN	STEP 1	STEP 2	STEP 3	STEP 4	STEP 5	STEP 6
+ .563	+0.000	+0.000	+0.000	+0.000	+0.000	+0.000
+0.000642	+0.013467	+0.000925	+0.000925	+0.000925	+0.002116	+0.002132
+0.003413						

## RECORDED TEMPERATURES (degrees F)

Tc	T16	Tb	Tv	Thb	Ts
+1166.8	+774.5	+1165.2	+1165.0	+1165.0	+11623.7
Ts1	Ts2	Ts3	Tcen	Th1	Th2
+1171.9	+958.8	+1173.2	+1209.1	+0.0	+2450.0

## REDUCED DATA

Area (exit)	A (throat)	A (throat)	Pf	Pg	Pelt
sq. in.	COLD	COR.			
sq. in.	sq. in.	sq. in.	psi	psi	psi
.06228	.060623	.000635	124.2	50.3	0.0
HEATER	HEATER	HEATER	HEATER	WIRE	Pelt
Volt	amps	watts	ohms	temp	Delta psi
23.50	27.676	650.4	.3490	4434.9	.0000

549  
•

SIGHT  
GLASS  
lbs/hr

THRUST  
MEAS  
lbs

VAC  
mlbs

45.731

CD

CF

ft/sec

1.4601

6400.5

2173.7

3198.3

14.215

270.5

304.0

Pelt (run)

Pelt (O)

Pelt (run)

Pelt (O)

1.4601

6400.5

2173.7

3198.3

14.215

270.5

304.0

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## RHENIUM HEAT EXCHANGER - BASELINE HEATER

SN# 001 RUN

THRUST RUN	THRUST ZERO	UNREDUCED DATA (volts)			THRUST ACCELERATION		
		STEP 1	STEP 2	STEP 3	STEP 4	STEP 5	STEP 6
+ .521	+ .336	+0.000	+0.000	+0.000	+0.000	+0.000	+0.000
Pf (run)	Pf (0)	Pg (run)	Pg (0)	Falt (run)	Falt (run)	Falt (run)	Falt (run)

+ .002102      -.000642      +.011362      +.000933      -.002132      -.0021

RECORDED TEMPERATURES (degrees F)							
Tc	T16	Tb	Tv	Thb	Thv	Ts	Ts2
+1192.1	+824.2	+1171.9	+203.1	+1767.7	+1615		
Ts1	Ts2	Tcs	Tcon	Th1	Th2		
+1224.3	+1015.0	+1232.8	+1248.9	+0.0	+0.0	+2806	

A (exit)	A (throat)	REDUCED DATA			Falt
		COR.	Ff	Pg	
sq. in.	sq. in.	psi a	psi a	psi a	psi a
.06228	.000623	.000633	33.6	41.8	0.0
HEATER	HEATER	HEATER	HEATER	HEATER	PF DELT A psi a
volt s	amp s	watt s	ohms	temp	
23.67	27.460	650.5	.8627	4507.4	.00000
SIGHT GLASS	MICRO MOTION	THRUST MEAS	THRUST VAC		O. O
1bs/hr	1bs/hr	m1bs	m1bs		
0.00000	.49462	37.081	37.104		
<b>.4456</b>					

CD	Cf	C*		REYNOLDS NUMBER	GAS TEMP. deg. f	PSP	ISP
		ft / sec	deg. f				
.9086	1.4241	6101.1	2023.4	2644.4	17.532	270.1	299.8

## HENIUM HEAT EXCHANGER - BASELINE HEATER

SNH 001 RUN# 10

THRUST RUN	UNREDUCED DATA (volts)			THRUST CALIBRATION		
	STEP 1	STEP 2	STEP 3	STEP 4	STEP 5	STEP 6
+4.589	+0.000	+0.000	+0.000	+0.000	+0.000	+0.000
.004862	Pf (0)	Pg (run)	Pg (0)	Falt (run)	Falt (0)	Falt (0)

+.000642 +.015421 +.000905 -.002119 -.002147

Tc	RECORDED TEMPERATURES (degrees F)			Thb	Tb	Tv
	T16	Ts2	Tss			
1160.7	+750.5	+1199.1		+182.0	+1679.6	+1737.5
Ts1						
1136.1	+929.2	+1127.8		+1205.3	+0.0	+3030.0

(exit)	REDUCED DATA			Pf	Pg	Palt
	A (throat)	A (throat)	COR.			
COLD	sq. in.	sq. in.				
sq. in.						
.06228	.0006223	.000634	1.63.3	58.2	0.0	.0006
HEATER	HEATER	HEATER	HEATER	HEATER	WIRE	Pf
VOLTS	amps	watts	ohms	temp	dia	DELTA
23.30	27.864	649.3	.6363	4367.9	.00000	psi
BRIGHT GLASS	MICRO MOTION	THRUST MEAS	THRUST VAC			
BS/HR	lbs/hr	lbs	lbs			
0.00000	.68211	53.823	53.862			
,648						

CD	REDUCED DATA			GAS TEMP, deg. f	PSF	ISP
	C*	ft/sec	ft/sec			
.9202	1.4852	6158.1	2713.6	2975.2	12.054	264.3
						302.6

		UNREDUCED DATA (volts)				STEP 5	
THRUST RUN	THRUST ZERO	STEP 1	STEP 2	STEP 3	STEP 4	Pt (run)	Palt (run)
+ .483	+ .327	+0.000	+0.000	+0.000	+0.000	+0.000	+0.000
Pf (run)	Pf (0)	Pg (run)	Pg (0)			- .002103	- .00021

		RECORDED TEMPERATURES (degrees F)				Tb		Tc	
		T16	Tb	Tv	Tb	Tc	Tb	Tc	Tb
+1122.9	+239.2		+1116.2		+121.0		+584.8		+616
Test 1	Test 2								
+266.0	+305.0		-4345.8		-4495.6		+163.2		+150

A (exit)	A (throat)	A (throat)	REDUCED DATA	Pf	Pg	Palt	Palt	Palt	Palt
sq. in.	COLD	COR.	sq. in.	psi	psi	psi	psi	psi	psi
.06228	.000623	.000623	.000623	123.0	34.2	0.0	0.0	.0005	
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER
VOLTS	amps	watts	amps	temp	temp	temp	temp	temp	temp
-0.00	0.000	0.0	0.000	0.0	0.000	0.000	0.000	0.000	0.000
SIGHT GLASS	MICRO MOTION 1bs/hr	THRUST MEAS mlbs	THRUST VAC mlbs						
0.0000 ,010	1.01067	31.300	31.330						
CD	Cf	C*	REYNOLDS	GAS NUMBER	GAS TEMP. deg. f	FSP	FSP	FSP	FSP
.9592	1.4723	2433.8	12063.2	147.3	0.000				

THRUST RUN	THRUST ZERO	UNREDUCED DATA (volts)			
		STEP 1	STEP 2	STEP 3	STEP 4
+ .599	+ .329	+0.000	+0.000	+0.000	+0.000
F <sub>f</sub> (run)	F <sub>f</sub> (0)	F <sub>g</sub> (run)	F <sub>g</sub> (0)	F <sub>alt</sub> (run)	F <sub>alt</sub> (0)
+ .000658	+ .004910	+ .014430	+ .000948	+ .002111	+ .002133

T <sub>C</sub>	T <sub>16</sub>	RECORDED TEMPERATURES (degrees F)			
		T <sub>b</sub>	T <sub>v</sub>	T <sub>b2</sub>	T <sub>s</sub>
+1171.2		+738.5	+1193.1	+175.5	+1673.4
T <sub>s1</sub>	T <sub>s2</sub>	T <sub>s3</sub>	T <sub>con</sub>	T <sub>b1</sub>	T <sub>b2</sub>
-1125.6	+918.6	+1113.2	+1197.0	+0.0	+2493.3

A (exit)	P (throat)	REDUCED DATA			
		A (throat) COLD	P <sub>f</sub>	P <sub>g</sub>	P <sub>alt</sub>
sq. in.	sq. in.	sq. in.	psia	psia	psia
.06226	.000623	.000625	165.0	54.1	0.0
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER
VOLTS	AMPS	WATTS	OHMS	TEMP	WIRE dia
23.37	27.310	649.9	.6403	4389.4	.0000
MICRO MOTION	THRUST MEAS	THRUST VAC			O <sub>2</sub> O
lb/s/hr	mlbs	mlbs			
0.00000 ,43	1.05466	54.210	54.241		
CD	C <sub>f</sub>	REYNOLDS NUMBER	GAS TEMP. deg. f	ISP	SEE
	ft/sec				
.7511	1.6104	3699.1	7953.6	11.582	135.1 303.6

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THRUST RUN	THRUST ZERO	UNREDUCED DATA (volts)			THRUST CALIBRATION	STEP 5	STEP 6
		STEP 1	STEP 2	STEP 3		STEP 4	
+ .554	+ .324	+0.000	+0.000	+0.000	+0.000	+0.000	+0.000
Pf (run)	Pf (0)	Pg (run)	Pg (0)	Pg (run)	Pg (0)	Palt (run)	Palt (0)

+ .00388 - .000662 + .012927 + .000887 - .002066 - .00210

RECORDED TEMPERATURES (degrees F)							
Tc	T16	Tb	Tv	Thb	Ts	Ts2	Ts3
+1184.8	+843.6	+1209.1	+201.5	+1617.6	+1357.		
Ts1	Ts2	Ts3	TCON	Thb	Tb1	Tb2	
+1273.1	+1051.5	+1265.8	+1269.5	+40.0	-42537.		

A(exit)	A(throat)	A(throat)	REDUCED DATA				Palt
			COR.	F4	Fg	Fg	
sq. in.	sq. in.	sq. in.	psi a	psi a	psi a	psi a	psi a
.06228	.000623	.000626	124.1	48.3	0.0	0.0	.0004
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	Pf DETA psi a
volt s	amps	watts	ohms	temp	wire dia	wire dia	
25.86	29.026	750.5	.8908	4655.3	.0000	0.0	

CD	C4	C*	REYNOLDS NUMBER	GAS TEMP. deg. f	ISP		SEC.
					ft / sec	ft / sec	
.9434	1.5344	4060.4	5738.7	1147.6	16.260	154.6	312.1

## RHENIUM HEAT EXCHANGER - BASELINE HEATER

SN# 001 FLIN# 14

		UNREDUCED DATA (volts)			
		STEP 1	STEP 2	STEP 3	STEP 4
THRUST RUN	ZERO	+0.000	+0.000	+0.000	+0.000
	+ .321	+0.000	+0.000	+0.000	+0.000
Pf (run)	Pf (O)	Pg (run)	Pg (O)	Palt (run)	Palt (O)
+ .002129	- .000650	+ .011471	+ .000937	- .002158	- .002174

		RECORDED TEMPERATURES (degrees F)			
		Tb	Tv	Tb	Ts
+1206.1	+876.3	+1205.7	+205.3	+1852.7	+1697.2
Ts1	Ts2	Ts3	Tc0n	Tm	Tf
+1313.3	+1092.1	+1265.1	+1284.8	+645.3	+95.8

		REDUCED DATA			
A(exit)	A(throat)	A(throat)	Pf	Pg	Palt
COLD	COR.	COR.			
sq. in.	sq. in.	sq. in.	psia	psia	psia
.06228	.000623	.000626	69.7	42.3	0.0
HEATER	HEATER	HEATER	HEATER	HEATER	Pf DELTA psia
volt	amps	watts	ohms	temp	
26.15	28.688	750.3	.9116	4765.3	.0000
SIGHT	MICRO	THRUST	THRUST		
GASS	MOTION	MEAS	VAC		
lbs/hr	1bs/hr	mlbs	mlbs		
0.0000	.77662	38.043	38.065		
.4438					

CD	Cf	C*	REYNOLDS NUMBER	GAS TEMP. deg. f	PSP	ISP
						sec.
.9423	1.4463	3925.2	5496.3	1026.8	19.710	176.5

## RHENIUM HEAT EXCHANGER - BASELINE HEATER

SHUT OFF RUN

		UNREDUCED DATA (volts)				
THRUST RUN	THRUST ZERO	STEP 1	STEP 2	STEP 3	STEP 4	STEP 5
+ .604	+ .332	+0.000	+0.000	+0.000	+0.000	+0.000
P <sub>f</sub> (run)	P <sub>f</sub> (0)	P <sub>g</sub> (run)	P <sub>g</sub> (0)	P <sub>g</sub> (run)	P <sub>g</sub> (run)	P <sub>g</sub> (run)
+ .004837	- .000650	+ .014525	+ .0000893	+ .002133	- .0021	- .0021

RECORDED TEMPERATURES (degrees F)						
T <sub>c</sub>	T <sub>16</sub>	T <sub>b</sub>	T <sub>v</sub>	T <sub>b</sub>	T <sub>m</sub>	T <sub>f</sub>
+1185.5	+798.7	+1206.2	+188.8	+1758.8	+1616	
T <sub>s1</sub>	T <sub>s2</sub>	T <sub>s3</sub>	T <sub>con</sub>	T <sub>m</sub>		
+1227.1	+1005.7	+1144.0	+1243.9	+591.9	+95	

REDUCED DATA						
A(exit)	A(throat)	A(throat)	F <sub>f</sub>	F <sub>f</sub>	P <sub>g</sub>	P <sub>g</sub>
sq. in.	COLD	COR.				
.06228	.000623	.000625	psi. in.	psi. in.	psi. in.	psi. in.
			162.9	162.9	54.7	54.7
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER
VOLTS	amps	watts	amps	amps	temp	temp
25.81	29.086	750.8	.8875	.8875	4633.0	.00000
SIGHT GLASS	MICRO MOTION	THRUST MEAS	•THRUST VAC			
1bs/hr	1bs/hr	mlbs	mlbs			
0.0000	1.07209	54.532	54.561			
4293						

CD	C*	REYNOLDS NUMBER	GAS TEMP. deg. f	PSF	ISP
.9516	1.6026	3678.1	8137.8	874.9	13.761
					163.2

## RHENIUM HEAT EXCHANGER - BASELINE HEATER

SN# 001 RUN# 16

UNREDUCED DATA (volts)					
THRUST RUN	ZERO	STEP 1	STEP 2	STEP 3	STEP 4
+ .549	+ .320	+ 0.000	+ 0.000	+ 0.000	+ 0.000
+ .003395	- .000650	Pf (O)	Pg (run)	Pg (O)	Palt (run)

+ .000650	+ .012910	+ .000863	- .002110	+ 0.000	+ 0.000
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RECORDED TEMPERATURES (degrees F)					
Tc	T16	Tb	Tv	Th	Ts
+1179.7	+840.4	+1208.9	+204.6	+1604.0	+1661.9
Ts1	Ts2	Ts3	Tc0n	Tm	Tf
+1056.5	+1174.9	+1258.7	+1258.7	+610.2	+93.2
+1285.9					

A (exit)	A (throat)	A (throat)	REDUCED DATA	Pg	Palt
COLD	CDR.	CDR.	Pf	Pf	Pf
.06228	.000623	.000623	.000623	40.3	0.0
HEATER	HEATER	HEATER	HEATER	HEATER	PF
volt	watts	watts	temp	wire	delta
25.87	29.010	751.1	.8925	4664.2	.0000
SIGHT	MICRO	THRUST	THRUST		
GASS	MOTION	MEAS	VAC		
lbs/hr	1bs/hr	lbs	• mils		
0.0000	.683388	46.005	46.026		

CD	C*	REYNOLDS NUMBER	GAS TEMP. deg. f	FSP	ISP
	ft / sec			sec.	
.9453	1.5292	3944.1	6191.1	1050.3	16.317
15295					167.5
					313.0

		UNREDUCED DATA (volts)			THRUST CALIBRATION			STEP 5			STEP	
THRUST RUN	ZERO	STEP 1	STEP 2	STEP 3	STEP 4							
+ .609	+ .331	+ 0.000	+ 0.000	+ 0.000	+ 0.000			+ 0.000	+ 0.000	+ 0.000		+ 0.0
Pf(run)	Ff(0)	Pg(run)	Pg(0)									
+ .004886	- .000650	+ .014670		+ .000863				- .002088				- .002

		RECORDED TEMPERATURES (degrees F)			Thermal			T <sub>E</sub>			
T <sub>C</sub>	T <sub>16</sub>	T <sub>b</sub>	T <sub>v</sub>								
+11191.2	+858.0		+1216.6		+202.1		+1826.0				+18E
T <sub>s1</sub>	T <sub>s2</sub>		T <sub>s3</sub>		T <sub>con</sub>		T <sub>m</sub>				T <sub>f</sub>
+1310.7	+1071.6		+1184.2		+1268.8		+626.0				+1

		REDUCED DATA			Ps			Ps			Ps	
A(exit)	A(throat)	A(throat)	Ps	Ps								
sq. in.	COLL. sq. in.	COLL. sq. in.	Psia	Psia								
.06228	.000623	.000626	.000626	.000626	.000626	.000626	.000626	.000626	.000626	.000626	.000626	.000626
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	F <sub>f</sub>	DELTA Psia
volts	amps	watts	ohms	temp	ohms	temp	ohms	temp	ohms	temp	ohms	temp
27.97	30.388	849.8	.9203	4811.0	.9203	4811.0	.9203	4811.0	.9203	4811.0	.00000	0.0

SIGHT	MICRO	THRUST	THRUST								
GLASS	MOTION	MEAS	VAC								
1bs/hr	1bs/hr	mlbs	mlbs								
0.0000	1.01524	55.881	55.915								

CD	Cf	C*	REYNOLDS NUMBER	GAS TEMP. deg. f	PSF	ISF
.9485	1.6210	3935.5	7099.6	1054.0	15.199	198.3

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HEMIUM HEAT EXCHANGER - BASELINE HEATER

SHP 001 RUN 1a

UNREDUCED DATA (volts)					
	THRUST ZERO	STEP 1	STEP 2	STEP 3	STEP 4
run	+ .322	+0.000	+0.000	+0.000	+0.000
Pf (0)	Pg (run)	Pg (0)	Palt (run)	Palt (0)	
.000659	.013074	.000921	.002097	.002113	
000355					

RECORDED TEMPERATURES (degrees F)					
	Tc	Tt6	Tb	Tv	Tg
	+898.3	+1223.2	+206.8	+1884.1	+1938.9
Ts2		Ts3			Tf
Ts1		Tcon		Tm	
	+1135.0	+1245.0	+1298.1	+655.1	+95.2
11370.8					

REDUCED DATA					
(exit)	A(throat) COLD	A(throat) COP.	Pf	Pg	Palt
9. in.	sq. in.	sq. in.	psi	psi	psi
.06228	.000623	.000626	123.1	48.8	0.0
HEATER	HEATER	HEATER	HEATER	HEATER	PF DELTA psi
volt	amps	watts	ohms	temp	
28.13	30.232	350.3	.9303	4863.8	.0000
					0.0
SIGHT GLASS	MICRO MOTION 1bs/hr	THRUST MEAS mlbs	THRUST VAC mlbs		
0.0000	.84353	46.440	46.462		
15267					
CD	C*	REYNOLDS NUMBER	GAS TEMP. deg. f	PSP	ISP
	ft / sec				sec.
.7424	1.5302	4169.2	5519.7	1213.2	18.301
					198.3
					317.4

THRUST RUN	THRUST ZERO	UNREDUCED DATA (volts)				STEP 5	STEP 6	Palt (run)	Palt (0)
		STEP 1	STEP 2	STEP 3	STEP 4				
.502	.281	+0.000	+0.000	+0.000	+0.000	+0.000	+0.000	+0.000	+0.000
P4 (run)	P4 (0)	Pg (run)	Pg (0)	Pg (run)	Pg (0)	Pg (run)	Pg (0)	Pg (run)	Pg (0)

TE	T <sub>14</sub>	RECORDED TEMPERATURES (degrees F)	T <sub>v</sub>	T <sub>b</sub>	T <sub>15</sub>	TE
+1158.5	+649.2	+1166.7	+165.1	+1557.1	+1645	+1645
T <sub>51</sub>	T <sub>52</sub>	T <sub>53</sub>	T <sub>54</sub>	T <sub>55</sub>	T <sub>56</sub>	+694.7
+1158.5	+649.2	+1166.7	+165.1	+1557.1	+1645	+1645

Axis	Exit COLD sq. in.	At throat COLD sq. in.	Reduced DATA P <sub>g</sub>	P <sub>g</sub> psi at P <sub>g</sub>
sq. in.	sq. in.	psi a		
.06228	.000623	.000625	46.7	.0004
HEATER	HEATER	HEATER	HEATER	HEATER WIRE dia
volt s	amps	watts	temp	temp
19.44	25.718	499.8	3943.4	.00000
SIGHT GLASS	MICRO MOTION 1bs/hr	THRUST MEAS mlbs	THRUST VAC mlbs	44.496 44.522
0.0000	01349			

CD	C*	REYNOLDS NUMBER	GAS TEMP. deg. F	F.S.F.	I.G.P.	SEC.
.9480	1.5310	3687.2	6945.2	871.5	1.1. 227	175.5
						769.7

## HELIUM HEAT EXCHANGER - BASELINE HEATER

SN# 001 ISN# '20

THRUST RUN	THRUST ZERO	UNREDUCED DATA (volts)			STEP 5	STEP 6
		STEP 1	STEP 2	THRUST CALIBRATION		
4.52	+ .268	+0.000	+0.000	+0.000	+0.000	+0.000
.002136	- .000647	Pf (0)	Pg (run)	Pg (0)	Pat (run)	Pat (0), +0.02154

Tc	RECORDED TEMPERATURES (degrees F)			Thb	Tb	Ts
	T16	Tb	Tv			
1153.9	+695.6	+1171.7	+184.5	+1619.1	+1701.9	
Ts1	Ts2	Ts3	Tc01	Tb	Tb	Ts
1081.5	+831.9	+935.2	+1176.1	+524.0	+65.4	

(exit)	A (throat)	A (throat)	REDUCED DATA			Pat.
			Pf	Fg	Pg	
sq. in.	sq. in.	sq. in.	psi a	psi a	psi a	psi a
.06228	.000623	.000626	89.9	41.1	0.0	.0003
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	Pat DELTA psi a
volt s	amps	watts	ohms	temp	volt s	
19.54	25.606	500.3	.7630	3981.9	.00000	0.0
SIGHT GLASS	MICRO MOTION lbs/hr	THRUST MEAS mlbs	VAC mlbs	THRUST VAC	REYNOLDS NUMBER	PSF
0.00000 .4572	.76561	36.803	36.620		deg. f	ISP
CD	Cf	C*	ft/sec			set.
.9424	1.4396	3669.4	5517.0	986.1	13.587	173.1

## RHENIUM HEAT EXCHANGER - BASELINE HEATER

SN# Q041 F014

UNREDUCED DATA (volts)					
THRUST RUN	THRUST ZERO	STEP 1	STEP 2	STEP 3	STEP 4
+ .539	+ .280	+0.000	+0.000	+0.000	+0.000
Pf (run)	Pf (O)	Pg (run)	Pg (O)	Palt (run)	Palt (O)

+ .004858 - .000647 + .013961 + .000885 - .002107 - .0021

RECORDED TEMPERATURES (degrees F)					
Tc	T16	Tb	Tv	Thb	Ts
+1167.3					
	+619.0	+1173.8	+165.5	+1502.4	+1586
Ts1	Ts2	Ts3	Tcon	Tm	Tf
+953.3	+771.4	+653.1	+1117.5	+463.4	+84

REDUCED DATA					
A (exit)	A (throat)	A (throat)	Pf	Pf	Pg
sq. in.	COLD	COR. sq. in.	psi	psi	
.06228	.000623	.000625	163.3	52.4	0.0
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER
volt	amps	watts	ohms	temp	wire dia.
19.17	26.084	500.0	.7348	3833.3	.00000
SIGHT GLASS	MICRO MOTION 1bs/hr	THRUST MEAS mlbs	THRUST VAC mlbs		
0.0000	1.06396	51.967	51.996		
.6466					
CD	Cf	C*	REYNOLDS NUMBER	GAS TEMP. deg. f	PSF
.9523	1.5920	3555.6	6416.8	791.2	9.616

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289.4

## HORIZONTAL HEAT EXCHANGER - BASELINE HEATER

SME 001 RUN# 202

		UNREDUCED DATA (volts)				
		STEP 1	STEP 2	STEP 3	STEP 4	
THRUST	ZERO	+0.000	+0.000	+0.000	+0.000	
RUN	+ .499	+ .271	+0.000	+0.000	+0.000	+0.000
+ (run)	Pf (O)	Pg (run)	Pg (O)	Palt (run)	Palt (O)	
	- .003360	+ .000647	+ .012890	+ .000844	- .002100	- .002117

		RECORDED TEMPERATURES (degrees F)				
		T <sub>c</sub>	T <sub>b</sub>	T <sub>v</sub>	T <sub>b</sub>	T <sub>c</sub>
		+841.1	+1207.9	+202.8	+1735.9	+1659.3
T <sub>s1</sub>	T <sub>s2</sub>	T <sub>s3</sub>	T <sub>c0n</sub>	T <sub>b</sub>	T <sub>f</sub>	
		+1051.6	+1135.7	+1249.7	+614.0	+91.3

		REDUCED DATA				
		A (throat)	A (throat)	Pg	Pg	
V (exit)	COLD	COR.	COR.			
sq. in.	sq. in.	sq. in.	sq. in.	psia	psia	
.06228	.000623	.000626	.000626	122.9	46.3	.0004
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	P <sub>b</sub>
Volts	amps	watts	amps	temp	wire	DETE.
24.73	30.326	749.9	3154	4257.8	.00000	dia
						psia
						0.0
SIGHT	MICRO	THRUST	THRUST	VAC	VAC	T <sub>b</sub>
GLASS	MOTION	MEAS	MEAS	mbs	mbs	sec.
lbs/hr	1bs/hr	lbs	lbs			
0.0000	.06439	45.886	45.908			
.5290						
CD	C*	REYNOLDS	GAS	PSP	PSP	
	ft/sec	NUMBER	TEMP. deg. f			
.941	1.5254	4032.8	5692.9	1113.3	16.334	174.2
						312.4-

THRUST RUN	THRUST ZERO	UNREDUCED DATA (volts)			STEP 5 *	STEP 6 *	STEP 7 *
		STEP 1	STEP 2	THRUST CALIBRATION			
+ .474	+ .287	+0.000	+0.000	+0.000	+0.000	+0.000	+0.000
Pf (run)	Pf (0)	Pg (run)	Pg (0)	Palt (run)	Palt (0)	Palt (run)	Palt (0)

+ .002109	- .000656	+ .011513	+ .000930
-----------	-----------	-----------	-----------

## RECORDED TEMPERATURES (degrees F)

Tc	Tt6	Tb	Tv	Thb	Tfe
+1193.8	+880.2	+1212.2	+212.6	+1832.4	+1914.
Ts1	Ts2	Ts3	Tcon	Tm	Tf
+1319.2	+1083.0	+1170.6	+1270.6	+646.7	+101.

A(exit)	A(throat)	A(throat)	REDUCED DATA	Pg	Palt
COLD sq. in.	COLD sq. in.	COR. sq. in.	Ff	Ff	Palt
.06228	.000623	.000627	psi a	psi a	psi a
HEATER	HEATER	HEATER	psi a	psi a	psi a
volt	amps	watts	temp	temp	temp
24.62	30.474	750.2	.8079	.4218.2	.00000
SIGHT GLASS	THRUST MEAS	THRUST VAC	HEATER WIRE dia	HEATER WIRE dia	HEATER WIRE dia
1bs/hr	lbs/hr	• milbs	• milbs	• milbs	• milbs
0.0000	.69227	37.440	37.458	37.458	37.458
4350					

CD	Cf	C*	REYNOLDS NUMBER	GAS TEMP. deg. f	PSF	ISP
.9350	1.4167	4.423.7	4226.2	1392.1	20.028	194.3
						310.0

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## UNREDUCED DATA (volts)

	THRUST ZERO RUN	STEP 1	STEP 2	STEP 3	STEP 4	STEP 5	STEP 6
+ .544	+ .273	+0.000	+0.000	+0.000	+0.000	+0.000	+0.000
+ (run)	Pf (0)	Fg (run)	Fg (0)	Falt (run)	Falt (0)		
- .004630	- .000656	+ .015679	+ .000869	- .002087	- .002109		

## RECORDED TEMPERATURES (degrees F)

	T <sub>c</sub>	T <sub>16</sub>	T <sub>b</sub>	T <sub>v</sub>	T <sub>tb</sub>	T <sub>f</sub>	T <sub>s</sub>
-1182.2		+801.5		+1209.3	+191.3	+1739.4	+1641.7
T <sub>s1</sub>		T <sub>s2</sub>		T <sub>cof</sub>		T <sub>m</sub>	
-1241.3		+1007.5		+1070.9	+1236.1	+590.7	+87.0

## REDUCED DATA

A(exit)	A(throat)	A(throat)	P <sub>cold</sub>	P <sub>cold</sub>	P <sub>f</sub>	P <sub>f</sub>	P <sub>alt</sub>
sq. in.	sq. in.	sq. in.	psi	psi	psi	psi	psi
.06228	.000623	.000627	162.8	59.4	0.0	0.0	0.005
HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	HEATER	Pf
Volts	amps	watts	ohms	temp	wire	diode	Delta
24.66	30.422	750.3	.8107	4233.4	.00000	0.0	per cent

SIGHT GLASS	MICRO MOTION	THRUST	THRUST VAC	THRUST lbs	PSP	ISP
1bs/hr	1bs/hr	meas	vac	lbs		
0.00000	1.00179	54.476	54.505			

CD	C*	REYNOLDS NUMBER	GAS TEMP. deg. f	PSP	ISP
.9457	1.4731	4277.9	6295.3	1314.2	13.766
					195.9

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## RHENIUM HEAT EXCHANGER - BASELINE HEATER

SN# 001 RLIN

THRUST RUN	THRUST ZERO	UNREDUCED DATA (volts)				STEP 5, STEP 6, STEP 7, STEP 8, STEP 9
		STEP 1	STEP 2	STEP 3	STEP 4	
+ .518	+ .268	+0.000	+0.000	+0.000	+0.000	+0.000
+ .003338	- .000652	+ .013580	+ .000959	+ .000959	+ .000959	+ .000959

Tc	T16	RECORDED TEMPERATURES (degrees F)				Thb	Tg
		Tb	Tv	Tg	Ts		
+1183.8	+238.9	+1216.2	+213.4	+213.4	+1216.2	+1820.3	+1939
Ts1	Ts2	Ts3	Tc1	Tc2	Tc3	Tm	Tf
+1276.0	+1039.4	+1260.1	+1277.9	+1277.9	+1260.1	+634.6	+634.6

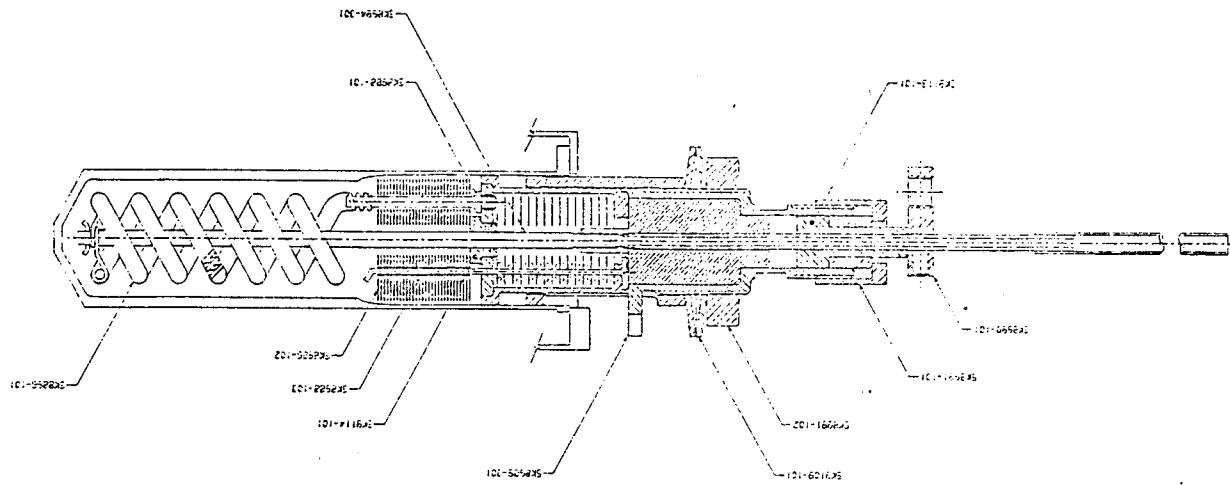
A (exit)	A (throat)	A (throat)	REDUCED DATA				Pd, d
			Pd	Pf	Pf	Pg	
sq. in.	COLD sq. in.	COR. sq. in.	psi	psi	psi	psi	psi
.06226	.000623	.000627	122.4	50.6	50.6	0.0	.0005
HEATER volts	HEATER amps	HEATER watts	HEATER deg	HEATER temp	HEATER WIRE dia	HEATER WIRE dia	HEATER WIRE dia
26.39	28.468	751.4	9271	4047.1	.00000	.00000	.0000

CD	C/F	C*	REYNOLDS NUMBER			GAS TEMP. deg. f.	PSF	TSF
			t/sec	t/sec	t/sec			
.9406	1.4649	4361.6	5166.3	1363.2	16.266	19E.6	19E.6	31Ae8

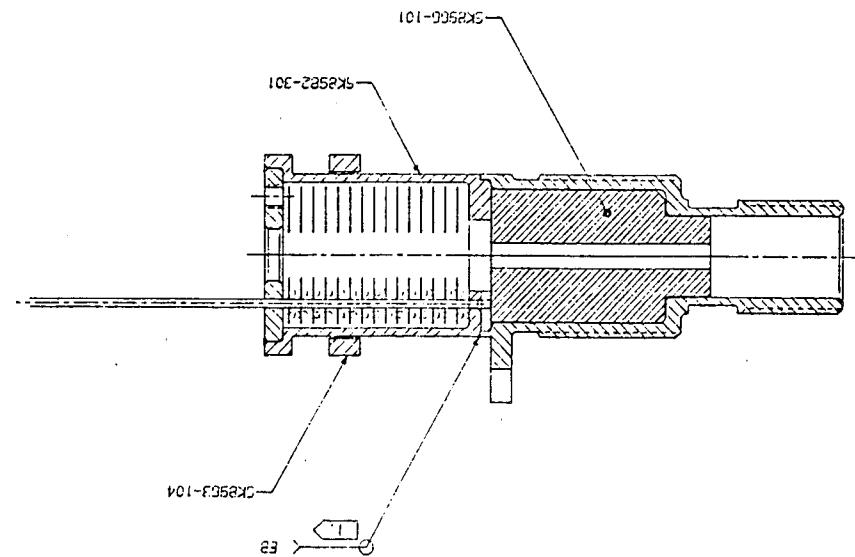
**APPENDIX C**  
**Sealed Cavity Design**

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C-2



C-3



COMPONENT LIST						
REF ID	NAME	QUANTITY	DESCRIPTION	MANUFACTURER / SPECIFICATION / DESIGNATION	DATE ISSUED	REVISION
SK856-101	SUPPORT	1				
SK856-104	RING	1				
SK852-301	LOWER BODY	1				
SK856-101	UPPER BODY	1				
SK856-104	RING	1				
SK856-101	SUPPORT	1				
SK856-101	UPPER BODY	1				
SK852-301	LOWER BODY	1				
SK856-104	RING	1				
SK856-101	SUPPORT	1				

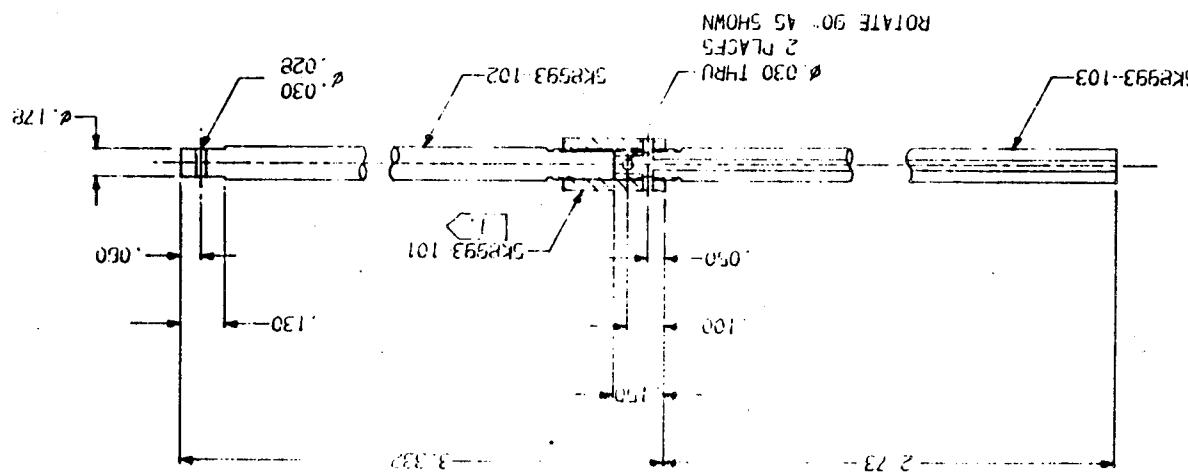
REF ID	DESCRIPTION	DATE ISSUED	REVISION
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MACHINE FLUSH TO DIAMETER AFTER MELTMENT.

PARTS LIST								
REF ID	DESCRIPTION	QUANTITY	UNIT	REMARKS	MANUFACTURER	MANUFACTURE DATE	EXPIRATION DATE	INVENTORY NUMBER
301-11		—						
1	SLEEVE	1			GKPG93-101			
1	SUPPORT	1			GKPG93-102			
1	TUBE	1			GKPG93-103			

B

A



273

C-4

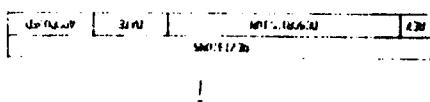
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C

D

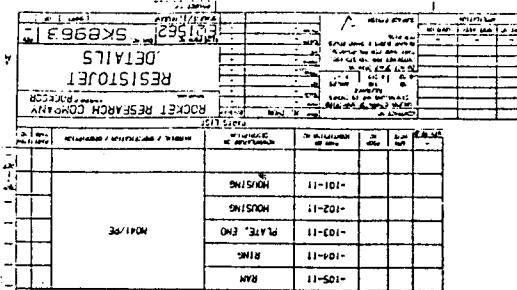
TRAP SKB993-101 TO SKB993-102 &amp; -103 USING CENTER PUNCH.



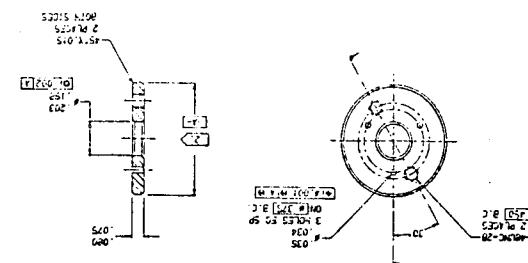
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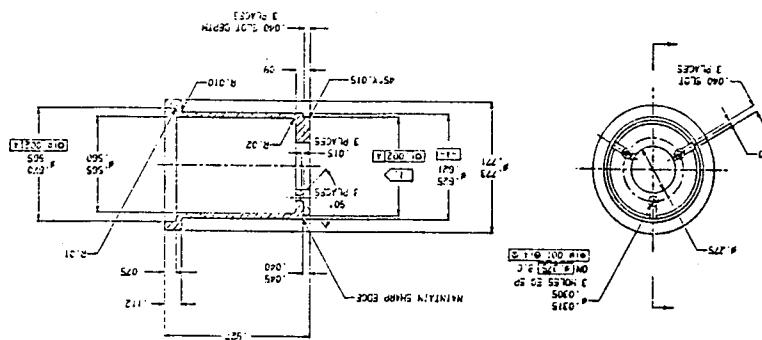
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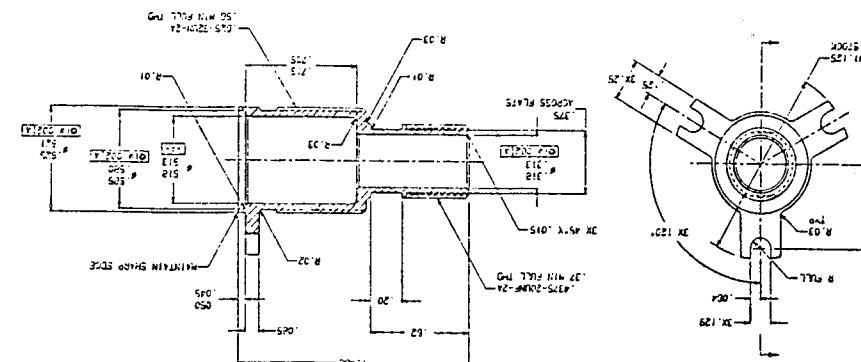
DETAIL - 103-11  
MACH DIA TO .5625/+.005 DIA OF DETAIL -102-11 TO MACHIN - .004/.002 DIA



DETAIL - 102-11  
MACH DIA TO .5625/+.005 DIA OF DETAIL -101-11 TO MACHIN - .004/.002 DIA



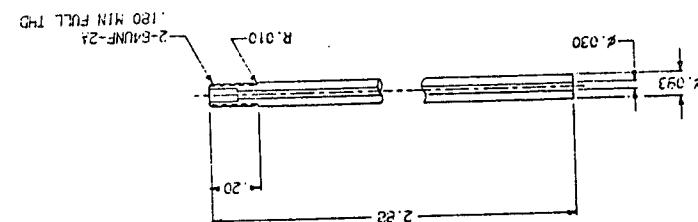
DETAIL - 101-11



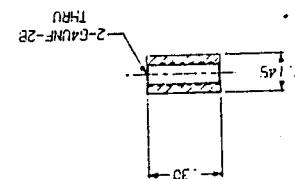
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DETAILS, (INCONEEL)		ROCKET RESEARCH COMPANY		POCCDR		PROJECT NUMBER		DESIGNER		DRAWN BY		APPROVED BY		REVIEWED BY		INITIALS	
-103-11	INLET TUBE	-102-11	SUPPORT	INCONEEL	E25												
-101-11	SLERVE																

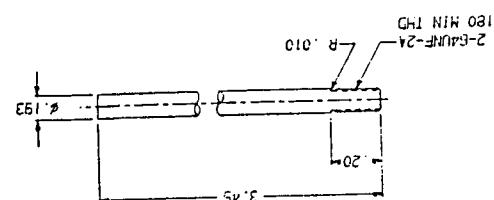
DETAIL - 103-11



DETAIL - 101-11



DETAIL - 102-11



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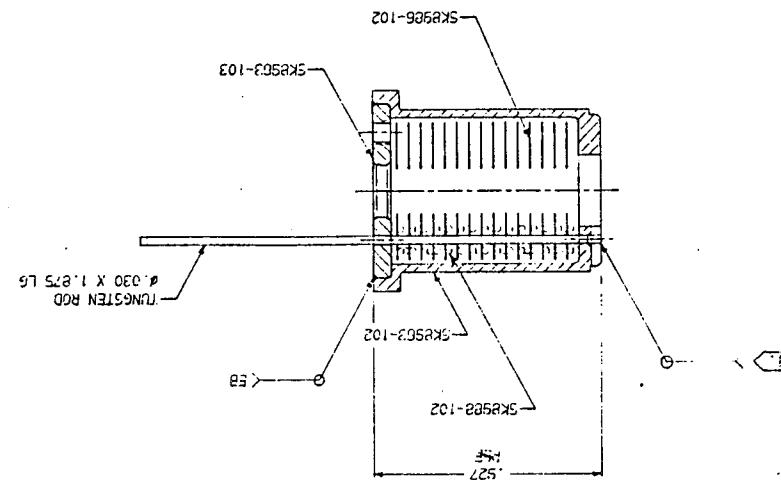
C-10

PARTS LIST									
REF ID	ITEM NO.	NAME	QUANTITY	DESCRIPTION	UNIT	QTY	NAME	QUANTITY	DESCRIPTION
		TUNGSTEN ROD	1	#.030 X 1.875 LG					
3			—						
15	SK8982-102	SPACER	—						
15	SK8966-102	SHIELD, HEAT	—						
1	SK8503-103	PLATE, END	—						
1	SK8965-102	HOUSING	—						
	201-11		—						

ROCKET RESEARCH COMPANY • ROCKCER

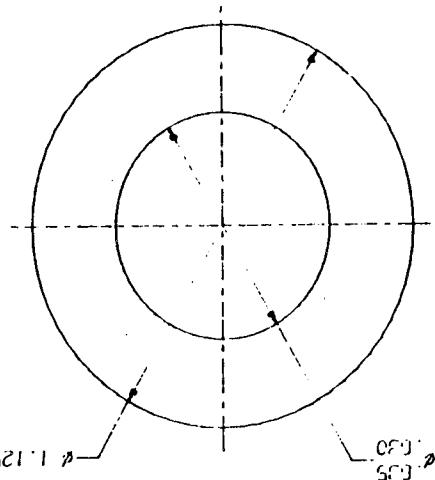
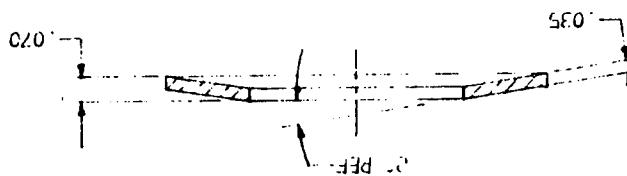
SUB-ASSEMBLY LOWER BODY

021562 SK8982



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ROCKET RESEARCH COMPANY		C-21562		SK9109	
WASHER,		BELLEVILLE			
PARTS LIST					
ITEM NO.	DESCRIPTION	QTY	UNIT	MANUFACTURER	REMARKS
17A	-101-11	—			
PRINTED IN U.S.A. BY THE AMERICAN STAMP CO., NEW YORK CITY					



ITEM NO.	DESCRIPTION	QTY	UNIT	MANUFACTURER
17B		—		

4

3

D

C

C-9

A

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ORIGINAL PAGE 2  
OF POOR QUALITY

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C-8

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A

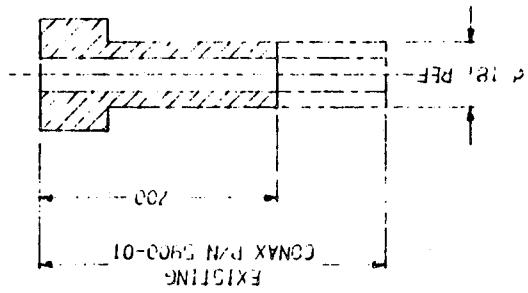
C

D

C

B

A



EXISTING  
CONAX P/N 5400-01

CONAX	ROCKET RESEARCH COMPANY	ROCKCR
MODULED	CONAX	CONAX
PARTS LIST	CONAX P/N 5400-01	-101-11
DESCRIPTION	ROCKET RESEARCH COMPANY	ROCKCR
ITEM NO.	5400-01	-101-11
MANUFACTURER	CONAX	CONAX
DATE OF MANUFACTURE	970101	970101
REVISION	0	0
QTY	1	1
REF		

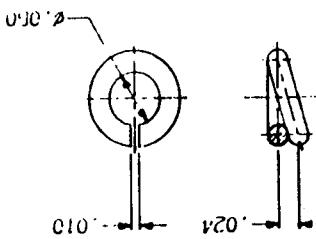
CONAX	ROCKET RESEARCH COMPANY	ROCKCR
MODULED	CONAX	CONAX
PARTS LIST	CONAX P/N 5400-01	-101-11
DESCRIPTION	ROCKET RESEARCH COMPANY	ROCKCR
ITEM NO.	5400-01	-101-11
MANUFACTURER	CONAX	CONAX
DATE OF MANUFACTURE	970101	970101
REVISION	0	0
QTY	1	1
REF		

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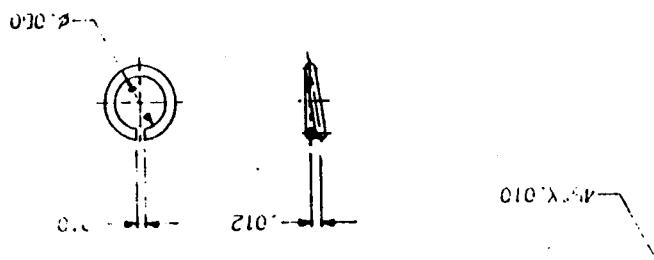
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PARTS LIST		QUANTITY		DESCRIPTION		MATERIAL		SIZE		TOLERANCE		NOTES	
ITEM	REF.	QTY.	UNIT	NAME	NUMBER	GRADE	TYPE	INCH	MM	INCH	MM	INCH	MM
A		1		ROCKET RESEARCH COMPANY	21562	SK8988	DETAILS, TUNGSTEN						
B		1		ROCKET RESEARCH COMPANY	21562	SK8988	DETAILS, TUNGSTEN						
C		1		TUNGSTEN, Ø .024	-102-11			.024	.610	.010	.254	.010	
D		1		TUNGSTEN, Ø .012	-103-11			.012	.305	.010	.254	.010	
E		1		TUNGSTEN, Ø .125	-101-11			.125	3.175				

### DETAIL - 102-11

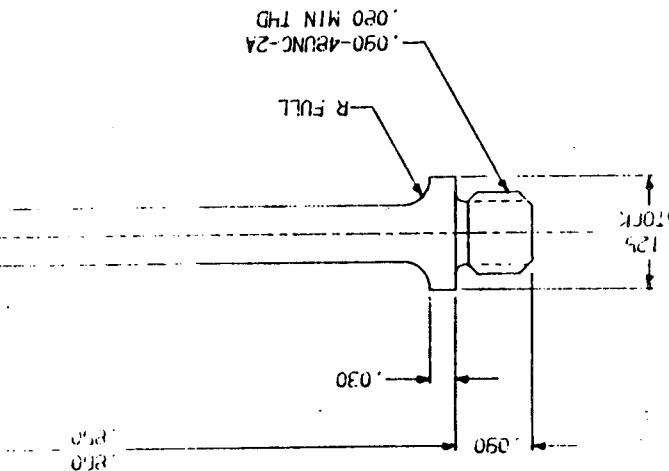


### DETAIL - 103-11



MATCH DIA TO INSIDE DIA (+.0005/.0001) OF SK8986. HEATFLR COIL

### DETAIL - 101-11



C-3

REVISION	DATE	REVISION	DATE
1	1	1	1

SK911A

21582

SPACER

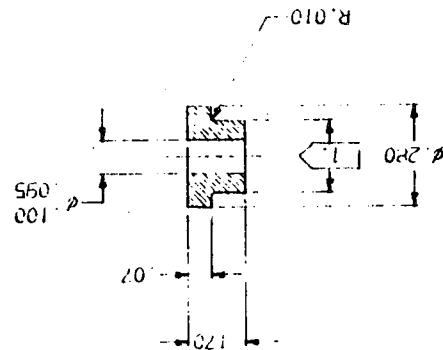
ROCKET RESEARCH COMPANY

A

Part No.	Description	Quantity	Unit	Notes
101-11	—	PN		

B

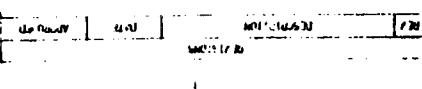
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C

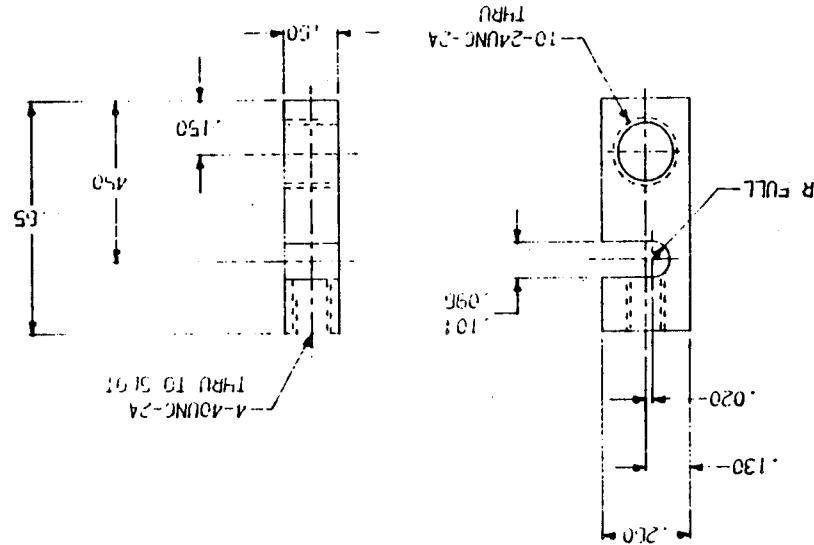
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MATCH DIA TO .203/.196 DIA ON SK95C3 103-11 TO WITHIN +.000/.002 DIA



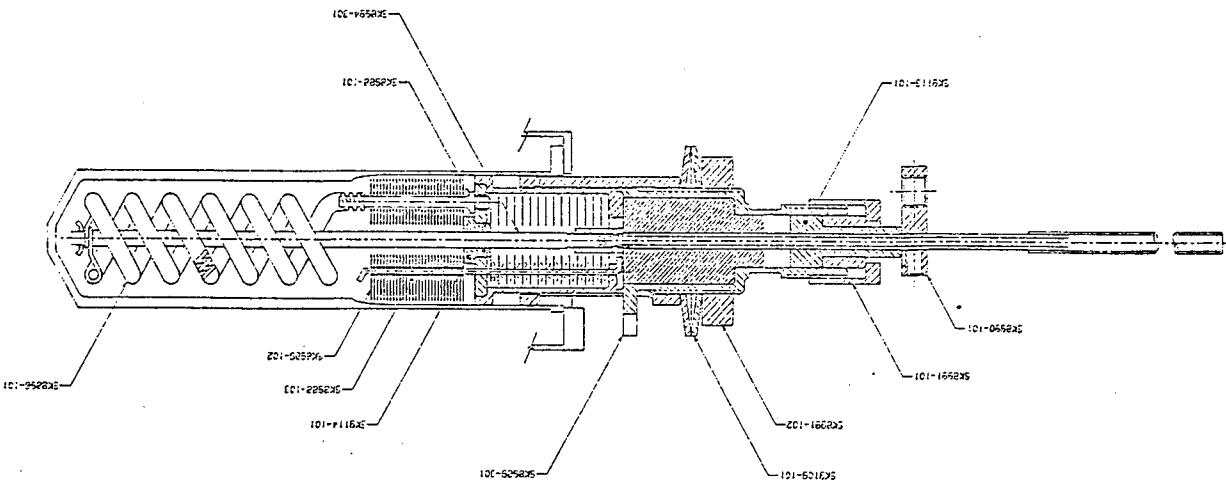
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ROCKET PESFAPCH COMPANY	CONNNECTOR	ELLECTRICAL	SK88950
Part No.	101-11	COPPER	
Designation			
Material No.			
Markings			
Quantity			
Unit			



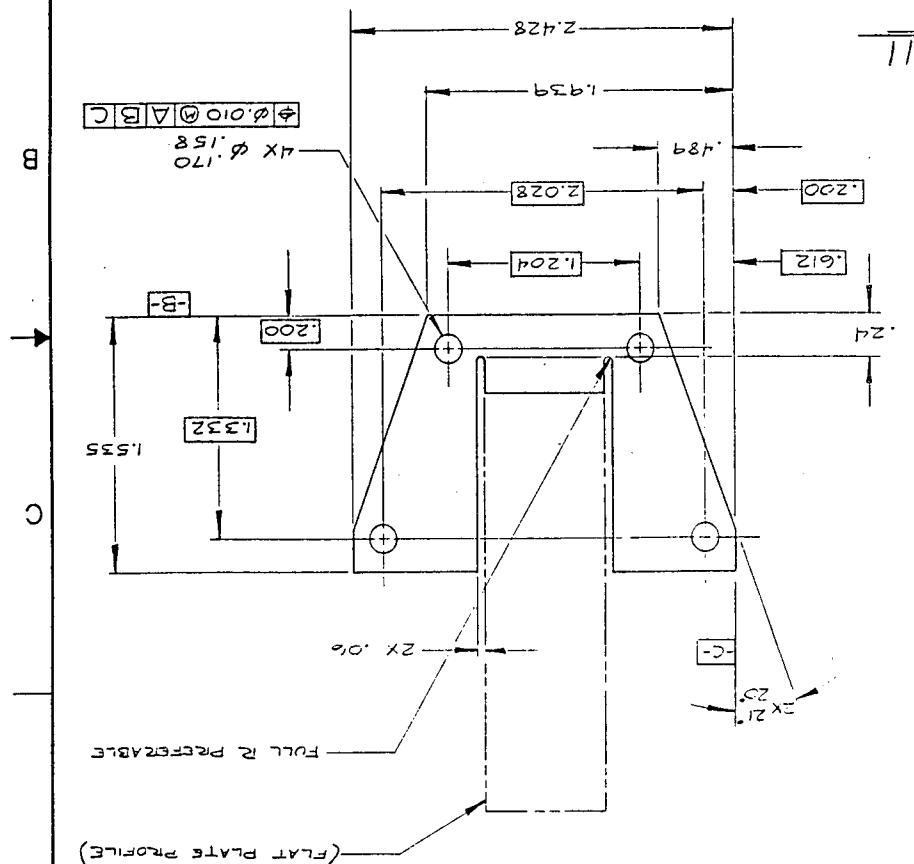
ROCKET PESFAPCH COMPANY	CONNNECTOR	ELLECTRICAL	SK88950
Part No.	101-11	COPPER	
Designation			
Material No.			
Markings			
Quantity			
Unit			

8-



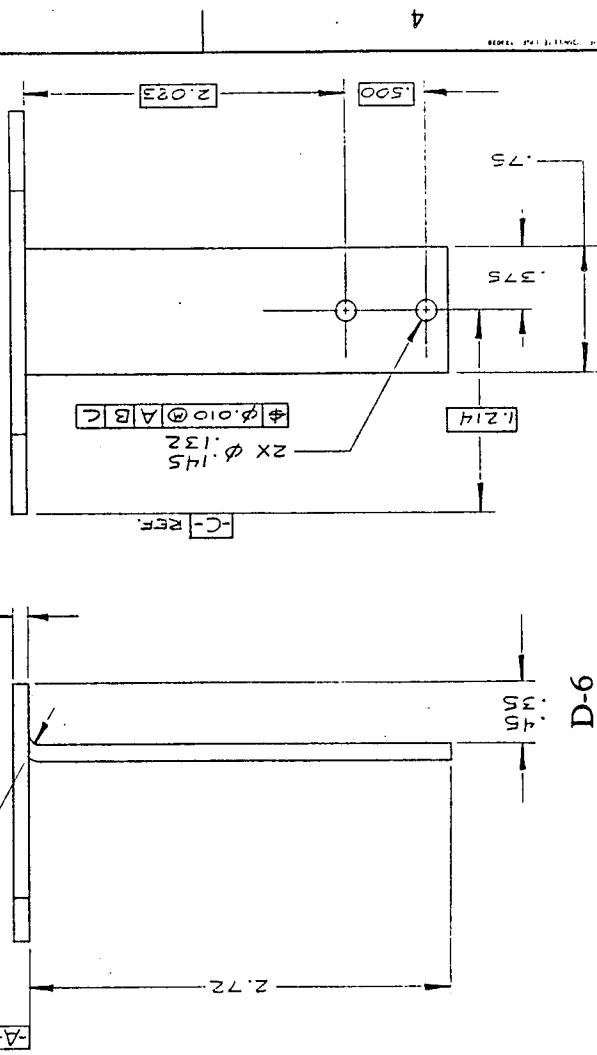
• **Wheat** 40.00 1 801-14465-83 1/26/1

<p>1. INTEGRAL DRAWING PER ANS = V14.5M - 1922.</p> <p>MATERIAL: 100 X 100 MESH, .004 DIA. WIRE, PLAIN SQUARE WAVE HAVES 25,</p> <p>TYPE 1, CLASS 1, PETR AMS 5794.</p> <p>2. </p>	<p>3. DIAMETER MEASURED TO TANGENT POINT OF FILLET RADII.</p> <p>4. REMOVE ALL HANGING STRANDS.</p> <p>5. SOLVENT DEGREASE PER TRC-PS-0205 USING M.E.K.</p> <p>6. VISUALLY INSPECT UNDER 10X MAGNIFICATION.</p> <p>NAME OF THE FOLLOWING CONDUCTIONS ARE ALLOWED:</p> <ul style="list-style-type: none"> <li>A.) TWO PARALLEL WIRES TOUCHING.</li> <li>B.) SEPARATE WIRES.</li> <li>C.) DELAMINATION OR FRAZER EDGES.</li> </ul>															
<p><b>SCREEN</b></p> <p><b>DIAGRAM</b></p>																
<p><b>PARTS LIST</b></p> <table border="1" style="width: 100%; border-collapse: collapse;"> <thead> <tr> <th>ITEM NO.</th> <th>REF ID</th> <th>DESCRIPTION</th> <th>QUANTITY</th> <th>NOTES</th> </tr> </thead> <tbody> <tr> <td>1</td> <td>-101-11</td> <td>—</td> <td>—</td> <td></td> </tr> <tr> <td>2</td> <td>Z.</td> <td>—</td> <td>—</td> <td></td> </tr> </tbody> </table>		ITEM NO.	REF ID	DESCRIPTION	QUANTITY	NOTES	1	-101-11	—	—		2	Z.	—	—	
ITEM NO.	REF ID	DESCRIPTION	QUANTITY	NOTES												
1	-101-11	—	—													
2	Z.	—	—													
<p><b>CONTRACT NO.</b> 5-12-A22-A <b>CHIN ROCKET RESEARCH COMPANY</b></p> <p><b>PRINTED SHEET NUMBER</b> 1 <b>PRINTED DATE</b> 10-10-88 <b>DESIGNER</b> J. S. <b>INSPECTOR</b> J. S. <b>REVIEWER</b> J. S. <b>APPROVING ENGINEER</b> J. S. <b>DATE ISSUED</b> 10-10-88 <b>EXPIRATION DATE</b> 10-10-90 <b>MANUFACTURER</b> J. S. <b>MANUFACTURE DATE</b> 10-10-88 <b>MANUFACTURE LOCATION</b> J. S. <b>MANUFACTURE NO.</b> J. S.</p>																
<p><b>APPENDIX A</b></p> <table border="1" style="width: 100%; border-collapse: collapse;"> <thead> <tr> <th>ITEM NO.</th> <th>REF ID</th> <th>DESCRIPTION</th> <th>QUANTITY</th> <th>NOTES</th> </tr> </thead> <tbody> <tr> <td>1</td> <td>C 21562</td> <td>—</td> <td>—</td> <td></td> </tr> <tr> <td>2</td> <td>C 21523</td> <td>—</td> <td>—</td> <td></td> </tr> </tbody> </table>		ITEM NO.	REF ID	DESCRIPTION	QUANTITY	NOTES	1	C 21562	—	—		2	C 21523	—	—	
ITEM NO.	REF ID	DESCRIPTION	QUANTITY	NOTES												
1	C 21562	—	—													
2	C 21523	—	—													



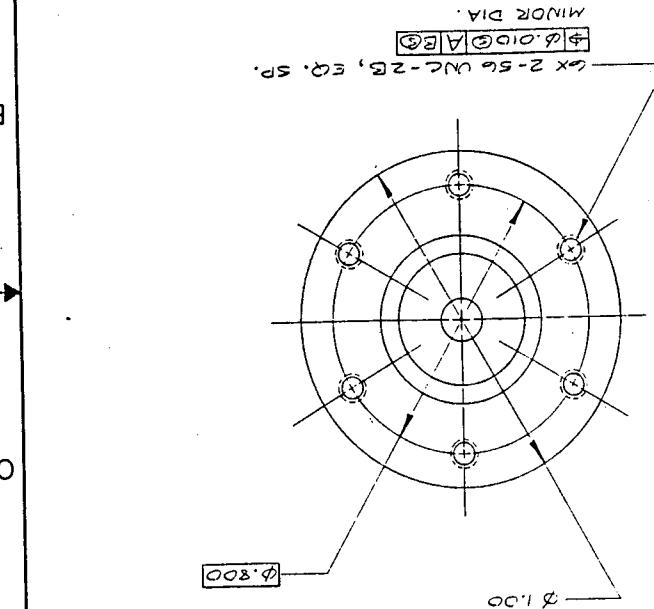
2. WIRE CLEAN USING M.E.K. (FLAT PLATE PROFILE)

1. MEDIUM-DISTANCE DRIVING PER ANSI Y14.5M - 1982.



2. WIPE CLEAN USING M.E.K.

PARTS LIST					
ITEM NO.	DESCRIPTION	QTY	SIZE	NOTES	REF.
-101-11	HASTELLOY 'B' PIPE RING-M-0135	—			



EX 2-56 UUC-2G, EQ. SP.  
φ.6.10(6)A(B)

MINOR DIA.

A

B

C

NAME	DESCRIPTION	QTY	NOTES
C	OUTLET / ADAPTER	1	

1. INTERPRET DIMENSIONS & TOLERANCES PER ANSI Y14.5M-1982.

3

2. SOLVENT CLEAN PIPE R2C-PS-0205.

4

3. ORIENT APPROXIMATELY AS SHOWN.



D-5

APPLICABILITY	
NAME NO.	REVISION
C 21562	SK10226
SCALE	4/1 RELEASE
DATE	SHEET
125	
PRINTED ON 11/12/1983 BY 2000 10:10 AM	
DRAFTS DRAWINGS ARE THE PROPERTY OF CHIN ROCKET RESEARCH COMPANY	

1

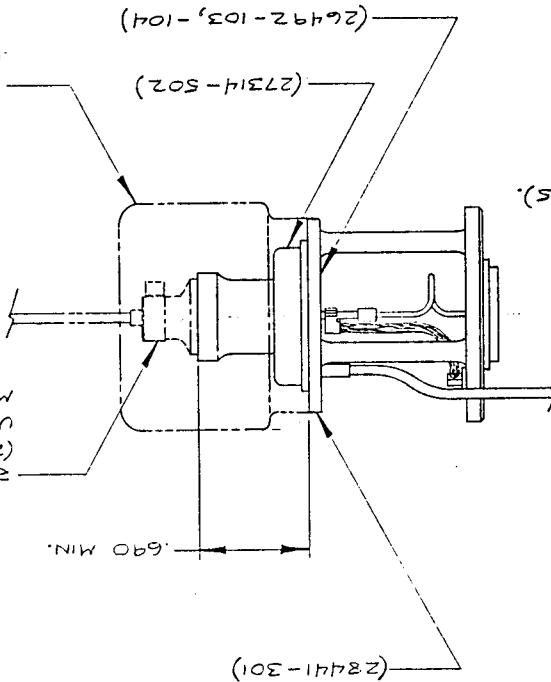
2

3

4

OPTIONAL PAGE 15  
OF POOR QUALITY

-101-11  
MAKE FROM ASSSEMBLY DETAILED  
BY REMOVING GAS GENERATOR  
FROM EHT ASSMBLY (Z8420-305).



PARTS LIST									
ITEM NO.	REF ID	DESCRIPTION	NOTES	REMARKS	DATE	APPROVED	NAME	DEPARTMENT	SECTION
1	C 21562	SHEET 2 OF 2	SCHEMATIC	REV B	12/27/62	DMG NO	SK10231	NOV 1962	DO NOT USE
2	C 21562	SHEET 2 OF 2	SCHEMATIC	REV B	12/27/62	DMG NO	SK10231	NOV 1962	DO NOT USE
3									
4									

A

B

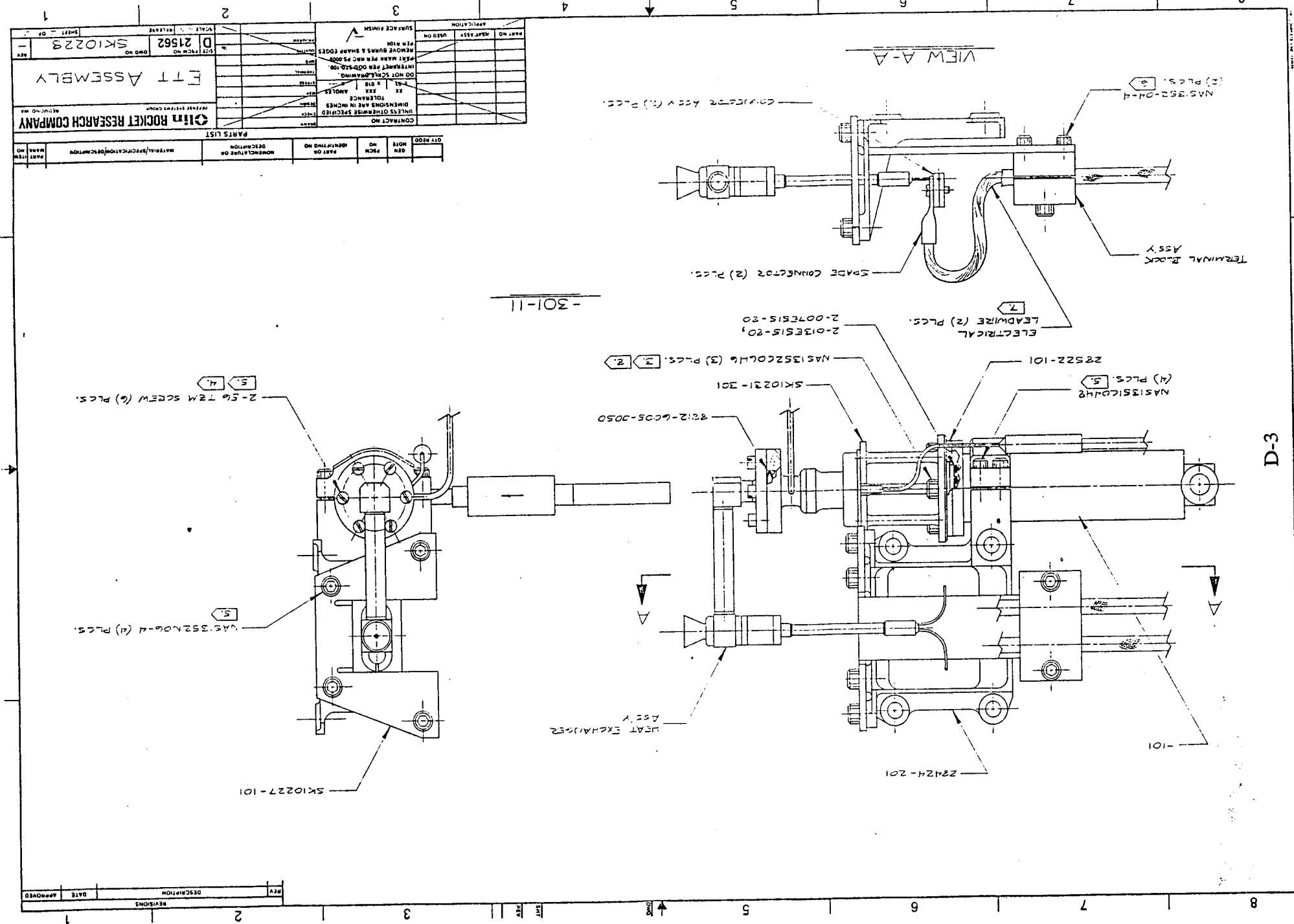
C

1

3

3

4



ORIGINAL PAGE IS  
OF POOR QUALITY

D-2

REF ID	DESCRIPTION	DATE APPROVED
1	INTERFACET SWING-BEAM & TOLERANCE D22 ASME Y14.5M-1972.	
2	VALVE/FLUID RESECTOR FROM EHT ASSEMBLY (24120 - 305).	
3	THIS PART IS THE ASSEMBLY CONTAINED BY REMOVING PROPELLANT TOLERANCE TO D22. W-L2S.	
4	APPLY BONK JITTER SPRAY TO TOLERANCES P20R	
5	TOGGLE TO D22. W-L2S.	
6	AN EQUIVALENT SCREW MAY BE SUBSTITUTED.	
7	STIR WIRE AND SEND APPROXIMATELY AS SHOWN ON ASSEMBLY.	
8	TRIM SCREW AS REQUIRED TO LIMIT PROTRUSION.	
A	AR	2-013 E1515-23 LEADWIRE ELECTRICAL AWG 10, HIGH TEMP.
B	2	2-007 E1515-23 SPADE CONECTOR AS5
C	2	2-010 E1515-23 COPURETOR AS5
D	1	17506 E1515-23 O-RING PARKER
E	2	2-013 E1515-23 NO.4-H NAS1352 SOCKET HEAD N04-H NAS1352 CAP SCREW NAS1351 COHES 6
F	6	— Z-S6 TEN SCREW
G	1	62259 2812-6005 V-SEAL PARKER
H	1	62259 2812-6005 V-SEAL PARKER
I	1	SK10227 TERMINAL BLOCK ASSY HEAT EXCHANG
J	1	— 101 28522 VALVE SPACER
K	1	— 101 2812A MOUNTING STRUCTURE
L	1	— 301 2201 MOUNTING
M	1	— 301-11 VALVE/
N	1	— 101-11 FLUID RESECTOR
O	1	— 301-11 VALVE/
P	1	— 101-11 FLUID RESECTOR
Q	1	— 301-11 VALVE/
R	1	— 101-11 FLUID RESECTOR
S	1	— 301-11 VALVE/
T	1	— 101-11 FLUID RESECTOR
U	1	— 301-11 VALVE/
V	1	— 101-11 FLUID RESECTOR
W	1	— 301-11 VALVE/
X	1	— 101-11 FLUID RESECTOR
Y	1	— 301-11 VALVE/
Z	1	— 101-11 FLUID RESECTOR

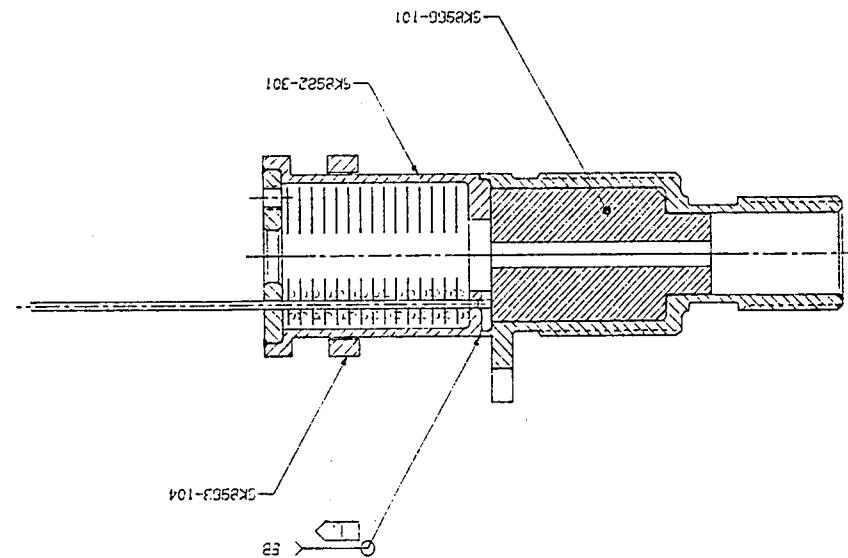
REF ID	DESCRIPTION	DATE APPROVED
1	INTERFACET SWING-BEAM & TOLERANCE D22 ASME Y14.5M-1972.	
2	THIS PART IS THE ASSEMBLY CONTAINED BY REMOVING PROPELLANT TOLERANCE TO D22. W-L2S.	
3	VALVE/FLUID RESECTOR FROM EHT ASSEMBLY (24120 - 305).	
4	TOGGLE TO D22. W-L2S.	
5	APPLY BONK JITTER SPRAY TO TOLERANCES P20R	
6	TOGGLE TO D22. W-L2S.	
7	AN EQUIVALENT SCREW MAY BE SUBSTITUTED.	
8	STIR WIRE AND SEND APPROXIMATELY AS SHOWN ON ASSEMBLY.	
A	AR	2-013 E1515-23 LEADWIRE ELECTRICAL AWG 10, HIGH TEMP.
B	2	2-007 E1515-23 SPADE CONECTOR AS5
C	2	2-010 E1515-23 COPURETOR AS5
D	1	17506 E1515-23 O-RING PARKER
E	2	2-013 E1515-23 NO.4-H NAS1352 SOCKET HEAD N04-H NAS1352 CAP SCREW NAS1351 COHES 6
F	6	— Z-S6 TEN SCREW
G	1	62259 2812-6005 V-SEAL PARKER
H	1	62259 2812-6005 V-SEAL PARKER
I	1	— 101 28522 VALVE SPACER
J	1	— 101 2812A MOUNTING STRUCTURE
K	1	— 301 2201 MOUNTING
L	1	— 301-11 VALVE/
M	1	— 101-11 FLUID RESECTOR
N	1	— 301-11 VALVE/
O	1	— 101-11 FLUID RESECTOR
P	1	— 301-11 VALVE/
Q	1	— 101-11 FLUID RESECTOR
R	1	— 301-11 VALVE/
S	1	— 101-11 FLUID RESECTOR
T	1	— 301-11 VALVE/
U	1	— 101-11 FLUID RESECTOR
V	1	— 301-11 VALVE/
W	1	— 101-11 FLUID RESECTOR
X	1	— 301-11 VALVE/
Y	1	— 101-11 FLUID RESECTOR
Z	1	— 301-11 VALVE/

**APPENDIX D**  
**Immersed Heater Design**

MACHINING FLUSH TO DIAMETER AFTER MOLDMENT.



PARTS LIST									
ITEM NO.	REF. NO.	DESCRIPTION	QUANTITY	UNIT	DESIGNATION NO.	SECTION	SECTION	SECTION	SECTION
		ROCKET RESEARCH COMPANY							
		SUB-ASSEMBLY							
		UPPER BODY							
		SUB-ASSEMBLY							
		ROCKET RESEARCH COMPANY							
		LOWER BODY	1		SK8502-301				
		RINGS	1		SK8503-104				
		SUPPORT	1		SK8506-101				
					301-11				

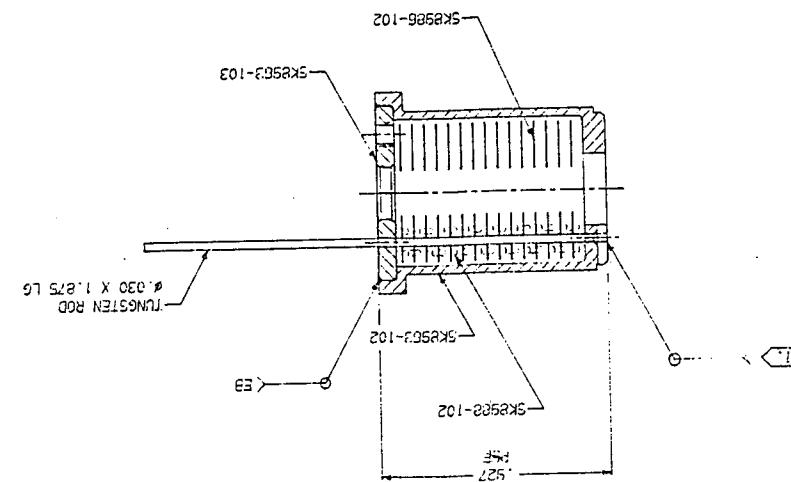


REF.	DESCRIPTION	DATE	REVISION

1	2	3	4	5	6	7	8

ITEM NO.	PARTS LIST	ROCKET RESEARCH COMPANY			
		ITEM NO.	NAME	DESCRIPTION	QUANTITY
3	—	—	SPACER	SK9982-102	15
15	—	—	SHIELD, HEAT	SK9986-102	15
1	PLATE, END	SK9983-103	—	SK9953-102	1
15	—	—	HEATSHIELD	SK9986-102	15
15	—	—	HOUSING	SK9953-102	15
301-11	—	—	—	—	1

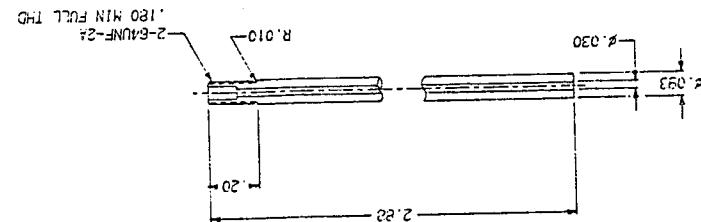
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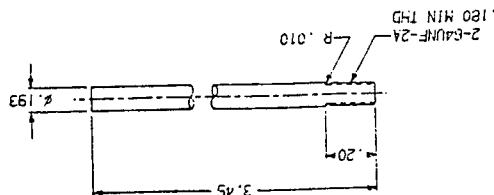
ROCKET RESEARCH COMPANY											
DETAILS, CINCONELE RESISTOJET											
COMPONENT NO. 103-11 INLET TUBE											
DATA SHEET NO.	NAME	DESIGNATION	DATE	REVISION							
103-11	SLAVEVE	INLET TUBE	10-25-62	1	1	1	1	1	1	1	1
102-11	SUPPORT	INCONELE E25	10-25-62	1	1	1	1	1	1	1	1
101-11	SLAVEVE	INCONELE E25	10-25-62	1	1	1	1	1	1	1	1

D-11

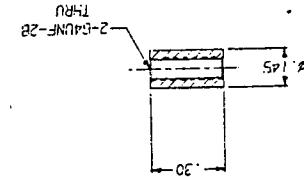
DETAIL - 103-11



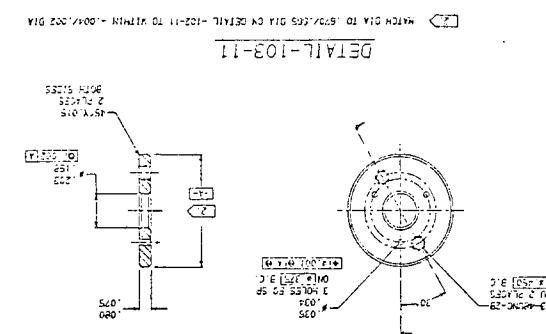
DETAIL - 102-11



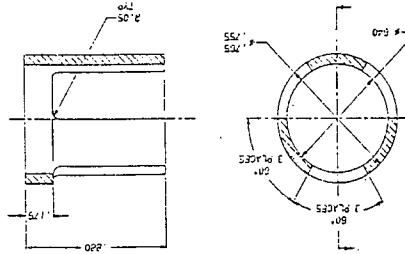
DETAIL - 101-11



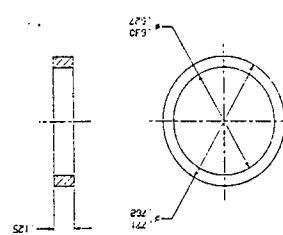
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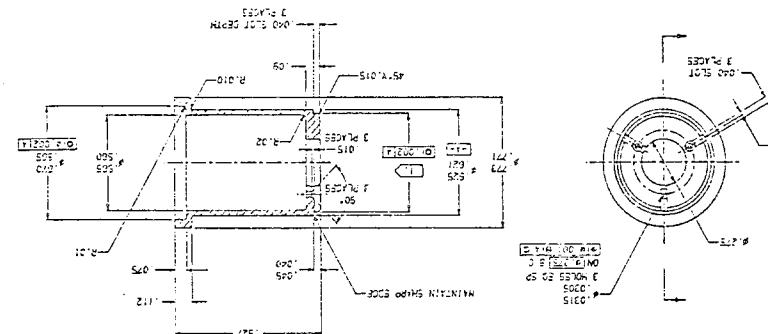
DETAIL - 105-11



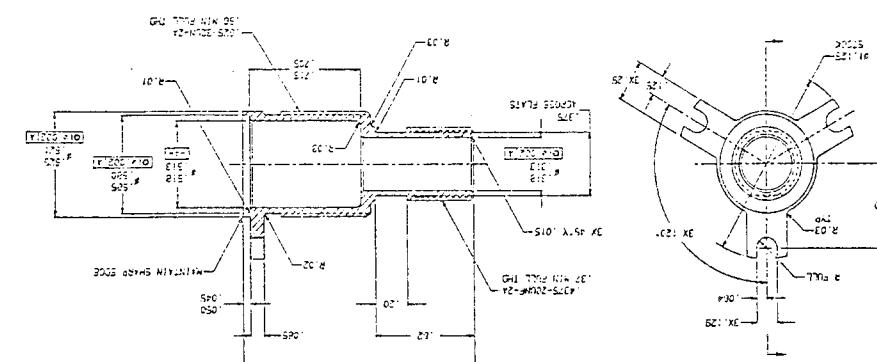
DETAIL - 104-11



ETAIL - 102-11



E:\A\16 - 101-11



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וְאֵת שָׁמֶן וְאֵת שְׁמַנִּים וְאֵת שְׁמַנִּים

TO ZONE TO T.B.D. W-LES.

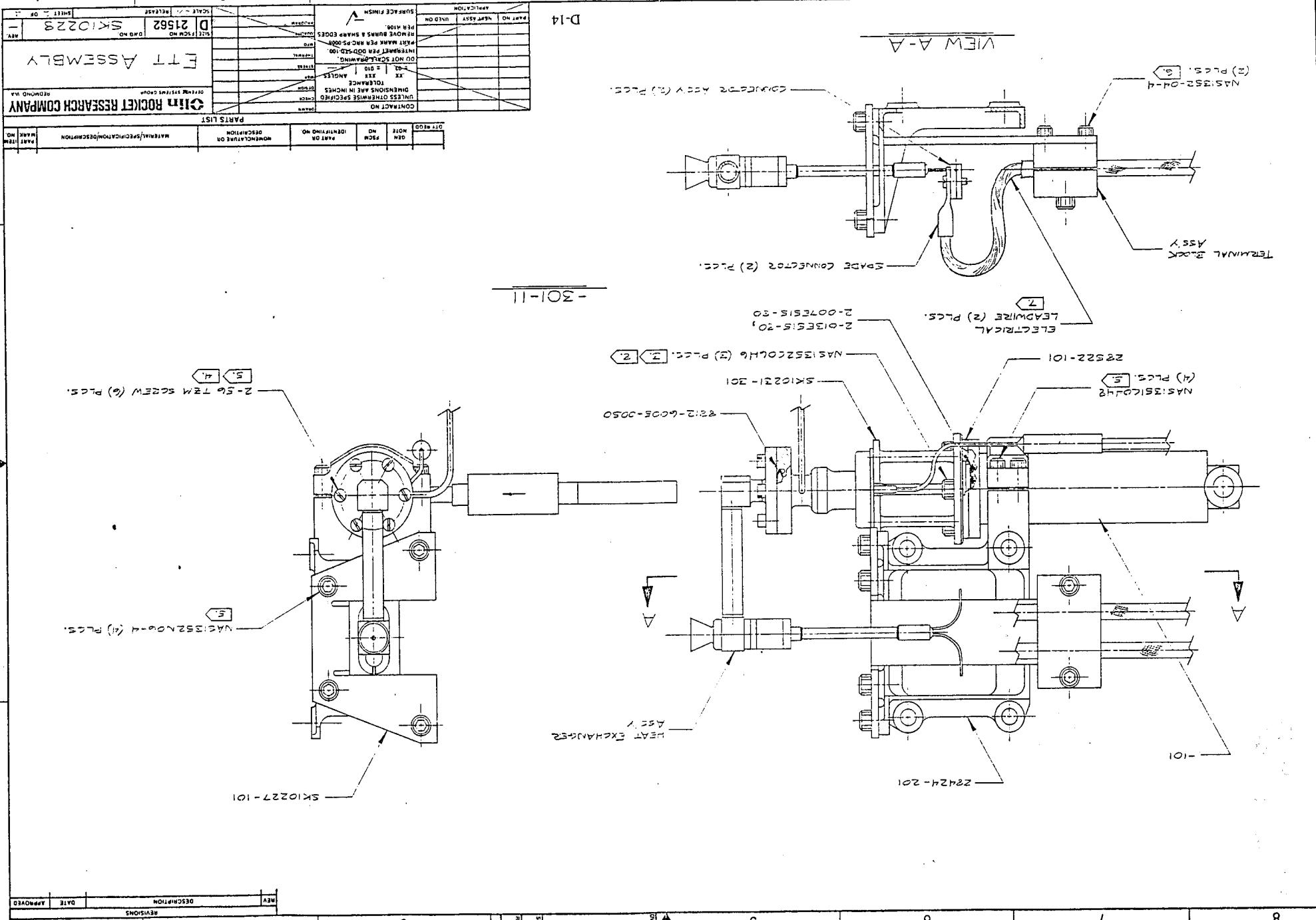
תְּרִיאָנָהָמָה וְאַתְּבָרָהָה

TO INSTALATION. APPENDIX FIVE.

STRIP WIRE AND SEND APPROXIMATELY AS SHOWN ON ASSEMBLE

REV.	SHS	GOALS	6	7
CCS				
6	5	▲		
REVISIONS	DESCRIPTION	DATE APPROVED		
EEV				

REF ID	DESCRIPTION	DATE	APPROVED
REV A	REVISIONS	18/01/2018	
1	1	1	1
2	2	2	2
3	3	3	3
4	4	4	4
5	5	5	5
6	6	6	6
7	7	7	7
8	8	8	8



PARTS LIST		DESCRIPTION		QUANTITY		NOTES	
ITEM NO.	SCN	PART NO.	SCN	DESCRIPTIVE NO.	MATERIAL/SPECIFICATION/DESCRIPTION	DATE ISSUED	DATE REV'D
<b>GAS GENERATOR ASSY</b>							
GAS GENERATOR ASSY							
SK10231							
SHEET 2 OF 2							
SCALE 2/1							
RELEASE							
C 21562							
SHEET NO. 0000 NO. 0000							
PART NO. MFR AS56							
USE ON							
PCB NO. 0000000000000000							
PRINT NO. 0000000000000000							
SHEET NO. 0000000000000000							
CHARACTERISTICS							

A

B

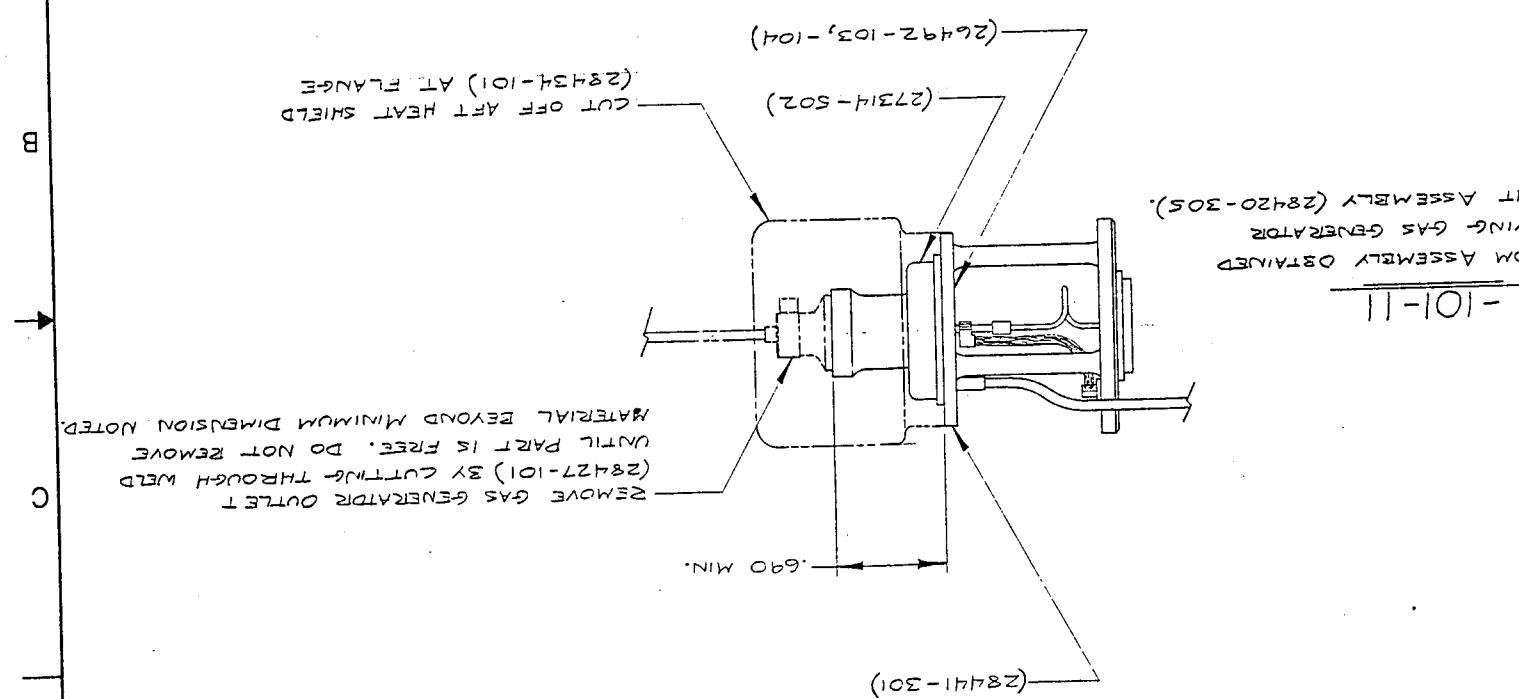
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1 2 3 4

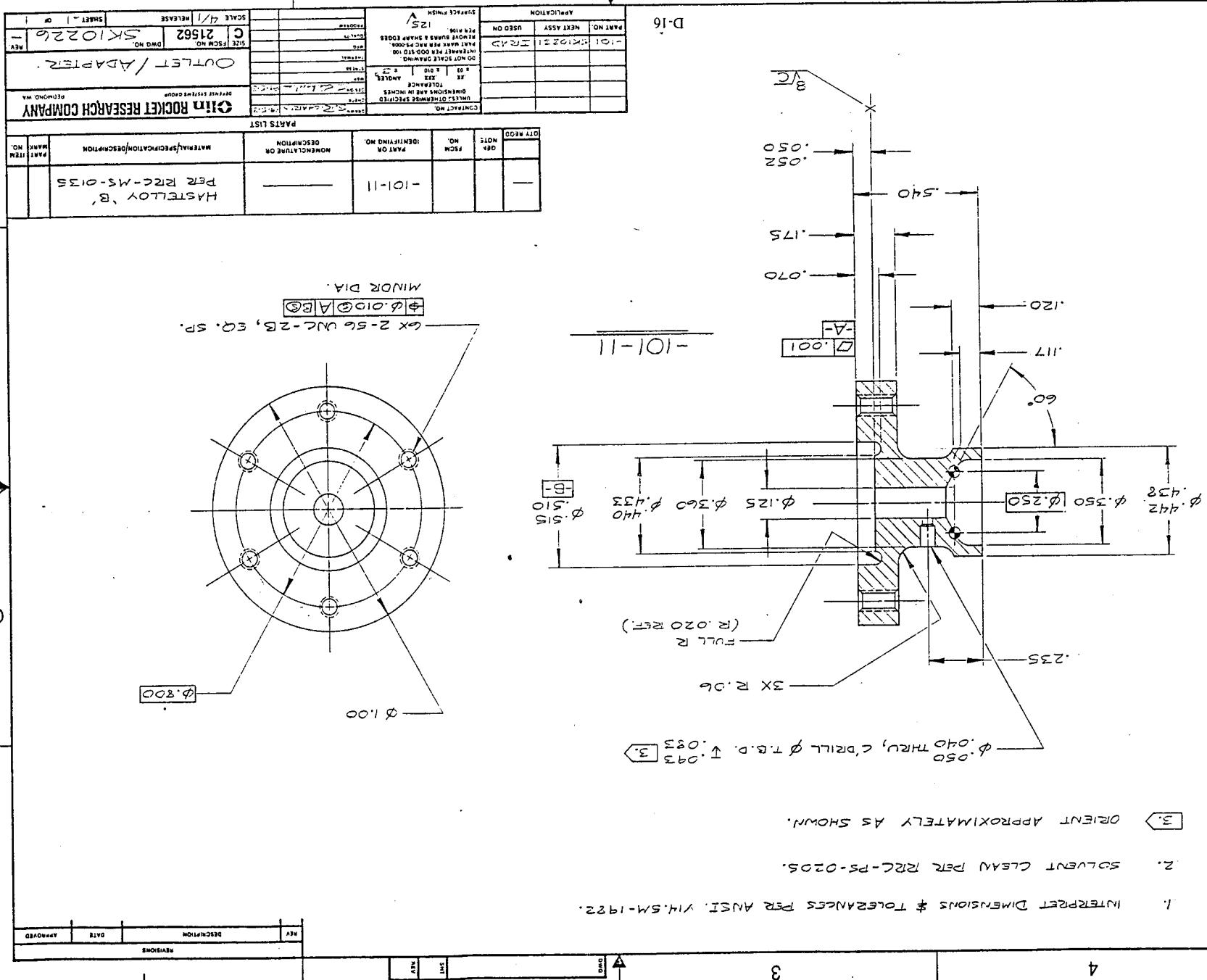
3 4

4

4



AMOUNT	DATE	APPROVED



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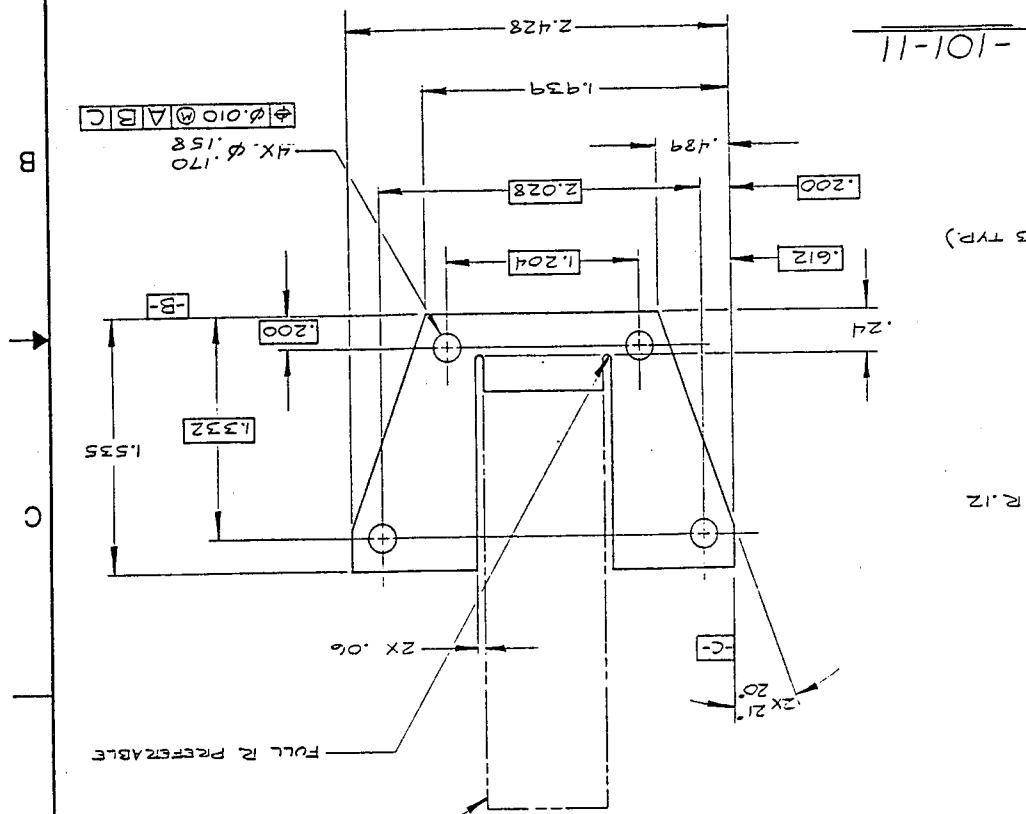
PARTS LIST			
ITEM NO.	DESCRIPTION	QTY REQD	NOTE
-101-11	PEZ AM'S 5540 INCORNL 600 SHEET 3/32" THICK	—	

2

DRAWING NO. C-21562 D/WG NO. SK10227		SCALE 2/1 RELEASE SHEET 1 OF 1
APPLICATON		
C-101 SK10227 Z-A-D		
CD FOR SCALE DRAWINGS INTERVALS ARE 0.00-0.10 INCHES NOTES ARE IN CHCKS		
CONTINUATION NO. 24 OF 25		
DESIGNERS NAME & COMPANY DRAWN BY: D. H. R. DATE: 10-10-82		
REVISIONS		
REV. A	10-10-82	APPROVED
REV. B	10-10-82	DESIGNER
REV. C	10-10-82	CHECKED
REV. D	10-10-82	REVIEWED
REV. E	10-10-82	APPROVED

3

4



1. INTERPRET DRAWING PER ANSI Y14.5M - 1982.  
2. WIRE CLEAN USING M.E.K.



**APPENDIX E**

**Immersed Heater Data**

OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121567-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

## PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-01 Sample Number 6

Parameter	Date	Units	Parameter	Date	Un
Flow	1.5278	Lbm/Hr	Tc	1251.9	De
Thrust (meas)	87.071	Nlbf	Tv	117.4	De
Thrust (corr)	87.071	Nlbf	Tf	67.7	De
Fuel Pressure	322.62	PsiA	T1	1232.4	De
Chamber Pressure	177.26	PsiA	T2	1287.4	De
Alt. Pressure	210.30	Motor	T3	1124.8	De
Elt:	0.002	Volts	T4	1174.2	De
Int.	0.011	Amps	T5	1296.6	De
Int. Power	0.000	Watts	T6	145.6	De
Int. Resistance	0.0000	Ohms			
Int Temperature	0.00	Degre F	CD	12.0000	
P/Mdot	0.00000	MJ/kg	Cf	1.34510	
Gas Temperature	0.0	Degre.	C*	4924.4	Ft
PSP	0.00000		Reynold#	0	
ISP	205.9	Secs.			

Remarks \_\_\_\_\_  
End of run Data \_\_\_\_\_Original  
Or Copy Page  
Quality

OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-01      Sample Number 12

DATE: 01-15-1970      11:18:57

Seq 2

Parameter	Data	Units	Parameter	Data	Units
Flow	1.3216	Lbm/Hr	Tc	1252.7	Degs F
Thrust (meas)	74.258	MlbF	Tv	121.8	Degs F
Thrust (corr)	74.258	MlbF	Tf	68.4	Degs F
Fuel Pressure	259.00	Psia	T1	1215.7	Degs F
Chamber Pressure	152.75	Psia	T2	1268.2	Degs F
Alt. Pressure	173.60	Mtorr	T3	1104.3	Degs F
E ht.	0.002	Volts	T4	1149.3	Degs F
I ht.	0.011	Amps	T5	1073.3	Degs F
Ht. Power	0.000	Watts	T6	156.6	Degs F
Ht. Resistance	0.00000	Ohms			
Ht Temperature	0.00	Degs F	CD	0.00000	
P/Mdot	0.00000	MJ/kg	Cf	1.33103	
Gas Temperature	0.0	Degs.	C*	4905.8	Ft/Sec
PSP	0.00000		Reynolds#	0	
ISP	203.0	Secs.			

Remarks \_\_\_\_\_  
End of run Data \_\_\_\_\_

OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-01      Sample Number 18

Parameter	Data	Units	Parameter	Data	Un
Flow	0.7584	Lbm/Hr	Tc	1188.1	De
Thrust (meas)	40.743	MlbF	Tv	137.6	De
Thrust (corr)	40.743	MlbF	Tf	69.6	De
Fuel Pressure	119.78	Psia	T1	1144.6	De
Chamber Pressure	86.74	Psia	T2	1189.1	De
Alt. Pressure	111.62	Mtorr	T3	1015.4	De
E ht.	0.002	Volts	T4	1046.3	De
I ht.	0.014	Amps	T5	977.0	De
Ht. Power	0.000	Watts	T6	150.6	De
Ht. Resistance	0.00000	Ohms			
Ht Temperature	0.00	Degs F	CD	0.00000	
P/Mdot	0.00000	MJ/Kg	Cf	1.28682	
Gas Temperature	0.0	Degs.	C*	4854.5	Ft
PSP	0.00000		Reynolds#	0	
ISP	194.2	Secs.			

Remarks \_\_\_\_\_  
End of run Data \_\_\_\_\_

COLIN/ROCKET RESEARCH CO.

DATE: 01-15-1970 12:57:05

CONTRACT NUMBER 121587-4333

SEG 4

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.02)

PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-01 Sample Number 24

Parameter	Data	Units	Parameter	Data	Units
Flow	0.6636	Lbm/Hr	Tc	1174.9	Degs F
Thrust (meas)	45.993	M1bf	Tv	144.6	Degs F
Thrust (corr)	45.993	M1bf	Tf	71.3	Degs F
Fuel Pressure	118.70	Psi <sub>e</sub>	Td	1135.8	Degs F
Chamber Pressure	93.72	Psi <sub>a</sub>	T2	1182.2	Degs F
Alt. Pressure	104.47	Millibar	T3	1323.4	Degs F
E Ht.	20.066	Volts	T4	1257.7	Degs F
I Ht.	0.265	Amps	T5	1217.7	Degs F
Ht. Power	165.830	Watts	T6	165.9	Degs F
Ht. Resistance	2.42794	Ohms			
Ht. Temperature	2404.71	Degs F	CD	0.00000	
P/Mdot	1.95356	MJ/Kg	C7	1.34353	
Gas Temperature	0.0	Degs.	C8	5994.8	ft/sec
FSP	3.59416		Reynolds#	0	
ISF	250.3	Secs.			

Remarks \_\_\_\_\_  
End of run Data \_\_\_\_\_Original Page  
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OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-01      Sample Number 29

Parameter	Data	Units	Parameter	Data	Unit
Flow	0.6156	Lbm/Hr	Tc	1168.4	Deg
Thrust (meas)	47.681	Nlbf	Tv	152.7	Deg
Thrust (corr)	47.681	Nlbf	Tf	72.5	Deg
Fuel Pressure	118.76	Psi <sub>a</sub>	Tl	1124.03	Deg
Chamber Pressure	97.34	Psi <sub>a</sub>	T2	1456.4	Deg
Alt. Pressure	100.43	Miller	T3	1572.9	Deg
E Ht.	29.943	Volt <sub>s</sub>	T4	1456.0	Deg
I Ht.	9.057	Amps	T5	1372.3	Deg
Ht. Power	271.100	Watt <sub>s</sub>	T6	1577.9	Deg
Ht. Resistance	3.30610	Ohms			
Ht. Temperature	3327.02	Degs F	CD	0.00000	
P/Mdot	3.49640	lbj/kg	C <sub>f</sub>	1.34225	
Gas Temperature	0.0	Degs.	C <sub>s</sub>	6723.4	
PSP	5.64690		Reynolds#	0	
ISP	280.8	Secs.			

Remarks  
End of run DataDate: 01-15-1990  
Run No.: 102-01  
Test No.: 29

E-6

Original Spec  
of Rocket's Quality

CLIN/ROCKET RESEARCH CO.

DATE: 01-15-1970 13:42:52

CONTRACT NUMBER 121507-4330

HI-PERF. REGISTO-JET PERFORMANCE MAP (Ver 1.022)

PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-01 Sample Number 34

Parameter	Data	Units	Parameter	Data	Units
Flow	0.5971	Lbm/lbf	Tc	1168.4	Degs F
Thrust (meas)	48.517	lbF	Tv	154.5	Degs F
Thrust (corr)	48.517	lbF	Tf	73.6	Degs F
Fuel Pressure	118.84	Psi'a	Tl	1435.6	Degs F
Chamber Pressure	98.83	Psi'a	T2	1187.6	Degs F
Alt. Pressure	97.94	PSIATR	T3	1640.7	Degs F
Elect. Volt	34.983	Volt	T4	1603.6	Degs F
I. Int.	0.236	Ampere	T5	1519.0	Degs F
Int. Power	323.123	Watts	T6	173.0	Degs F
Int. Resistance	3.73757	Ohms			
Int. Temperature	3601.33	Degs F	CD	0.0000	
Flowdot	4.27450	lb/lkg	Cf	1.34350	
Sus. Temperature	0.0	Degs.	C*	-7024.9	Ft/Sec
PSP	6.64061		Reynoldst	0	
ISP	273.3	Secs.			

Remarks \_\_\_\_\_  
End of run Date \_\_\_\_\_ORIGINAL PAGE IS  
OF POOR QUALITY

OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121567-43330

HI FERF. RESISTO-JET PERFORMANCE MAP (VER 1.00)

PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-02 Sample Number 2

Parameter	Data	Units	Parameter	Data	Unit
Flow	0.5931	Lbm/Hr	Tc	1143.9	Deg
Thrust (meas)	50.071	M1bf	Tv	152.3	Deg
Thrust (corr)	50.071	M1bf	Tf	74.3	Deg
Fuel Pressure	121.16	PsiA	T1	957.8	Deg
Chamber Pressure	101.30	PsiA	T2	1071.6	Deg
Alt. Pressure	97.62	Atorr	T3	1772.7	Deg
E ht.	39.922	Volts	T4	1672.4	Deg
I ht.	9.590	Amps	T5	1562.4	Deg
Ht. Power	382.847	Watts	T6	153.3	Deg
Ht. Resistance	4.16362	Ohms			
Ht. Temperature	4187.50	Deg F	CD	0.02762	
P/Mdot	5.12264	MJ/Kg	C†	1.35261	
Gas Temperature	0.0	Degs.	C*	7248.9	Ft.
PSP	7.62496	Reynoldss#		0	
IEP	304.7	Secs.			

2.4 min Run Time

Remarks \_\_\_\_\_  
End of run Date \_\_\_\_\_

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OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4330

HI FERR. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

## PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-022      Sample Number 5

DATE: 01-15-1972

14:16:22  
SEQ 85

Parameter	Data	Units	Parameter	Data	Units
Flow	0.7612	Lbm/Hr	Tc	4193.9	Degs F
Thrust (meas)	40.966	M1BF	Tv	142.9	Degs F
Thrust (corr)	40.966	M1BF	Tf	73.4	Degs F
Fuel Pressure	120.99	PsiA	Td	1143.8	Degs F
Chamber Pressure	67.33	PsiA	T2	1162.8	Degs F
Alt. Pressure	110.60	Atmorr	T3	1017.3	Degs F
Elt. ht.	0.002	Volts	T4	1045.3	Degs F
I ht.	0.014	Amps	T5	975.7	Degs F
Ht. Power	0.000	Watts	T6	157.2	Degs F
Ht. Resistance	0.20000	Ohms			
Ht Temperature	4187.50	Degs F	CD	2.02302	
P/Mdot	0.00000	MJ/Kg	Cf	1.28526	
Gas Temperature	0.0	Degs.	C*	485.4	ft./sec
PSFP	0.00000		Reynolds#	0	
ISP	194.5	Secs.			

Remarks \_\_\_\_\_  
End of run Data \_\_\_\_\_

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CLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121567-4332

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-02 Sample Number 11

Parameter	Data	Units	Parameter	Data	Unit
Flow	1.2441	Lbm/Hr	Tc	1246.7	Degs
Thrust (meas)	83.226	Nlbf	Tv	126.8	Degs
Thrust (corr)	83.226	Nlbf	Tf	71.3	Degs
Fuel Pressure	258.34	PsiA	Ts	1246.2	Degs
Chamber Pressure	165.05	PsiA	T2	126.9	Degs
Alt. Pressure	157.23	Atmos	T3	1320.3	Degs
E ht.	26.056	Volts	T4	1272.7	Degs
I ht.	9.428	Amps	T5	1241.3	Degs
Ht. Power	187.077	Watts	T6	197.3	Degs
Ht. Resistance	2.42739	Ohms			
Ht. Temperature	2095.59	Degs F	Cd	0.00000	
P/Mdot	1.20744	NJ/Kg	Cf	1.37762	
Gas Temperature	0.0	Degs.	Cx	5644.3	Ft/E
FSP	2.26578		Reynolds#	0	
ISF	242.1	Secs.			

Remarks  
End of run DataOriginal Map  
On Poor Quality

BLIN/ROCKET RESEARCH CO.

DATE: 01-15-1970 15:00:52

CONTRACT NUMBER 121587-4332

HI FREQ. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-02 Sample Number 16

Parameter	Data	Units	Parameter	Data	Units
Flow	1.1670	Lbm/Hr	Tc	1244.8	Degs F
Thrust (meas)	87.989	Nlbf	Tv	133.9	Degs F
Thrust (corr)	87.989	Nlbf	Tf	72.2	Degs F
Fuel Pressure	252.36	PsiA	Ti	1211.0	Degs F
Chamber Pressure	172.67	PsiA	T2	1264.6	Degs F
Alt. Pressure	153.36	inHg	T3	1413.9	Degs F
Int. ht.	29.978	Volt	T4	1339.6	Degs F
Ext. ht.	14.177	inHg	T5	1323.3	Degs F
Int. Power	335.079	Watts	T6	1224.1	Degs F
Int. Resistance	2.48227	Ohms			
Int. Temperature	2645.83	Degs F	CD	0.020035	
P/Mdot	2.22472	NJ/Kg	Cf	1.39410	
Gas Temperature	0.0	Degs.	Ck	5122.4	ft., Sec
PSF	3.79981		Reynolds#	0	
TEP	265.3	Secs.			

Remarks \_\_\_\_\_

End of run Data \_\_\_\_\_

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OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-02 Sample Number 22

Parameter	Data	Units	Parameter	Date	Unit
Flow	1.1775	Lbm/Hr	Tc	1252.8	Degs
Thrust (meas)	91.194	MlbF	Tv	126.3	Degs
Thrust (corr)	91.194	MlbF	Tf	67.7	Degs
Fuel Pressure	259.52	PsiA	Ti	1206.9	Degs
Chamber Pressure	177.09	PsiA	T2	1261.7	Degs
Alt. Pressure	133.02	Atmorr	T3	1492.4	Degs
E ht.	35.049	Volts	T4	1369.4	Degs
I ht.	11.723	Amps	T5	1361.7	Degs
Ht. Power	410.662	Watts	T6	195.1	Degs
Ht. Resistance	2.98975	Ohms			
H2 Temperature	2981.97	Degs F	CD	0.0000	
F/Mdot	2.76941	MJ/Kg	Cf	1.40821	
Gas Temperature	0.2	Degs.	Cg	6383.2	Ft/S
PSP	4.49628		Reynolds#	2	
ISP	279.4	Secs.			

Remarks \_\_\_\_\_

End of run Data \_\_\_\_\_

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OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (VER 1.00)

## PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-02      Sample Number 28

DATE: 01-16-1970      07:47:32

SEQ 12

Parameter	Data	Units	Parameter	Data	Units
Flow	1.1538	Lbm/Hr	Tc	1245.3	Degs F
Thrust (meas)	92.919	M1bf	Tv	131.6	Degs F
Thrust (corr)	92.919	M1bf	Tf	69.7	Degs F
Fuel Pressure	259.83	Psiia	T1	1212.6	Degs F
Chamber Pressure	180.91	Psiia	T2	1267.3	Degs F
Alt. Pressure	128.46	Micro	T3	1587.1	Degs F
E ht.	40.047	Volts	T4	1456.0	Degs F
I ht.	12.083	Amps	T5	1444.3	Degs F
Ht. Power	483.983	Watts	T6	213.3	Degs F
Ht. Resistance	3.31428	Ohms			
Ht Temperature	3515.42	Degs F	CD	0.30000	
P/Mdot	3.32336	MJ/Kg	Cf	1.40439	
Gas Temperature	0.0	Degs.	C*	6654.7	ft/sec
PSP	5.19737		Reynolds#	0	
ISP	290.5	Secs.			

Remarks \_\_\_\_\_

End of run Data \_\_\_\_\_

CLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.02)

PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-02 Sample Number 34

Parameter	Data	Units	Parameter	Data	Unit
Flow	1.1330	Lbm/Hr	Tc	1246.9	Degs
Thrust (meas)	94.765	MlbF	Tv	137.2	Degs
Thrust (corr)	94.765	MlbF	Tf	71.2	Degs
Fuel Pressure	260.12	psia	T1	1212.0	Degs
Chamber Pressure	194.37	psia	T2	1255.9	Degs
Alt. Pressure	124.35	Atorr	T3	1684.5	Degs
E ht.	44.985	Volts	T4	1525.8	Degs
I ht.	12.377	Amps	T5	1508.4	Degs
Ht. Power	556.767	Watts	T6	272.0	Degs
Ht. Resistance	3.63469	Ohms			
Ht Temperature	3644.64	Degs F	CD	0.00000	
P/Mdot	3.90013	lb/KS	Cf	1.40531	
Gas Temperature	0.0	Degs.	C*	6736.5	Ft/S
FSP	5.86436		Reynolds#		R
ISP	301.7	secs.			

Remarks \_\_\_\_\_

End of Run Date \_\_\_\_\_

OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.0@)

## PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-02

Sample Number 36

Parameter	Date	Units	Parameter	Date	Units
Flow	1.1724	Lbm/Hr	Tc	1227.1	Degs F
Thrust (meas)	91.715	M1bf	Tv	140.8	Degs F
Thrust (corr)	91.715	M1bf	Tf	71.6	Degs F
Fuel Pressure	260.18	Psi <sub>a</sub>	T1	1144.4	Degs F
Chamber Pressure	179.15	Psi <sub>a</sub>	T2	1231.4	Degs F
Alt. Pressure	128.13	Motor	T3	1436.6	Degs F
E ht.	39.141	Volt <sub>e</sub>	T4	1356.3	Degs F
I ht.	12.157	Amps	T5	1263.6	Degs F
Ht. Power	475.851	Watts	T6	201.1	Degs F
Ht. Resistance	3.21947	Ohms			
Ht. Temperature	3218.01	Degs F	CD	0.00020	
P/ndot	3.22129	MJ/Kg	Cf	1.39968	
Gas Temperature	0.0	Degs.	C*	485.4	Ft/Sec
FSP	5.17813		Reynolds#	0	
ISP	282.2	Secs.			

Remarks \_\_\_\_\_ AT 50V NOT UP TD TEMP

End of run Data \_\_\_\_\_

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CLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4352

## H2 PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

## PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-02 Sample Number 42

Parameter	Data	Units	Parameter	Data	Unit
Flow	1.119	Lbm/Hr	Tc	1251.2	Degs
Thrust (meas)	96.450	M1bF	Tv	140.4	Degs
Thrust (curr)	96.450	M1bF	Tf	72.6	Degs
Fuel Pressure	260.23	PsiA	T1	1242.4	Degs
Chamber Pressure	187.71	PsiA	T2	1266.1	Degs
Alt. Pressure	120.89	Motorr	T3	1831.4	Degs
E ht.	50.073	Volts	T4	1612.3	Degs
I ht.	12.626	Amps	T5	1527.3	Degs
Ht. Power	6332.274	Watts	T6	1579.6	Degs
Ht. Resistance	3.96586	Ohms			
Ht. Temperature	3984.92	Degs F	CD	0.00000	
P/Mdot	4.51287	MJ/Kg	Cf	1.40472	
Gas Temperature	0.0	Degs.	C*	7165.2	Ft/c
PSP	5.54345		Reynolds#	G	
ISP	312.8	Secs.			

Remarks \_\_\_\_\_  
End of run Data \_\_\_\_\_Original  
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QUALITY

CLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4330

## HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.020)

## PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-02 Sample Number 48

Parameter	Data	Units	Parameter	Data	Units
FLOW	1.3860	Lbm/Hr	Tc	1246.2	Degs F
Thrust (meas)	104.782	MlbF	Tv	132.3	Degs F
Thrust (corr)	104.782	MlbF	Tf	71.9	Degs F
Fuel Pressure	322.93	PsiA	Ti	1235.6	Degs F
Chamber Pressure	204.86	PsiA	T2	1270.7	Degs F
Alt. Pressure	140.00	IntrP	T3	1486.4	Degs F
E ht.	35.123	Volts	T4	1360.9	Degs F
I ht.	12.276	Amps	T5	1372.2	Degs F
Ht. Power	431.238	Watts	T6	234.1	Degs F
Hc. Resistance	2.86061	Ohms			
Ht Temperature	2849.27	Degs F	CD	0.00000	
P/Mdot	2.46944	MJ/Kg	Cf	1.39850	
Gas Temperature	0.0	Degs.	Ck	6273.3	Ft./Sec
PSF	4.12779		Reynold#	0	
ISP	272.7	Secs.			

Remarks \_\_\_\_\_

End of run Data \_\_\_\_\_

*Quality*

OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4330

## H2 PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

## PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-02 Sample Number 54

Parameter	Data	Units	Parameter	Date	Unit
Flow	1.3627	Lbm/Hr	Tc	1277.6	Degs
Thrust (meas)	107.849	Nlbf	Tv	136.4	Degs
Thrust (corr)	107.849	Nlbf	Tf	72.6	Degs
Fuel Pressure	322.9E	Psia	Tz	1252.6	Degs
Chamber Pressure	200.9E	Psia	T2	1257.4	Degs
Alt. Pressure	136.99	Atm	T3	1567.3	Degs
E ht.	39.992	Volts	T4	1452.4	Degs
I ht.	12.736	Amps	T5	1477.6	Degs
Ht. Power	509.335	Watts	T6	243.0	Degs
Ht. Resistance	7.14012	Ohms			
Ht. Temperature	3136.47	Degs F	OD		
P/Mdot	2.96633	MJ/Kg	C%	1.41102	
Gas Temperature	0.0	Degs.	C%	652E-3	Per cent
PSP	4.71417		Reynolds#	②	
ISP	285.4	Secs.			

Remarks  
End of run DataORIGINALLY  
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QUALITY

CLIN/OCKET RESEARCH CO.

CONTRACT NUMBER 121587-4332

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

## PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-02 Sample Number 60

Parameter	Data	Units	Parameter	Data	Units
Flow	1.3395	Lbm/Hr	Tc	1246.3	Degs F
Thrust (meas)	110.199	Mlbf	Tv	135.7	Degs F
Thrust (corr)	110.199	Mlbf	Tf	73.8	Degs F
Fuel Pressure	323.15	Psi	Tl	1234.7	Degs F
Chamber Pressure	243.12	Psi	T2	1291.0	Degs F
Alt. Pressure	133.65	Motor	T3	1643.7	Degs F
E ht.	4.4954	Volts	T4	1482.5	Degs F
I ht.	13.080	Amps	T5	1490.5	Degs F
Ht. Power	550.009	Watts	T6	246.1	Degs F
Ht. Resistance	3.43632	Ohms			
Ht. Temperature	3441.34	Degs F	CD	0.00000	
P/Mdot	3.48410	MJ/Kg	Cf	1.41349	
Gas Temperature	0.0	Degs.	Cx	6753.2	ft./sec
ISF	5.32672		Reynolds#	0	
ISF	296.7	Secs.			

Remarks \_\_\_\_\_

End of run Data \_\_\_\_\_

CLIN/ROCKET RESEARCH CO.

DATE: 01-16-1990 100:44:17

CONTRACT NUMBER 121587-4332

H2 PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-02 Sample Number 66

Parameter	Data	Units	Parameter	Data	Unit
Flow	1.3156	Lbm/Hr	Tc	1276.4	Deg
Thrust (meas)	113.938	M1bf	Tv	137.3	Deg
Thrust (corr)	113.938	M1bf	Tf	74.8	Deg
Fuel Pressure	323.36	PsiA	T1	1234.7	Deg
Chamber Pressure	217.11	PsiA	T2	1291.7	Deg
Ait. Pressure	130.75	Atorr	T3	1737.4	Deg
E ht.	49.965	Volts	T4	1545.7	Deg
I ht.	13.370	Amperes	T5	1550.2	Deg
Hc. Power	563.046	Watts	T6	251.3	Deg
Hc. Resistance	3.73700	Ohms			
Ht Temperature	3749.77	Deg F	CD	0.00000	
F/Mdot	4.03200	MJ/Kg	Cf	1.43450	
Gas Temperature	0.0	Degs.	C*	7004.1	Ft/s
RSP	5.85370		Reynolds#	6	
ISP	312.3	Secs.			

Remarks post thrust calibration error  
End of Run Date \_\_\_\_\_Original Page  
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OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4332

DATE: 21-16-1970 12:12:47

SEQ 2C

## HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.20)

## PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-02 Sample Number 72

Parameter	Date	Units	Parameter	Date	Units
Flow	1.2945	Lbm/Hr	Tc	1265.8	Degs F
Thrust (meas)	114.188	Mlbf	Tv	140.5	Degs F
Thrust (corr)	114.188	Mlbf	Tf	76.0	Degs F
Fuel Pressure	323.55	PsiA	T1	4233.6	Degs F
Chamber Pressure	220.69	PsiA	T2	1236.0	Degs F
Alt. Pressure	128.68	Intrr	T3	1836.9	Degs F
E ht.	54.995	Volts	T4	1545.0	Degs F
I ht.	13.620	Amps	T5	1645.0	Degs F
Ht. Power	747.463	Watts	T6	258.0	Degs F
Ht. Resistance	4.27556	Ohms			
Ht Temperature	4056.54	Degs F	CD	0.00000	
P/Mdot	4.57431	MJ/Kg	Cf	1.41424	
Gas Temperature	0.0	Degs.	C*	7235.7	Rate/Sec
PSP	6.55292		Reynolds#	0	
ISP	318.1	Secs.			

Remarks \_\_\_\_\_

End of Run Date \_\_\_\_\_

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CONTRACT NUMBER 121557-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.02)

PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-02 Sample Number 74

Parameter	Data	Units	Parameter	Data	Unit
Flow	1.3337	Lbm/Hr	Tc	1255.3	Degs
Thrust (meas)	110.235	Mlbf	Tv	144.6	Degs
Thrust (corr)	110.285	Mlbf	Tf	76.3	Degs
Fuel Pressure	323.59	PsiA	T1	1147.0	Degs
Chamber Pressure	214.38	PsiA	T2	1247.6	Degs
Alt. Pressure	133.39	Atorr	T3	1625.7	Degs
E ht.	48.597	VoltE	T4	1437.5	Degs
I ht.	13.466	Amps	T5	1363.4	Degs
Ht. Power	684.395	Watts	T6	242.0	Degs
Ht. Resistance	3.60669	Ohms		3.00000	
Ht Temperature	3618.14	Degs F	CD		
P/Mdot	3.89431	MJ/Kg	C4	1.40630	
Gas Temperature	0.0	Degs.	C8	6822.5	Ft/E
PSF	5.92353		Reynolds#	2	
ISP	298.2	Secs.			

Remarks Not steady-state values  
End of Run Date \_\_\_\_\_

OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4330

DATE: 01-16-1970 14:57:46

SEQ 22

H2 PERS. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-02 Sample Number 79

Parameter	Data	Units	Parameter	Data	Units
Flow	1.3158	Lbm/Hr	Tc		Degs F
Thrust (meas)	111.995	M1bf	Tv	137.2	Degs F
Thrust (corr)	111.995	M1bf	Tf	75.9	Degs F
Fuel Pressure	323.24	PsiA	T1	1235.7	Degs F
Chamber Pressure	247.47	PsiA	T2	1233.3	Degs F
Alt. Pressure	130.32	Motor	T3	1739.6	Degs F
E ht.	49.956	Volts	T4	1544.2	Degs F
I ht.	13.356	Amps	T5	1555.2	Degs F
Net. Power	667.232	Watts	T6	247.9	Degs F
H2. Resistance	3.74018	Ohms			
Ht Temperature	3753.03	Degs F	CD	0.02300	
P/MCot	4.02447	MJ/Kg	Cf	1.407469	
Gas Temperature	0.0	Degs.	C*	7004.6	Foot/Sec
PGP	5.94787		Reynolds#	0	
IEP	306.9	Secs.			

Remarks \_\_\_\_\_  
End of run Data \_\_\_\_\_

OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.02)

PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-04 Sample Number 4

Parameter	Data	Units	Parameter	Data	Unit
Flow	0.7506	Lbm/Hr	Tc	1173.8	Deg
Thrust (mass)	40.219	NlbF	Tv	118.5	Deg
Thrust (corr)	40.219	NlbF	Tf	64.6	Deg
Fuel Pressure	119.30	PsiA	T1	122.9	Deg
Chamber Pressure	83.47	PsiA	T2	1151.7	Deg
Alt. Pressure	103.64	PSI	T3	1047.4	Deg
Elt. Ht.	0.022	Volt	T4	1097.3	Deg
I Ht.	0.013	Amps	T5	1213.9	Deg
Ht. Power	0.020	Watts	T6	127.4	Deg
Ht. Resistance	0.0000	Ohms			
Ht. Temperature	0.00	Deg F	CD	C.00000	
Prdct	0.0000	MJ/KG	CF	1.28371	
Gas Temperature	0.0	Deg	C*	4332.7	Ft/lb
PSF	0.0000		Reynolds#	0	
ISP	193.6	Secs.			

Remarks  
End of run Date \_\_\_\_\_

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CLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 124537-4338

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-04 Sample Number 10

DATE: 01-17-1990 12:21:25

SEG 25

Parameter	Data	Units	Parameter	Data	Units
Flow	1.3116	Lbm/Hr	Tc	1259.6	Degs F
Thrust (mass)	73.857	MlbF	Tv	120.5	Degs F
Thrust (corr)	73.857	MlbF	Tf	67.1	Degs F
Fuel Pressure	253.61	PsiA	Tl	1216.6	Degs F
Chamber Pressure	152.76	PsiA	T2	1269.7	Degs F
Ait. Pressure	142.83	PsiA	T3	1145.6	Degs F
E ht.	0.302	Volts	T4	1212.6	Degs F
I ht.	0.314	Amps	T5	1145.0	Degs F
Ht. Power	2.000	Watts	T6	1211.8	Degs F
Ht. Resistance	0.30000	Ohms			
Ht. Temperature	0.20	Degs F	CD	0.00000	
P/Mdot	0.00000	MJ/Kg	C%	1.32384	
Gas Temperature	0.0	Degs.	C%	4945.2	ft/sec
PSF	0.00000		Reynolds#	0	
ISP	203.3	Secs.			

Remarks \_\_\_\_\_

End of run Data \_\_\_\_\_

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CLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121537-43332

HT. PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-04 Sample Number 14

DATE: 01-17-1990 10:40:  
S592

Parameter	Data	Units	Parameter	Data	Uni
Flow	1.5208	Lbm/Hr	Tc	1281.3	Deg
Thrust (mass)	86.039	M1bf	Tv	118.9	Deg
Thrust (corr)	86.039	M1bf	Tf	67.7	Deg
Fuel Pressure	323.15	PsiA	T1	1235.8	Deg
Chamber Pressure	177.42	PsiA	T2	1292.7	Deg
Ait. Pressure	160.23	Atorr	T3	1468.9	Deg
Elt.	0.202	Volts	T4	1240.0	Deg
Ilt.	0.010	Amps	T5	1374.3	Deg
Ht. Power	0.000	Watts	T6	167.3	Deg
Ht. Resistance	0.00000	Ohms			
Ht. Temperature	0.00	Deg F	CD	0.00002	
P/Mdot	0.00000	MJ/Kg	Cf	1.32657	
Gas Temperature	0.0	Deg.	C%	4951.5	Ft,
PSP	0.00000		Reynold#	0	
ISP	204.2	Secs.			

Remarks \_\_\_\_\_  
End of run Date \_\_\_\_\_

CLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4330

DATE: 04-17-1999 11:00:34

SEQ 27

## HI PERF. RESISTO-JET PERFORMANCE MAP (VER 1.00)

## PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-04 Sample Number 17

Parameter	Data	Units	Parameter	Data	Units
Flow	1.0974	Lbm/Hr	Tc	1247.0	Degs F
Thrust (mass)	95.691	M1bf	Tv	132.7	Degs F
Thrust (corr)	95.691	M1bf	Tf	70.0	Degs F
Fuel Pressure	259.19	PsiA	T1	1210.2	Degs F
Chamber Pressure	167.13	PsiA	T2	1266.6	Degs F
Alt. Pressure	122.71	Atmorr	T3	1891.3	Degs F
Elt. Ht.	49.957	Volt	T4	1741.0	Degs F
I Ht.	1.425	Amps	T5	1741.0	Degs F
Ht. Power	621.199	Watts	T6	234.1	Degs F
Ht. Resistance	4.01149	Ohms			
Ht. Temperature	40231.52	Degs F	CD	2.00000	
P/Mdot	4.47269	MJ/Kg	Cf	1.397772	
Gas Temperature	0.0	Degs.	Cx	7239.3	Ft./Sec
PSP	6.47987		Reynolds#	0	
ISP	314.5	Secs.			

Remarks  
End of run DataOriginal PAGE 13  
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CONTRACT NUMBER 121587-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (VER 1.00)

PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-04      Sample Number 25

DATE: 01-17-1970

SEG 2

Parameter	Data	Units	Parameter	Data	Un
Flow	1.2939	Lbm/Hr	Tc	1282.6	De
Thrust (meas)	114.320	Mlbf	Tv	134.8	De
Thrust (corr)	114.320	Mlbf	Tf	73.3	De
Fuel Pressure	323.50	PsiA	T1	1234.4	De
Chamber Pressure	221.22	PsiA	T2	-	De
Alt. Pressure	134.80	Miller	T3	1273.5	De
E. ht.	54.993	Volt	T4	10417.7	De
I. ht.	13.473	Amps	T5	1775.5	De
Ht. Power	742.300	Watts	T6	240.0	De
Ht. Resistance	4.07557	Ohms			
Ht. Temperature	4097.64	Degs F	CD	3.00000	
F/Mdot	4.55154	NJ/Kg	Cf	4.41267	
Gas Temperature	0.0	Degs	C*	7256.5	Fr
PSP	6.47995		Reynold's	7256.5	
ISP	318.6	Secs.		0	

Remarks  
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CONTRACT NUMBER 1215G7-4530

HT. PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 122-04 Sample Number 34

DATE: 04-17-1978 12:38:15

SEQ 29

Parameter	Data	Units	Parameter	Data	Units
Flow	1.5625	lbm/lbm	Tc	1274.6	Degs F
Thrust (meas)	126.314	M1bf	Tv	131.4	Degs F
Thrust (corr)	126.314	M1bf	Tf	73.1	Degs F
Fuel Pressure	399.45	PsiA	Tl	1252.1	Degs F
Chamber Pressure	244.83	PsiA	T2	-	Degs F
Alt. Pressure	155.65	Atorr	T3	1662.0	Degs F
E ht.	45.121	Volt	T4	1535.5	Degs F
I ht.	13.635	Amps	T5	1592.5	Degs F
Ht. Power	6.14.959	watts	T6	247.8	Degs F
Ht. Resistance	3.36774	Ohms			
Ht. Temperature	3308.57	Degs F	CD	0.02320	
P/Mdot	3.12362	lb./kg	C*	1.44217	
Gas Temperature	0.0	Degs.	C*	642.3	Fl. Sec
FSP	4.66230		Reynolds#	Q	
IEP	291.5	secs.			

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End of run Date \_\_\_\_\_ORIGINAL PAGE  
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CONTRACT NUMBER 121587-4330

HI PERC. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-04 Sample Number 37

DATE: 01-17-1992 13:32:14  
SEG 3c

Parameter	Data	Units	Parameter	Data	Unit
Flow	1.5442	Lbm/Hr	Tc	1392.0	Deg
Thrust (meas)	128.834	Nlbf	Tv	135.9	Deg
Thrust (corr)	128.354	Nlbf	Tf	74.4	Deg
Fuel Pressure	379.76	PsiA	T1	1252.7	Deg
Chamber Pressure	249.13	PsiA	T2	1309.6	Deg
Alt. Pressure	149.94	Wtorr	T3	1746.6	Deg
Alt.	49.94%	Volt	T4	1500.3	Deg
I Alt.	1.3.972	Amps	T5	1647.1	Deg
Ht. Power	697.833	Watts	T6	162.0	Deg
Ht. Resistance	3.57479	Ohms			
Ht. Temperature	3583.07	Degs F	CD		
P/Molat	3.52669	MJ/Kg	Cf	1.44359	
Gas Temperature	0.0	Degs.	C*	6847.6	F <sub>21</sub>
PSP	5.40760		Reynolds#	6	
ISP	300.9	Secs.			

Remarks \_\_\_\_\_

End of run Date \_\_\_\_\_

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CLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121567-4332

DATE: 01-17-1970 13:23:52

Seq 31

HI FERF, RESISTO-JET PERFORMANCE MAP (Ver 1.02)

PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-04 Sample Number 42

Parameter	Data	Units	Parameter	Data	Units
Flow	1.5159	Lbm/Hr	Tc	1263.9	Degs F
Thrust (meas)	131.359	Mlbf	Tv	138.6	Degs F
Thrust (corr)	131.359	Mlbf	Tf	76.0	Degs F
Fuel Pressure	402.25	Psi	Ti	1253.7	Degs F
Chamber Pressure	263.86	Psi	T2	1311.4	Degs F
Alt. Pressure	144.42	Atmorr	T3	1840.1	Degs F
E hi.	54.930	Volt	T4	1655.6	Degs F
I hi.	1.4.262	Amps	T5	1714.2	Degs F
Ht. Power	750.123	Watts	T6	272.7	Degs F
Ht. Residue	3.65476	Ohms		0.00000	
Ht. Temperature	3370.97	Degs C	CD		
F/Mdot	4.10544	MJ/Kg	Cf	1.41431	
Gas Temperature	0.0	Degs C	C*	7167.6	Ft./Sec
PSR	5.96012		Reynolds#	0	
ISP	312.4	Secs			

Remarks \_\_\_\_\_  
End of run Date \_\_\_\_\_Original PAGE 3  
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CLEAN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-04      Sample Number 47

Parameter	Data	Units	Parameter	Data	Unit
Flow	1.4955	Lbm/Hr	Tc	1301.7	Deg
Thrust (mass)	133.413	M16F	Tv	142.4	Deg
Thrust (curr)	133.413	M16F	Tf	77.3	Deg
Fuel Pressure	400.42	PSia	T1	1253.9	Deg
Chamber Pressure	258.37	PSia	T2	1305.2	Deg
Alt. Pressure	140.76	PSia	T3	1536.3	Deg
Int.	39.587	Volt	T4	1724.5	Deg
Int.	14.514	Amps	T5	1764.3	Deg
Ht. Power	870.678	Watt	T6	2053.3	Deg
Ht. Resistance	4.43215	Ohms			
Ht. Temperature	4156.01	Deg F	CD	0.03630	
P/Indst	4.622079	Kg/Kg	CF	1.41127	
Gas Temperature	0.0	Degs.	CS	1332.6	ft
FSP	6.51646		Reynoldsd		
Imp	321.6	Secs.		2	

Remarks \_\_\_\_\_

End of run Date \_\_\_\_\_

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CLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121537-4332

DATE: 01-17-1962 14:53:52

SEQ 33

## WT PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

## PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-04 Sample Number 53

Parameter	Data	Units	Parameter	Data	Units
Flow	1.2766	Lbm/Hr	Tc	1258.6	Degs F
Thrust (meas)	113.920	Mlbf	Tv	138.4	Degs F
Thrust (corr)	113.920	Mlbf	Tf	78.4	Degs F
Fuel Pressure	323.25	Psis	T1	1237.0	Degs F
Chamber Pressure	222.38	Psi <sub>atm</sub>	T2	1292.1	Degs F
Alt. Pressure	127.58	Atmos	T3	1952.5	Degs F
Elt. %	56.474	Volt	T4	1765.7	Degs F
I lt.	13.620	Amp	T5	1794.4	Degs F
Ht. Power	763.611	Watts	T6	217.2	Degs F
Ht. Resistance	4.17723	Ohms			
Ht. Temperature	4202.10	Degs F	CR	0.00000	
Prndot	4.74664	Mg/kg	C†	1.40020	
Gas Temperature	0.0	Degs.	C‡	7392.6	Ft./Sec
PSP	6.69456	Reynolds#		0	
ISP	321.8	Secs.			

Remarks \_\_\_\_\_

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CONTRACT NUMBER 121587-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

## PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-05      Sample Number 8

DATE: 01-18-1990      07:3  
SEQ 34

Parameter	Data	Units	Parameter	Data	U
Flow	0.5782	Lbm/Hr	Tc	1232.1	D
Thrust (meas)	62.765	M1bf	Tv	129.4	D
Thrust (corr)	32.765	M1bf	Tf	66.3	D
Fuel Pressure	220.29	Psi <sub>a</sub>	T1	1192.8	D
Chamber Pressure	164.61	Psi <sub>a</sub>	T2	1249.1	D
Alt. Pressure	118.48	Motorr	T3	1845.0	D
E ht.	44.981	Volts	T4	1710.9	D
I ht.	11.604	Amps	T5	1701.2	D
Ht. Power	521.942	Watts	T6	197.5	D
Ht. Resistance	3.87651	Ohms			
Ht. Temperature	3895.11	Degs F	CD	0.00000	
P/Mdot	4.23475	MJ/Kg	Cf	1.37321	
Gas Temperature	0.0	Degs.	C*	7151.0	F
PSP	6.29355		Reynolds#		Z
ISP	305.2	Secs.			

Remarks \_\_\_\_\_

End of Run Date \_\_\_\_\_

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OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121567-4330

DATE: 01-18-1970 07:58:23  
SEQ 55

## HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

## PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-05 Sample Number 13

Parameter	Data	Units	Parameter	Data	Units
Flow	0.9526	Lbm/Hr	Tc	1241.2	Degs F
Thrust (meas)	84.053	MlbF	Tv	137.8	Degs F
Thrust (corr)	84.053	MlbF	Tf	68.1	Degs F
Fuel Pressure	220.44	Psia	T1	1197.7	Degs F
Chamber Pressure	167.49	Psia	T2	1255.0	Degs F
Alt. Pressure	115.38	Millibar	T3	1988.9	Degs F
E ht.	50.004	Volts	T4	1879.5	Degs F
I ht.	11.805	Amps	T5	1824.6	Degs F
Ht. Power	592.305	Watts	T6	219.9	Degs F
Ht. Resistance	4.23585	Ohms			
Ht Temperature	4262.34	Degs F	CD	0.00000	
P/Mdot	4.91791	MJ/Kg	Cf	1.37223	
Gas Temperature	0.0	Degs.	Cx	7461.9	Ft/Sec
FSP	7.00936		Reynolds#	0	
ISP	348.3	Secs.			

Remarks \_\_\_\_\_  
End of run Date \_\_\_\_\_Original PAGE 18  
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OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4336

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-06      Sample Number 9

Parameter	Data	Units	Parameter	Data	Un
Flow	0.9585	Lbm/Hr	Tc	1243.5	De
Thrust (meas)	82.812	MibF	Tv	138.7	De
Thrust (corr)	82.812	MibF	Tf	69.4	De
Fuel Pressure	220.11	PsiA	Ti	1194.3	De
Chamber Pressure	166.47	PsiA	T2	1247.9	De
Alt. Pressure	115.54	Atmorr	T3	1917.3	De
E ht.	47.473	Volts	T4	1773.4	De
I ht.	11.693	Amps	T5	1760.8	De
Ht. Power	555.117	Watts	T6	214.5	De
Ht. Resistance	4.05992	Ohms			
Ht. Temperature	4081.57	Degs F	ED	0.00000	
P/Mdot	4.59630	MJ/Kg	Cf	1.36011	
Gas Temperature	0.0	Degs.	Cx	7371.6	Ft
PSP	6.69010		Reynolds#	0	
ISF	311.6	Secs.			

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CONTRACT NUMBER 121587-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-07 Sample Number 7

DATE: 01-22-1990 08:05:15  
SEQ 37

Parameter	Date	Units	Parameter	Date	Units
Flow	1.3988	Lbm/Hr	Tc	1273.1	Degs F
Thrust (meas)	78.737	Mlbf	Tv	116.2	Degs F
Thrust (corr)	78.737	Mlbf	Tf	62.8	Degs F
Fuel Pressure	285.39	Psia	T1	1221.3	Degs F
Chamber Pressure	164.63	Psia	T2	1274.7	Degs F
Alt. Pressure	155.88	Motor	T3	1156.2	Degs F
E ht.	0.004	Volts	T4	1219.5	Degs F
I ht.	0.012	Amps	T5	1155.2	Degs F
Ht. Power	0.000	Watts	T6	164.3	Degs F
Ht. Resistance	0.00000	Ohms			
Ht Temperature	0.00	Degs F	CD	0.00000	
P/Mdot	0.00000	MJ/Kg	Cf	1.30888	
Gas Temperature	0.0	Degs.	C*	4995.1	Ft/Sec
FSP	0.00000		Reynolds#	0	
IEP	203.2	Secs.			

Remarks \_\_\_\_\_

End of run Date: \_\_\_\_\_

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OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

## PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-07      Sample Number 13

Parameter	Data	Units	Parameter	Data	Unit
Flow	1.1727	Lbm/Hr	Tc	1249.1	Deg
Thrust (meas)	102.677	Mbf	Tv	130.5	Deg
Thrust (corr)	102.677	Mbf	Tf	66.7	Deg
Fuel Pressure	285.68	Psi	T1	1220.2	Deg
Chamber Pressure	203.68	Psi	T2	1278.6	Deg
Alt. Pressure	132.56	Atorr	T3	1922.1	Deg
Elt.	52.558	Volts	T4	1746.9	Deg
I ht.	12.684	Amperes	T5	1765.5	Deg
Ht. Power	677.131	Watts	T6	231.9	Deg
Ht. Resistance	4.07941	Ohms			
Ht Temperature	4101.60	Degs F	CD	0.00000	
P/Mdot	4.58254	MJ/Kg	Cf	1.38091	
Gas Temperature	0.0	Degs.	C*	7371.4	Ft
PSP	6.56995		Reynolds#	0	
ISP	316.4	Secs.			

Remarks \_\_\_\_\_

End of run Date \_\_\_\_\_

BLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 1215E7-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-07 Sample Number 19

DATE: 01-22-1990 09:07:31  
SEQ 39

Parameter	Date	Units	Parameter	Data	Units
Flow	1.0936	Lbm/Hr	Tc	1245.9	Degs F
Thrust (meas)	95.558	MlbF	Tv	135.8	Degs F
Thrust (corr)	95.558	MlbF	Tf	68.7	Degs F
Fuel Pressure	262.42	PsiA	T1	1210.4	Degs F
Chamber Pressure	171.43	PsiA	T2	1267.0	Degs F
Alt. Pressure	124.05	Atmorr	T3	1926.1	Degs F
E ht.	50.981	Volts	T4	1759.3	Degs F
I ht.	12.453	Amps	T5	1767.3	Degs F
Ht. Power	635.366	Wattes	T6	232.6	Degs F
Ht. Resistance	4.09057	Ohms			
Ht Temperature	4113.06	Degs F	CD	0.00000	
F/Mdot	4.61124	MJ/KG	Cf	1.36476	
Gas Temperature	0.0	Degs.	C*	7429.8	Foot/Sec
PSP	6.63680		Reynolds#	0	
ISP	315.2	Secs.			

Remarks \_\_\_\_\_  
End of run Data \_\_\_\_\_Original page 3  
of four pages

OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121537-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

## PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-07      Sample Number 25

Parameter	Data	Units	Parameter	Data	U
FLOW	1.0201	Lbm/Hr	Tc	1235.2	D
THRUST (meas)	88.701	NlbF	Tv	141.0	D
THRUST (corr)	88.701	NlbF	Tf	72.2	D
FUEL PRESSURE	240.36	PsiA	T1	1205.4	D
Chamber Pressure	179.10	PsiA	T2	1262.1	D
Alt. Pressure	117.42	Ptorr	T3	1929.0	D
E ht.	49.285	VoltS	T4	1773.4	D
I ht.	12.052	Amps	T5	1770.7	D
Ht. Power	593.966	Watts	T6	234.9	D
Ht. Resistance	4.08734	Ohms			
Ht Temperature	4111.80	Degs F	CD	0.00000	
P/Mdot	4.62131	MJ/KG	Cf	1.35411	
Gas Temperature	0.0	Degs.	C*	7451.6	F
FSP	6.68392		Reynolds#		
ISF	313.6	Secs.			

Remarks  
End of run Data

OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121567-4330

## HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

## PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-07 Sample Number 30

DATE: 01-22-1990 10:04:11  
SEQ#1

Parameter	Data	Units	Parameter	Data	Units
FLOW	0.9529	Lbm/Hr	Tc	1239.7	Degs F
Thrust (meas)	82.637	M1bF	Tv	143.9	Degs F
Thrust (corr)	82.637	M1bF	Tf	71.2	Degs F
Fuel Pressure	221.28	Psia	T1	1197.7	Degs F
Chamber Pressure	168.00	Psia	T2	1253.8	Degs F
Alt. Pressure	112.33	MTorr	T3	1934.9	Degs F
Eht.	47.759	Volts	T4	1783.9	Degs F
I ht.	11.659	Amps	T5	1776.3	Degs F
Ht. Power	556.928	Watts	T6	231.0	Degs F
Ht. Resistance	4.097733	Ohms			
Ht Temperature	4120.01	Degs F	CD	0.00000	
P/Mdot	4.63878	MJ/Kg	Cf	1.34494	
Gas Temperature	0.0	Degs.	C*	7483.1	Ft./Sec
PSP	6.72652		Reynolds#	0	
ISP	312.9	Secs.			

Remarks \_\_\_\_\_  
End of run Date \_\_\_\_\_

OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4330

## HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

## PERFORMANCE MAP ON MODIFIED ETT S/N 201

Run Number 102-07 Sample Number 36

Parameter	Data	Units	Parameter	Data	Un
Flow	0.8983	Lbm/Hr	Tc	1220.6	De
Thrust (meas)	76.920	Mlbf	Tv	146.3	De
Thrust (corr)	76.920	Mlbf	Tf	72.2	De
Fuel Pressure	266.71	Psia	T1	1191.5	De
Chamber Pressure	159.53	Psia	T2	1247.8	De
Alt. Pressure	108.56	Millorr	T3	1934.5	De
E ht.	46.313	Volts	T4	1797.8	De
I ht.	11.327	Amps	T5	1777.6	De
Ht. Power	524.574	Watts	T6	228.8	De
Ht. Resistance	4.06380	Ohms			
Ht Temperature	4111.24	Degs F	CD	0.00000	
P/Mdot	4.63452	MJ/Kg	Cf	1.31843	
Gas Temperature	0.0	Degs.	C*	7537.4	Ft
PSP	6.80610		Reynolds#		G
ISP	308.9	Secs.			

Remarks \_\_\_\_\_

End of Run Data \_\_\_\_\_

OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4330

## HI PERF. RESISTO-JET PERFORMANCE MAP (VER 1.00)

## PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-08      Sample Number 10

DATE: 01-30-1990      09:53:50  
SEQ 43

Parameter	Data	Units	Parameter	Data	Units
Flow	1.1688	Lbm/Hr	Tc	1259.9	Degs F
Thrust (mass)	105.293	Mbf	Tv	131.0	Degs F
Thrust (corr)	105.293	Mbf	Tf	66.7	Degs F
Fuel Pressure	293.66	PsiA	T1	1220.4	Degs F
Chamber Pressure	213.47	PsiA	T2	1277.1	Degs F
Alt. Pressure	134.58	Atorr	T3	1960.9	Degs F
E ht.	54.813	Volts	T4	1772.4	Degs F
I ht.	13.078	Amps	T5	1795.1	Degs F
Ht. Power	716.827	Watts	T6	232.1	Degs F
Ht. Resistance	4.19141	Ohms			
Ht Temperature	4216.67	Degs F	CD	0.00000	
P/Mdot	4.78578	MJ/Kg	Cf	1.34710	
Gas Temperature	0.0	Degs.	C*	7629.5	Ft/Sec
PSP	6.79560		Reynolds#	0	
ISP	319.4	Secs.			

Remarks \_\_\_\_\_  
End of run Date \_\_\_\_\_

OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4330

## HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

## PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-08      Sample Number 16

Parameter	Data	Units	Parameter	Data	Un
Flow	1.1040	Lbm/Hr	Tc	1249.8	De
Thrust (meas)	97.684	M1bf	Tv	138.1	De
Thrust (corr)	97.084	M1bf	Tf	69.0	De
Fuel Pressure	272.71	Psia	T1	1215.9	De
Chamber Pressure	200.41	Psia	T2	1272.8	De
Alt. Pressure	124.57	Mtorr	T3	1962.0	De
E ht.	52.569	Volts	T4	1786.8	De
I ht.	12.572	Amps	T5	1796.9	De
Ht. Power	660.923	Watts	T6	238.6	De
Ht. Resistance	4.18130	Ohms			
Ht Temperature	4206.29	Degs F	CD	0.00000	
P/Mdot	4.75117	MJ/Kg	Cf	1.32439	
Gas Temperature	0.0	Degs.	C*	7704.5	Ft
PSF	6.79536		Reynolds#	0	
ISP	317.1	Secs.			

( Remarks \_\_\_\_\_

End of run Data \_\_\_\_\_

CLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4330

HIGH PERFORMANCE RESISTO-JET MAF (Ver 1.00)

PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-08      Sample Number 22

Parameter	Data	Units	Parameter	Data	Units
Flow	1.0258	Lbm/Hr	Tc	1238.5	Degs F
Thrust (meas)	89.797	MlbF	Tv	142.6	Degs F
Thrust (corr)	89.797	MlbF	Tf	70.4	Degs F
Fuel Pressure	250.66	PsiA	T1	1268.2	Degs F
Chamber Pressure	168.65	PsiA	T2	1264.8	Degs F
Alt. Pressure	117.71	Mtorr	T3	1971.3	Degs F
E ht.	50.965	Volts	T4	1806.2	Degs F
I ht.	12.143	Amps	T5	1804.5	Degs F
Ht. Power	618.844	Watts	T6	233.6	Degs F
Ht. Resistance	4.19716	Ohms			
Ht Temperature	4222.58	Degs F	CD	0.00000	
P/Mdot	4.76786	MJ/Kg	Cf	1.30138	
Gas Temperature	0.0	Degs.	C*	7805.4	Foot/Sec
PSP	6.87693		Reynolds#	0	
ISP	315.7	Secs.			

Remarks \_\_\_\_\_  
End of run Data \_\_\_\_\_

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OF FOUR QUARTER

OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-08      Sample Number 28

DATE: 01-30-1970      11:45  
SEQ

Parameter	Data	Units	Parameter	Data	Unit
Flow	0.9574	Lbm/Hr	Tc	1227.3	Deg
Thrust (meas)	83.389	M1bf	Tv	146.0	Deg
Thrust (corr)	83.389	M1bf	Tf	71.4	Deg
Fuel Pressure	231.75	Psia	T1	1200.6	Deg
Chamber Pressure	177.67	Psia	T2	1256.7	Deg
Alt. Pressure	112.25	Mtorr	T3	1965.4	Deg
E ht.	48.996	Volts	T4	1812.6	Deg
I ht.	11.732	Amps	T5	1801.2	Deg
Ht. Power	574.815	Watts	T6	235.7	Deg
Ht. Resistance	4.17627	Ohms			
Ht. Temperature	4201.12	Degs F	CD	0.00000	
F/Mdot	4.76507	MJ/kg	Cf	1.28331	
Gas Temperature	0.0	Degs.	C*	7876.2	Ft
PSP	6.88005		Reynolds#	0	
ISP	314.2	Secs.			

Remarks \_\_\_\_\_

End of run Data \_\_\_\_\_

OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-08      Sample Number 34

Parameter	Data	Units	Parameter	Data	Units
Flow	0.8950	Lbm/Hr	Tc	1229.2	Degs F
Thrust (meas)	77.647	M1bF	Tv	146.1	Degs F
Thrust (corr)	77.647	M1bF	Tf	72.2	Degs F
Fuel Pressure	214.70	Psia	T1	1193.1	Degs F
Chamber Pressure	167.68	Psia	T2	1249.4	Degs F
Alt. Pressure	109.66	Motorr	T3	1973.8	Degs F
E ht.	47.568	Volts	T4	1832.5	Degs F
I ht.	11.350	Amps	T5	1810.9	Degs F
Ht. Power	539.879	Watts	T6	219.4	Degs F
Ht. Resistance	4.19117	Ohms			
Ht Temperature	4216.43	Degs F	CD	0.00000	
P/Mdot	4.78756	MJ/kg	Cf	1.26621	
Gas Temperature	0.0	Degs.	C*	7952.0	Ft/Sec
PSP	6.93911		Reynolds#	0	
ISP	313.0	Secs.			

Remarks \_\_\_\_\_  
End of run Data \_\_\_\_\_

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OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

PERFORMANCE MAP ON MODIFIED ETT S/N 001

Run Number 102-08 Sample Number 40

Parameter	Data	Units	Parameter	Data	Uni
Flow	0.8399	Lbm/Hr	Tc	1220.2	Deg
Thrust (meas)	72.379	MlbF	Tv	149.7	Deg
Thrust (corr)	72.379	MlbF	Tf	72.6	Deg
Fuel Pressure	197.63	Psia	T1	1187.5	Deg
Chamber Pressure	153.16	Psia	T2	1243.4	Deg
Alt. Pressure	105.70	Mtorr	T3	1973.5	Deg
E ht.	45.981	Volts	T4	1544.5	Deg
I ht.	10.982	Amps	T5	1814.4	Deg
Ht. Power	504.969	Watts	T6	224.5	Deg
Ht. Resistance	4.18683	Ohms			
Ht. Temperature	4211.97	Degs F	CD	0.00000	
P/Mdot	4.77159	MJ/Kg	Cf	1.25148	
Gas Temperature	0.0	Degs.	C*	7992.0	Ft,
PSP	6.96233		Reynolds#	0	
ISP	310.9	Secs.			

Remarks \_\_\_\_\_  
End of run Data \_\_\_\_\_

OLIN/ROCKET RESEARCH CO.

DATE: 01-31-1990 12:19:43

CONTRACT NUMBER 121587-4330

HI PERF. RESISTO-JET PERFORMANCE MAF (Ver 1.00)

HEALTH CHECK ON MODIFIED ETT S/N 001 AT START OF LIFE TEST

Run Number 103-01 Sample Number 8

Parameter	Data	Units	Parameter	Data	Units
Flow	1.4832	Lbm/Hr	Tc	1264.9	Degs F
Thrust (meas)	83.415	MlfF	Tv	118.2	Degs F
Thrust (corr)	83.415	MlfF	Tf	63.4	Degs F
Fuel Pressure	323.46	PsiA	T1	1232.8	Degs F
Chamber Pressure	186.14	PsiA	T2	1267.8	Degs F
Alt. Pressure	162.90	Atmorr	T3	1172.1	Degs F
El. ht.	0.002	Volts	T4	1237.5	Degs F
I. ht.	0.216	Amps	T5	1177.0	Degs F
Ht. Power	0.000	Watts	T6	174.8	Degs F
Ht. Resistance	0.00000	Ohms			
Ht Temperature	0.00	Degs F	CD	0.00000	
P/Mdot	0.00000	MJ/Kg	Cf	1.22635	
Gas Temperature	0.0	Degs.	C*	5326.4	Ft/Sec
PSF	0.00000		Reynolds#	0	
ISP	203.0	Secs.			

Remarks \_\_\_\_\_

End of run Data \_\_\_\_\_

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COLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4330

HI PERF. RESISTO-JET PERFORMANCE MAF (Ver 1.00)

HEALTH CHECK ON MODIFIED ETT S/N 001 AT START OF LIFE TEST

Run Number 103-01      Sample Number 16

DATE: 01-31-1990      13:03  
Seq:

Parameter	Data	Units	Parameter	Data	Un
Flow	1.2511	Lbm/Hr	Tc	1256.0	De
Thrust (meas)	106.826	M1bf	Tv	135.0	De
Thrust (corr)	106.826	M1bf	Tf	68.0	De
Fuel Pressure	323.68	Psia	T1	1229.7	De
Chamber Pressure	229.08	Psia	T2	1286.2	De
Alt. Pressure	136.70	Mtorr	T3	1345.2	De
Eht.	50.070	Volts	T4	1675.7	De
I ht.	12.971	Amps	T5	1726.4	De
Ht. Power	649.442	Watts	T6	248.9	De
Ht. Resistance	3.86028	Ohms		0.00000	
Ht Temperature	3876.44	Degs F	CD		
P/Mdot	4.11991	MJ/Kg	Cf	1.27490	
Gas Temperature	0.0	Degs.	C*	7771.6	Ft
PSP	6.06841		Reynolds#	0	
ISP	307.9	Secs.			

Remarks

End of run Data

COLIN/ROCKET RESEARCH CO.

DATE: 01-31-1970 14:54:08

SEQ/3

CONTRACT NUMBER 121587-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (ver 1.00)

PERFORMANCE MAP AN MODIFIED ETT S/N 001 AT 300 PSIA

Run Number 103-02 Sample Number 8

Parameter	Data	Units	Parameter	Data	Units
Flow	1.1445	Lbm/Hr	Tc	1263.1	Degs F
Thrust (meas)	100.743	Mlbf	Tv	141.2	Degs F
Thrust (corr)	100.743	Mlbf	Tf	70.7	Degs F
Fuel Pressure	298.65	Psia	T1	1223.1	Degs F
Chamber Pressure	220.51	Psia	T2	1277.9	Degs F
Alt. Pressure	126.41	Millibar	T3	1962.8	Degs F
Eht.	53.452	Volts	T4	1779.9	Degs F
I ht.	12.760	Amps	T5	1796.5	Degs F
Ht. Power	683.251	Watts	T6	250.4	Degs F
Ht. Resistance	4.18378	Ohms		0.00000	
Ht Temperature	4206.42	Degs F	CD		
F/Mdot	4.73778	KJ/Kg	Cf	1.24898	
Gas Temperature	0.0	Degs.	C*	8177.3	Ft/Sec
FSP	6.76990		Reynolds#	0	
ISP	317.4	Secs.			

Remarks \_\_\_\_\_

End of run Data \_\_\_\_\_

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Quality

## WORK SHEET

Date 4-9-5

Page \_\_\_\_\_

## USING DRY ARGON + Hydrogen + Ammonia

RUN #	SEG#	Flow Lbs/hr	Heater temp					
104-01	10	0.174	0°					
104-02	1	Dry Argon	2.2944	3809	VS116	DRY ARGON	16	
104-02	2		2.3275	3904				
104-02	3		2.2892	4023				
104-02	4		2.3107	4117				
		4 - 10 - 90						
104-03	5	2.3108	4188					
104-03	6	2.2943	0					
104	4 - 11 - 90		0					
104-04	7	2.6840	0					
		4 - 30 - 90						
104-05	10	DATA						
104-06	8	0.1306	3806	VS116	HYDROGEN	1		
104-07	9	0.1342	3889					
104-07	10	0.1316	4004					
104-07	11	0.1302	4102					
		5 - 15 - 90						
104-08	12	0.1304	4211					
104-08	13	0.1306	4228	→ 3 min run only				
104-09	14	0.1294	4320					
104-09	15	0.2602	4004					
		5 - 16 - 90						
104-10	16	0.2587	4113					
104-10	17	0.2623	4202					
		5 - 16 - 90						
104-11	18	0.5119	3708	VS116	AMMONIA	1		
104-11	19	0.5030	3778	→ 20 min run time				
104-11	20	0.5030	3896					
		5 - 17 - 90						
104-12	21	0.5005	4003					
104-12	22	0.4980	4091					
104-13	23	0.4967	4198					
105-1	1			WAugmented.				
	2			NG				
105-2	3			4000				
105-2	4			4100				

OLIN/ROCKET RESEARCH CO.

DATE: 04-07-1990 09:47:33

CONTRACT NUMBER 121587-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

## PERFORMANCE MAP USING ARGON

Run Number 104-01

Sample Number 10

Parameter	Data	Units	Parameter	Data	Unit:
Flow	2.3474	Lbm/Hr	Tc	67.7	Degs
Thrust (meas)	34.137	MlbF	Tv	65.9	Degs
Thrust (corr)	34.137	MlbF	Tf	69.3	Degs
Fuel Pressure	14.65	PsiA	Ti	69.5	Degs
Chamber Pressure	82.00	PsiA	T2	67.6	Degs
Alt. Pressure	71.93	Atorr	T3	64.0	Degs
E ht.	-0.027	Volts	T4	66.1	Degs
I ht.	0.003	Amperes	T5	66.7	Degs
Ht. Power	-0.006	Watts	T6	65.6	Degs
Ht. Resistance	6.00000	Ohms			
Ht. Temperature	0.00	Degs F	CD		
P/Mdot	0.00000	MJ/kg	Cf	1.13747	
Gas Temperature	0.0	Degs.	Cg	1462.7	FT/S
PSP	-0.00000		Reynoldst	5	
ISP	32.5	Secs.			

Remarks \_\_\_\_\_

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OLIN/ROCKET RESEARCH CO.

DATE: 04-09-1990 09:49:

CONTRACT NUMBER 121587-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

## PERFORMANCE MAP USING ARGON

Run Number 104-01 Sample Number 12

Parameter	Data	Units	Parameter	Data	Units
Flow	2.6788	Lbm/Hr	Tc	67.6	De
Thrust (meas)	39.685	MlbF	Tv	65.5	De
Thrust (corr)	39.685	MlbF	Tf	70.0	De
Fuel Pressure	14.67	PsiA	T1	69.5	De
Chamber Pressure	93.76	PsiA	T2	69.6	De
Alt. Pressure	77.18	Mtorr	T3	63.9	De
E ht.	-0.010	Volts	T4	66.4	De
I ht.	-0.002	Amps	T5	66.7	De
Ht. Power	0.000	Watts	T6	64.7	De
Ht. Resistance	0.00000	Ohms			
Ht. Temperature	0.00	Degs F	CD	0.00000	
P/Mdot	0.00000	MJ/Kg	C†	1.15623	
Gas Temperature	0.0	Degs.	C*	1465.6	Ft
PSF	0.00000		Reynolds#	0	
ISP	53.5	Secs.			

Remarks \_\_\_\_\_

CLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121567-4330

HI FERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

## PERFORMANCE MAP USING DRY ARGON

Run Number 104-02      Sample Number 8

Parameter	Data	Units	Parameter	Data	Unit
Flow	2.2944	Lbm/Hr	Tc	95.8	Degs
Thrust (mes)	78.418	NlbF	Tv	75.0	Degs
Thrust (corr)	78.418	NlbF	Tf	76.3	Degs
Fuel Pressure	14.49	PsiA	T1	126.1	Degs
Chamber Pressure	166.84	PsiA	T2	177.6	Degs
Alt. Pressure	71.14	Atorr	T3	1023.0	Degs
E ht.	28.813	Volts	T4	954.8	Degs
I ht.	7.592	Amps	T5	521.4	Degs
Ht. Power	218.764	Watts	T6	114.8	Degs
Ht. Resistance	3.79497	Ohms			
Ht Temperature	3809.33	Degs F	CD	0.00000	
P/Mdot	0.75674	MJ/Kg	Cf	1.28453	
Gas Temperature	0.0	Degs.	C*	3035.8	Ft/s
PSF	2.78613		Reynolds#	0	
ISP	123.2	Secs.			

Remarks SEQ 1    @ 3800 of 2.3 lbf/lb

End of run Data \_\_\_\_\_

CLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121557-4336

## HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

## PERFORMANCE MAP USING DRY ARGON

Run Number 104-02 Sample Number 19

Parameter	Data	Units	Parameter	Data	L
Flow	2.3275	Lbm/Hr	Tc	104.3	I
Thrust (meas.)	80.318	MlbF	Tv	89.3	F
Thrust (corr)	80.318	MlbF	Tf	81.4	I
Fuel Pressure	14.49	PsiA	Ti	133.4	I
Chamber Pressure	170.53	PsiA	T2	105.2	I
Alt. Pressure	71.31	Atorr	T3	1043.7	I
E ht.	27.514	Volts	T4	974.9	I
Ht. Power	7.593	Amps	T5	520.5	D
Ht. Power	224.098	Watts	T6	136.2	D
Ht. Resistance	3.88705	Ohms			
Ht. Temperature	3903.94	Degs F	CD	0.00000	
P/Mdot	0.76415	MJ/Kg	Cf	1.28696	
Gas Temperature	0.0	Degs.	C*		
PSP	2.78664		Reynold#	3107.6	F
ISP	124.4	Secs.		0	

Remarks Seq 2 @ 3900°F 2.5 lbf/in  
End of run Data

CLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

## PERFORMANCE MAP USING DRY ARGON

Run Number 104-02      Sample Number 29

Parameter	Data	Units	Parameter	Data	Units
Flow	2.2892	Lbm/Hr	Tc	103.4	Degs
Thrust (meas)	79.596	Mlbf	Tv	86.9	Degs
Thrust (corr)	79.996	Mlbf	Tf	79.7	Degs
Fuel Pressure	14.47	PsiA	T1	133.9	Degs
Chamber Pressure	170.37	PsiA	T2	188.0	Degs
Alt. Pressure	71.01	Atorr	T3	1069.6	Degs
E ht.	30.098	Volts	T4	1002.6	Degs
I ht.	7.518	Amps	T5	498.4	Degs
Ht. Power	226.239	Watts	T6	132.9	Degs
Ht. Resistance	4.00324	Ohms			
Ht Temperature	4023.33	Degre F	CD	0.00000	
P/Mdot	0.78454	MJ/Kg	Cf	1.28297	
- Gas Temperature	0.0	Degre.	C*	3158.8	Ft/Sq
PSP	2.82519		Reynolds#		
ISP	126.0	Secs.		2	

Remarks SEQ 3 @ 4000 °F = 2.3 lbm/hr

End of run Data

ORIGINAL PAGE  
OF  
FOUR

OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

## PERFORMANCE MAP USING DRY ARGON

Run Number 104-02

Sample Number 39

Parameter	Data	Units	Parameter	Data	Units
Flow	2.3107	Lbm/Hr	Tc	166.9	De
Thrust (meas)	80.638	M1bf	Tv	95.2	De
Thrust (corr)	80.639	M1bf	Tf	63.6	De
Fuel Pressure	14.46	PsiA	T1	139.4	De
Chamber Pressure	172.47	PsiA	T2	194.3	De
Alt. Pressure	70.05	Atorr	T3	1038.7	De
Elt. Power	30.689	Volts	T4	10.23 .5	De
I ht.	7.495	Amps	T5	534.9	De
Ht. Power	230.025	Watts	T6	141.2	De
Ht. Resistance	4.07435	Ohms			
Ht. Temperature	4116.94	Degs F	CD	0.00000	
P/Mdot	0.79009	MJ/Kg	Cf	1.27749	
Gas Temperature	0.0	Degs.	Cx	3165.0	Ft
PSP	2.84907		Reynolds#		V
ISP	125.8	Secs.			

Remarks  
End of run Data

SEQ 4 @ 4100 of 23 / Sy/6a

OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

## PERFORMANCE MAP USING DRY ARGON

Run Number 104-03      Sample Number 39

Parameter	Date	Units	Parameter	Date	Units
Flow	2.3108	Lbm/Hr	Tc	109.5	Degs
Thrust (meas)	78.800	MlbF	Tv	100.1	Degs
Thrust (corr)	78.800	MlbF	Tf	97.6	Degs
Fuel Pressure	14.00	PsiA	Ti	135.4	Degs
Chamber Pressure	167.91	PsiA	T2	181.6	Degs
Alt. Pressure	68.70	Atmorr	T3	963.5	Degs
E ht.	29.837	Volt	T4	999.9	Degs
I ht.	7.166	Amps	T5	482.0	Degs
Ht. Power	213.813	Watts	T6	139.2	Degs
Ht. Resistance	4.16377	Ohms			
Ht. Temperature	4188.27	Degs F	CD	0.00000	
P/Mdot	0.73435	MJ/Kg	Cf	1.28229	
Gas Temperature	0.0	Degs.	C*	3084.0	Ft/Sq
PSP	2.71001		Reynolds#	0	
ISP	122.9	Secs.			

Remarks SEQ 5 @ 4200 ft 2.3 lb/in²

End of run Date \_\_\_\_\_

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NEXT PAGE  
QUALITY

OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

## PERFORMANCE MAP USING DRY ARGON

Run Number 104-03      Sample Number 45

Parameter	Data	Units	Parameter	Data	U
Flow	2.2943	Lbm/Hr	Tc	78.8	D
Thrust (meas)	33.424	Nlbf	Tv	80.5	D
Thrust (corr)	33.424	M1bf	Tf	79.4	D
Fuel Pressure	13.71	PsiA	T1	60.4	D
Chamber Pressure	82.25	PsiA	T2	60.4	D
Alt. Pressure	67.10	Atmorr	T3	74.6	D
E Ht.	-0.016	Voltts	T4	76.9	D
I Ht.	0.002	Amps	T5	77.4	D
Ht. Power	-0.000	Watts	T6	77.2	D
Ht. Resistance	0.00000	Ohms			
Ht Temperature	0.00	Degs F	CD		
P/Mdot	0.00000	MJ/Kg	Cf	1.11209	
Gas Temperature	0.0	Degs.	C*	1521.6	F
PSP	-0.00000		Reynolds#		G
ISP	52.6	Secs.			

Remarks SEQ 6 Change ment  
End of run DataOriginal page  
of four pages

CLIN/ROCKET RESEARCH CO.

DATE: 04-11-1952 07:08:33

CONTRACT NUMBER 121587-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

PERFORMANCE MAP USING DRY ARGON

Run Number 104-04 Sample Number 4

Parameter	Date	Units	Parameter	Data	Units
Flow	2.6840	Lbm/Hr	Tc	70.9	Degs F
Thrust (meas)	39.266	MlbF	Tv	70.5	Degs F
Thrust (corr)	39.266	MlbF	Tf	71.0	Degs F
Fuel Pressure	14.64	Psi <sub>a</sub>	T1	73.0	Degs F
Chamber Pressure	94.63	Psi <sub>a</sub>	T2	72.9	Degs F
Alt. Pressure	74.57	Atm <sub>air</sub>	T3	77.4	Degs F
E ht.	-0.014	Volt <sub>s</sub>	T4	69.0	Degs F
I ht.	0.002	Amps	T5	74.1	Degs F
Ht. Power	-0.0005	Watt <sub>s</sub>	T6	69.7	Degs F
Ht. Resistor <sub>ave</sub>	0.00000	Ohms			
Ht. Temperature	0.03	Degs F	CD		
Fl/Mdot	0.00000	lb <sub>s</sub> /kg	C <sub>f</sub>		
Gas Temperature	0.0	Degs.	C <sub>g</sub>		
FSP	-0.00200		Reynold <sub>s</sub> #		
ISP	52.3	Secs.			

Remarks Seq 7 change noted

End of run Date

*On time*

OLIN/ROCKET RESEARCH CO.

DATE: 04-30-1960 13:21

CONTRACT NUMBER 121587-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

## PERFORMANCE MAP WITH DRY HYDROGEN

Run Number 104-05 Sample Number 6

Parameter	Data	Units	Parameter	Data	Un
Flow	0.1326	Lbm/Hr	Tc	69.0	De
Thrust (meas)	6.489	Mlbf	Tv	67.4	De
Thrust (corr)	6.489	Mlbf	Tf	68.4	De
Fuel Pressure	14.57	PsiA	Ti	71.5	De
Chamber Pressure	22.93	PsiA	T2	71.1	De
Alt. Pressure	101.71	Millirr	T3	62.1	De
)E ht.	2.000	Volts	T4	66.9	De
I ht.	0.019	Amps	T5	68.6	De
Ht. Power	0.000	Watts	T6	66.2	De
Ht. Resistance	0.00000	Ohms			
Ht. Temperature	6.00	Degs F	CD	0.00000	
P/Mdot	0.00000	MJ/Kg	Cf	0.78962	
Gas Temperature	0.0	Degs.	C*	7336.6	Ft
PSP	0.00000		Reynolds#	0	
ISP	180.1	Secs.			

Remarks \_\_\_\_\_

This is not EOR Data

OLIN/ROCKETT RESEARCH CO.

DATE: 04-30-1970 15:47:03

CONTRACT NUMBER 1245E7-4332

HI-PEPF. REBILTO-GET PERFORMANCE MAP (Ver 1.00)

## PERFORMANCE MAP USING HYDROGEN

Run Number 104-06 Sample Number 17

Parameter	Date	Units	Parameter	Date	Units
Flow	6.1206	lbm/hr	Tc	23.8	Degs F
Thrust (meas)	16.897	lbF	Tv	25.1	Degs F
Thrust (cont'd)	16.8572	M1bf	Tf	80.2	Degs F
Fuel Pressure	24.73	psia	T1	95.6	Degs F
Chamber Pressure	39.51	psia	T2	96.7	Degs F
Air. Temperature	95.90	degC	T3	142.6	Degs F
Int. Int.	27.155	Voltts	T4	472.6	Degs F
Int. Amps	0.219	Amps	T5	253.1	Degs F
Int. Power	223.489	Watts	T6	101.6	Degs F
Int. Resistance	3.30373	Ohms			
Int. Temperature	3805.77	Degs F	CD	0.02200	
P/Mdot	1.3.55237	lb/kg	Cf	1.15605	
Gas Temperature	2.0	Degs.	Cx	12840.9	ft/sec
PSP	13.33337		Reynolds#	2	
ISP	461.4	Secs.			

Remarks \_\_\_\_\_  
End of run Data \_\_\_\_\_

SEQ 8 @ 3800 ft 13 lbf/in

GLEN/REGG RESEARCH CO.

DATE: 25-24-1992 06:56

DIRECTOR: 225-537-4772

HT PIERS, FREESTO-3-TEC PERFORMANCE MAP (Ver. 1.02)

## PERFORMANCE MAP USING HYDROGEN

Run Number 104-07 Sample Number 17

Parameter	Date	Units	Parameter	Date	Un
Flow	07-23-92	Lbm/Hr	Tc	02-1	De
Thrust (gross)	17-243	N.LB	Tv	02-5	De
Thrust (net)	17-243	NLB	Tf	75-9	De
Flight pressure	14-33	PSIA	T1	22-6	De
Chamber pressure	12-19	PSIA	T2	57-4	De
Alt. (feet)	123-61	Feet	T3	03-5	De
He. Alt.	27-753	Voltages	T4	404-7	De
T. Int.	0-234	Sample	T5	572-4	De
Ht. Power	231-062	Watts	T6	99-9	De
H2. Resistor	3-274-92	Ohms			
Ht. Temperature	3689-72	Degs F	CD	0-00200	
P/Mdot	13-69276	kg/kg	CF	1-195-42	
Gas Temperature	0.0	Degs	C*	1.2707-02	Fit
Pop	43-15233		Reynolds*	0	
Tdp	470-2	Secs.			

Run Number  
E72 C7 Date 2/26/92

SEG 9 @ 3900°F 13 May/92

Original  
of 2001 Pac  
Run

OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

## PERFORMANCE MAP USING HYDROGEN

Run Number 104-07 Sample Number 32

Parameter	Data	Units	Parameter	Data	Units
Flow	0.1316	Lbm/Hr	Tc	82.9	Degs F
Thrust (mass)	17.179	MlbF	Tv	86.0	Degs F
Thrust (corr)	17.179	MlbF	Tf	73.1	Degs F
Fuel Pressure	14.01	PsiA	T1	64.9	Degs F
Chamber Pressure	40.00	PsiA	T2	70.7	Degs F
Alt. Pressure	110.93	AtmOr	T3	594.6	Degs F
E ht.	28.573	Volts	T4	524.0	Degs F
I ht.	8.230	Amps	T5	293.6	Degs F
Ht. Power	235.156	Watts	T6	103.7	Degs F
Ht. Resistance	3.47170	Ohms			
Ht Temperature	4004.31	Degs F	CD	0.22000	
P/Mdot	14.17700	MJ/Kg	Cf	1.18270	
Gas Temperature	0.0	Degs.	Cx	12896.4	ft/sec
PSP	13.56470		Reynolds#	0	
ISP	474.1	Secs.			

Remarks SEG 10 @ 4000 ft .1316 m/hh

End of run Data

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OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121597-4339

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.0E)

## PERFORMANCE MAP USING HYDROGEN

Run Number 104-07      Sample Number 47

Parameter	Data	Units	Parameter	Data	U
Flow	0.1302	Lbm/Hr	Tc	85.3	D
Thrust (meas)	17.265	Mlbf	Tv	88.9	D
Thrust (corr)	17.265	Mlbf	Tf	79.8	D
Fuel Pressure	13.99	Psi <sub>a</sub>	T1	67.4	D
Chamber Pressure	4.0.13	Psi <sub>a</sub>	T2	93.8	D
Alt. Pressure	100.57	Miller	T3	629.6	D
Eht.	29.193	Volts	T4	559.4	D
I ht.	8.214	Amps	T5	305.2	D
Ht. Power	239.784	Watts	T6	107.2	D
Ht. Resistance	3.55425	Ohms			
Ht. Temperature	4102.02	Degs F	CD	0.00000	
P/Mdot	14.61624	MJ/Kg	Cf	1.18472	
Gas Temperature	2.0	Degs.	C*	13000.5	F*
FSP	13.75468		Reynolds#		D
ISP	401.7	Secs.			

Remarks SEQ 11 @ 4100°F - 13 sec/h  
End of run Data

BLIN/ROCKET RESEARCH CO.

DATE: 35-15-1990 69:38:59

CONTRACT NUMBER 21457-A736

HI-PEFF. RESISTO-JET PERFORMANCE MAP (Ver 1.20)

## PERFORMANCE MAP USING HYDROGEN

Run Number 104-08 Sample Number 28

Parameter	Date	Units	Parameter	Data	Units
FLOW	0.15.04	Lbm/Hr	Tc	96.9	Degs F
THRUST (mass)	17.593	Mlbf	Tv	91.8	Degs F
THRUST (corr.)	17.593	Mlbf	Tf	81.6	Degs F
Fuel Pressure	14.91	Psi	T1	98.8	Degs F
Chamber Pressure	42.41	Psi	T2	98.4	Degs F
Airt. Pressure	104.32	MDot	T3	652.9	Degs F
Ht.	32.374	Volt	T4	879.7	Degs F
I ht.	0.329	Amperes	T5	345.4	Degs F
Ht. Power	252.923	Watt	T6	109.2	Degs F
Ht. Resistance	7.6545	Ohms			
Ht. Temperature	4212.42	Degs F	CD	3.2E0000	
P/MDot	15.35363	MJ/Kg	Cf	1.16924	
Gas Temperature	0.0	Degs.	C*	13476.7	ft./Sec
PEP	14.25878		Reynolds#	0	
ISP	489.8	Secs.			

Remarks SEQ 12 @ 4200 ft. 13 lbf/in

End of run Date

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GLENROCKET RESEARCH CO.

CONTRACT NUMBER 122587-4330

H2 PERFO. RESISTO-JET PERFORMANCE MAP (Ver 1.00)

PERFORMANCE MAP USING HYDROGEN

Run Number 104-03 Sample Number 30

Parameter	Data	Units	Parameter	Data	Units
Flow	2.1306	Lbm/Hr	Tc	85.7	De
Thrust (mass)	17.461	Mlbf	Tv	85.4	De
Thrust (corr)	17.461	Mlbf	Tf	78.2	De
Fuel Pressure	14.72	PSia	T1	85.4	De
Chamber Pressure	41.31	PSia	T2	90.9	De
Alt. Pressure	134.57	Atmos	T3	610.0	De
Alt.	36.778	Volts	T4	530.6	De
I Alt.	8.406	Amps	TS	120.7	De
Ht. Power	258.749	Watts	TS	95.6	De
Ht. Resistance	3.64133	Ohms			
Ht Temperature	4228.70	Degs F	CD	0.00000	
P/Mdot	15.72593	MJ/Kg	Cf	1.16319	
Gas Temperature	6.0	Degs.	C*	13429.2	Ft
FSP	14.69215		Reynolds#	0	
ISP	485.5	Secs.			

Remarks  
End of Run DataSEQ 13 @ 4300 OF 1316m/hr  
(only 3 runs)

CLIN/ROCKET RESEARCH CO.

DATE: 05-15-1970 11:49:57

CONTRACT NUMBER 124587-4330

NO PERC. REGISTO-JET PERFORMANCE MAP (Ver 1.00)

## PERFORMANCE MAP USING HYDROGEN

Run Number 104-09 Sample Number 17

Parameter	Data	Units	Parameter	Data	Units
Flow	0.1294	LBm/HR	Tc	87.0	Degs F
Thrust (meas)	17.225	Nlbf	Tv	91.3	Degs F
Thrust (corr)	17.229	Nlbf	Tf	81.1	Degs F
Fuel Pressure	14.41	PsiA	T1	89.2	Degs F
Chamber Pressure	41.25	PsiA	T2	96.3	Degs F
Ait. Pressure	104.25	Atm	T3	695.0	Degs F
E ht.	30.993	Volts	T4	623.6	Degs F
I ht.	0.291	Amps	T5	339.2	Degs F
Ht. Power	256.947	Watts	T6	112.6	Degs F
Ht. Resistance	3.73835	Ohms			
Ht. Temperature	4349.63	Degs F	CD	0.00000	
P/Ndot	1.5.75588	MJ/Kg	Cf	1.14766	
Gas Temperature	0.0	Degs.	C*	13526.1	Ft/Sec
PSP	14.76669		Reynolds#	0	
ISP	483.3	Secs.			

Remarks SEP 14 @ 4300 ft

• End of run Data • 13 16m / hr

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CLINT/ROCKET RESEARCH CO.

DATE: 05-15-1970 14:45:

CONTRACT NUMBER 154587-4330

HT PERFOR. RESISTO-JET PERFORMANCE MAP (Ver 1.02)

PERFORMANCE MAP USING HYDROGEN

Run Number 104-09 Sample Number 33

Parameter	Date	Units	Parameter	Date	Uni
Flow	2.2602	Lbm/Hr	Tc	90.3	Dec
Thrust (mass)	37.842	M1bf	Tv	102.1	Dec
Thrust (corr)	37.842	M1bf	Tf	82.0	Dec
Fuel Pressure	14.53	PSia	Tz	85.6	Dec
Chamber Pressure	31.25	PSia	Tx	87.3	Dec
Alt. Pressure	105.44	PSI	Tg	80.5	Dec
E. Hr.	42.654	Volts	Td	351.7	Dec
I. Hr.	13.272	Amperes	Ts	266.5	Dec
Ht. Power	425.853	Watts	Tb	132.0	Dec
Ht. Resistance	3.47149	Ohms		0.02000	
Ht. Temperature	4284.06	Degs F	Cd		
P/mdot	15.95059	MJ/kg	Cf	1.27074	
Gas Temperature	2.0	Degs.	Cx	13350.4	Ft
PSF	13.72147		Reynolds#	0	
ISF	527.3	Secs.			

Remarks SEQ 15 @ 4000 °F • 26 lsm/hr

End of run Date

OLIN/ROCKET RESEARCH CO.

DATE: 08-16-1990    08:17:37

CONTRACT NUMBER 121567-4330

HI PERF. RESISTO-JET PERFORMANCE MAP (ver 1.0C)

## PERFORMANCE MAP USING HYDROGEN

Run Number 104-10    Sample Number 16

Parameter	Data	Units	Parameter	Data	Units
Flow	0.2587	Lbm/Hr	Tc	85.1	Degs F
Thrust (meas)	33.293	Nlbf	Tv	95.1	Degs F
Thrust (corr.)	38.295	Nlbf	Tf	77.0	Degs F
Fuel Pressure	14.62	Psi	Tl	81.4	Degs F
Chamber Pressure	81.88	Psi	T2	82.4	Degs F
Alt. Pressure	172.86	Micro	T3	84.6	Degs F
Ht.	43.563	Vents	T4	83.2	Degs F
Ht.	12.926	Arc ft	T5	87.4	Degs F
Ht. Power	552.572	Watts	T6	105.8	Degs F
Ht. Resistance	3.56355	Ohms			
Ht Temperature	4113.04	Degs F	CD	0.00000	
P/Mdot	16.34495	lb/kg	Cf	1.2E534	
Gas Temperature	0.0	Degs.	C*	13436.2	Ft/Sec
PSP	13.81212		Reynolds#	0	
ISP	536.8	Secs.			

Remarks \_\_\_\_\_  
End of Run Data \_\_\_\_\_

SEP 16 @ 4100 °F    0.26 lbf/in/lb

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CLIN/ROCKET RESEARCH CO.

DATE: 05-16-1990 PAGE 2

CONTRACT NUMBER: 121557-4332

H1 PERC - RESISTO-JET PERFORMANCE MAP (Ver 1.0Z)

PERFORMANCE MAP USING HYDROGEN

Run Number 104-10 Sample Number 32

Parameter	Date	Unit	Parameter	Date
Flow	2.2623	Lbm/Hr	TG	89.8
Thrust (msec)	30.799	Nlbf	TV	103.2
Thrust (lbft)	30.797	Nlbf	TF	80.9
Gas Pressure	5.0152	PSia	T1	95.0
Chamber Pressure	83.07	PSia	T2	66.7
Alt. Pressure	1.0311	PSia	T3	93.9
Alt. Height	44.867	Volts	T4	57.6
Alt. Height	12.313	Amps	T5	28.6
Alt. Power	35.1733	Watts	T6	43.8
Alt. Resistance	3.63915	Ohms		
Alt Temperature	4202.45	Degs F	CD	0.00000
Product	15.69426	MJ/kg	Cf	1.27523
Gas Temperature	2.0	Degs.	C*	13525.4
PSP	14.12503		Reynolds#	0
ISP	536.4	Secs.		

Remarks: SEQ 17 @ 4200 °F . 26 /Sec/

Enc of Run Data

OLIN/ROCKET SESSIONS 50

CONTRACT NUMBER 124587-4330

DATE: 08-14-1990

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## H2 PERF - RESISTO-JET PERFORMANCE MAP (Ver 1.02)

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Run Number 104-11 Sample Number 19

Parameter	Data	Units	Parameter	Data	Units
Flow	0.5119	Lbm/Hr	Tc	89.1	Degs F
Thrust (mesa)	34.905	Mile	Tv	34.7	Degs F
Thrust (corr)	34.925	Mile	Tf	77.4	Degs F
Fuel Pressure	14.74	PSI@6	Tz	434.0	Degs F
Chamber Pressure	66.26	PSI@6	Tg	427.0	Degs F
Alt. Pressure	59.60	PSI@6	Tx	65.0	Degs F
E-Tc	32.455	Volt	T4	50.4	Degs F
E-Tv	37.452	Volt	T5	54.9	Degs F
E-Tf	257.968	Volt	T6	54.9	Degs F
Alt. Pressure	59.745	PSI@6	T7	44.9	Degs F
E-Tc	3.22237	Ohms			
Ht. Temperature	2738.94	Degs F			
P/Wt/G	4.45321	MJ/KG	Cf	0.000023	
Gas Temperature	2.0	Degs.	C*	1.44278	
Reynolds#	5493.7	Ft/Sec			
Secs.	246.4				6

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SEP 18 @ 3800°F . 50 lbm/lhr

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OF POOR QUALITY

OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121537-4332

HI PEER, REACTO-JET PERFORMANCE MAP (VER 2, 6/6)

PERFORMANCE MAP USING ANOMA

Run Number 104-11      Sample Number 25

Parameter	Data	Units	Parameter	Data	Units
FLOW	0.5032	Lbm/HR	Tc	75.3	Degs
Thrust (mass)	33.564	Nlbf	Tv	93.4	Degs
Thrust (corr)	33.564	Nlbf	Tf	80.4	Degs
Fuel Pressure	14.72	PSia	T1	100.6	Degs
Chamber Pressure	64.52	PSia	T2	135.8	Degs
Ait. Pressure	56.80	PSI	T3	60.9	Degs
Alt. in. ft.	32.374	Voltas	T4	617.5	Degs
Ht. Power	9.250	Amperes	T5	373.6	Degs
Ht. Resistance	201.478	Wattts	T6	124.2	Degs
Ht Temperature	3.28743	Cent			
P/Mdot	3777.97	Degs F	CD	0.00000	
Gas Temperature	4.43653	MJ/KG	Cf	1.42529	
PPR	8.34670	Degs	C*	5442.6	Ft/Sec
ISP	241.1	Secs.	Reynolds#	0	

Remarks  
End of run date

RUN 104-11      SEP 19 @ 3800 °F      .50 lbf/l

Original  
Or  
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Quality

BLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4332

DATE: 05-16-1970 13:57:51

HT PERC., RESISTO-JET PERFORMANCE MAP (Ver 1.02)

PERFORMANCE MAP USING ANONIA

Part Number 124-11 Sample Number 40

Parameter	Date	Units	Parameter	Data	Units
Flow	0.5030	Lbm/Hr	Tc	98.6	Degs F
Thrust (meas)	33.402	MIBF	Tv	96.6	Degs F
Thrust (corr)	33.422	MIBF	Tf	82.3	Degs F
Fuel Pressure	4.473	PSia	T1	143.5	Degs F
Chamber Pressure	64.48	PSia	T2	142.2	Degs F
Ait. Pressure	85.81	PSia	T3	719.7	Degs F
E Ht.	31.114	Volts	T4	650.1	Degs F
Ht.	9.206	Atms	T5	384.6	Degs F
Ht. Power	256.443	Watts	T6	127.2	Degs F
Ht. Resistance	3.38000	Ohms			
Ht. Temperature	3295.82	Degs F	CD	0.02000	
S/Ndot	4.51957	MJ/Kg	Cf	1.41682	
Gas Temperature	6.2	Degs.	C*	5440.0	Ft./Sec
ESP	6.54462	Reynoldst#		0	
ESP	239.9	Secs.			

Remarks  
End of run Data

SEQ 20 @ 3900°F .50 15m/km

Original Part  
E75

CETN/MERCHANT RESEARCH CO.

DATE: 05-17-1990 060:34

CONTINUATION NUMBER 124587-4332

PERF. PREDICT-O-GET PERFORMANCE MAP (VER 1.00)

PERFORMANCE MAP USING AMMONIA

Run Number 104-12 Sample Number 17

Parameter	Date	Units	Parameter	Date	Units
Flow	0.5025	Lbm/Hr	Tc	90.8	Deg
Thrust (mass)	33.526	M1bf	Tv	37.5	Deg
Thrust (crry)	33.526	M1bf	Tf	75.0	Deg
Fuel Pressure	14.72	PsiA	T4	105.9	Deg
Chamber Pressure	65.24	PsiA	T2	137.5	Deg
Airs. Pressure	87.35	PSI	T3	74.3	Deg
E. Rte.	21.970	Volt	T4	674.6	Deg
I. Rte.	7.214	Amps	T5	402.0	Deg
Alt. Power	294.627	Watts	T6	121.6	Deg
Alt. Resistance	3.47073	Ohms			
Alt. Temperature	4203.16	Degs F	CD	0.0232	
P/Mdot	4.67183	MJ/Kg	Cf	1.40241	
Gas Temperature	2.0	Degs.	C%	5529.4	F/t/
FSP	0.75497		Reynolds#	0	
ISP	242.0	Secs.			

SEP 21 @ 4000 °F .50 lbm/hr

Emissions Date

E-76

CETN/MERCHANT  
RESEARCH CO.  
PAGE 4

BLINK/ROCKET RESEARCH CO.

DATE: 05-17-1962 (9:42:47)

CONTRACT NUMBER 424527-072

NO. PEPF - RESISTO-CET PERFORMANCE MAP (Ver 1.0C)

PERFORMANCE MAP USING ANTENNA

Run Number 124-12      Sample Number 32

Parameter	Date	Units	Parameter	Date	Units
Flow	0.4783	Lbm/Hr	Tc	94.8	Degs F
Thrust (meas)	34.734	M1BF	Tv	92.7	Degs F
Thrust (corr)	34.734	M1BF	Tf	77.7	Degs F
Fuel Pressure	15.73	PsiA	T1	140.9	Degs F
Chamber Pressure	67.08	PsiA	T2	141.5	Degs F
Ait. Pressure	55.27	Atmorr	T3	763.3	Degs F
E. ht.	33.234	Volts	T4	672.8	Degs F
I. ht.	9.316	AmPS	T5	404.6	Degs F
Ht. Power	307.729	Watts	T6	125.8	Degs F
Ht. Resistance	3.54655	Ohms			
Ht Temperature	4291.70	Degs F	CD	0.00000	
P/Mdot	4.92406	kg/kg	Cf	1.41825	
Gas Temperature	30.0	Degs	Cx	5716.5	Ft/Sec
PSP	6.82758		Reynolds*	0	
ZSP	252.0	Secs			

Remarks \_\_\_\_\_  
End of run Date \_\_\_\_\_

SEP 22 @ 4100 ft . 916 m/hr

Cylindrical  
Antenna

OLIN/ROCKET RESEARCH CO.

CONTRACT NUMBER 121587-4332

HI PERF. RESISTO-JET PERFORMANCE MAP (Ver 1.03)

PERFORMANCE MAP USING AMMONIA

DATE: 05-17-1992

14:54

Run Number 104-12      Sample Number 48

Parameter	Data	Units	Parameter	Data	Units
Flow	0.4967	Lbm/Hr	Tc	96.1	%
Thrust (meas)	35.172	M1bf	Tv	94.0	D
Thrust (corr)	35.172	M1bf	Tf	78.3	D
Fuel Pressure	14.74	Psi	T1	112.7	D
Chamber Pressure	63.06	Psi	T2	144.7	D
Ait. Pressure	39.37	Mtorr	T3	791.4	D
E ht.	31.964	Volt	T4	724.4	D
I ht.	9.343	Amps	T5	427.1	D
Ht. Power	317.345	Watt	T6	128.5	D
Ht. Resistance	3.63526	Ohms			
Ht Temperature	4197.65	Degs F	CD	0.00000	
P/Mdot	5.06982	MJ/Kg	Cf	1.41498	
Gas Temperature	2.0	Degs	C*	5816.5	F
Psp	8.98984		Reynolds#	2	
ISP	255.8	Secs.			

Remarks  
End of run Data  
SEP 23 @ 4200 °F .50 lba/s

**APPENDIX F**  
**Immersed Heater Injector Redesign**

**Applied Technology  
Division  
TRW Space & Technology  
Group**

One Spacel Park  
Redondo Beach, CA 90278  
213.535.4321

**Mail Station:** 01/2260

89.K536.1-051  
12 July 1989

Mr. Mark Simon  
Rocket Research Company  
11441 Willows Rd. N.E.  
Redmond, WA 98073 - 9709

Dear Mark:

The long-awaited heat exchanger analysis report has finally emerged and I am enclosing a copy for your reading pleasure. I have requested additional copies with color reproduction of figures and will supply one to NASA-Lewis as soon as available.

Sincerely,



Rein Grabbi

Enclosures (30)

ORIGINAL PAGE  
OF POOR QUALITY

# Interoffice Correspondence

## TRW Space & Technology Group



Subject: Recommendations for  
Increasing the ISP of the  
HiPEHT Thruster

To: Rein Grabbi  
cc: H. W. Behrens  
D. E. Fritz  
R. L. Sackheim

Date: July 11, 1989  
89.K314.3-017

From: J. E. Eninger  
P. D. Lohn  
E. Y. Wong

Location/Phone:  
R1/1038  
Ext. 20478

### 1. BACKGROUND AND SUMMARY

We were asked to make recommendations for design modifications to the HiPEHT thruster for increasing the specific impulse. The ground rules were to focus on increasing the heat transfer between the electrical heater coil and the gas. Also, it is beyond the scope of the task to consider any changes in the design of the heating coil itself.

The first step was to review the thermodynamics, heat transfer, and gasdynamics of the current thruster design. The results are presented in Section 2. An estimate was made of the potential for increasing specific impulse by enhancing the heat-transfer coefficient between the heater coil and the gas, and by operating the coil at a higher temperature. The estimate shows, for example, that for the mass flow rate of  $3 \times 10^{-4}$  lb/s and for the filament temperature maintained at the operating temperature of 3700°F, a doubling of the heat-transfer coefficient would increase the specific impulse from the current measured value of 306 seconds to 328 seconds. If, in addition, the filament temperature can be increased to 5000°F, then the potential specific impulse is estimated to be 380 seconds. Although 5000°F is probably above the practical operating temperature for the coil (tungsten deposits were observed in the nozzle throat after

tests at this temperature), the calculations serve as an upper bound for the specific impulse that can be expected.

The nature of the flow in the heating section has a major effect on the heat transfer from the coil. For efficient heat transfer, one wants:

- 1) a high gas velocity over the coil,
- 2) good mixing so all the gas is uniformly processed by the coil,
- 3) small-scale disturbances in the main flow<sup>1</sup>, and
- 4) minimum radial flow at the ends of the chamber that allows gas to leave unheated.

Internal rotating flows are complicated and difficult to predict. It is often advisable to evaluate candidate designs with a flow-simulation experiment. Such an experiment, however, is beyond the scope of the present task. Insight into the features of the flow was obtained from visualization experiments of similar flows reported in the literature. The understanding obtained from the body of past work was crucial in formulating our recommendations. A critical review of the rotational-flow literature pertaining to the current situation is presented in Section 3. References cited include a videotape of classic flow-visualization experiments of rotational flows. A copy in VHS format is included as part of this memorandum.

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<sup>1</sup> "It should be noted that, if the turbulence level in the oncoming air is increased by placing a grid or some other type of turbulence promoter upstream of the cylinder, the surface conductance may increase by as much as 50 percent."

It is challenging to recommend improvements in the current design, which works well and is the result of much excellent engineering development at TRW. Recent improvements in the mechanical support of the heating coil, however, should allow it to withstand more severe flow-field forces at higher operating temperatures. This will allow more heat to be transferred to the gas.

Our recommendation concerns the method by which momentum is transferred from the injected gas to the vortex flow. The specific recommendations are presented in Section 4. The current design employs a narrow slit extending over only 20% of the chamber length that injects at high velocity exactly tangential to the inner cylindrical surface. Wall friction can be expected to immediately dissipate as much as 50% of the injected momentum (Section 3). We recommend that the slit be replaced with a row of 8 orifices, equally spaced along the entire length of the heating chamber. The orifices are sized so the total discharge is identical to that of the slit. They should be angled inward 26° from the tangential direction. The intent is to maximize the momentum transferred to the vortex by minimizing the momentum dissipated by the wall. In addition to higher velocity over the coil, discrete injection promotes mixing of the gas and introduces small-scale disturbances that should further enhance heat transfer. Extending the row of orifices to the ends of the chamber lessens the effect of end-wall boundary layer and should minimize the amount of flow leaving the chamber without being processed by the coil.

## 2. ESTIMATE OF THE POTENTIAL SPECIFIC-IMPULSE INCREASE

An analysis was carried out 1) to assess the heat-transfer from the coil to the gas in the current design, and 2) to estimate the potential for increasing specific impulse by increasing the heat-transfer to the gas.

### 2.1 Gas Properties Entering the Heating Chamber

The first step in the analysis is to calculate the gas temperature and composition leaving the catalyst bed and entering the heating chamber. We assume that hydrazine is decomposed adiabatically in the catalyst bed according to the reaction:



where  $\alpha$  is the  $\text{NH}_3$  dissociation fraction, which is defined as the fraction of the maximum possible  $\text{NH}_3$  that decomposes. Thus  $\alpha = 1$  corresponds to complete dissociation of  $\text{NH}_3$ , and  $\alpha = 0$  corresponds to no dissociation (maximum possible  $\text{NH}_3$  in the gas). We focused on a dissociation fraction of 0.6, which is representative of the Rocket Research Corporation (RRC) catalyst bed. The gas temperature exiting the catalyst bed is found by 1) calculating the net heat of reaction for decomposition as a function of temperature, and 2) selecting the temperature where the net is zero.

Table I gives the enthalpies and heats of formation required for the calculation, as well as the heat of reaction for  $\text{N}_2\text{H}_4$  decomposition as a function of temperature. The calculation shows that for a dissociation fraction  $\alpha = 0.6$ , the heat of reaction vanishes at  $1186^\circ\text{K}$  ( $1675^\circ\text{F}$ ). The composition exiting the catalyst bed is given by:

$N_2H_4$ , HEAT OF FORMATION AT 298K: 12.1 KCAL/MOL

THERMODYNAMIC DATA FOR  $N_2H_4$ , DECOMPOSITION WITH A  $NH_3$  DECOMPOSITION FRACTION OF 0.60

TABLE I

T (K)	$N_2$	$H_2$	$NH_3$	HEAT OF FORMAT'N REACTION ENTHALPY	$C_p$	$C_p/C_v$	ENTHALPY	$C_p$	$C_p/C_v$	KCAL/MOL	$NH_3$ DISSOCIATED
298	0.000	0.000	-10.970	17.951	0.000					0.000	

	$H(T) - H(298K)$	HEAT OF	HEAT OF	WITH $NH_3$							
400	0.710	0.707	0.903	-11.482	16.100	0.750	7.47	1.36	0.708	6.98	1.40

	$N_2$	$H_2$	$NH_3$	HEAT OF	HEAT OF	WITH $NH_3$					
500	1.413	1.406	1.867	-11.919	14.232	1.508	7.72	1.35	1.408	7.07	1.39

	$N_2$	$H_2$	$NH_3$	HEAT OF	HEAT OF	WITH $NH_3$					
600	2.152	2.106	2.909	-12.282	12.294	2.293	7.89	1.34	2.121	7.07	1.39

	$N_2$	$H_2$	$NH_3$	HEAT OF	HEAT OF	WITH $NH_3$					
700	2.853	2.808	4.027	-12.582	12.294	2.293	8.06	1.33	2.823	7.10	1.39

	$N_2$	$H_2$	$NH_3$	HEAT OF	HEAT OF	WITH $NH_3$					
800	3.596	3.514	5.215	-12.824	8.315	3.085	8.32	1.31	3.541	7.23	1.38

	$N_2$	$H_2$	$NH_3$	HEAT OF	HEAT OF	WITH $NH_3$					
900	4.355	4.226	6.470	-13.016	6.235	4.750	8.54	1.30	4.269	7.32	1.38

	$N_2$	$H_2$	$NH_3$	HEAT OF	HEAT OF	WITH $NH_3$					
1000	5.129	4.944	7.787	-13.163	4.104	5.614	8.75	1.30	4.269	7.32	1.38

	$N_2$	$H_2$	$NH_3$	HEAT OF	HEAT OF	WITH $NH_3$					
1100	5.917	5.670	9.163	-13.271	1.921	6.499	8.95	1.28	5.752	7.52	1.36

	$N_2$	$H_2$	$NH_3$	HEAT OF	HEAT OF	WITH $NH_3$					
1200	6.718	6.404	10.592	-13.343	0.310	7.403	9.14	1.28	6.509	7.61	1.35

	$N_2$	$H_2$	$NH_3$	HEAT OF	HEAT OF	WITH $NH_3$					
1300	7.529	7.148	12.071	-13.385	0.326	9.326	9.14	1.28	6.509	7.71	1.35

	$N_2$	$H_2$	$NH_3$	HEAT OF	HEAT OF	WITH $NH_3$					
1400	8.350	7.902	13.596	-13.402	4.906	9.266	9.49	1.26	8.051	7.82	1.34

	$N_2$	$H_2$	$NH_3$	HEAT OF	HEAT OF	WITH $NH_3$					
1500	9.179	8.668	15.162	-13.399	7.269	10.224	9.66	1.26	8.038	7.92	1.33

	$N_2$	$H_2$	$NH_3$	HEAT OF	HEAT OF	WITH $NH_3$					
1600	10.015	9.446	16.765	-13.381	9.670	11.198	9.81	1.26	8.02	1.02	1.33

	$N_2$	$H_2$	$NH_3$	HEAT OF	HEAT OF	WITH $NH_3$					
1700	11.707	10.233	18.402	-13.347	12.106	12.185	9.94	1.25	8.104	1.02	1.32

	$N_2$	$H_2$	$NH_3$	HEAT OF	HEAT OF	WITH $NH_3$					
1800	11.707	11.030	20.066	-13.303	14.572	13.185	10.13	1.24	11.256	8.10	1.32

	$N_2$	$H_2$	$NH_3$	HEAT OF	HEAT OF	WITH $NH_3$					
1900	12.560	12.560	21.868	-13.250	17.104	14.211	10.16	1.24	12.07	8.22	1.32

	$N_2$	$H_2$	$NH_3$	HEAT OF	HEAT OF	WITH $NH_3$					
2000	13.418	12.651	23.465	-13.191	19.585	15.217	10.19	1.24	12.07	8.22	1.32

	$N_2$	$H_2$	$NH_3$	HEAT OF	HEAT OF	WITH $NH_3$					
2100	14.280	14.280	24.745	-13.097	20.877	17.104	10.16	1.24	14.280	8.40	1.31

	$N_2$	$H_2$	$NH_3$	HEAT OF	HEAT OF	WITH $NH_3$					
2200	15.146	14.307	26.950	-13.053	24.698	17.290	10.46	1.24	15.146	8.46	1.31

	$N_2$	$H_2$	$NH_3$	HEAT OF	HEAT OF	WITH $NH_3$					
2300	16.015	16.015	28.723	-12.973	27.288	18.340	10.54	1.23	16.015	8.52	1.30

	$N_2$	$H_2$	$NH_3$	HEAT OF	HEAT OF	WITH $NH_3$					
2400	16.886	15.993	30.515	-12.877	29.899	19.398	10.63	1.23	16.291	8.58	1.30

	$N_2$	$H_2$	$NH_3$	HEAT OF	HEAT OF	WITH $NH_3$					
2500	17.761	16.848	32.326	-12.797	32.532	20.466	10.71	1.23	17.761	8.64	1.30

	$N_2$	$H_2$	$NH_3$	HEAT OF	HEAT OF	WITH $NH_3$					
2600	18.638	17.708	34.154	-12.697	35.182	21.540	10.78	1.22	18.638	8.68	1.30

	$N_2$	$H_2$	$NH_3$	HEAT OF	HEAT OF	WITH $NH_3$					
2700	19.517	18.575	35.975	-12.692	37.851	22.622	10.92	1.22	19.517	8.73	



Table I also includes the enthalpy, specific heat at constant pressure, and the ratio of specific heats, both for the mixture exiting the catalyst bed at  $\alpha = 0.6$ , and for a mixture after the remaining  $\text{NH}_3$  dissociates in the heating chamber. These data are used in the performance calculations below.

## 2.2 Performance of the Current Design

The electrical heating of the gas in the current design was found by calculating the gas total-temperature increase that gives the experimentally measured specific impulse. For this purpose, the isentropic-flow formula for specific impulse was modified by a multiplicative efficiency factor to account for the neglected viscous effects. A factor of 0.90 was found by taking the ratio of the specific impulse computed in Reference 2 by a full viscous analysis of the thruster to the specific impulse computed by the simple isentropic-flow formula.

The gas-heating computations were carried out based on two assumptions regarding  $\text{NH}_3$  dissociation in the heating chamber:

- 1) no additional  $\text{NH}_3$  dissociation, and 2) complete  $\text{NH}_3$  dissociation. The actual situation was assessed by computing, according to Reference 3, the time for 99% dissociation as a function of temperature. The results are given in Table II.

TABLE II - 99% DISSOCIATION TIME FOR  $\text{NH}_3$

Temp. (°K)	1000	1200	1500	1700	2000
Temp. (°F)	1340	1700	2240	2600	3140
$\tau_{99}$ (s)	$3.0 \times 10^8$	$3.6 \times 10^8$	4.3	0.06	$5 \times 10^{-4}$

The gas residence time was estimated based on an average axial velocity in the heater section given by

$$u_a = \dot{m}/(\rho A) ,$$

where  $\dot{m}$  is the mass flow rate,  $\rho$  is the gas density, and  $A$  is the cross-sectional area of the heater section. The residence time is given by

$$\tau = L/u_a ,$$

where  $L$  is the length of the heater section. For the nominal  $3.0 \times 10^{-4}$  lb/s mass flow, the residence time is estimated to be 0.001 seconds. Therefore, for gas temperatures below  $1700^{\circ}\text{K}$  ( $2600^{\circ}\text{F}$ ), the assumption of no additional  $\text{NH}_3$  dissociation is the proper one. Ammonia dissociation in the heater section becomes significant as  $2000^{\circ}\text{K}$  ( $3140^{\circ}\text{F}$ ) is approached.

Results of the performance assessment of the current design are summarized in Table III for the particular operating condition specified by a mass flow of  $3.0 \times 10^{-4}$  lb/s and a dissociation fraction of  $\alpha = 0.6$  out of the catalyst bed. Table III is actually a portion of a PC spreadsheet used to carry out the computations.

Section 1 of Table III gives conditions just before the gas is electrically heated. The total temperature is  $1186^{\circ}\text{K}$  ( $1675^{\circ}\text{F}$ ). Note that the unheated gas gives a predicted specific impulse of 227 seconds.

Section 2 of Table III gives the heat transfer for the case when all the  $\text{NH}_3$  in the heater section dissociates. The total temperature of the gas is adjusted until the predicted specific impulse matches the RRC measured value of 306 seconds. The final

TABLE III  
PERFORMANCE ASSESSMENT OF CURRENT DESIGN AT MASS FLOW OF  
 $3 \times 10^{-4}$  LB/S AND DISSOCIATION FRACTION OF  $\alpha = 0.6$

NOZZLE EXIT MACH NO.:	7.2
NOZZLE EFFICIENCY FACTOR:	0.9
<b>1. CONDITIONS BEFORE GAS IS ELECTRICALLY HEATED</b>	
1.000 N <sub>2</sub> H <sub>4</sub> → 0.533 NH <sub>3</sub> + 0.733 N <sub>2</sub> + 1.200 H <sub>2</sub>	
MOLECULAR WEIGHT:	12.97 G/MOLE
GAS TEMPERATURE:	1186 K
ENTHALPY:	7.28 KCAL/MOL
GAMMA (CP/CV):	1.28
SPECIFIC IMPULSE:	2231 M/S = 227 SECONDS
<b>2. CONDITIONS AFTER GAS ELECTRICALLY HEATED (TOTAL NH<sub>3</sub> DISSOCIATION)</b>	
RRC MEASURED	
SPECIFIC IMPULSE:	3003 M/S = 306 SECONDS
MOLECULAR WEIGHT:	10.67 G/MOLE
INITIAL TEMPERATURE:	1186 K
NH <sub>3</sub> HEAT OF FORMATION:	-13.33 KCAL/MOL
ENTHALPY (NO NH <sub>3</sub> ):	6.40 KCAL/MOL
FINAL TEMPERATURE:	1922 K
FINAL ENTHALPY:	12.28 KCAL/MOL
GAMMA (C <sub>P</sub> /C <sub>V</sub> ):	1.32
HEAT TRANSFER TO GAS NH <sub>3</sub> DISSOCIATION:	0.030 KCAL/S = 127 WATTS
SENSIBLE HEATING:	0.062 KCAL/S = 258 WATTS
TOTAL:	0.092 KCAL/S = 384 WATTS
FILAMENT TEMPERATURE:	2311 K
ENTHALPY:	15.53 KCAL/MOLE
HX EFFECTIVENESS:	0.729
<b>3. CONDITIONS AFTER GAS ELECTRICALLY HEATED (NO ADDITIONAL NH<sub>3</sub> DISSOCIATION)</b>	
RRC-MEASURED	
SPECIFIC IMPULSE:	3003 M/S = 306 SECONDS
FINAL TEMPERATURE:	1959 K
FINAL ENTHALPY:	14.80 KCAL/MOL
GAMMA (C <sub>P</sub> /C <sub>V</sub> ):	1.24
SENSIBLE HEATING:	0.079 KCAL/S = 331 WATTS
FILAMENT TEMPERATURE:	2311 K
ENTHALPY:	18.46 KCAL/MOLE
HX EFFECTIVENESS:	0.673
NUMBER OF TRANSFER UNITS:	1.12

temperature is 1922°K (3000°F). To reach this temperature, 127 watts are needed to dissociate the NH<sub>3</sub>, and 258 watts of sensible heating are needed to raise the temperature. The total of 384 watts is below the experimentally measured input of 530 watts. The difference can be attributed in part to conductive heat loss. The calculations are based on the assumption of no heat transfer to the surroundings from 1) the catalyst chamber, 2) the heating chamber, or 3) the plumbing connecting them. A detailed assessment of the heat loss from the hardware in the test configuration was not made, and we cannot say at this time how much of the difference between the estimated and measured power to the heating section is due to heat losses.

The stated heat-transfer effectiveness of 0.729 is the ratio of the 384 watts transferred to the gas to the amount that would be transferred if the gas were heated to the measured filament temperature of 2311°K (3700°F).

Section 3 of Table III gives the heat transfer for the case when no additional NH<sub>3</sub> in the heater section dissociates. Again, the total temperature of the gas is adjusted until the predicted specific impulse matches the measured value of 306 seconds. The final temperature in this case is 1959°K (3067°F). To reach this temperature, 331 watts of sensible heating are needed. This is only 14% less than the case of total NH<sub>3</sub> dissociation. The heat-transfer effectiveness in this case is 0.673.

The estimated gas temperatures are in a range where it is not clear whether significant NH<sub>3</sub> dissociation will take place in the heating chamber. Table III shows, however, that the performance of the current design, in terms of total temperature of the gas, the total amount of heat transferred to the gas, and the heat-transfer effectiveness, is nearly the same under the widely different assumptions of 1) no additional NH<sub>3</sub>

dissociation and 2) total  $\text{NH}_3$  dissociation. This is in spite of the fact that there is a large difference in the final molecular weights and in final specific-heat ratios under the two assumptions. The simpler assumption is no additional dissociation in the heating chamber. Then, all heat transfer is in the form of sensible heating of the gas. Because of the demonstrated insensitivity of the performance to the assumption employed and for reasons of simplicity, the estimated potential for specific-impulse improvement in the next section is based on the assumption of no additional  $\text{NH}_3$  dissociation. This allows a simple single-pass heat-exchanger model to be employed.

### 2.3 Potential for Specific-Impulse Improvement

An estimate of the improvement in specific impulse by enhancement of heat transfer between the coil and the gas can be made by treating the heating chamber as a simple single-stream heat exchanger. The effectiveness  $\epsilon$ , defined as the actual heat transferred divided by the maximum possible is given by (Reference 4):

$$\epsilon = 1 - e^{-Ntu}, \quad (1)$$

where the  $Ntu$  is the "number of transfer units" given by the dimensionless quantity

$$Ntu = (h A) / (\dot{m} c_p) \quad (2)$$

Here,  $h$  is the overall heat-transfer coefficient between the coil and the gas,  $A$  is the surface area of the coil,  $\dot{m}$  is the mass flow rate, and  $c_p$  is the specific heat of the gas. For a single-pass heat exchanger, the final temperature difference between the heating element and the fluid stream is reduced by a factor of  $\epsilon^{-1} = 0.368$  for every  $Ntu$ . When the effectiveness

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has been calculated, such as in our assessment of the performance of the current design in Table III, then the number of transfer units can be found from

$$N_{tu} = -\ln(1-\epsilon) . \quad (3)$$

For example, in Table III, Section 3 (no additional NH<sub>3</sub> dissociation), for an effectiveness of 0.673, the number of transfer units is 1.12 .

The NTUs are directly proportional to the heat-transfer coefficient  $h$  (Equation 2). Therefore, a specified percentage enhancement in the heat-transfer coefficient results in the same percentage increase in the NTUs, and Equation (1) can be used to calculate the enhanced effectiveness. Then the new gas temperature  $T_{exit}$  exiting the heating chamber can be determined from the definition of effectiveness:

$$\epsilon = [H(T_{exit}) - H(T_{entrance})]/[H(T_{filament}) - H(T_{entrance})] , \quad (4)$$

where  $H(T)$  is the enthalpy of the gas at temperature  $T$ . The exiting gas temperature, along with the molecular weight and ratio of specific heats, is used to calculate the enhanced specific impulse.

The results are shown in Figures 1 and 2 for a mass flow of  $3.0 \times 10^{-4}$  lb/s for two cases: 1) the filament is maintained at the current measured temperature of 3700°F, and 2) the filament is increased to 5000°F.

An estimate for the heat-transfer coefficient for the current design can be obtained from Equation (2) with the following data:

# GAS-TEMPERATURE INCREASE FROM HEAT-TRANSFER ENHANCEMENT

MASS FLOW: 3E-04 LBM/S - FILAMENT: 3700F & 5000F  
EXIT-GAS TEMP: FILAMENT: 3700F FILAMENT 5000F  
ENTRANCE-GAS: 1675F

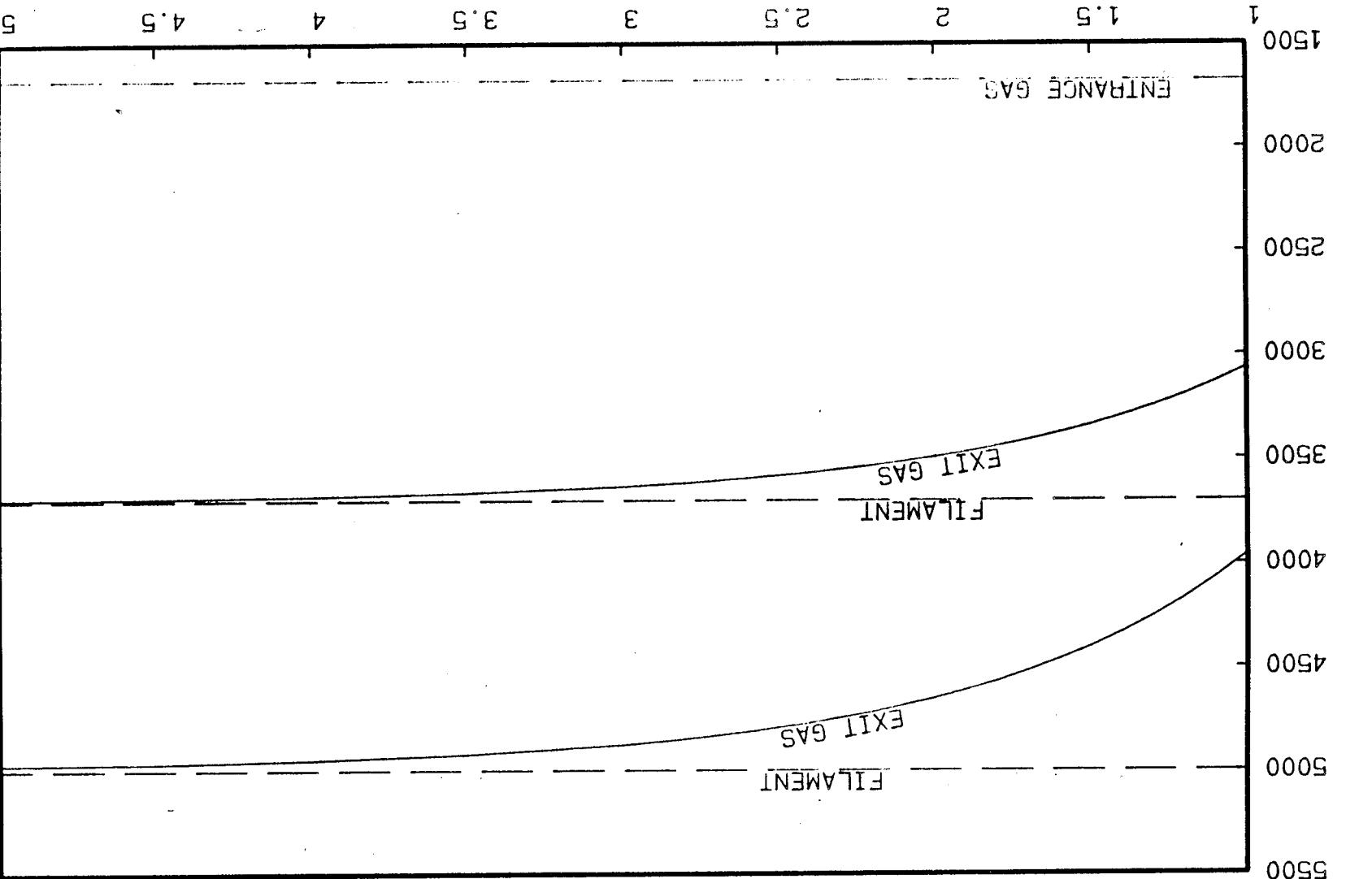


FIGURE 1

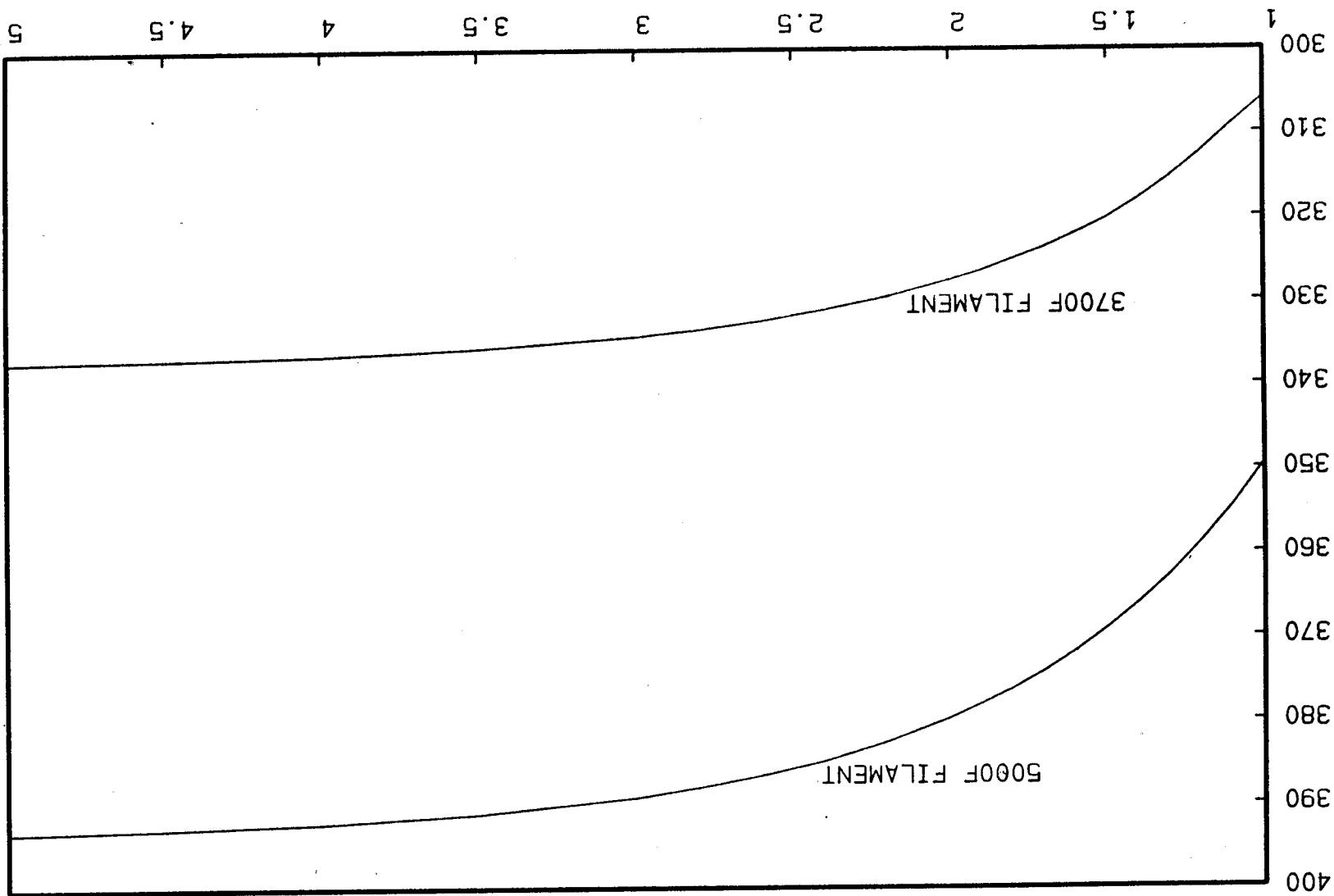
## ISP INCREASE FROM HEAT-TRANSFER ENHANCEMENT

MASS FLOW:  $3E-04 \text{ LBM/S}$  - FILAMENT: 3700F & 5000F

FILAMENT: 3700F  
FILAMENT: 5000F

FILAMENT:

FIGURE 2



Number of Transfer Units . . . . .	$N_{tu}$	=	1.12
Total Wire Surface Area			
of Heating Coil . . . . .	$A$	=	$1.82 \times 10^{-4} \text{ m}^2$
Mass Flow Rate . . . . .	$m$	=	$3.0 \times 10^{-4} \text{ lb/s}$
Specific Heat . . . . .	$c_p$	=	$1.4 \times 10^{-4} \text{ kg/s}$
			$0.73 \text{ kcal/kg-K}$

The calculated heat-transfer coefficient is

$$h = 0.61 \text{ kcal/s-m}^2\text{-K} . \quad (5)$$

If the heat-transfer from the heater element is idealized as resulting from flow over a cylinder, then an estimate can be made of the flow velocity over the coil, and for the heat-transfer enhancement resulting from a velocity increase. The Nusselt number  $Nu_b = h D/k$ , based on the cylinder diameter  $D$ , is related to the Prandtl number  $Pr$  and Reynolds number  $Re_b$  by (Reference 1):

$$Nu_b = 1.1 C Re_b^n Pr^{0.31} \quad (6)$$

where  $C = 0.615$  and  $n = 0.466$  for Reynolds numbers in the range  $40 - 4,000$ . The following average data are used to carry out the calculations:

Gas Conductivity. . . . .	$K$	=	$2.8 \times 10^{-5} \text{ kcal/s-m-K}$
Gas Viscosity . . . . .	$\mu$	=	$2.8 \times 10^{-5} \text{ kg/m-s}$
Gas Density . . . . .	$\rho$	=	$1.2 \text{ kg/m}^3$
Heater-Wire Diameter . . . . .	$D$	=	0.009 in.
			$2.29 \times 10^{-4} \text{ m}$

With these data, we calculate:

Nusselt Number . . . . .	$Nu_D = 5.0$
Reynolds Number . . . . .	$Re_D = 93$
Gas Velocity Over Wire . . . . .	$u = 9.1 \text{ m/s}$

where the gas velocity is calculated from the Reynolds number by  $u = \mu Re_D / (\rho D)$ . The velocity of 9.1 m/s over the coil falls between the estimated velocity of 240 m/s exiting the injection slit and the estimated average axial velocity of 2.6 m/s in the heating chamber. Equation (6) shows that the heat-transfer coefficient varies with the 0.47<sup>th</sup> power of the Reynolds number, and hence of the velocity. Thus to enhance the heat-transfer coefficient by a factor of two, the velocity would have to be increased by a factor of 4.4 from 9.1 m/s to 40 m/s.

A realistic expectation would be to double the velocity over the coil. The heat-transfer coefficient would be increased by the factor  $2^{0.47} = 1.4$ . According to Figure 2, without increasing the filament temperature from its current operating level of 3700°F the expected specific impulse would increase from 306 seconds to 318 seconds. Higher values would result from the beneficial effects of:

- 1) small-scale turbulence induced by discrete injection,
- 2) improvements in preventing flow exiting unheated at the ends of the chamber, and
- 3) operating at a higher filament temperature.

### 3. FLOW-FIELD CONSIDERATIONS

The design modification we propose is to replace the current single-slit injector located axially near the center of the chamber with an array of orifices distributed along the length of the chamber. The orifices are angled inward relative to the tangential direction of the chamber wall to minimizing momentum loss from the jets due to viscous drag on the wall. For the proposed injection configuration, there will be an optimum both for the number of orifices and for the injection angle. The optimal design of the thruster configuration from the heat-transfer standpoint is the one that maximizes the vortex strength.

The optimum angle for injection is a compromise between two opposing factors. For an injection angle  $\beta$  measured inward from the tangential direction, the jet velocity can be decomposed into to a tangential component  $V \cos \beta$  and a radial component  $V \sin \beta$ . Only the tangential component transfers momentum to the vortex, and for this reason  $\beta$  should be as small as possible. On the other hand, if  $\beta$  is too small, momentum is lost to the wall due to viscous drag. An optimum angle could be determined by a flow simulation experiment. For example, vortex strength measured by pressure depression at the core could be found as a function of injection angle. A flow simulation experiment is beyond the scope of the current work. A review of rotating-flow literature, however, produced much data from past experiments that help support the proposed design. A summary is given below, and selected reference to the rotating-flow literature is given at the end.

The friction loss due to the cylindrical wall has been investigated by Kendall and Roschke (References 1 & 3-6). In the hope of reducing momentum loss to the side wall, Roschke had the

jet orifices drilled tangent to a circle having a diameter 92% of the chamber diameter. Although not mentioned by Roschke, the 92%-configuration appears to be consistent with experimental results obtained by Keyes (Reference 7), which showed optimal vortex strength occurs when the injection is tangent to a circle having a diameter approximately 90% of the chamber diameter. For this case, the injection angle is given by

$$\beta = \cos^{-1} (0.90) = 25.8^\circ$$

The penalty paid for angling the jets inward is the reduction of the tangential component of the injection velocity, which in this case is reduced by the factor  $\cos \beta = 0.90$ .

In addition to the angle of injection, the number of orifices must be specified. The proposed design modification is to replace the current 0.008-in. ~~x~~ 0.073-in. single-slit injector an array of 8 orifices distributed along the entire length of the chamber. The center-to-center orifice spacing is 0.045 in. The diameter of the orifices is selected to give the same total mass flow as the single slit. If the discharge coefficient were the same for the orifices as for the slit, then the diameter of the orifices would be 0.00964 in. The actual diameter should be increased in the final mechanical design to account for the smaller discharge coefficient for the orifices than for the slit.

The selection of the number of orifices was based on the maximum number that gives an orifice diameter of the same order as the slit width. There is concern that a larger number of very small diameter orifices would be dominated by viscous effects. The proposed configuration has a spacing-to-diameter ratio of 4.5, which is close to the ratio of 4 used by Roschke in the construction of his vortex tube (Reference 5).

In rotating flows, the primary circumferential flow dominates. An idealized two-dimensional rotating flow with no end walls is relatively simple. In actual hardware, however, the core-flow interaction with the end-wall boundary layers results in complex radial and axial secondary flows (References 9 - 10). The complexity of these secondary flows is illustrated in Figures 6 - 12 reproduced from Reference 9, and in the videotape accompanying this memorandum. Some observations are made below of flow features that pertain to the heating chamber of the current application.

The effect of the chamber end walls is shown in Figure 7. According to the theoretical studies of interaction of the end-wall boundary layers with the rotating core flow (References 11-20), the flow features can be summarized as follows. In the boundary layer, the tangential velocity is slower than the outer flow, thus reducing the centrifugal force which can no longer balance the radial pressure gradient. As a result, fluid flows radial inward at the end walls to a radial position where its reduced centrifugal force can balance the radial pressure gradient. For reasons of continuity, the radial flow is accompanied by an axial flow in a cylindrical layer across the core region. The process repeats at the other end wall. In this way, a serpentine secondary flow pattern is set up as the path by which injected fluid finds its way to the central core and exits. For a given diameter, shorter chambers have more flow reversals. For a length-to-diameter ratio  $L/D = 1$ , which is close to the current application, Figure 7 shows seven axial flow reversals.

The secondary flow pattern is very sensitive to hardware protruding into the chamber. The Figure 11 shows a dramatic change in the flow pattern resulting from small wires of various

diameters stretched across the diameter. Figure 12 shows similar major effects from a probe inserted diametrically into the flow. In the current application, the presence of the heating coil can be expected to have a major effect on the flow. Any analyses that neglect its presence are suspect.

#### 4. SUMMARY OF RECOMMENDATIONS FOR DESIGN MODIFICATIONS

Based on the literature review and private communications with J. Kendall and E. Roschke we would like to recommend the following:

- 1) Replace the current slit injector with an array of 8 circular orifices extending along the entire length of the chamber.
- 2) Size the orifice diameters at approximately 0.010 inches so they have the same discharge as the slit.
- 3) Angle the injection orifices inward  $26^{\circ}$  from the tangential direction to minimize drag on the chamber wall.
- 4) Position the end orifices close to end walls in order to energize the boundary layer and minimize end loss of unheated gas.

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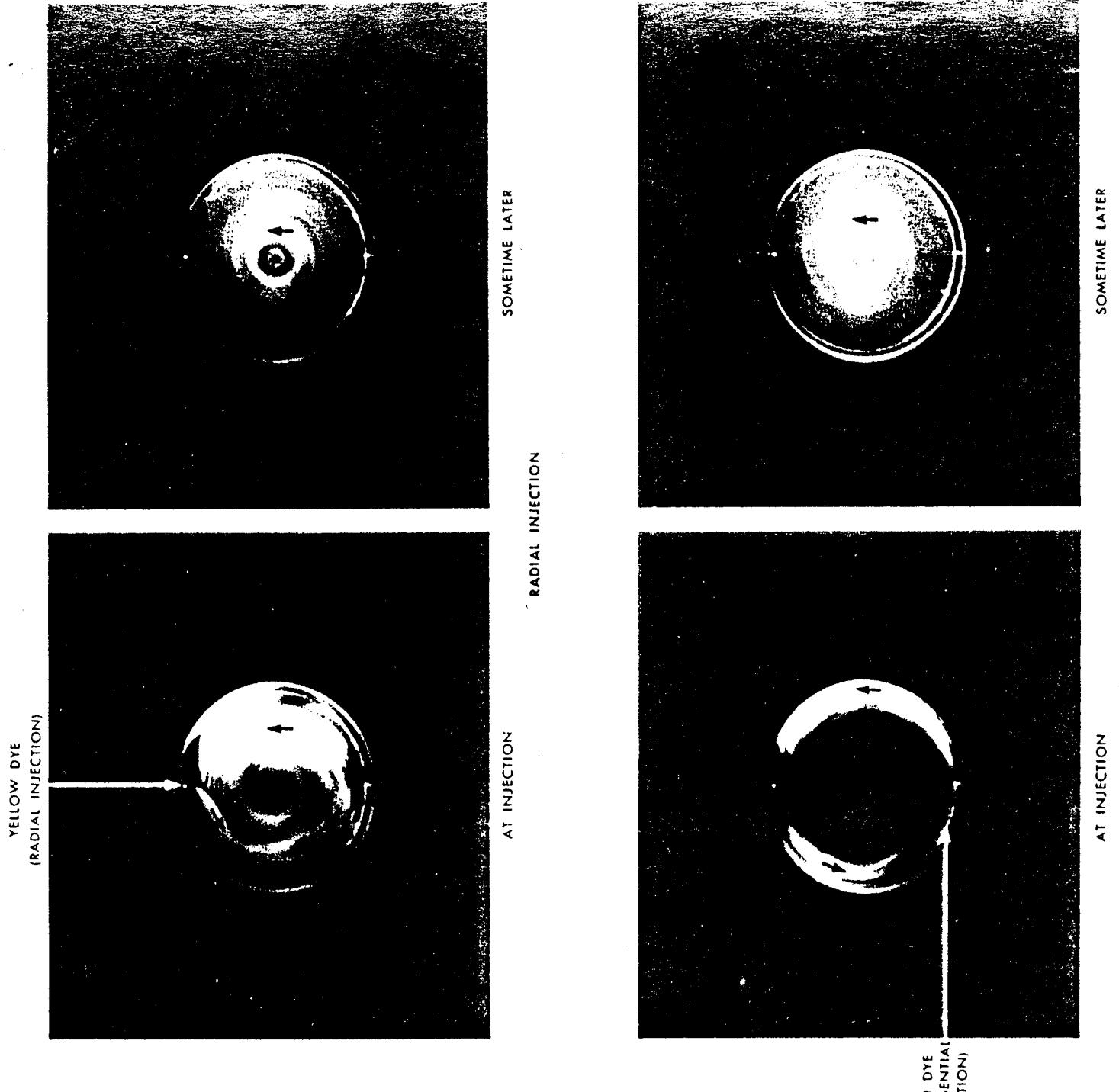
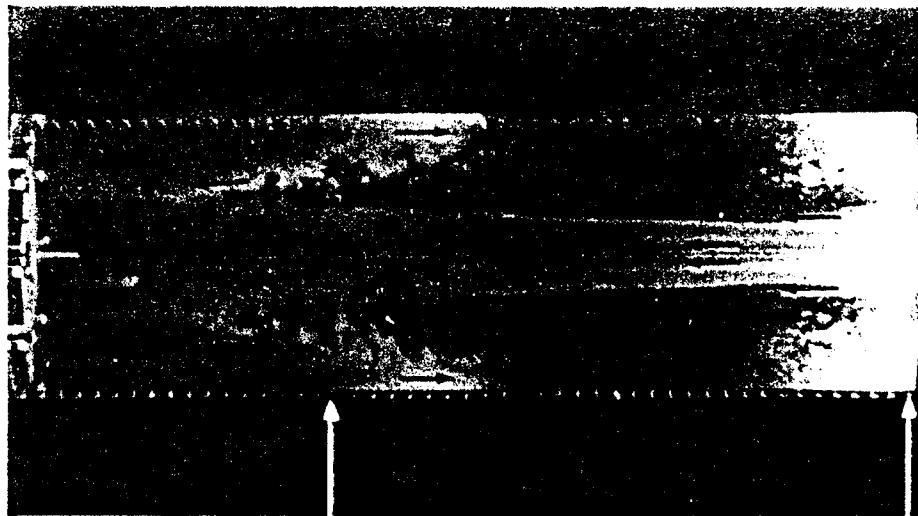


Fig. 6. End views of standard configuration with nominal  $\frac{1}{2}$ -in. exit hole  $L/D = 6$ ;  $P_w - P_d = 5$  psig; illumination through  $\frac{1}{4}$ -in. slit at  $L/D = 2$ ; dye released at  $L/D = 2$

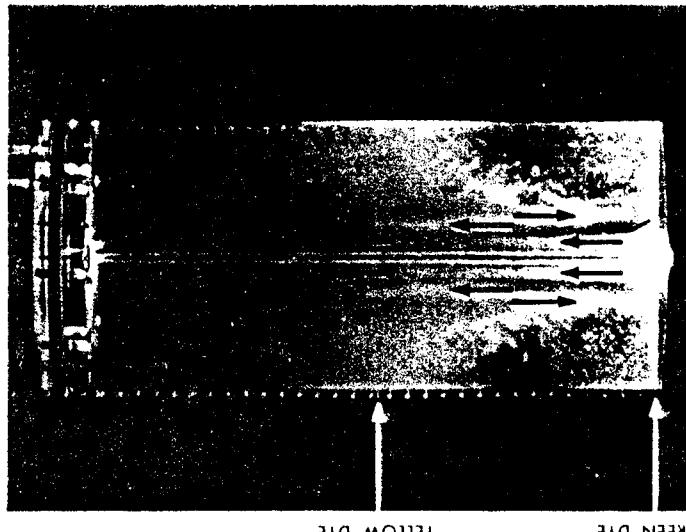
$p_u = p_d = 5$  psig with nominal  $\frac{1}{2}$ -in. exit hole (at  $L/D = 3$ ,  $p_u - p_d = 2$  psig)

Fig. 7. Dye studies in standard configuration: Effect of aspect ratio at low values of  $L/D$  and

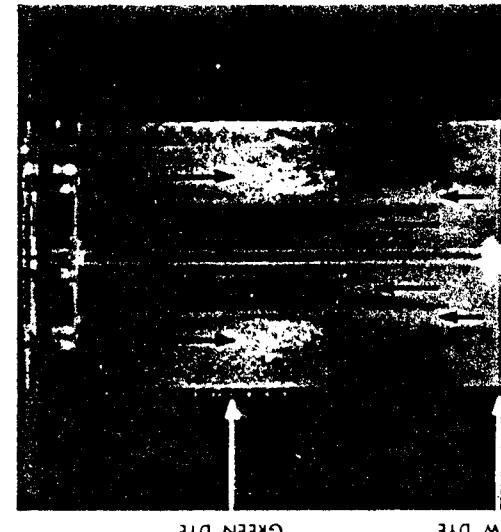
$L/D = 3$



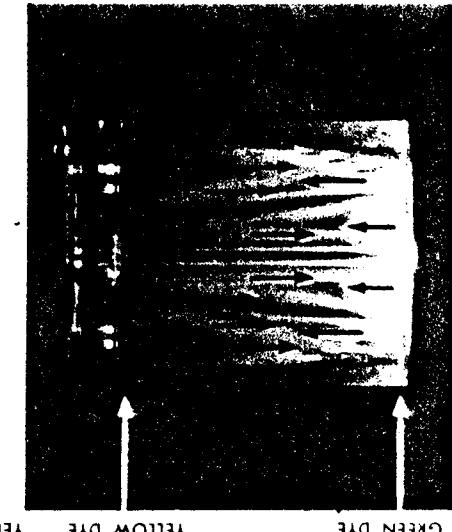
$L/D = 2$

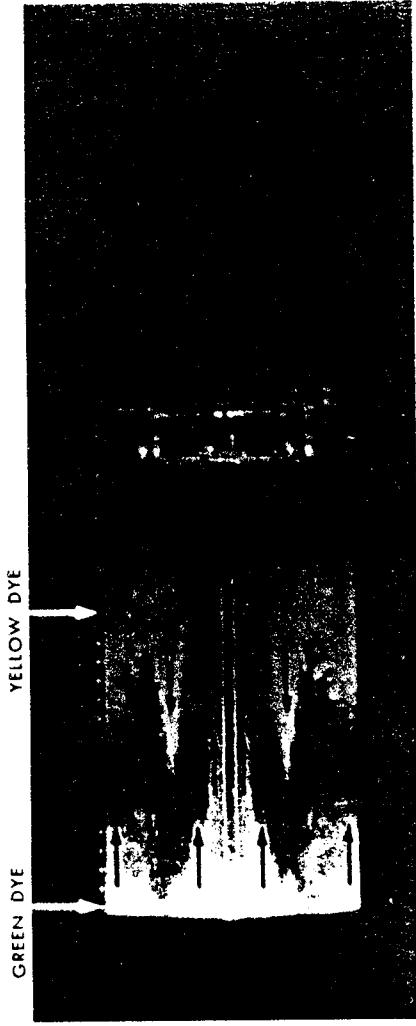


$L/D = 1.5$

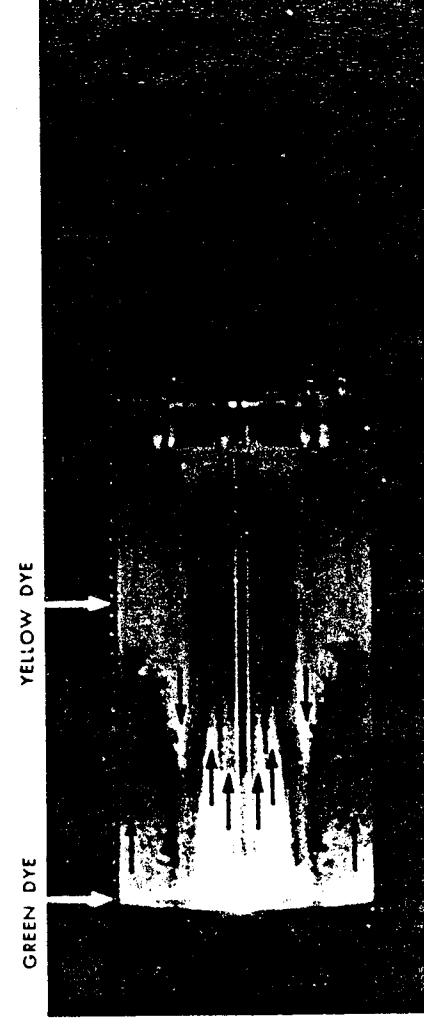


$L/D = 1$

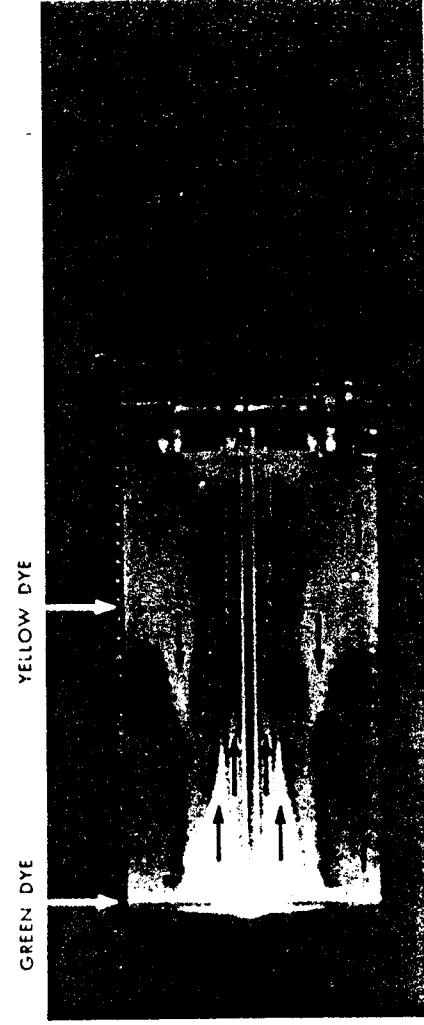




$p_e - p_n = 2 \text{ psig}$



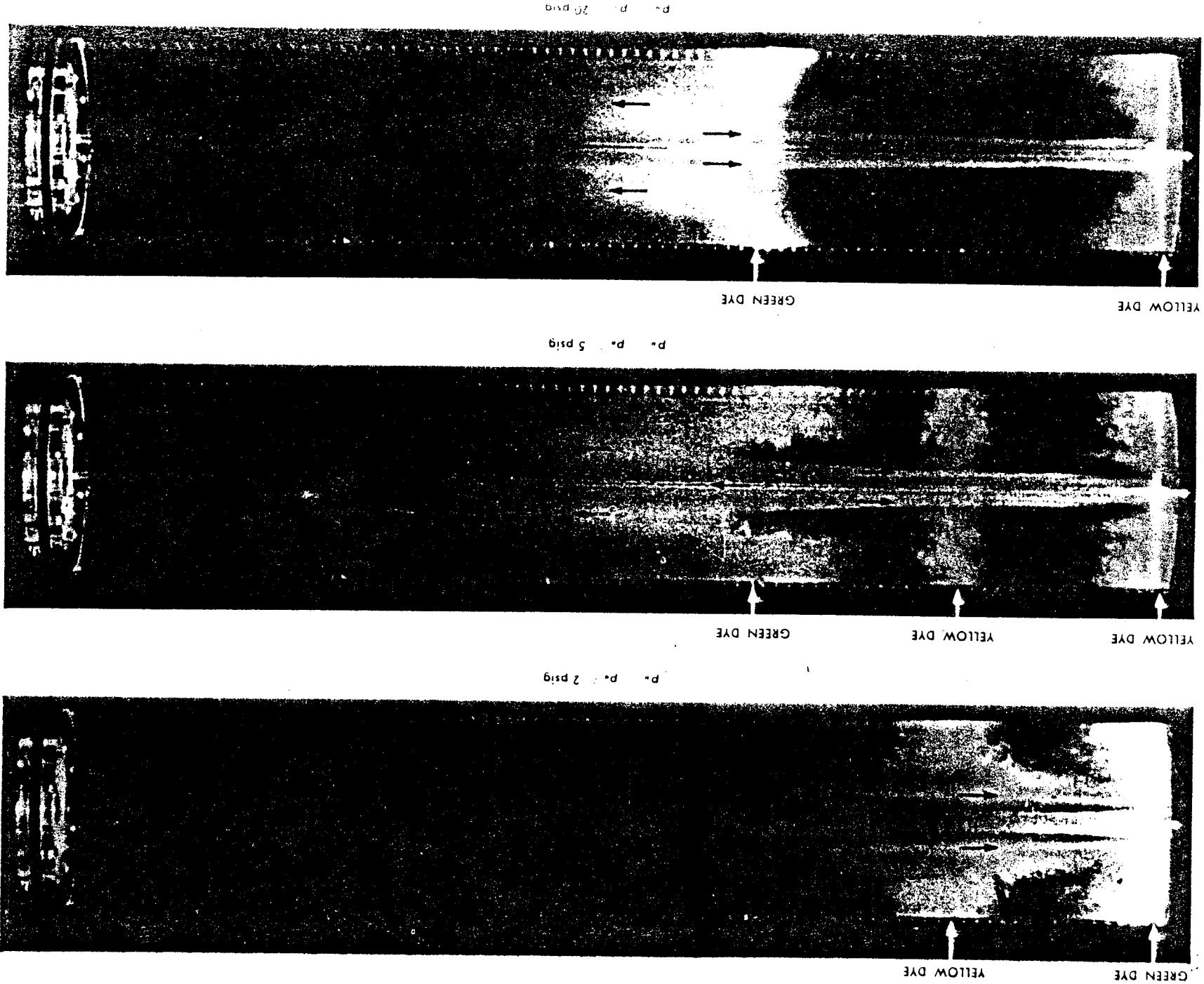
$p_e - p_n = 5 \text{ psig}$

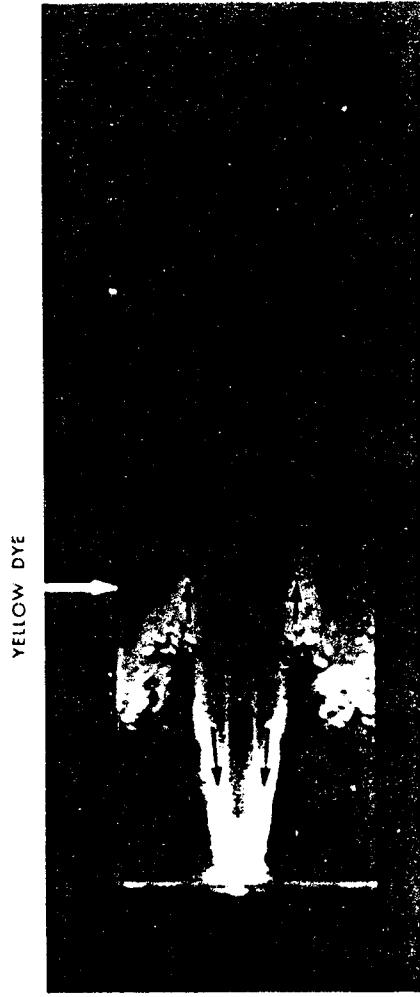


$p_e - p_n = 20 \text{ psig}$

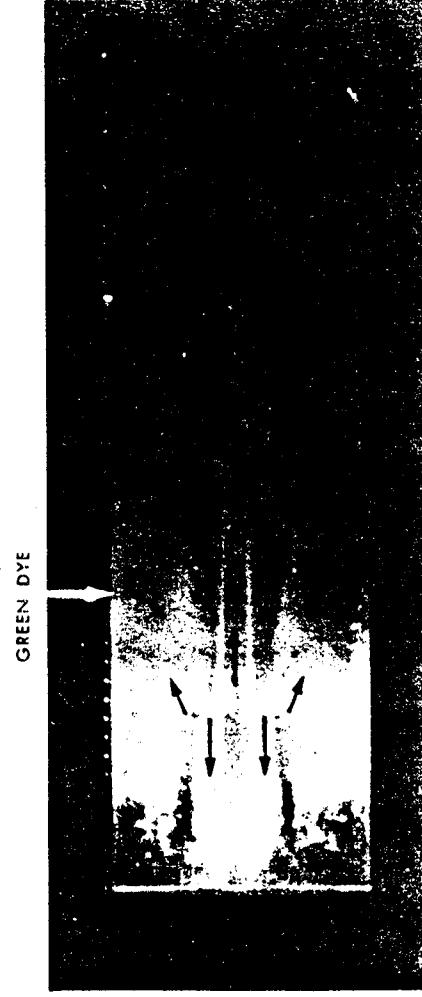
Fig. 8. Dye studies in standard configuration: Effect of static wall pressure at  $L/D = 1.5$  with nominal  $\frac{1}{2}$ -in. exit hole

Fig. 9. Dye studies in standard configuration; effect of static wall pressure at  $L/D = 5.33$  with nominal 9/16-in. exit hole.

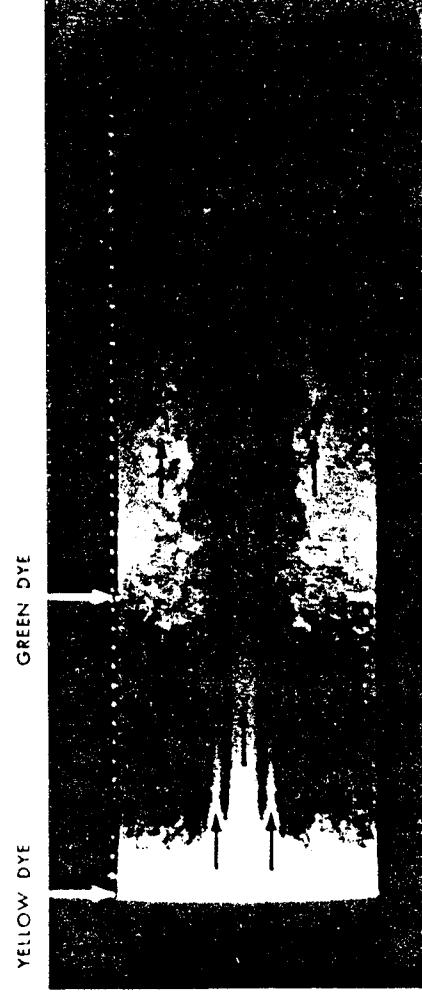




$p_v = p_e = 5 \text{ psig}$



$p_v = p_e = 5 \text{ psig}$



$p_v = p_e = 20 \text{ psig}$

Fig. 10. Dye studies in standard configuration: Views of a portion of the vortex near the closed endwall, at  $L/D = 6$  with nominal  $1/2$ -in. exit hole

Q005 100 dyes w/tpg

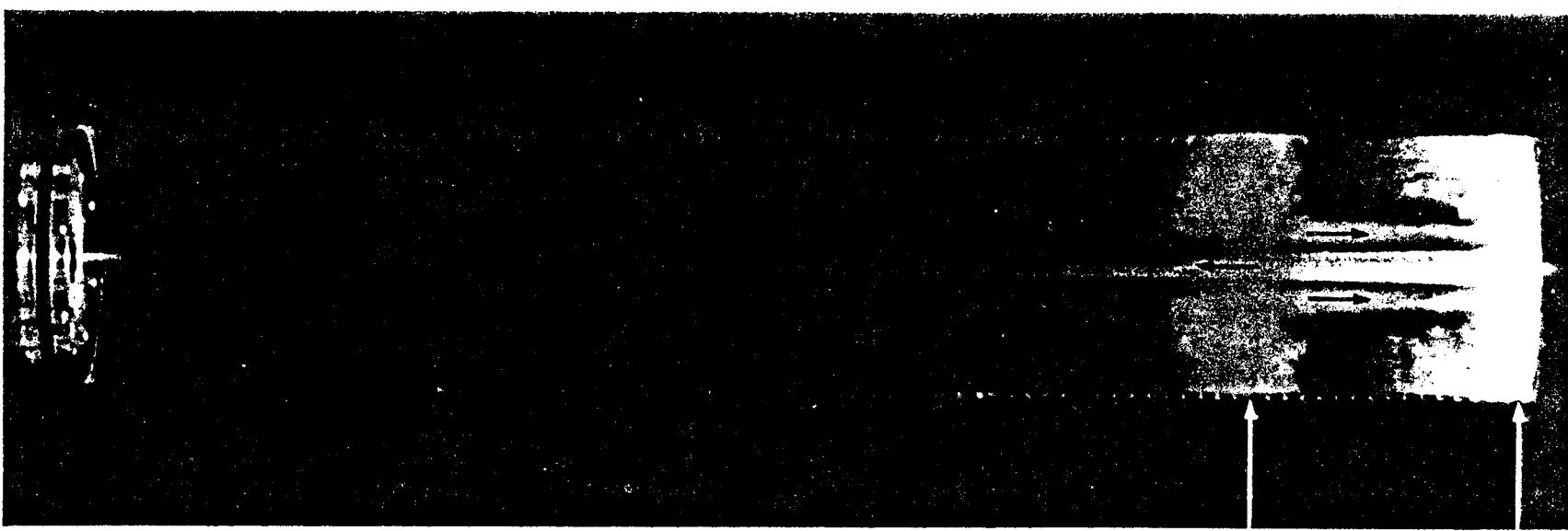
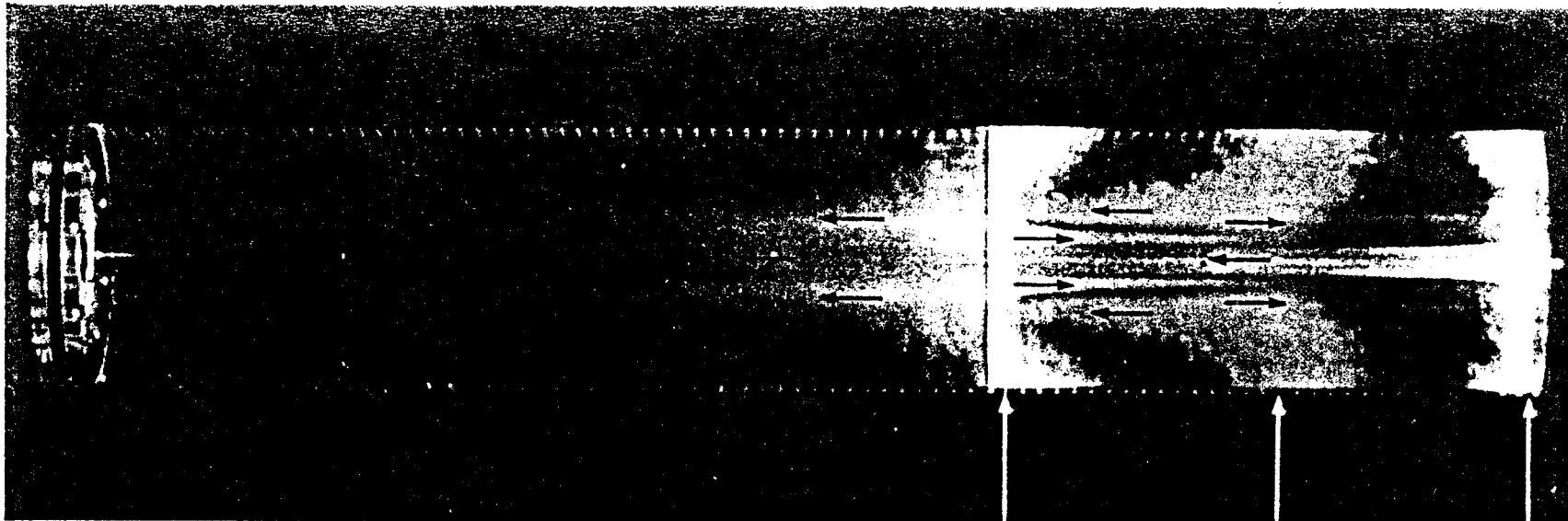
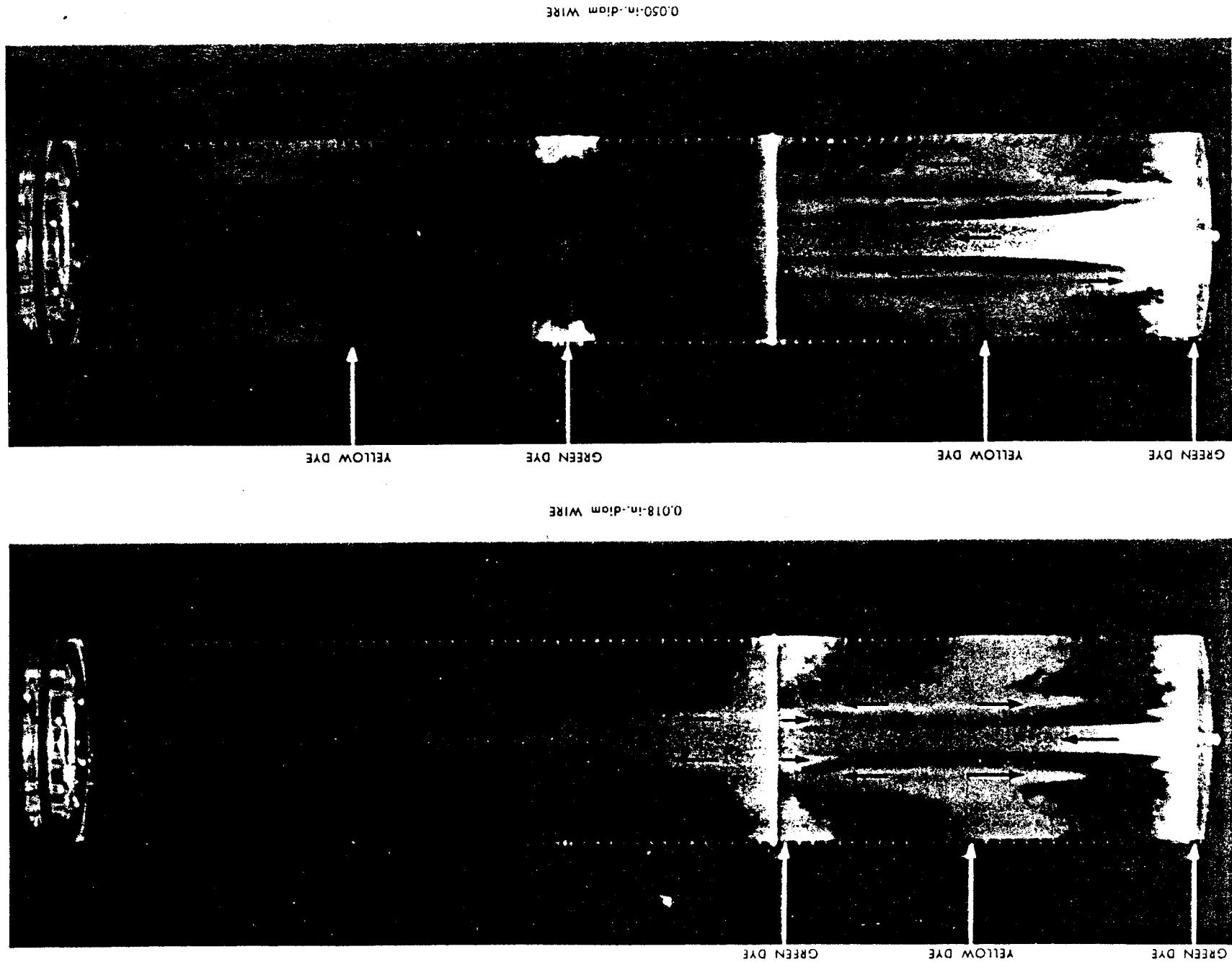
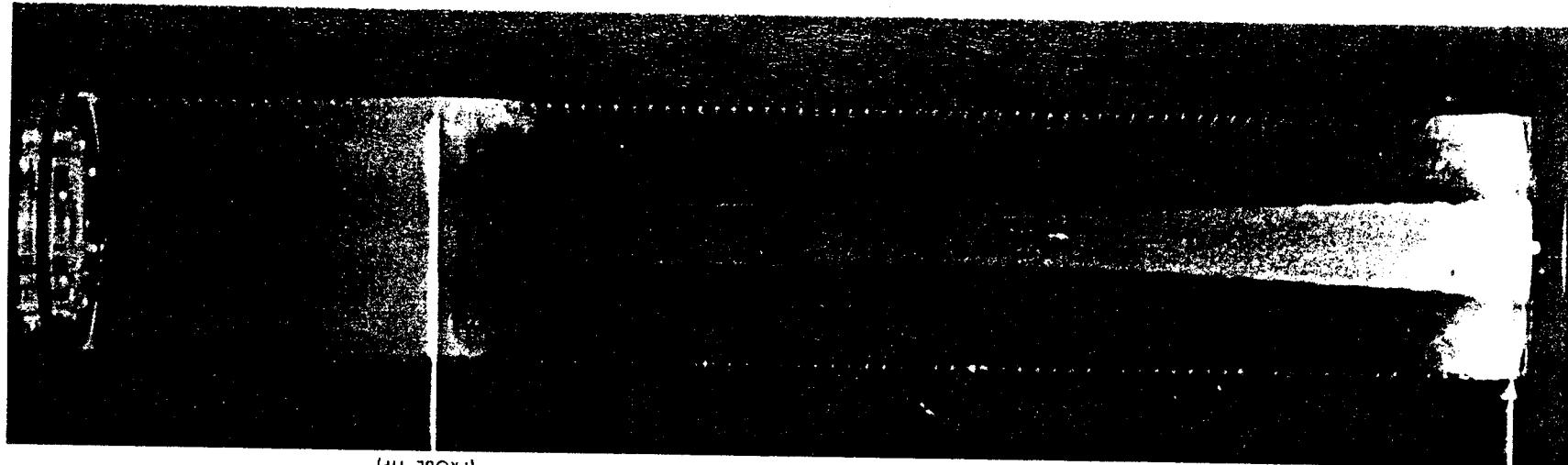


Fig. 11. Dye studies in standard configuration: Effect of various-size-diameter-probe wires stretched across a vortext diameter, at  $L/D = 5.33$  and  $p_1 = p_2 = 5$  psig with nominal 9/10-in. exit hole



probes, at  $L/D = 5.33$  with nominal 9/16-in. exit hole

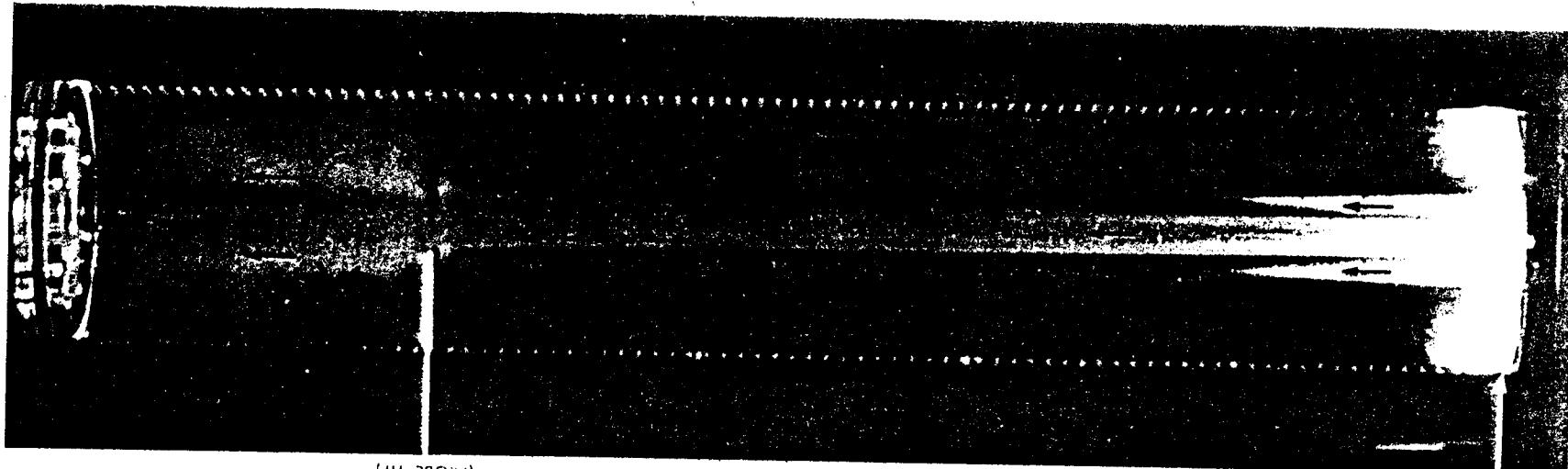
Fig. 12. Dye studies in standard configuration: Effect of cantilever-mounted  
PROBE TIP LOCATED NEAR OPPOSITE WALL;  $p_a$  ...  $p_c$  ... 5 psig



YELLOW DYE  
(PROBE TIP)

GREEN DYE

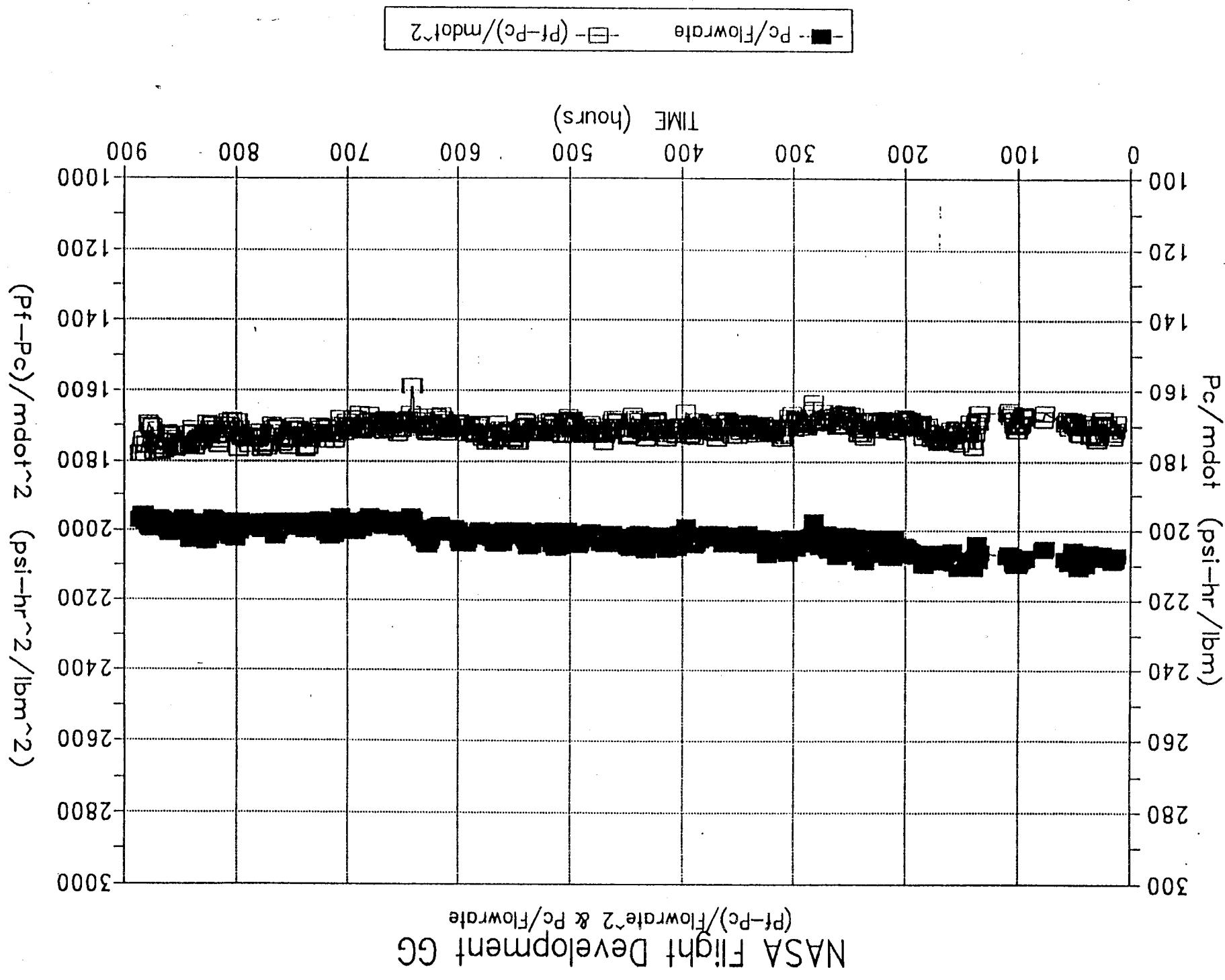
PROBE TIP LOCATED NEAR VORTEX CORE;  $p_a$  ...  $p_c$  ... 10 psig

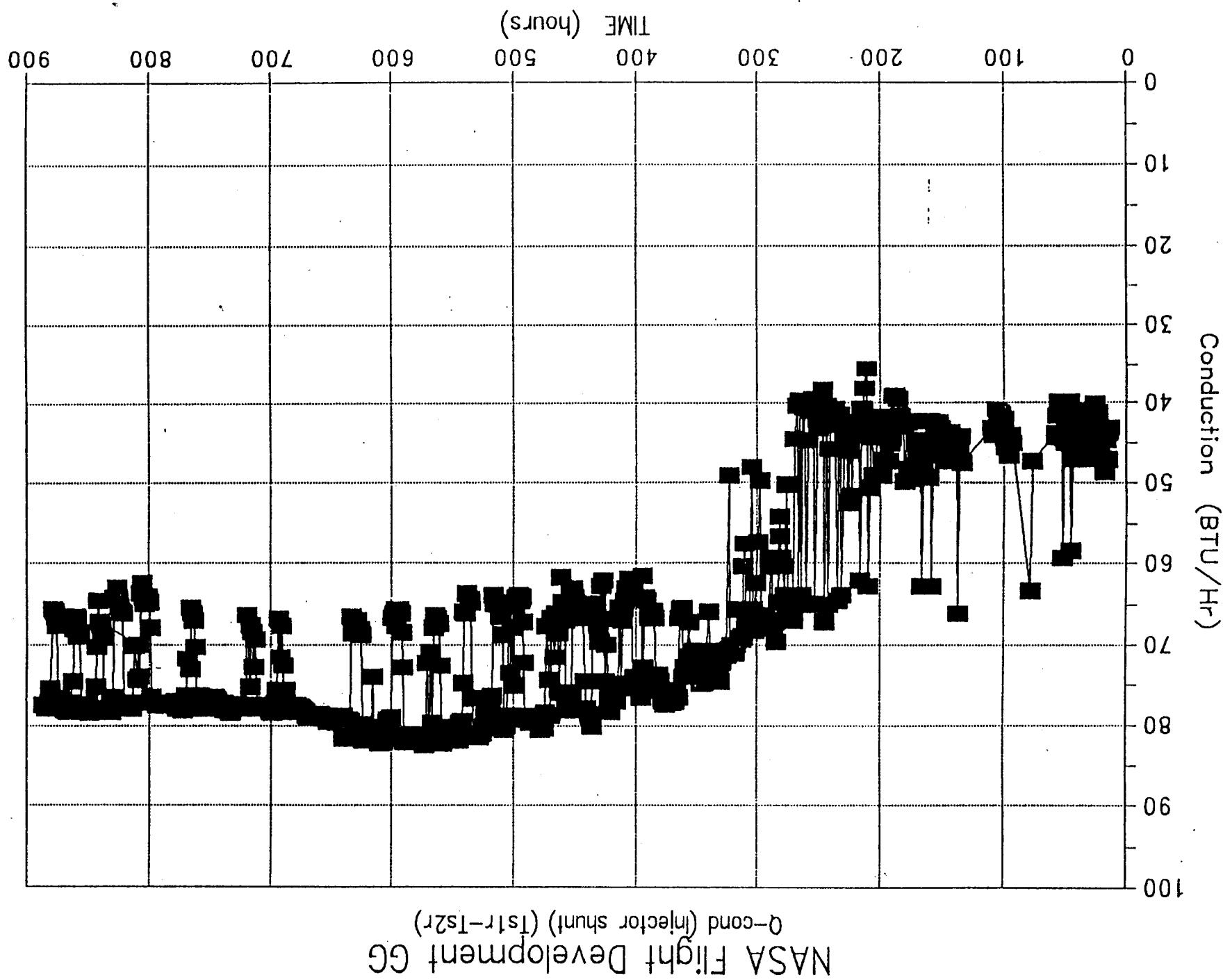


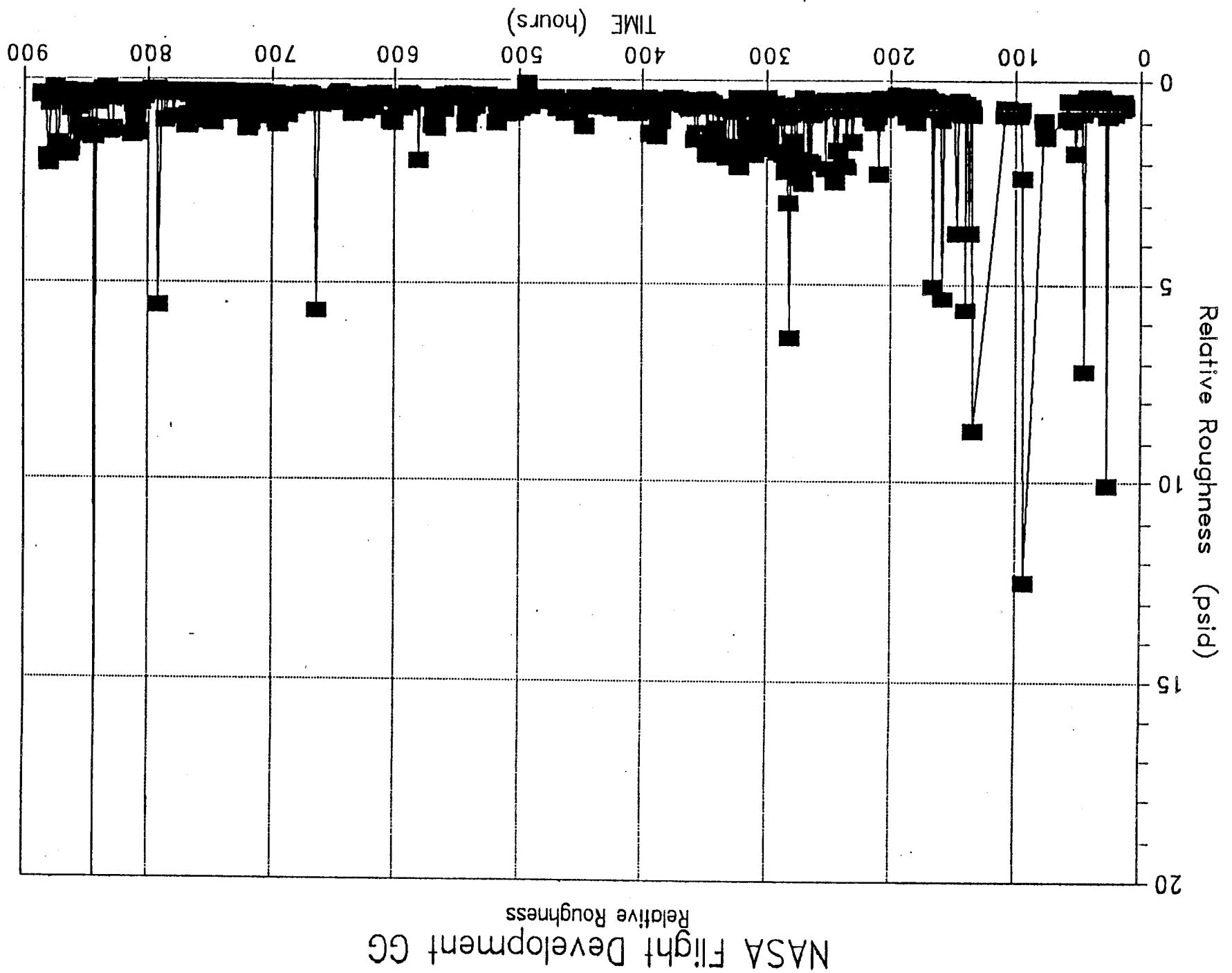
YELLOW DYE  
(PROBE TIP)

GREEN DYE

**APPENDIX G**  
**Advanced Gas Generator Data**









## REPORT DOCUMENTATION PAGE

*Form Approved*

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Public reporting burden for this collection of information is estimated to average 1 hour per response, including the time for reviewing instructions, searching existing data sources, gathering and maintaining the data needed, and completing and reviewing the collection of information. Send comments regarding this burden estimate or any other aspect of this collection of information, including suggestions for reducing this burden, to Washington Headquarters Services, Directorate for Information Operations and Reports, 1215 Jefferson Davis Highway, Suite 1204, Arlington, VA 22202-4302, and to the Office of Management and Budget, Paperwork Reduction Project (0704-0188), Washington, DC 20503.

### **1. AGENCY USE ONLY (Leave Blank)**

**2. REPORT DATE** January 1992

### **3. REPORT TYPE AND DATES COVERED**

Final Contractor Report

### **4. TITLE AND SUBTITLE**

High Performance Storable Propellant Resistojet, Final Report

### **6. AUTHOR(S)**

Prepared by C.E. Vaughan

### **7. PERFORMING ORGANIZATION NAME(S) AND ADDRESS(Es)**

Rocket Research Company  
11441 Willows Rd. N.E.  
Redmond, WA 98073-9709

### **8. PERFORMING ORGANIZATION REPORT NUMBER**

E-7825

### **9. SPONSORING/MONITORING AGENCY NAME(S) AND ADDRESS(ES)**

National Aeronautics and Space Administration  
Lewis Research Center  
Cleveland, Ohio 44135-3191

### **10. SPONSORING/MONITORING AGENCY REPORT NUMBER**

NASA CR-191128

### **11. SUPPLEMENTARY NOTES**

Project Manager, David C. Byers, Space Propulsion Technology Division, NASA Lewis Research Center, organization code 5330, (216) 977-7543.

### **12a. DISTRIBUTION/AVAILABILITY STATEMENT**

Unclassified - Unlimited  
Subject Category 20

This publication is available from the NASA Center for Aerospace Information, (301) 621-0390.

### **13. ABSTRACT (Maximum 200 words)**

From 1965 until 1985 resistojets were used for a limited number of space missions. Capability increased in stages from an initial application using a 90 W gN<sub>2</sub> thruster operating at 123 sec specific impulse (Isp) to a 830 W N<sub>2</sub>H<sub>4</sub> thruster operating at 305 sec Isp. Prior to 1985 fewer than 100 resistojets were known to have been deployed on spacecraft. Building on this base NASA embarked upon the High Performance Storable Propellant Resistojet (HPSR) program to significantly advance the resistojet state-of-the-art. Higher performance thrusters promised to increase the market demands for resistojets and enable space missions requiring higher performance. During the program three resistojets were fabricated and tested. High temperature wire and coupon materials tests were completed. A life test was conducted on an advanced gas generator.

### **14. SUBJECT TERMS**

Electric propulsion; Stationkeeping; Resistojets

### **15. NUMBER OF PAGES**

331

### **16. PRICE CODE**

A15

### **17. SECURITY CLASSIFICATION OF REPORT**

Unclassified

### **18. SECURITY CLASSIFICATION OF THIS PAGE**

Unclassified

### **19. SECURITY CLASSIFICATION OF ABSTRACT**

Unclassified

### **20. LIMITATION OF ABSTRACT**

Unlimited



